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ELASTIC-PLASTIC AND CREEP  
ANALYSIS OF CASINGS  
FOR THERMAL WELLS

by

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May 1990

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**ELASTIC-PLASTIC AND CREEP ANALYSIS  
OF CASINGS FOR THERMAL WELLS**

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**Research Report To  
The Centre for Frontier Engineering Research**

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## ABSTRACT

This report considers an idealized model of a horizontal slice through a thermal recovery well subject to the constraint of zero strain in the longitudinal direction, develops the governing equations for the stress analysis for a number of different types of responses, presents a special-purpose computer code for the analysis of the problem, and gives numerical results for some example problems.

The model idealizes the casing as a thin cylinder supported laterally by the geological formation. The loading consists of temperature and internal pressure. In Chapter 1 the elastic case is considered and closed form expressions are derived for all components of casing stress and strain. This basic approach is then modified in subsequent chapters to predict:

- (a) Combinations of pressure and temperature that initiate yielding.
- (b) Stress and strain histories for elastic-plastic strain hardening material response.
- (c) Stress and strain histories for elastic-plastic strain hardening material response, with a Bailey-Norton creep relationship, in which the thermal coefficient, modulus of elasticity, yield strength and strain hardening moduli are all temperature dependent.

These formulations and solutions were developed as the initial phase for a study directed at predicting creep and relaxation effects on casings in thermal recovery wells. Funding for the study was provided by the Center for Frontier Engineering Research. However, funding was not available to permit a study of the application of these developments to realistic time-history situations.

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### Notation

$p(\Delta p)$	:	internal pressure of the casing
$T(\Delta T)$	:	temperature
$t(\Delta t)$	:	time
$\sigma_{\theta}$	:	tangential stress
$\sigma_r$	:	radial stress
$\sigma_z$	:	axial stress
$\sigma_{ij}$	:	stress tensor
$\sigma_1, \sigma_2, \sigma_3$	:	principal stresses
$\bar{\sigma}$	:	effective stress
$\sigma_Y^o$	:	initial yield limit
$\epsilon_{\theta}$	:	tangential strain (total)
$\epsilon_r$	:	radial strain (total)
$\epsilon_z$	:	axial strain (total)
$\epsilon_{ij}$	:	strain tensor
$\epsilon_1, \epsilon_2, \epsilon_3$	:	principal strains
$\epsilon_{\theta}^p$	:	plastic tangential strain
$\epsilon_r^p$	:	plastic radial strain
$\epsilon_z^p$	:	plastic axial strain
$\epsilon_{ij}^p$	:	plastic strain tensor
$\epsilon_{ij}^T$	:	thermal strain tensor
$\epsilon_{\theta}^c$	:	creep tangential strain
$\epsilon_r^c$	:	creep radial strain
$\epsilon_z^c$	:	creep axial strain
$\epsilon_{ij}^c$	:	creep strain tensor
$\dot{\epsilon}_{ij}^c$	:	tensor of the creep strain rates
$\bar{\epsilon}$	:	effective strain

$S_{ij}$  : deviatoric stress tensor  
 $S_1, S_2, S_3$  : principal-stress deviators  
 $J_2$  : second invariant of the deviator of the stress tensor  
 $d\lambda$  : a positive scale factor in flow rule  
 $R$  : fraction of total strain increment in elastic region  
 $u$  : radial displacement  
 $v$  : tangential displacement  
 $w$  : axial displacement  
 $r$  : internal radius of the casing  
 $t$  : wall thickness of the casing  
 $E$  : elastic modulus of the casing  
 $E^t$  : plastic modulus of the casing (tangent modulus)  
 $\mu$  : Poisson ratio of the casing  
 $G$  : shear modulus of the casing ( $G = E/2(1+\mu)$ )  
 $\alpha$  : thermal expansion coefficient of the casing  
 $[\alpha]$  :  $[\alpha] = (\alpha, \alpha, \alpha)^T$   
 $D(D^E)$  : elastic stiffness matrix  
 $D^P$  : plastic stiffness matrix  
 $D^{EP}$  : elastic-plastic stiffness matrix ( $D^{EP} = D^E - D^P$ )  
 $\bar{E}$  : elastic modulus of rock  
 $\bar{\mu}$  : Poisson ratio of rock  
 $C_{ijkl}$  : 4th order stiffness tensor of material  
 $\delta_{ij}$  : Kronecker delta  
 $\bar{\nabla}$  : expression that denotes some formula is abridged

## CHAPTER 1 INTRODUCTION

### 1.1 Background

In order to investigate relaxation effects on the stress in the casing of a thermal recovery well it is necessary to be able to model a casing in such a way that the elastic-plastic-temperature-stress-strain response of the casing is properly simulated. Subsequently, the creep can be developed. The model should be capable of determining all stress and strain components for time-dependent problems for thermoplastic materials in a simple and convenient manner. In defining a tractable study which might contribute in a meaningful way to useful predictions for casing analysis the following constraints were adopted for the model.

1. The simplest possible model which might predict a reasonable simulation of stress-strain history in the casing has been adopted.
2. No attempt has been made to predict temperature or pressure variation along the casing. It has been assumed that the temperature and pressure at the point of interest would be available as input quantities.
3. It has been assumed that a reasonable simulation for this limited study can be obtained from an axisymmetric horizontal slice.
4. The response of the slice can be adequately represented as
  - i) a thin cylinder, for which
  - ii) thermal expansion of surrounding rock can be neglected,
  - iii) extensional strains in the axial direction are zero.



Within the above constraints a simple analysis for an elastic-strain hardening plasticity solution, including thermal effects in the casing, has been developed. The simple analysis is based directly on the equilibrium equation for rock-casing interaction rather than the FEM which would require a more complex computer program. In this way simple numerical solutions can be obtained which are "exact" within the above constraints.

This report embraces both elastic-plastic temperature dependent problems and the time dependent creep problem. Computation procedure, programming and numerical results are presented as well as the creation of the model.

## 1.2 Idealized Model

In polar coordinates the general strain-dependent equations are as follows.

$$\epsilon_r = \frac{\partial u}{\partial r} ; \quad \epsilon_\theta = \frac{1}{r} \frac{\partial v}{\partial \theta} + \frac{u}{r} ; \quad \epsilon_z = \frac{\partial w}{\partial z} ; \quad (1.1a-1.1c)$$

$$\epsilon_{r\theta} = \frac{1}{2} \left( \frac{1}{r} \frac{\partial u}{\partial \theta} + \frac{\partial v}{\partial r} - \frac{u}{r} \right) ; \quad (1.1d)$$

$$\epsilon_{rz} = \frac{1}{2} \left( \frac{\partial u}{\partial z} + \frac{\partial w}{\partial r} \right) ; \quad (1.1e)$$

$$\epsilon_{\theta z} = \frac{1}{2} \left( \frac{\partial v}{\partial z} + \frac{1}{r} \frac{\partial w}{\partial \theta} \right) ; \quad (1.1f)$$

where u represents the displacement in the radial direction, v in the tangential direction and w in the axial direction. All notation is defined in the list following the Table of Contents.

Using the assumptions of Section 1.1, we isolate a segment from the

casing system in the well and model it as a hollow cylinder, with internal and external pressure, which is inextensible in the longitudinal direction, as shown in Fig. 1.1. The internal pressure is the pressure of the injected steam. The external pressure comes from the surrounding rock which can be regarded as a spring.

Thus, we have the following constraints

$$i) \quad w = 0, \quad \varepsilon_z = 0 ; \quad (1.2a-b)$$

$$ii) \quad \frac{\partial v}{\partial \theta} = 0, \quad \varepsilon_\theta = \frac{u}{r} ; \quad (1.2c-d)$$

iii) There are not any shearing stresses.

Obviously this is a plane strain problem for which the normal stresses,  $\sigma_r$ ,  $\sigma_\theta$  and  $\sigma_z$ , are principal stresses (Fig. 1.2). The matrices of the stress and strain state are

$$\begin{bmatrix} \sigma_\theta \\ \sigma_r \\ \sigma_z \end{bmatrix} = \frac{E}{(1+\mu)(1-2\mu)} \begin{bmatrix} 1-\mu & \mu & \mu \\ \mu & 1-\mu & \mu \\ \mu & \mu & 1-\mu \end{bmatrix} \begin{bmatrix} \varepsilon_\theta \\ \varepsilon_r \\ \varepsilon_z \end{bmatrix} \quad (1.3)$$

and

$$\begin{bmatrix} \varepsilon_\theta \\ \varepsilon_r \\ \varepsilon_z \end{bmatrix} = \frac{1}{E} \begin{bmatrix} 1 & -\mu & -\mu \\ -\mu & 1 & -\mu \\ -\mu & -\mu & 1 \end{bmatrix} \begin{bmatrix} \sigma_\theta \\ \sigma_r \\ \sigma_z \end{bmatrix} \quad (1.4)$$

in which E is elastic modulus of the casing and  $\mu$  is the Poisson ratio of the casing.

If we consider the internal surface,  $\sigma_r = -p$ , and the top and bottom boundary conditions require that  $\varepsilon_z = 0$ . It follows that a closed form solution can be obtained for (1.3) and (1.4) for the elastic case.

Before proceeding further we first need to define the spring constant of supporting rock. For thick cylinders (Fig. 1.3) the displacement along the

radial direction is (Timoshenko, 1956)

$$u = \frac{(1+\mu)(1-2\mu)}{E} \frac{p_i a^2 - p_o b^2}{b^2 - a^2} r + \frac{1}{r} \frac{1+\mu}{E} \frac{(p_i - p_o) a^2 b^2}{b^2 - a^2} \quad (1.5)$$

For a hole in an infinite slice the displacement at the internal surface is obtained by setting  $r = a$ ,  $p_o = 0$  and letting  $b \rightarrow \infty$ . The expression then becomes

$$u = \frac{1+\mu}{E} a p_i \quad (1.6)$$

Introducing  $\bar{E}$  and  $\bar{\mu}$  for rock properties

$$p_i = \frac{\bar{E}}{(1+\bar{\mu}) r} u \quad (1.7a)$$

Now we define  $\bar{K}$  for the supporting rock as follows

$$\bar{K} = \frac{\bar{E}}{(1+\bar{\mu})} \quad (1.7b)$$

Thus, for the casing

$$p_o = \bar{K} \frac{u}{r_o} = K \epsilon_\theta \quad (1.8)$$

in which  $r_o$  is the external radius of the casing and  $\epsilon_\theta$  is the circumferential strain in the casing.

Returning to the basic problem of achieving a closed form solution for the casing system we consider a thin cylinder as shown in Fig. 1.4. When the wall thickness is small, so that normal stresses are distributed uniformly

throughout the thickness, the equilibrium equation for the system can be written as (Popov, 1976)

$$2\sigma_{\theta}t = 2pr_i - 2p_o r_o \quad (1.9a)$$

or

$$\sigma_{\theta} = (pr_i - p_o r_o)/t = \left(p - \bar{K} \frac{r_o}{r_i} \varepsilon_o\right) \frac{r_i}{t} \quad (1.9b)$$

where  $r_i$  is the internal radius. Now we define afresh the spring coefficient for the supporting rock in the following form

$$K = \bar{K} \frac{r_o}{r_i} \nabla \alpha_m \bar{K} = \alpha_m \frac{\bar{E}}{(1 + \bar{\mu})} \quad (1.9c)$$

where  $\alpha_m = r_o/r_i$ . In addition, for simplicity,  $r$  is used for the internal radius of the casing rather than  $r_i$  in what follows.

Since displacement is constrained in the vertical direction

$$\varepsilon_z = \frac{1}{E} (\sigma_z - \mu(\sigma_r + \sigma_{\theta})) = 0$$

Thus  $\sigma_z = \mu(\sigma_r + \sigma_{\theta}) \quad (1.9b)$

and  $\sigma_r = -p \quad (1.9c)$

The displacement  $u$  can be obtained by substituting the above relations into the first equation in (1.4) leading to

$$\varepsilon_{\theta} = \frac{1-\mu^2}{E} \left(\sigma_{\theta} - \frac{\mu}{1-\mu} \sigma_r\right) = \frac{u}{r}$$

in which the latter equality arises from (1.2d).

Solving for u,

$$u = \frac{\left(\frac{r}{t} + \frac{\mu}{1-\mu}\right)}{\frac{E}{r(1-\mu^2)} + \frac{K}{r}} p \nabla a p \quad (1.10a)$$

in which

$$a = \frac{[r(1-\mu) + \mu t] r (1+\mu)}{Et + Kr(1-\mu^2)} \quad (1.10b)$$

Substituting (1.10a) into (1.9a) yields

$$\sigma_{\theta} = \frac{1 - \frac{\mu(1+\mu)K}{E}}{1 + \frac{r}{t} \frac{(1-\mu^2)}{E} K} \frac{r}{t} p \nabla b p \quad (1.11a)$$

in which  $b = \frac{r}{t} \left(1 - \frac{K}{r} a\right)$  (1.11b)

When there is temperature change, but the physical properties of the casing system are independent of temperature, equations (1.3) and (1.4) are replaced by

$$\begin{bmatrix} \sigma_{\theta} \\ \sigma_r \\ \sigma_z \end{bmatrix} = \frac{E}{(1+\mu)(1-2\mu)} \begin{bmatrix} 1-\mu & \mu & \mu \\ \mu & 1-\mu & \mu \\ \mu & \mu & 1-\mu \end{bmatrix} \begin{bmatrix} \epsilon_{\theta} - \alpha \Delta T \\ \epsilon_r - \alpha \Delta T \\ \epsilon_z - \alpha \Delta T \end{bmatrix} \quad (1.12)$$

and

$$\begin{bmatrix} \epsilon_{\theta} \\ \epsilon_r \\ \epsilon_z \end{bmatrix} = \frac{1}{E} \begin{bmatrix} 1 & -\mu & -\mu \\ -\mu & 1 & -\mu \\ -\mu & -\mu & 1 \end{bmatrix} \begin{bmatrix} \sigma_{\theta} \\ \sigma_r \\ \sigma_z \end{bmatrix} + \begin{bmatrix} \alpha \Delta T \\ \alpha \Delta T \\ \alpha \Delta T \end{bmatrix} \quad (1.13)$$

The radial displacement becomes

$$u = a p + a^{\circ} \Delta T \quad (1.14a)$$

where 
$$a^0 = \frac{r(1+\mu)E\alpha t}{Et + Kr(1-\mu^2)} \quad (1.14b)$$

Then the tangential stress  $\sigma_\theta$  is

$$\sigma_\theta = b p + b^0 \Delta T \quad (1.15a)$$

in which  $b^0 = -\frac{K}{t} a^0 \quad (1.15b)$

It is clear that as long as the pressure is given all stress and strain components will follow. The concise computing formulas of the elastic solution are listed in the following

$$u = pa + \Delta T a^0 \quad (a)$$

$$\sigma_\theta = pb + \Delta T b^0 \quad (b)$$

$$\sigma_r = -p \quad (c)$$

$$\sigma_z = p\mu(b-1) + \Delta T (ub^0 - E\alpha) \quad (d) \quad (1.16)$$

$$\epsilon_\theta = pc + \Delta T c^0 \quad (e)$$

$$\epsilon_r = - (pd + \Delta T d^0) \quad (f)$$

$$\epsilon_z = 0 \quad (g)$$

where 
$$a = \frac{[r(1-\mu) + \mu t] r(1+\mu)}{Et + Kr(1-\mu)^2}$$

$$b = r\left(1 - \frac{K}{r} a\right) / t$$

$$c = (1+\mu) [(1-\mu)b + \mu] / E$$

$$d = (1+\mu) [(b-1)\mu + 1] / E$$

$$K = \frac{\alpha}{m} \bar{E} / (1+\bar{\mu}) \quad (1.17)$$

$$a^0 = \frac{r(1+\mu)E\alpha t}{Et + Kr(1-\mu^2)}$$

$$b^0 = -Ka^0/t$$

$$c^0 = [(1-\mu^2)b^0 / E] + (1+\mu)\alpha$$

$$d^0 = [(1+\mu)\mu b^0 / E] - (1+\mu)\alpha$$

The above process is summarized as follows. We first establish the stress-strain relations like (1.3) and (1.4). Then the equilibrium equation is combined with them to yield the closed form solution to the simple model. In the later chapters, all formulations that we will discuss are based on this simple procedure and the elastic-plastic-temperature-stress-strain formulation of the casing system is obtained in the same way.

## CHAPTER 2 TEMPERATURE INDEPENDENT ANALYSIS

### 2.1 Prediction of Initial Yield

This chapter considers solution for elastic-plastic problems where the material properties are not dependent upon temperature. Because  $\sigma_\theta$ ,  $\sigma_r$  and  $\sigma_z$  are principal stresses the Von-Mises condition for initial yield of the casing can be written as

$$\bar{\sigma} \equiv \frac{1}{\sqrt{2}} [(\sigma_\theta - \sigma_r)^2 + (\sigma_r - \sigma_z)^2 + (\sigma_z - \sigma_\theta)^2]^{1/2} = \sigma_Y \quad (2.1)$$

where  $\sigma_Y$  represents the yield strength and  $\bar{\sigma}$  is defined as the 'effective stress'.

Substituting (1.16b,c,d) into (2.1) it follows that the pressure at the onset of the yield of the casing under different temperatures satisfies a second-degree equation

$$Ap_Y^2 + Bp_Y + C = 0 \quad (2.2a)$$

$$\text{where } A = b(b+1) + \mu(\mu-1)(b-1)^2 + 1 \quad (2.2b)$$

$$B = \Delta T b^0(2b+1) + \Delta T(\mu b^0 - E\alpha) [(2\mu-1)(b-1)] - \Delta T b^0 \mu(b-1) \quad (2.2c)$$

$$C = (\Delta T b^0)^2 + (\Delta T)^2(\mu b^0 - E\alpha)(\mu b^0 - E\alpha - b^0) - \sigma_Y^2 \quad (2.2d)$$

Obviously, only the positive root of Eq. (2.2a) is meaningful, i.e.,

$$p_Y = \frac{-B + \sqrt{B^2 - 4AC}}{2A} \quad (2.2e)$$



Then the stresses and strains under the critical yield condition will come from the pressure  $p_y$  using (1.16) from (h) to (g). So, we have

$$\sigma_{\theta} = p_y b + \Delta T b^0 \quad (2.3a)$$

$$\sigma_r = - p_y \quad (2.3b)$$

$$\sigma_z = p_y \mu(b-1) + \Delta T(\mu b^0 - E\alpha) \quad (2.3c)$$

$$\epsilon_{\theta} = p_y c + \Delta T c^0 \quad (2.3d)$$

$$\epsilon_r = - (p_y d + \Delta T d^0) \quad (2.3e)$$

$$\epsilon_z = 0 \quad (2.3f)$$

To use simple nondimensional ratios and present the results in a systematic manner we define  $p_y$  as the pressure in an unconstrained casing which produces the yield stress  $\sigma_y$ , i.e.,

$$p_y = \frac{t}{r} \sigma_y \quad (2.4)$$

Define  $\Delta T_y$  as the temperature increment that causes the longitudinally constrained pipe to yield in compression

$$- E\alpha \Delta T_y = - \sigma_y$$

So, 
$$\Delta T_y = \frac{\sigma_Y}{E\alpha} \quad (2.5)$$

We also introduce a nondimensional ratio for the rock stiffness, which one may call the foundation coefficient, defined as

$$c_f = \frac{K}{E/(1+\mu)} = \frac{\bar{E}(1+\mu)}{E(1+\mu)} \quad (2.6)$$

Now we can use  $p_y$ ,  $\Delta T_y$  and  $c_f$  to nondimensionalize equations (2.3)

$$\tilde{\sigma}_\theta = \frac{\sigma_\theta}{\sigma_Y} = \frac{1-\mu}{1+(1-\mu)c_f} \frac{c_f}{\frac{r}{t}} \frac{p}{\sigma_Y} - \frac{E\alpha c_f \frac{r}{t}}{1+(1-\mu)c_f \frac{r}{t}} \frac{1}{\sigma_Y} \Delta T \quad (2.7a)$$

$$= a_p \tilde{p} - a_T \tilde{\Delta T} \quad (2.7b)$$

in which  $\tilde{p} = p/p_y \quad (2.8a)$

$$\tilde{\Delta T} = \Delta T/\Delta T_y \quad (2.8b)$$

and

$$a_p = (1-\mu c_f) / [1+(1-\mu)c_f \frac{r}{t}] \quad (2.9a)$$

$$a_T = c_f / [\frac{t}{r} + (1-\mu)c_f] \quad (2.9b)$$

Similarly,

$$\tilde{\sigma}_r = \frac{\sigma_r}{\sigma_Y} = -\frac{t}{r} \tilde{p} \quad (2.10a)$$

$$\tilde{\sigma}_z = \frac{\sigma_z}{\sigma_Y} = b_p \tilde{p} - b_T \tilde{\Delta T} \quad (2.10b)$$

in which

$$b_p = \mu \left(1 - c_f - \frac{t}{r}\right) / \left[1 + (1 - \mu)c_f \frac{r}{t}\right] \quad (2.11a)$$

$$b_T = \left(1 + c_f \frac{r}{t}\right) / \left[1 + (1 - \mu)c_f \frac{r}{t}\right] \quad (2.11b)$$

The yield condition (2.1) can be written as

$$2 \frac{\bar{\sigma}_Y^2}{\sigma_Y^2} = (\tilde{\sigma}_\theta - \tilde{\sigma}_r)^2 + (\tilde{\sigma}_r - \tilde{\sigma}_z)^2 + (\tilde{\sigma}_z - \tilde{\sigma}_\theta)^2 \equiv 2 \quad (2.12)$$

Substituting (2.7b), (2.10a) and (2.10b) into (2.12), a second-degree equation in terms of nondimensional ratio  $\tilde{p}$  (when  $\tilde{\Delta T}$  is kept constant) yields

$$A(\tilde{p})^2 + B\tilde{p} + C = 0 \quad (2.13a)$$

where  $A = a_p^2 + b_p^2 + (a_p + b_p) \frac{r}{t} - a_p b_p \quad (2.13b)$

$$B = [(b_T - 2a_T)a_p + (a_T - 2b_T)b_p - (a_T + b_T) \frac{r}{t}] \tilde{\Delta T} \quad (2.13c)$$

$$C = (a_T^2 + b_T^2 - a_T b_T)(\tilde{\Delta T})^2 - 1 \quad (2.13d)$$

Because for a given casing  $r$  and  $t$  are fixed, and the constants  $E$ ,  $\mu$  and  $\alpha$  don't vary for any casing, we can consider the nondimensional ratios  $\tilde{p}$  and  $\tilde{\Delta T}$  to be the independent variables and the nondimensional ratio  $c_f$  as a parameter which affects the yield condition expressed in terms of  $\tilde{p}$  and  $\tilde{\Delta T}$ .

From Eq. (2.13a) the ratio  $\tilde{p}$  under initial yield limit is a function of  $\tilde{\Delta T}$  and  $c_f$ , i.e.,

$$\tilde{p} = f(\Delta\tilde{T}, c_f) . \quad (2.13e)$$

With this relation it is convenient to plot the initial yield condition against the nondimensional ratios  $\Delta\tilde{T}$  and  $c_f$ , as shown in Chapter 6.

## 2.2 Derivation of Governing Equations for Plastic Behavior

For this chapter, elastic-plastic behavior is defined as time independent but loading history dependent. The constitutive equations are written in incremental form. To simplify later derivations tensor notation is introduced. The stress increment tensor and strain increment tensor can be written, respectively, as

$$d\sigma_{ij} \quad \text{and} \quad d\varepsilon_{ij} \quad i, j = 1, 2, 3$$

Now we may derive the constitutive law for plastic behavior. It is usual for isotropic hardening materials to employ the Von-Mises yield condition (Chen 1982 and Wong 1982), i.e.,

$$f(\sigma_{ij}, \sigma_Y) = \bar{\sigma} - \sigma_Y = 0 \quad (2.14)$$

where  $\bar{\sigma} = \sqrt{3J_2}$  is the effective stress.  $J_2$  represents the second invariant of the deviatoric stress tensor  $S_{ij}$ . We have

$$J_2 = \frac{1}{2} S_{ij} S_{ij} \quad (2.15)$$

$$\text{in which } S_{ij} = \sigma_{ij} - \frac{1}{3} (\sigma_{11} + \sigma_{22} + \sigma_{33}) \delta_{ij} \quad (2.16)$$

The usual form for the flow rule (Normality principle) is

$$d\epsilon_{ij}^p = d\lambda \frac{\partial f}{\partial \sigma_{ij}} \quad (2.17)$$

where  $d\lambda$  is a positive scale factor.

The total strain  $d\epsilon$  can be expressed as the sum of elastic, plastic and thermal strains (Bushnell 1973)

$$d\epsilon = d\epsilon^e + d\epsilon^p + d\epsilon^T \quad (2.18a)$$

$$\text{so} \quad d\epsilon^e = d\epsilon - d\epsilon^p - d\epsilon^T \quad (2.18b)$$

in which  $d\epsilon$  is total strain vector

$$d\epsilon = [d\epsilon_\theta, d\epsilon_r, d\epsilon_z]^T$$

$d\epsilon^e$  is elastic strain vector

$$d\epsilon^e = [d\epsilon_\theta^e, d\epsilon_r^p, d\epsilon_z^e]^T$$

$d\epsilon^p$  is plastic strain vector

$$d\epsilon^p = [d\epsilon_\theta^p, d\epsilon_r^p, d\epsilon_z^p]^T$$

$d\epsilon^T$  is thermal strain vector

$$d\epsilon^T = [\alpha] dT$$

$$[\alpha] = [\alpha, \alpha, \alpha]^T$$

The stress increments are expressed as

$$\begin{aligned} d\sigma_{ij} &= C_{ijkl} d\epsilon_{kl}^e \\ &= C_{ijkl} (d\epsilon_{kl} - d\epsilon_{kl}^p - d\epsilon_{kl}^T) \end{aligned} \quad (2.19)$$

where  $C_{ijkl}$  is a 4th-order symmetrical tensor (Chen 1982)

$$C_{ijkl} = 2G \delta_{ik} \delta_{jl} + \lambda_E \delta_{ij} \delta_{kl} \quad (2.20)$$

We take Taylor expansion for the function (2.14) in the neighbourhood of point  $f_0 (\sigma_{ij}, \sigma_Y)$

$$f_1 = f_0 + \frac{\partial f}{\partial \sigma_{ij}} d\sigma_{ij} + \frac{\partial f}{\partial \sigma_Y} d\sigma_Y + o[(d\sigma_{ij})^2 + (d\sigma_Y)^2]$$

Because the yield surface  $f_1$  and  $f_0$  are zero, and we can neglect the second-order term, Prager's consistency condition is

$$\frac{\partial f}{\partial \sigma_{ij}} d\sigma_{ij} + \frac{\partial f}{\partial \sigma_Y} d\sigma_Y = 0 \quad (2.21)$$

Substituting (2.19) into (2.21) we have

$$\begin{aligned} & \frac{\partial f}{\partial \sigma_{ij}} C_{ijkl} (d\epsilon_{kl} - d\epsilon_{kl}^P - d\epsilon_{kl}^T) + \frac{\partial f}{\partial \sigma_Y} \frac{d\sigma_Y}{d\bar{\epsilon}^P} d\bar{\epsilon}^P \\ &= \frac{\partial f}{\partial \sigma_{ij}} C_{ijkl} (d\epsilon_{kl} - d\epsilon_{kl}^T) - \frac{\partial f}{\partial \sigma_{ij}} C_{ijkl} d\epsilon_{kl}^P + (-1)H' d\bar{\epsilon}^P \\ &= \frac{\partial f}{\partial \sigma_{ij}} C_{ijkl} (d\epsilon_{kl} - d\epsilon_{kl}^T) - \left( \frac{\partial f}{\partial \sigma_{ij}} C_{ijkl} \frac{\partial f}{\partial \sigma_{kl}} + H' \frac{d\bar{\epsilon}^P}{d\lambda} \right) d\lambda \end{aligned}$$

Solving for  $d\lambda$

$$d\lambda = \frac{\frac{\partial f}{\partial \sigma_{ij}} C_{ijkl} (d\epsilon_{kl} - d\epsilon_{kl}^T)}{\frac{\partial f}{\partial \sigma_{ij}} C_{ijkl} \frac{\partial f}{\partial \sigma_{kl}} + H'} \quad (2.22a)$$

in which

$$H' = \frac{d\sigma_Y}{d\bar{\epsilon}^P} = \frac{E E^t}{(E - E^t)} \quad (2.22b)$$

$E^t$  is plastic modulus of the casing, and  $d\bar{\epsilon}^P$  is effective plastic strain defined in (2.35).

Now substituting (2.22a) into the flow rule (2.17) we have

$$d\epsilon_{mn}^P = \frac{\frac{\partial f}{\partial \sigma_{mn}} \frac{\partial f}{\partial \sigma_{ij}} C_{ijkl}}{\frac{\partial f}{\partial \sigma_{ij}} C_{ijkl} \frac{\partial f}{\partial \sigma_{kl}} + H'} (d\epsilon_{kl} - d\epsilon_{kl}^T)$$

$$\nabla P_{mnkl} (d\epsilon_{kl} - d\epsilon_{kl}^T) \quad (2.23)$$

in which

$$P_{mnkl} = \frac{\frac{\partial f}{\partial \sigma_{mn}} \frac{\partial f}{\partial \sigma_{ij}} C_{ijkl}}{\frac{\partial f}{\partial \sigma_{ij}} C_{ijkl} \frac{\partial f}{\partial \sigma_{kl}} + H'} \quad (2.24a)$$

Evaluating the factors in (2.22a) and (2.24a), we have

$$\frac{\partial f}{\partial \sigma_{ij}} = \frac{\partial \bar{\sigma}}{\partial \sigma_{ij}} = \frac{\sqrt{3}}{2\sqrt{J_2}} \frac{\partial (J_2)}{\partial \sigma_{ij}} = \frac{\sqrt{3}}{2} \frac{S_{ij}}{\sqrt{J_2}} = \frac{3}{2} \frac{S_{ij}}{\bar{\sigma}}$$

$$\frac{\partial f}{\partial \sigma_{ij}} C_{ijkl} = \frac{3}{2} \frac{S_{ij}}{\bar{\sigma}} (2G\delta_{ik}\delta_{jl} + \lambda_E \delta_{ij}\delta_{kl}) = \frac{3G}{\bar{\sigma}} S_{kl} \quad (2.24b)$$

$$\frac{\partial f}{\partial \sigma_{ij}} C_{ijkl} \frac{\partial f}{\partial \sigma_{kl}} = \frac{3G}{\bar{\sigma}} S_{kl} \frac{3}{2} \frac{S_{kl}}{\bar{\sigma}} = \frac{9GJ_2}{\bar{\sigma}^2} = 3G$$

So that (2.24a) becomes

$$P_{mnkl} = \frac{\frac{3}{2} \frac{S_{mn}}{\bar{\sigma}} S_{kl}}{\bar{\sigma} \left(1 + \frac{H'}{3G}\right)} = \frac{3 S_{mn} S_{kl}}{2 \left(1 + \frac{H'}{3G}\right) \bar{\sigma}^2} \quad (2.25a)$$

and (2.22a) becomes

$$d\lambda = \frac{S_{k\ell}}{\bar{\sigma} \left(1 + \frac{H'}{3G}\right)} (d\varepsilon_{k\ell} - d\varepsilon_{k\ell}^T) \quad (2.25b)$$

Substituting (2.23) into (2.19) we get  $d\sigma_{ij}$  as

$$\begin{aligned} d\sigma_{ij} &= C_{ijmn} [d\varepsilon_{mn} - p_{mnk\ell} (d\varepsilon_{k\ell} - d\varepsilon_{k\ell}^T) - d\varepsilon_{mn}^T] \\ &= C_{ijmn} (\delta_{km} \delta_{\ell n} - p_{mnk\ell}) (d\varepsilon_{k\ell} - d\varepsilon_{k\ell}^T) \end{aligned}$$

We can write this as

$$d\sigma_{ij} = C_{ijkl}^{EP} (d\varepsilon_{k\ell} - d\varepsilon_{k\ell}^T) \quad (2.26a)$$

in which  $C_{ijkl}^{EP} = C_{ijmn} (\delta_{km} \delta_{\ell n} - p_{mnk\ell})$

$$\begin{aligned} &= (2G\delta_{im} \delta_{jn} + \lambda_E \delta_{ij} \delta_{mn}) (\delta_{km} \delta_{\ell n} - \frac{3S_{mn} S_{k\ell}}{2(1 + \frac{H'}{3G}) \bar{\sigma}^2}) \\ &= (2G\delta_{ik} \delta_{j\ell} + \lambda_E \delta_{ij} \delta_{k\ell}) - \frac{3}{2\bar{\sigma}^2 (1 + \frac{H'}{3G})} (2GS_{ij} S_{k\ell} + \lambda_E \delta_{ij} S_{mn} S_{k\ell}) \\ &= (2G\delta_{ik} \delta_{j\ell} + \lambda_E \delta_{ij} \delta_{k\ell}) - \frac{3G}{\bar{\sigma}^2 (1 + \frac{H'}{3G})} S_{ij} S_{k\ell} \end{aligned} \quad (2.26b)$$

The first term in (2.26b) is the elastic part and the second is the plastic part. Eq. (2.26a) is the constitutive equation for elastic-plastic behavior. In order to express it in matrix form we expand  $S_{ij} S_{k\ell}$ . In general,



$$s_{ij} = \sigma_{ij} - \frac{1}{3} (\sigma_{11} + \sigma_{22} + \sigma_{33}) \delta_{ij}$$

As there are principal stresses only we have

$$s_{11} = \sigma_{\theta} - \frac{1}{3} (\sigma_{\theta} - \sigma_r - \sigma_z) = \frac{1}{3} (2\sigma_{\theta} - \sigma_r - \sigma_z) \equiv \bar{s}_{\theta}$$

$$s_{22} = \frac{1}{3} (2\sigma_r - \sigma_{\theta} - \sigma_z) \equiv \bar{s}_r \quad (2.27)$$

$$s_{33} = \frac{1}{3} (2\sigma_z - \sigma_r - \sigma_{\theta}) \equiv \bar{s}_z$$

So,

$$s_{ij} s_{kl} = \begin{bmatrix} \bar{s}_{\theta}^2 & \bar{s}_{\theta} \bar{s}_r & \bar{s}_{\theta} \bar{s}_z \\ & \bar{s}_r^2 & \bar{s}_r \bar{s}_z \\ \text{(Sym)} & & \bar{s}_z^2 \end{bmatrix}$$

and

$$2G \delta_{ij} \delta_{j\ell} + \lambda_E \delta_{ij} \delta_{k\ell} = \frac{E}{(1+\mu)(1-2\mu)} \begin{bmatrix} 1-\mu & \mu & \mu \\ \mu & 1-\mu & \mu \\ \mu & \mu & 1-\mu \end{bmatrix}$$

In addition, using (2.22b), the coefficient of the last term in (2.26b) can be expressed as

$$\frac{3G}{\sigma^2 \left(1 + \frac{H'}{3G}\right)} = \frac{9E(E-E^t)}{2(1+\mu) \sigma^2 [3(E-E^t) + 2E^t(1+\mu)]}$$

Now we can write (2.26a) in matrix form as

$$d\sigma = D^{EP} (d\varepsilon - d\varepsilon^T) \quad (2.28a)$$

in which

$$d\sigma = (d\sigma_{\theta}, d\sigma_r, d\sigma_z)^T \quad (2.28b)$$

$$d\varepsilon = (d\varepsilon_{\theta}, d\varepsilon_r, d\varepsilon_z)^T \quad (2.28c)$$

$$d\varepsilon^T = (d\varepsilon_{\theta}^T, d\varepsilon_r^T, d\varepsilon_z^T)^T \quad (2.28d)$$

and  $D^{EP} = D^E - D^P$

$$= \begin{bmatrix} \lambda' & \lambda & \lambda \\ \lambda & \lambda' & \lambda \\ \lambda & \lambda & \lambda' \end{bmatrix} - \begin{bmatrix} s_{\theta\theta} & s_{\theta r} & s_{\theta z} \\ s_{r\theta} & s_{rr} & s_{rz} \\ s_{z\theta} & s_{zr} & s_{zz} \end{bmatrix} \quad (2.28e)$$

Consequently, (2.28a) may be written in expanded form as

$$\begin{bmatrix} d\sigma_{\theta} \\ d\sigma_r \\ d\sigma_z \end{bmatrix} = \begin{bmatrix} \lambda' & \lambda & \lambda \\ \lambda & \lambda' & \lambda \\ \lambda & \lambda & \lambda' \end{bmatrix} \begin{bmatrix} d\varepsilon_{\theta} - \Delta T\alpha \\ d\varepsilon_r - \Delta T\alpha \\ d\varepsilon_z - \Delta T\alpha \end{bmatrix} - \begin{bmatrix} s_{\theta\theta} & s_{\theta r} & s_{\theta z} \\ s_{r\theta} & s_{rr} & s_{rz} \\ s_{z\theta} & s_{zr} & s_{zz} \end{bmatrix} \begin{bmatrix} d\varepsilon_{\theta} - \Delta T\alpha \\ d\varepsilon_r - \Delta T\alpha \\ d\varepsilon_z - \Delta T\alpha \end{bmatrix}$$

(2.28f)

in which

$$\lambda = \frac{E\mu}{(1+\mu)(1-2\mu)}$$

$$\lambda' = \frac{E(1-\mu)}{(1+\mu)(1-2\mu)}$$

$$\begin{aligned}
 S_{\theta\theta} &= \frac{E(E-E^t)(2\sigma_{\theta}-\sigma_r-\sigma_z)^2}{2(1+\mu)\sigma^2(3(E-E^t)+2(1+\mu)E^t)} \\
 S_{rr} &= \frac{E(E-E^t)(2\sigma_r-\sigma_{\theta}-\sigma_z)^2}{2(1+\mu)\sigma^2(3(E-E^t)+2(1+\mu)E^t)} \\
 S_{zz} &= \frac{E(E-E^t)(2\sigma_z-\sigma_r-\sigma_{\theta})^2}{2(1+\mu)\sigma^2(3(E-E^t)+2(1+\mu)E^t)} \\
 S_{\theta r} &= \frac{E(E-E^t)(2\sigma_{\theta}-\sigma_r-\sigma_z)(2\alpha_z-\sigma_{\theta}-\sigma_r)}{2(1+\mu)\sigma^2(3(E-E^t)+2(1+\mu)E^t)} = S_{r\theta} \\
 S_{\theta z} &= \frac{E(E-E^t)(2\alpha_{\theta}-\sigma_r-\sigma_z)(2\sigma_z-\sigma_{\theta}-\sigma_r)}{\alpha(1+\mu)\sigma^2(3(E-E^t)+2(1+\mu)E^t)} = S_{z\theta} \\
 S_{rz} &= \frac{E(E-E^t)(2\sigma_r-\sigma_{\theta}-\sigma_z)(2\sigma_z-\sigma_{\theta}-\sigma_r)}{2(1+\mu)\sigma^2(3(E-E^t)+2(1+\mu)E^t)} = S_{zr}
 \end{aligned} \tag{2.28g}$$

Now tht the constiutive matrix is completely defined, solutions for stress and strain may be obtained by proceeding in the same way as in Chapter One. If the loading increment is  $dp$ , we can write the equilibrium equations in increment form. These are (see equation (1.9))

$$d\sigma_{\theta} = (dp - kd\varepsilon_{\theta}) \frac{r}{t} \tag{2.29a}$$

$$d\sigma_r = - dp \tag{2.29b}$$

and  $d\varepsilon_z = 0$  (2.29c)

Using (2.28b), (2.29) becomes

$$-dp = (\lambda - S_{r\theta})(d\varepsilon_{\theta} - \alpha dT) + (\lambda' - S_{rr})(d\varepsilon_r - \alpha dT) - (\lambda - S_{rz}) \alpha dT$$

Solving for  $(d\varepsilon_r - \alpha dT)$

$$d\varepsilon_r - \alpha dT = \frac{1}{(\lambda' - s_{rr})} [(\lambda - s_{r\theta})(d\varepsilon_{\theta} - \alpha dT) + dp - (\lambda - s_{rz}) \alpha dT]$$

Substituting  $d\varepsilon_r$  into the first line of (2.28b) yields

$$d\sigma_{\theta} = (\lambda' - s_{\theta\theta})(d\varepsilon_{\theta} - \alpha dT) + \frac{\lambda - s_{\theta r}}{\lambda' - s_{rr}} [(\lambda - s_{r\theta})(d\varepsilon_{\theta} - \alpha dT) + dp - (\lambda - s_{rz}) \alpha dT]$$

Collecting the above terms and solving for  $d\varepsilon_{\theta}$  we obtain

$$d\varepsilon_{\theta} = \frac{\lambda' - s_{rr}}{(\lambda' - s_{\theta\theta})(\lambda' - s_{rr}) - (\lambda - s_{\theta r})^2} d\sigma_{\theta} + \frac{(\lambda' - s_{\theta\theta})(\lambda' - s_{rr}) - (\lambda - s_{r\theta})^2 - (\lambda - s_{\theta r})(\lambda - s_{rz})}{(\lambda' - s_{\theta\theta})(\lambda' - s_{rr}) - (\lambda - s_{\theta r})^2} \alpha dT + \frac{\lambda - s_{\theta r}}{(\lambda' - s_{\theta\theta})(\lambda' - s_{rr}) - (\lambda - s_{\theta r})^2} dp \quad (2.30)$$

Finally the tangential stress increment  $d\sigma_{\theta}$  is obtained by substituting (2.30) into the equilibrium equation (2.29a)

$$d\sigma_{\theta} = \frac{(1 - \frac{(\lambda - s_{\theta r})}{B} k)}{1 + \frac{r}{t} k \frac{(\lambda' - s_{rr})}{B}} \frac{r}{t} dp + \frac{(\lambda' - s_{rr})((\lambda' - s_{\theta\theta}) - (\lambda - s_{\theta z})) - (\lambda - s_{r\theta})((\lambda - s_{r\theta}) + (\lambda - s_{rz}))}{B + (\lambda' - s_{rr}) k \frac{r}{t}} \frac{kr}{t} \alpha dT \quad (2.31)$$

in which  $B = (\lambda' - s_{\theta\theta})(\lambda' - s_{rr}) - (\lambda - s_{\theta r})^2$ .

Now it follows immediately that we can solve all stress and strain increments in a similar way as Chapter One. For simplicity, to compute the

concise formulas for plastic behavior, when the pressure increment and temperature change are prescribed, we have

$$d\sigma_{\theta} = \left( \frac{1 + ka_2^0}{1 + ka_1^0 \frac{r}{t}} \right) \frac{r}{t} dp + \left( \frac{a_T^0 k \frac{r}{t}}{1 + a_1^0 k \frac{r}{t}} \right) \alpha dT \quad (2.32a)$$

$$d\sigma_r = - dp \quad (2.32b)$$

$$d\sigma_z = c_1 d\sigma_{\theta} + c_2 d\sigma_r + c_T \alpha dT \quad (2.32c)$$

and

$$d\varepsilon_{\theta} = a_1^0 d\sigma_{\theta} + a_2^0 d\sigma_r + a_T^0 \alpha dT \quad (2.33a)$$

$$d\varepsilon_r = b_1^0 d\sigma_{\theta} + b_2^0 d\sigma_r - b_T^0 \alpha dT \quad (2.33b)$$

$$d\varepsilon_z = 0 \quad (2.33c)$$

in which

$$a_1 = \lambda' - S_{\theta\theta}$$

$$a_2 = \lambda - S_{\theta r}$$

$$b_1 = \lambda - S_{r\theta} = a_2$$

$$b_2 = \lambda' - S_{rr}$$

$$c_1' = \lambda - S_{z\theta}$$

$$c_2' = \lambda - S_{zr}$$

$$c_1 = c_1' a_1^0 + c_2' b_1^0$$

$$c_2 = c_1' a_2^0 + c_2' b_2^0 \quad (2.34)$$

$$a_1^0 = b_2 / (a_1 b_2 - b_1 a_2) \quad a_2^0 = -a_2 / (a_1 b_2 - b_1 a_2)$$

$$b_1^0 = -b_1 / (a_1 b_2 - b_1 a_2) \quad b_2^0 = a_1 / (a_1 b_2 - b_1 a_2)$$

$$a_T = (S_{\theta\theta} + S_{\theta r} + S_{\theta z} - 2\lambda - \lambda')$$

$$b_T = (S_{r\theta} + S_{rr} + S_{rz}) - 2\lambda - \lambda'$$

$$c_T' = (S_{z\theta} + S_{zr} + S_{zz} - 2\lambda - \lambda')$$

$$a_T^0 = a_1^0 a_T + a_2^0 a_T$$

(2.34)

$$b_T^0 = b_1^0 a_T + b_2^0 b_T$$

$$c_T = c_T' - c_1' a_T^0 - c_2' b_T^0$$

$$k = \alpha \bar{E} / (1 + \bar{\mu})$$

$$H' = E E^t / (E - E^t)$$

$$\bar{\sigma} = \frac{1}{\sqrt{2}} [(\sigma_r - \sigma_\theta)^2 + (\sigma_\theta - \sigma_z)^2 + (\sigma_z - \sigma_r)^2]^{1/2}$$

and

$\lambda, \lambda', S_{\theta\theta}, S_{\theta r}, S_{\theta z}, S_{r\theta}, S_{rr}, S_{rz}, S_{z\theta}, S_{zr}, S_{zz}$  are defined in (2.28g).

Once we have determined the plastic solution from (2.32) to (2.34) we may determine the plastic strains as follows. First we note that the effective plastic strain can be defined as

$$d\bar{\epsilon}^p = c \sqrt{d\epsilon_{ij}^p d\epsilon_{ij}^p} = cd\lambda \sqrt{\frac{\partial f}{\partial \sigma_{ij}} \frac{\partial f}{\partial \sigma_{ij}}}$$

in which the flow rule of (2.17) has been used.

For  $d\bar{\epsilon}^p$  to identify with  $d\epsilon_1^p$  in uniaxial test

$$d\bar{\epsilon}^p = c \sqrt{(d\epsilon_1^p)^2 + 2\left(-\frac{d\epsilon_1^p}{2}\right)^2} = c \frac{\sqrt{3}}{2} d\epsilon_1^p \equiv d\epsilon_1^p$$

Therefore  $c = \frac{\sqrt{3}}{2}$

$$\text{Then } d\bar{\epsilon}^p = d\lambda \sqrt{\frac{2}{3} \frac{\partial f}{\partial \sigma_{ij}} \frac{\partial f}{\partial \sigma_{ij}}} \quad (2.35)$$

From Eq. (2.24)b

$$\frac{\partial f}{\partial \sigma_{ij}} = \frac{3}{2} \frac{s_{ij}}{\bar{\sigma}}$$

Substituting this into (2.35) yields

$$d\bar{\epsilon}^p = d\lambda \sqrt{\frac{2}{3} \frac{9}{4} \frac{s_{ij} s_{ij}}{\bar{\sigma}^2}} = d\lambda \quad (2.36a)$$

Then, using (2.36a) in flow rule we have

$$d\epsilon_{ij}^p = d\lambda \frac{\partial f}{\partial \sigma_{ij}} = d\bar{\epsilon}^p \frac{\partial f}{\partial \sigma_{ij}} = \frac{3}{2} \frac{s_{ij}}{\bar{\sigma}} d\bar{\epsilon}^p \quad (2.36b)$$

From (2.25b) and (2.36a) we have

$$d\bar{\epsilon}^P = d\lambda = \frac{S_{k\ell}}{\bar{\sigma} \left(1 + \frac{H'}{3G}\right)} (d\epsilon_{k\ell} - d\epsilon_{k\ell}^T)$$

Because only principal stresses exist

$$S_{ij} = 0 \quad \text{when } i \neq j$$

and

$$S_{11} = \frac{1}{3} (2\sigma_\theta - \sigma_r - \sigma_z) \nabla \bar{S}_\theta$$

$$S_{22} = \frac{1}{3} (2\sigma_r - \sigma_\theta - \sigma_z) \nabla \bar{S}_r$$

$$S_{33} = \frac{1}{3} (2\sigma_z - \sigma_\theta - \sigma_r) \nabla \bar{S}_z$$

So  $d\bar{\epsilon}^P$  becomes

$$d\bar{\epsilon}^P = \frac{1}{\bar{\sigma} \left(1 + \frac{H'}{3G}\right)} \left[ \bar{S}_\theta (d\epsilon_\theta - \alpha dT) + \bar{S}_r (d\epsilon_r - \alpha dT) + \bar{S}_z (d\epsilon_z - \alpha dT) \right] \quad (2.37a)$$

Similarly, (2.36b) becomes

$$d\epsilon_{ij}^P = \frac{3}{2\bar{\sigma}^2 \left(1 + \frac{H'}{3G}\right)} \left[ \bar{S}_\theta (d\epsilon_\theta - \alpha dT) + \bar{S}_r (d\epsilon_r - \alpha dT) + \bar{S}_z (d\epsilon_z - \alpha dT) \right] S_{ij} \quad (2.37b)$$



### 2.3 Elastic-Plastic Solution - Computational Procedure

We can first find the condition at which yield occurs from (2.2) and (2.3) for any given temperature. Then, the prediction of initial yield is obtained.

Then, suppose that the computer has just completed the calculations for a certain loading or unloading state and the load has just been increased by increments  $\Delta T$  and  $\Delta p$ . For generalization we also suppose that the initial point doesn't fall on the yield surface. Referring to Fig. 2.1, the initial conditions are in the following

- i)  $\sigma_{Y0}$  is known;  $\sigma_{Y1}$  is known too, but  $f(\sigma_{ij}^{(1)}, \sigma_{Y1}) < 0$ ;  $\sigma_{ij}^{(1)}$  are known, say point A is given;  $\Delta p$  and  $\Delta T$  are given.
- ii)  $\Delta\sigma_{ij}$  are unknown. That is to say the point B or point C needs to be determined.

The subsequent computational steps follow.

1. Solve for  $\Delta\sigma_{ij}^{(e)}$  and  $\Delta\epsilon_{ij}^{(e)}$  using the computational formulas of elastic solution from (b) to (g) of (1.16), in which  $\Delta p$  replaces  $p$ .

$$\Delta\sigma_{\theta} = \Delta p b + \Delta T b^0 \quad (2.38a)$$

$$\Delta\sigma_r = -\Delta p \quad (2.38b)$$

$$\Delta\sigma_z = \Delta p \mu (b-1) + \Delta T (\mu b^0 - E\alpha) \quad (2.38c)$$

$$\Delta\epsilon_{\theta} = \Delta p c + \Delta T c^0 \quad (2.38d)$$

$$\Delta\epsilon_r = -(-\Delta p d + \Delta T d^0) \quad (2.38e)$$

$$\Delta \epsilon_z = 0 \quad (2.38f)$$

in which all parameters are defined in (1.17) of Chapter 1.

2. For stress increments  $\Delta \sigma_{ij}$  calculate

$$\sigma_{ij}^{(e)} = \sigma_{ij}^{(1)} + \Delta \sigma_{ij}^{(e)} \quad (2.39)$$

and

$$f(\sigma_{ij}^{(e)}, \sigma_{Y1}) = \bar{\sigma} - \sigma_{Y1} \quad (2.40)$$

3. Check if  $f(\sigma_{ij}^{(e)}, \sigma_{Y1}) \leq 0$ . If so, the values of  $\Delta \sigma_{ij}^{(e)}$  are correct and it is an elastic loading or unloading state. Therefore

$$\sigma_{ij}^{(2)} = \sigma_{ij}^{(e)} = \sigma_{ij}^{(1)} + \Delta \sigma_{ij}^{(e)}$$

Then, go to next load increment, i.e., return to step 1.

4. If  $f(\sigma_{ij}^{(e)}, \sigma_{Y1}) > 0$  it indicates a plastic loading state. It shows that the stress increments  $\Delta \sigma_{ij}^{(e)}$  should include two portions which are the elastic and the plastic. We can express the elastic portion with a scale factor "R", i.e.,

$$R \Delta \sigma_{ij}^{(e)} \text{ and } R \Delta \epsilon_{ij}^{(e)}$$

To find the scalar R we directly employ the yield condition (Wong 1982)

$$f(\sigma_{ij}^{(1)} + R \Delta \sigma_{ij}^{(e)}, \sigma_{Y1}) = 0 \quad (2.41a)$$

The expansion can be written as

$$\frac{1}{2} (S_{ij}^{(1)} + R\Delta S_{ij}^{(e)}) (S_{ij}^{(1)} + R\Delta S_{ij}^{(e)}) - \frac{1}{3} \sigma_{Y1}^2 = 0$$

or

$$(\Delta S_{ij}^{(e)} \Delta S_{ij}^{(e)}) R^2 + 2S_{ij}^{(1)} \Delta S_{ij}^{(e)} R + S_{ij}^{(1)} S_{ij}^{(1)} - \frac{2}{3} \sigma_{Y1}^2 = 0 \quad (2.41b)$$

When only principal stresses exist we have

$$S_{ij} S_{ij} = S_{ii} S_{jj} = S_{11}^2 + S_{22}^2 + S_{33}^2 = S_{\theta}^2 + S_r^2 + S_z^2 \quad (2.41c)$$

$$\Delta S_{ij} \Delta S_{ij} = \Delta S_{\theta}^2 + \Delta S_r^2 + \Delta S_z^2 \quad (2.41d)$$

$$S_{ij} \Delta S_{ij} = S_{\theta} \Delta S_{\theta} + S_r \Delta S_r + S_z \Delta S_z \quad (2.41e)$$

where  $S_{\theta}$ ,  $S_r$ ,  $S_z$  and  $\Delta S_{\theta}$ ,  $\Delta S_r$ ,  $\Delta S_z$  are defined in (2.27). Thus, Eq. (2.41b) can be written as a second-degree algebra equation

$$AR^2 + BR + C = 0 \quad (2.42a)$$

where  $A = \Delta S_{ij}^{(e)} \Delta S_{ij}^{(e)} = (\Delta S_{\theta}^{(e)})^2 + (\Delta S_r^{(e)})^2 + (\Delta S_z^{(e)})^2 \quad (2.42b)$

$$B = 2S_{ij}^{(1)} \Delta S_{ij}^{(e)} = 2(S_{\theta}^{(1)} \Delta S_{\theta}^{(e)} + S_r^{(1)} \Delta S_r^{(e)} + S_z^{(1)} \Delta S_z^{(e)}) \quad (2.42c)$$

$$C = S_{ij}^{(1)} S_{ij}^{(1)} - \frac{2}{3} \sigma_{Y1}^2 = (S_{\theta}^{(1)})^2 + (S_r^{(1)})^2 + (S_z^{(1)})^2 - \frac{2}{3} \sigma_{Y1}^2 \quad (2.42d)$$

Obviously, just the positive root of Eq. (2.42a) is applicable and it follows that if

$R = 0$  the increment is entirely plastic;

$R = 1$  the increment is entirely elastic;

$0 < R < 1$  the increment is elastic-plastic.

5. Now supply the increments

$$\sigma_{ij}^{(1)} \rightarrow \sigma_{ij}^{(1)} + R\Delta\sigma_{ij}^{(e)}$$

(2.43)

$$\varepsilon_{ij}^{(1)} \rightarrow \varepsilon_{ij}^{(1)} + R\Delta\varepsilon_{ij}^{(1)}$$

to get to the yield surface.

6. Let

$$\Delta p^* = (1-R)\Delta p$$

(2.44)

$$\Delta T^* = (1-R)\Delta T$$

7. Solve for  $\Delta\sigma_{ij}$  and  $\Delta\varepsilon_{ij}$  using the formulas of plastic solution (2.32) and (2.33), in which  $\Delta p^*$  and  $\Delta T^*$  replace  $\Delta p$  and  $\Delta T$  respectively.

Note that the  $\Delta\sigma_{ij}$  and  $\Delta\varepsilon_{ij}$  here should be consistent with the constitutive relations (2.28b). Consequently

$$\Delta\sigma_{\theta} = \left( \frac{1+ka_2^0}{1+ka_1^0} \right) \frac{r}{t} \Delta p^* + \left( \frac{a_T^0 k \frac{r}{t}}{1+a_1^0 k \frac{r}{t}} \right) \alpha \Delta T^* \quad (2.45a)$$

$$\Delta\sigma_r = -\Delta p^* \quad (2.45b)$$

$$\Delta\sigma_z = c_1 \Delta\sigma_{\theta} + c_2 \Delta\sigma_r + c_T \alpha \Delta T^* \quad (2.45c)$$

and

$$\Delta \epsilon_{\theta} = a_1^0 \Delta \sigma_{\theta} + a_2^0 \Delta \sigma_r - a_T^0 \alpha \Delta T^* \quad (2.46a)$$

$$\Delta \epsilon_r = b_1^0 \Delta \sigma_{\theta} + b_2^0 \Delta \sigma_r - b_T^0 \alpha \Delta T^* \quad (2.46b)$$

$$\Delta \epsilon_z = 0 \quad (2.46c)$$

8. Compute effective plastic strain from (2.37a) and (2.37b) as

$$\overline{\Delta \epsilon}^P = \frac{1}{\overline{\sigma} \left(1 + \frac{H'}{3G}\right)} \left[ \overline{S}_{\theta} (\Delta \epsilon_{\theta} - \alpha \Delta T^*) + \overline{S}_r (\Delta \epsilon_r - \alpha \Delta T^*) + \overline{S}_z (\Delta \epsilon_z - \alpha \Delta T^*) \right]$$

$$\Delta \epsilon_{ij}^P = \frac{1}{2\overline{\sigma}^2 \left(1 + \frac{H'}{3G}\right)} \left[ \overline{S}_{\theta} (\Delta \epsilon_{\theta} - \alpha \Delta T^*) + \overline{S}_r (\Delta \epsilon_r - \alpha \Delta T^*) + \overline{S}_z (\Delta \epsilon_z - \alpha \Delta T^*) \right] S_{ij}$$

9. Use above results to update the values of the previous step.

$$\sigma_{ij}^{(2)} = \sigma_{ij}^{(1)} + \Delta \sigma_{ij} \quad (2.47a)$$

$$\epsilon_{ij}^{(2)} = \epsilon_{ij}^{(1)} + \Delta \epsilon_{ij} \quad (2.47b)$$

$$\epsilon_{ij}^{P(2)} = \epsilon_{ij}^{P(1)} + \Delta \epsilon_{ij}^P \quad (2.47c)$$

$$\overline{\epsilon}^P(2) = \overline{\epsilon}^P(1) + \overline{\Delta \epsilon}^P \quad (2.47d)$$

$$\sigma_Y^{(2)} = \sigma_Y^{(1)} + H' \overline{\Delta \epsilon}^P(2) \quad (2.47e)$$

10. Go back to Step 1 for the next load increment.

11. Use the following relations to check the consistency of the results after each loading cycle

$$\sigma_{\theta}^C = \frac{E}{(1+\mu)(1-2\mu)} \left[ (1-\mu)(\epsilon_{\theta} - \epsilon_{\theta}^P - \alpha\Delta T) + \mu(\epsilon_r - \epsilon_r^P - \alpha\Delta T) + \mu(\epsilon_z - \epsilon_z^P - \alpha\Delta T) \right]$$

$$\sigma_z^C = \frac{E}{(1+\mu)(1-2\mu)} \left[ (1-\mu)(\epsilon_z - \epsilon_z^P - \alpha\Delta T) + \mu(\epsilon_{\theta} - \epsilon_{\theta}^P - \alpha\Delta T) + \mu(\epsilon_r - \epsilon_r^P - \alpha\Delta T) \right]$$

(2.48)

If  $\sigma_{ij}^C$  are consistent with computational values  $\sigma_{ij}$ , the results are correct.

### CHAPTER 3 TEMPERATURE DEPENDENT ANALYSIS

#### 3.1 Modification to Governing Equations

In the previous chapter we have been concerned with a solution which took no account of the change of material properties as the temperature varied. In fact, physical properties of Algoma Steel MN-80 which is representative of steels from which Canadian casings for thermal recovery wells are made, show that initial yield limit  $\sigma_{Y0}$ , modulus of elasticity  $E$ , tangent modulus  $E^t$ , and thermal expansion coefficient  $\alpha$ , all vary with the temperature. Adding these variations of properties to the computation is necessary.

We first consider the elastic phase in which  $\sigma_{Y0}$ ,  $E$  and  $\alpha$  are functions of temperature, while Poisson ratio  $\mu$  is temperature independent.

The total strain can be expressed with the differential

$$d\epsilon = d\epsilon^e + d\epsilon^T = d\epsilon^e + [\alpha]dT \quad (3.1)$$

From Hooke's Law

$$\sigma = D\epsilon^e$$

and 
$$\epsilon^e = D^{-1} \sigma .$$

Differentiating with respect to temperature  $T$

$$d\epsilon^e = \frac{dD^{-1}}{dT} \sigma dT + D^{-1} d\sigma \quad (3.2)$$

Substituting (3.2) into (3.1) and solving for  $d\sigma$  we have

$$d\sigma = D[d\varepsilon - d\varepsilon^T - \frac{dD^{-1}}{dT} \sigma dT]$$

This can be written as

$$d\sigma = D[d\varepsilon - [\alpha]^* dT] = D[d\varepsilon - d\varepsilon^{*T}] \quad (3.3)$$

where  $[\alpha]^*$  is a matrix which has the following increment form and

$$d\varepsilon^{*T} \equiv [\alpha]^* dT. \quad (3.4a)$$

$$[\alpha]^* = \begin{bmatrix} \alpha_\theta^* \\ \alpha_r^* \\ \alpha_z^* \end{bmatrix} = \begin{bmatrix} \alpha \\ \alpha \\ \alpha \end{bmatrix} + \frac{1}{\Delta T \Delta E} \begin{bmatrix} 1 & -\mu & -\mu \\ -\mu & 1 & -\mu \\ -\mu & -\mu & 1 \end{bmatrix} \begin{bmatrix} \sigma_\theta \\ \sigma_r \\ \sigma_z \end{bmatrix} \quad (3.4b)$$

in which  $\Delta E = E^{T+\Delta T} - E^T$ .

Now the constitutive relations (1.12) of Chapter 1 are replaced by the following

$$\begin{bmatrix} \Delta\sigma_\theta \\ \Delta\sigma_r \\ \Delta\sigma_z \end{bmatrix} = \frac{E}{(1+\mu)(1-2\mu)} \begin{bmatrix} 1-\mu & \mu & \mu \\ \mu & 1-\mu & \mu \\ \mu & \mu & 1-\mu \end{bmatrix} \begin{bmatrix} \Delta\varepsilon_\theta - \alpha_\theta^* \Delta T \\ \Delta\varepsilon_r - \alpha_r^* \Delta T \\ \Delta\varepsilon_z - \alpha_z^* \Delta T \end{bmatrix} \quad (3.5)$$

For elastic solution and prediction of initial yield the computational procedure of using the temperature dependent model can obviously follow the procedure described previously. That is, formulas (1.16) of Chapter 1 and formulas (2.2) and (2.3) of Chapter 2 are still applicable except that the



thermal expansion coefficient  $[\alpha]$  should be replaced by  $[\alpha]^*$  of (3.4b), and it is necessary to use an increment form in the computation.

For the plastic temperature dependent problem we modify Eq. (2.19), namely,

$$d\sigma_{ij} = C_{ijkl}(d\epsilon_{kl} - d\epsilon_{kl}^p - d\epsilon_{kl}^T)$$

as follows. First,  $d\epsilon_{kl}^{*T}$  of (3.4a) should replace  $d\epsilon_{kl}^T$ , with  $d\epsilon_{kl}^{*T}$  defined in (3.4)

$$d\epsilon_{kl}^{*T} = \alpha_{kl}^* dT \quad (3.6)$$

As yield now depends on the temperature, the Mises hardening rule should be written as

$$\bar{\sigma} = \sigma_Y \left( \int d\bar{\epsilon}_p, T \right) \quad (3.7)$$

It follows that the consistency condition (2.21) needs to be modified such that

$$\frac{\partial f}{\partial \sigma_{ij}} d\sigma_{ij} + \frac{\partial f}{\partial \sigma_Y} d\sigma_Y = 0$$

in which  $d\sigma_Y = \frac{\partial \sigma_Y}{\partial \bar{\epsilon}_p} d\bar{\epsilon}_p + \frac{\partial \sigma_Y}{\partial T} dT$  (3.8)

Then

$$\frac{\partial f}{\partial \sigma_{ij}} d\sigma_{ij} + \frac{\partial f}{\partial \sigma_Y} \left( \frac{\partial \sigma_Y}{\partial \bar{\epsilon}_p} d\bar{\epsilon}_p + \frac{\partial \sigma_Y}{\partial T} dT \right) = 0$$

Following the same procedure as from (2.21) to (2.23) of Chapter 2 the plastic strain is given as

$$d\epsilon_{mn}^p = \frac{\frac{\partial f}{\partial \sigma_{mn}} \frac{\partial f}{\partial \sigma_{ij}} C_{ijkl}}{\frac{\partial f}{\partial \sigma_{ij}} C_{ijkl} \frac{\partial f}{\partial \sigma_{kl}} + H'} (d\epsilon_{kl} - d\epsilon_{kl}^{*T}) - \frac{\frac{\partial f}{\partial \sigma_{mn}} \frac{\partial \sigma_Y}{\partial T} dT}{\frac{\partial f}{\partial \sigma_{ij}} C_{ijkl} \frac{\partial f}{\partial \sigma_{kl}} + H'}$$

$$\equiv P_{mnkl} (d\epsilon_{kl} - d\epsilon_{kl}^{*T}) - \frac{\frac{\partial f}{\partial \sigma_{mn}} \frac{\partial \sigma_Y}{\partial T} dT}{\frac{\partial f}{\partial \sigma_{ij}} C_{ijkl} \frac{\partial f}{\partial \sigma_{kl}} + H'} \quad (3.9)$$

The expressions from (2.24a) to (2.28a) of Chapter 2 are still applicable and we obtain the constitutive relations with thermoplastic material as follows

$$d\sigma = D^{EP} (d\epsilon - d\epsilon^{*T}) + d\sigma^{*T} \quad (3.10)$$

in which 
$$d\sigma_{ij}^{*T} = \frac{3 \frac{\partial \sigma_Y}{\partial T} dT}{2(1 + \frac{H'}{3G}) \sigma^2} S_{ij}$$

or 
$$\begin{bmatrix} d\sigma_{\theta}^{*T} \\ d\sigma_r^{*T} \\ d\sigma_z^{*T} \end{bmatrix} = \frac{3 \frac{\partial \sigma_Y}{\partial T} dT}{2(1 + \frac{H'}{3G}) \sigma^2} \begin{bmatrix} \bar{S}_{\theta} \\ \bar{S}_r \\ \bar{S}_z \end{bmatrix}$$

(For  $\bar{S}_{\theta}, \bar{S}_r, \bar{S}_z$  see (2.27))

$$d\epsilon_{ij}^{*T} = \alpha_{ij}^* dT \quad (3.11)$$

$$\text{or } \begin{bmatrix} d\varepsilon_{\theta}^{*T} \\ d\varepsilon_r^{*T} \\ d\varepsilon_z^{*T} \end{bmatrix} = dT \begin{bmatrix} \alpha_{\theta}^* \\ \alpha_r^* \\ \alpha_z^* \end{bmatrix}$$

in which the  $\alpha^*$  are specified by (3.4b). The elastic-plastic matrix  $D^{EP}$  now has the same form as in (2.28), namely,

$$D^{EP} = D^E - D^P = \begin{bmatrix} \lambda' & \lambda & \lambda \\ \lambda & \lambda' & \lambda \\ \lambda & \lambda & \lambda' \end{bmatrix} - \begin{bmatrix} s_{\theta\theta} & s_{\theta r} & s_{\theta z} \\ s_{r\theta} & s_{rr} & s_{rz} \\ s_{z\theta} & s_{zr} & s_{zz} \end{bmatrix}$$

Once the constitutive relations (3.10) are established it is clear that the formulas (2.32), (2.33) and (2.34) with the computational procedure in 2.3 of Chapter 2 can be exactly used to solve for temperature dependent problems as long as we define  $(d\sigma - d\sigma^{*T})$  as an unknown. The only difference is to consider  $\sigma_{Y0}$ ,  $E$ ,  $E^t$  and  $\alpha$  to be the functions of temperature. As a consequence, the final results will come from adding  $d\sigma^{*T}$  which can be determined without any difficulty.

### 3.2 Solution Strategy

In the above section the method for temperature dependent problems was presented. The temperature effect is treated by replacing the thermal expansion coefficient  $\alpha$  by an "effective thermal expansion coefficient  $\alpha_{ij}^*$ " and the stress response  $d\sigma$  is replaced by "effective thermal stress  $(d\sigma - d\sigma^{*T})$ ". For each temperature increment we have

$$\begin{bmatrix} \alpha_{\theta}^* \\ \alpha_r^* \\ \alpha_z^* \end{bmatrix} = \begin{bmatrix} \alpha \\ \alpha \\ \alpha \end{bmatrix} + \frac{1}{\Delta T \Delta E} \begin{bmatrix} 1 & -\mu & -\mu \\ \mu & 1 & -\mu \\ -\mu & -\mu & 1 \end{bmatrix} \begin{bmatrix} \sigma_{\theta} \\ \sigma_r \\ \sigma_z \end{bmatrix} \quad (3.12)$$

$$\begin{bmatrix} d\sigma_{\theta}^{*T} \\ d\sigma_r^{*T} \\ d\alpha_z^{*T} \end{bmatrix} = \frac{3 \frac{\partial \sigma_Y}{\partial T} \Delta T}{2 \left(1 + \frac{H'}{3G}\right) \sigma^2} \begin{bmatrix} \bar{s}_{\theta} \\ \bar{s}_r \\ \bar{s}_z \end{bmatrix} \quad (3.13)$$

However, there exists a simple alternative way to deal with the same problem. If the change of physical properties with temperature is gradual and, in addition, the step size of the temperature increment is small enough, we can drop the derivative terms  $dD^{-1}/dT$  in (3.2) and  $\partial \sigma_Y / \partial T$  in (3.8). Instead, we may use directly a Lagrangian interpolation of  $\sigma_{Y0}$ ,  $E$ ,  $E^T$  and  $\alpha$ , which are tabulated functions of temperature. The error caused from this procedure would be acceptable as long as a small step size can be guaranteed. This simple method is adopted in the current program CASEPT-1 of temperature dependent problems while the former, in which the derivative terms are taken into account, will be employed in the creep model, i.e., the time dependent problem to be addressed subsequently.

The strategy employed with Lagrangian interpolation is shown in Fig. 3.1. The available data for MN80 steel, for  $\sigma_{Y0}$ ,  $E$ ,  $E^t$  and  $\alpha$ , have been tabulated. We use one-dimensional interpolation for  $E$  and  $\sigma_{Y0}$ , curve fitting for  $\alpha$ , and two-dimensional interpolation for  $E^t$ . This is because the plastic modulus  $E^t$  is a bivariate function of temperature  $T$  and total strain  $\epsilon$  (or effective strain  $\bar{\epsilon}$ ), i.e.,

$$E^t = E^t(T, \varepsilon)$$

(3.14)

The two-dimensional interpolation scheme for  $E^t$  will be given in Chapter 5.

## CHAPTER 4 TIME DEPENDENT ANALYSIS - CREEP

### 4.1 Creep Phenomena and Their Mathematical Modeling

As we know, the creep problem refers to time dependent deformation and (or) failure of components and structures. Time dependence is the chief characteristic of creep. A typical creep curve consists of three regimes, designated as primary, secondary and tertiary, as shown in Fig. 4.1. The instantaneous response to  $\epsilon_0$ , at  $t=0$ , depends on the magnitude of the stress and can be elastic or elastic-plastic. The creep curve is strongly influenced by stress and temperature. The influence of stress at constant temperature is shown in Fig. 4.2 while the influence of temperature at constant stress is shown in Fig. 4.3. We note that both show thresholds below which no noticeable creep is observed.

For casings for thermal wells, typical working conditions are  $T = 350^\circ\text{C}$  and  $P = 16.5 \text{ MPa}$ , and the operating period, in general, lasts about three months. So the primary and secondary creep are important in such problems.

To model the creep process in one dimension, most work is based on the equation of state approach, which is popular within the engineering community. An expression of the form called the Bailey-Norton Law (Harry, K., 1980) is

$$\epsilon^C = A \sigma^m t^n \quad (4.1)$$

Here  $\epsilon^C$  is the creep strain, and  $A$ ,  $m$  and  $n$  are constants that are a function of temperature. This law is intended to model primary and secondary creep only. If we ignore the time derivative of the stress the strain rate can be obtained by differentiating Eq. (4.1) with respect to time. This may result

in two forms (Harry, K., 1980). The time hardening formulation is

$$\dot{\epsilon}^c = \frac{\partial \epsilon^c}{\partial t} = A \sigma^n t^{n-1} \quad (4.2)$$

and the strain hardening formulation is

$$\dot{\epsilon}^c = A^{1/n} \sigma^{m/n} (\epsilon^c)^{(n-1)/n} \quad (4.3)$$

For multiaxial stress conditions it is assumed that the tensor of the creep strain rates can be written in terms of the stress deviator tensor as follows

$$\dot{\epsilon}_{ij}^c = \lambda s_{ij} \quad i, j = 1, 2, 3 \quad (4.4)$$

in which

$$s_{ij} = \sigma_{ij} - \frac{1}{3} \sigma_{kk} \delta_{ij}$$

and  $\lambda$  is a factor of proportionality, which is determined as (Harry, K., 1980)

$$\lambda = \frac{3}{2\bar{\sigma}} \frac{d\bar{\epsilon}^c}{dt} \quad (4.5)$$

Here  $\bar{\sigma}$  is the effective stress and  $\bar{\epsilon}^c$  is an effective creep strain, extended from the uniaxial creep model to the multiaxial case, and expressed as

$$\bar{\epsilon}^c = A \bar{\sigma}^m t^n \quad (4.6)$$

So, using (4.2) and (4.6) in (4.5), now we may write

$$\lambda = \frac{3}{2\bar{\sigma}} \frac{d\bar{\epsilon}^c}{dt} = \frac{3}{2} A n \bar{\sigma}^{m-1} t^{n-1} \quad (4.7)$$

Then the time hardening formation is represented by

$$\dot{\epsilon}_{ij}^c = \frac{3}{2} A n \bar{\sigma}^{m-1} t^{n-1} S_{ij} \quad (4.8)$$

and the strain hardening formation is rewritten, similarly, as

$$\dot{\epsilon}_{ij}^c = \frac{3}{2} A \frac{1}{n} \bar{\sigma}^{\left(\frac{m}{n} - 1\right)} \epsilon_c^{\left(\frac{n-1}{n}\right)} S_{ij} \quad (4.9)$$

Both the time hardening and strain hardening rules can be used for computation, however, the time hardening formation is easier to use than is the strain hardening formation. In this report we prefer to adopt the time hardening rule. In practice, we make use of another incremental form for computations, which stem directly from Eq. (4.4), in the form

$$\dot{\epsilon}_{ij}^c = \lambda S_{ij} = \frac{3}{2\bar{\sigma}} \frac{d\bar{\epsilon}^c}{dt} S_{ij}$$

This incremental form is approximately

$$\Delta \epsilon_{ij}^c = \frac{3}{2\bar{\sigma}} \frac{\Delta \bar{\epsilon}^c}{\Delta t} S_{ij} \Delta t = \frac{3}{2\bar{\sigma}} \Delta \bar{\epsilon}^c S_{ij} \quad (4.10)$$

For small changes of stress the formula is valid to obtain creep strain increments using an iterative process.



#### 4.2 Formulation of the Creep Problem for Casings

We note first that the total strain, in tensor form, can be decomposed as

$$\varepsilon_{ij} = \varepsilon_{ij}^e + \varepsilon_{ij}^p + \varepsilon_{ij}^c + \varepsilon_{ij}^T \quad (4.11a)$$

or, in vector form,

$$\varepsilon = \varepsilon^e + \varepsilon^p + \varepsilon^c + \varepsilon^T \quad (4.11b)$$

in which  $\varepsilon_{ij}^T = \alpha \Delta T \sigma_{ij}$

The incremental form is

$$\Delta\varepsilon = \Delta\varepsilon^e + \Delta\varepsilon^p + \Delta\varepsilon^c + \Delta\varepsilon^T \quad (4.11c)$$

where the superscripts e and p refer to the elastic strain and plastic strain, respectively, which have been obtained from Chapter 3. The superscript c refers to the creep strain which is unknown. The constant,  $\alpha$ , is the coefficient of linear thermal expansion and  $\Delta T$  is the temperature change.

From (4.11c) we have

$$\Delta\varepsilon^e = \Delta\varepsilon - \Delta\varepsilon^p - \Delta\varepsilon^c - \Delta\varepsilon^T \quad (4.12a)$$

and

$$\Delta\sigma = D\Delta\varepsilon^e = D[\Delta\varepsilon - \Delta\varepsilon^p - \Delta\varepsilon^c - \Delta\varepsilon^T] \quad (4.12b)$$

Expressing (4.12b) in matrix form, yields

$$\begin{bmatrix} \Delta\sigma_{\theta} \\ \Delta\sigma_r \\ \Delta\sigma_z \end{bmatrix} = \frac{E(T)}{(1+\mu)(1-2\mu)} \begin{bmatrix} 1-\mu & \mu & \mu \\ \mu & 1-\mu & \mu \\ \mu & \mu & 1-\mu \end{bmatrix} \begin{bmatrix} \Delta\epsilon_{\theta} - \Delta\epsilon_{\theta}^P - \Delta\epsilon_{\theta}^C - \alpha \Delta T \\ \Delta\epsilon_r - \Delta\epsilon_r^P - \Delta\epsilon_r^C - \alpha \Delta T \\ \Delta\epsilon_z - \Delta\epsilon_z^P - \Delta\epsilon_z^C - \alpha \Delta T \end{bmatrix} \quad (4.13)$$

in which  $E(T)$  is the temperature dependent elastic modulus.

As in Chpaters 1 to 3, we assume that the total axial strain  $\epsilon_z = 0$ . The components of stress in the matrix (4.13) can then be expressed as

$$\Delta\sigma_{\theta} = \lambda' \Delta\epsilon_{\theta} + \lambda \Delta\epsilon_r + a^C \quad (4.14a)$$

$$\Delta\sigma_r = \lambda \Delta\epsilon_{\theta} + \lambda' \Delta\epsilon_r + b^C \quad (4.14b)$$

$$\Delta\sigma_z = \lambda \Delta\epsilon_{\theta} + \lambda \Delta\epsilon_r + c^C \quad (4.14c)$$

The the inverse form is

$$\Delta\epsilon_{\theta} = a_1^C \Delta\sigma_{\theta} + a_2^C \Delta\sigma_r + a_T^C \quad (4.15a)$$

$$\Delta\epsilon_r = b_1^C \Delta\sigma_{\theta} + b_2^C \Delta\sigma_r + b_T^C \quad (4.15b)$$

$$\Delta\epsilon_z = 0 \quad (4.15c)$$

in which

$$\lambda = E(T) \mu / (1+\mu)(1-2\mu) \quad (4.16a)$$

$$\lambda' = E(T)(1-\mu) / (1+\mu)(1-2\mu) \quad (4.16b)$$

$$a^c = -\lambda' (\Delta\varepsilon_\theta^c + \Delta\varepsilon_\theta^p + \alpha\Delta T) - \lambda (\Delta\varepsilon_r^c + \Delta\varepsilon_z^c + \Delta\varepsilon_r^p + \Delta\varepsilon_z^p + 2\alpha\Delta T) \quad (4.16c)$$

$$b^c = -\lambda' (\Delta\varepsilon_r^c + \Delta\varepsilon_r^p + \alpha\Delta T) - \lambda (\Delta\varepsilon_\theta^c + \Delta\varepsilon_z^c + \Delta\varepsilon_\theta^p + \Delta\varepsilon_z^p + 2\alpha\Delta T) \quad (4.16d)$$

$$c^c = -\lambda' (\Delta\varepsilon_z^c + \Delta\varepsilon_z^p + \alpha\Delta T) - \lambda (\Delta\varepsilon_\theta^c + \Delta\varepsilon_r^c + \Delta\varepsilon_\theta^p + \Delta\varepsilon_r^p + 2\alpha\Delta T) \quad (4.16e)$$

and

$$a_1^c = \frac{\lambda'}{\lambda'^2 - \lambda^2} = b_2^c \quad (4.17a)$$

$$a_2^c = \frac{-\lambda}{\lambda'^2 - \lambda^2} = b_1^c \quad (4.17b)$$

$$a_T^c = -a_1^c a^c - a_2^c b^c \quad (4.17c)$$

$$b_T^c = -b_1^c a^c - b_2^c b^c \quad (4.17d)$$

In the above coefficients, just the creep strains  $\Delta\varepsilon_\theta^c$ ,  $\Delta\varepsilon_r^c$  and  $\Delta\varepsilon_z^c$  are unknown.

Recalling the equilibrium equations for the casing system (i.e. Eqn. (1.9)) in incremental form,

$$\Delta\sigma_\theta = (\Delta p - K \Delta\varepsilon_\theta) \frac{r}{t} \quad (4.18a)$$

$$\Delta\sigma_r = - \Delta p \quad (4.18b)$$

Substituting the expression of  $\Delta\varepsilon_\theta$  in (4.15) into Eq. (4.18) yields

$$\Delta\sigma_\theta = \left( \frac{1 + K a_2^c}{1 + K \frac{r}{t} a_1^c} \right) \frac{r}{t} \Delta p - \frac{K \frac{r}{t} a_T^c}{1 + K \frac{r}{t} a_1^c} \quad (4.19a)$$

$$\Delta\sigma_r = - \Delta p \quad (4.19b)$$

$$\Delta\sigma_z = \lambda (a_1^c + b_1^c) \Delta\sigma_\theta + \lambda (a_2^c + b_2^c) \Delta\sigma_r + \lambda (a_T^c + b_T^c) + c^c \quad (4.19c)$$

It is clear that as long as  $\Delta\varepsilon_{ij}^c$  is formulated properly the problem can be solved.

### 4.3 Calculation Procedure for Creep Prediction

Since the creep strains involved in the computation formulas (4.19) depend on the stresses, through the correspondent coefficients, the process in Sect. 4.3 is nonlinear. To solve the problem the widely accepted method of initial strains is employed.

The elastic-plastic responses, without regard for creep are first computed. That is to say, the loading is applied at the instant,  $t=0$ , when time-dependent creep has not yet occurred. It then becomes possible to solve for the first time interval. The size of this interval depends on the particular problem and the principal considerations are that the computation be convergent and that the stress change be kept small in any one interval.

Next, the procedure for computing creep begins with some initial estimates of the creep increments during the first interval. The sequential computation steps are indicated below.

1. Chose an appropriate time increment  $\Delta t$  in hours. The initial accumulated creep strains  $\epsilon_{\theta}^C$ ,  $\epsilon_r^C$  and  $\epsilon_z^C$  are taken as zero, namely

$$\epsilon_{\theta}^C = \epsilon_r^C = \epsilon_z^C = 0$$

while the initial estimates of creep strain increments are prescribed. For example, assume

$$\Delta \epsilon_{\theta}^C = 2.5E - 3$$

$$\Delta \epsilon_r^C = 4.0E - 4$$

$$\Delta \epsilon_z^C = -1.0E - 3$$

These estimates are not critical since subsequent iterations quickly produce the correct values. However, it should be noted that, if the estimates for the three components are equal in magnitude to one another, the computation will fail.

To make the estimates in the first time period be reasonable, the following procedure may be used. Assume that the stresses do not change during the time interval. Then  $\sigma_{\theta}$ ,  $\sigma_r$ ,  $\sigma_z$  and  $\bar{\sigma}$ , are known from the previous elastic-plastic computation. The value of  $\bar{\sigma}$  is used together with  $\Delta t$  in the incremental form of Eq. (4.6), which is

$$\Delta \bar{\epsilon}^C = A \bar{\sigma}^m n t^{n-1} \Delta t \tag{4.20}$$

to obtain the effective creep strain increment  $\Delta \bar{\epsilon}^C$ . Eq. (4.10) is then used,

together with this effective creep strain and the stresses, to obtain estimates of incremental creep strain components, namely,  $\Delta \epsilon_{\theta}^C$ ,  $\Delta \epsilon_r^C$  and  $\Delta \epsilon_z^C$ . These initial estimates of the creep strain increments are now used to recompute

$$\bar{\Delta \epsilon}^C = \frac{\sqrt{2}}{3} \sqrt{(\Delta \epsilon_{\theta}^C - \Delta \epsilon_r^C)^2 + (\Delta \epsilon_r^C - \Delta \epsilon_z^C)^2 + (\Delta \epsilon_z^C - \Delta \epsilon_{\theta}^C)^2} \quad (4.21)$$

2. Solve for  $\Delta \sigma_{\theta}$ ,  $\Delta \sigma_r$  and  $\Delta \sigma_z$  from Eqs. (4.15), then

$$\sigma_{\theta} = \sigma_{\theta}^{(o)} + \Delta \sigma_{\theta}$$

$$\sigma_r = \sigma_r^{(o)} + \Delta \sigma_r$$

$$\sigma_z = \sigma_z^{(o)} + \Delta \sigma_z$$

Here  $\sigma_{\theta}^{(o)}$ ,  $\sigma_r^{(o)}$  and  $\sigma_z^{(o)}$  refer to the instantaneous stresses at  $t=0$ . The values of  $\sigma_{ij}$  are used to obtain the deviatoric stress tensor

$$S_{ij} = \sigma_{ij} - \frac{1}{3} \sigma_{KK} \delta_{ij}$$

3. Compute the effective  $\bar{\sigma}$  using a creep law like Eq. (4.20). That is,

$$\bar{\sigma} = \left( \frac{\bar{\Delta \epsilon}^C}{A n t^{n-1} \Delta t} \right)^{\frac{1}{m}} \quad (4.22)$$

Note that one doesn't use the values from step 2 for computing  $\bar{\sigma}$ .

4. Using the values of  $\bar{\sigma}$ ,  $\bar{\Delta \epsilon}^C$  and  $S_{ij}$ , compute a new effective creep strain increment

$$\Delta \epsilon_{ij}^c = \frac{3}{2\bar{\sigma}} \Delta \bar{\epsilon}^c S_{ij}$$

5. Repeat step 1 through step 4 using the new values of  $\Delta \epsilon_{ij}^c$  as a starting point until the components of the creep strain increments converge.

The convergence criterion is

$$\left| \Delta \epsilon_{ij}^{c_0} - \Delta \epsilon_{ij}^c \right| < \epsilon \quad (4.23)$$

If the iteration process fails to converge, it is necessary to change the initial estimates of the creep strain increments or shorten the time interval.

After convergent solutions are reached we have

$$\sigma_{ij}^{(1)} = \sigma_{ij}^{(0)} + \Delta \sigma_{ij} \quad (4.24a)$$

$$\epsilon_{ij}^{(1)} = \epsilon_{ij}^{(0)} + \Delta \epsilon_{ij}^c \quad (4.24b)$$

in which  $\sigma_{ij}^{(0)}$  and  $\epsilon_{ij}^{(0)}$  are elastic-plastic responses.

6. Another time increment is added and step 1 is repeated. But this time  $\epsilon_{\theta}^c$ ,  $\epsilon_r^c$  and  $\epsilon_z^c$  were no longer zero and the initial values for iteration  $\Delta \epsilon_{\theta}^c$ ,  $\Delta \epsilon_r^c$  and  $\Delta \epsilon_z^c$  should be taken to be equal to those obtained in the previous interval, while the  $\sigma_{\theta}^{(0)}$ ,  $\sigma_r^{(0)}$  and  $\sigma_z^{(0)}$  in Step 2 are replaced by those from convergent solution of Step 5.

7. Continue until the required time span for solution has been completed.

The above procedure is shown in the block diagram of Fig. 4.4.

Since creep data are not available for casing steel, we have selected the

five creep expressions from the monographs and papers which are listed in Table 4.1. Preliminary calculations for each indicate that just two among them are adaptable to casing systems. These are: formula (2) in Table 4.1,

$$\bar{\epsilon}^c = 1.5 * 10^{-32} * \bar{\sigma}^{6.0} * t^{2/3}$$

and, formula (4) in Table 4.1

$$\bar{\epsilon}^c = 2.72 * 10^{-18} * \bar{\sigma}^{-2.99} * t^{0.45}.$$

In these equations stress is in psi and time is in hours.

The results using these two formulas in the computer program for the creep prediction do not show much difference.



Table 4.1 Listing of Creep Laws

Test Objective	(1) 1980 H. Kraus	(2) 1980 H. Kraus	(3) 1983 "Design for Creep"	(4) 1982 "Design for Creep"	(5) 1980 D. Snyder, K. Bathe
Pressure vessel with an ellipsoidal head Internal pressure=3 MPa	Stress redistribution in a rotating disc	Creep of a beam in bending	Creep of an internally pressurized thick tube	A thick-wall cylinder Internal pressure=25.5 MPa	
Test Material	Poisson's Ratio = 0.3	12% Chrome steel	Commercially pure aluminum	0.19% Carbon steel	Yield stress = 70 Mpa
Young's Modulus	140 GPa	126 GPa	70 GPa	170 GPa	157 GPa
Test Temperature	Room temperature	530°C	Room temperature	450°C	550°C
Creep Law	$19.8 \cdot 10^{-16} \bar{\sigma}^{3.61} \cdot t^{1.06}$	$15 \cdot 10^{-32} \cdot \bar{\sigma}^{6.0} \cdot t^{2/3}$	$2.21 \cdot 10^{-15} \cdot \bar{\sigma}^{3.2} \cdot t^{0.3}$	$2.72 \cdot 10^{-18} \bar{\sigma}^{2.99} \cdot t^{0.45}$	$F \cdot (1 - e^{-Rt}) + Gt$
$\bar{\epsilon}^c =$					$\left\{ \begin{array}{l} F=1.608 \cdot 10^{-10} \cdot \bar{\sigma}^{-1.843} \\ R=5.929 \cdot 10^{-5} \\ * e^{2.029 \cdot 10^{-4} \bar{\sigma}} \\ G=6.73 \cdot 10^{-9} \end{array} \right.$
Comment	Applied to small load short time	Applied to casing system		Applied to casing system	

## CHAPTER 5 STRUCTURE OF PROGRAM CASEPC

### 5.1 Flow of Operation

In this section we shall consider some aspects of the computer program, called CASEPC for CAsing Elastic-Plastic and Creep analysis, which can be utilized to predict creep response and the elastic-plastic stresses and strains for both loading and unloading. As mentioned in the previous chapters, the program has two main functions. The first function is to determine elastic-plastic solutions, while the second is to predict creep effects in combination with elastic-plastic response. Both of these types of solutions are temperature dependent.

A schematic chart of the program is shown in Fig. 5.1.

### 5.2 Structure of Program

Program CASEPC is arranged in 27 subprogram segments. The main program contains just eight subroutines which are supported by the other subroutines. A flow chart of CASEPC is given in Fig. 5.2.

In the program CASEPC the two most important segments are subroutine EPSOL and subroutine CRSOL. The first is specially designed to deal with elastic-plastic stress analysis including loading and unloading. The flow chart of this subroutine is shown as in Fig. 5.3. The second is responsible for creep computation which is shown in the block diagram of Fig. 5.4.

### 5.3 Additional Subroutines for Temperature Dependent Solutions

To take account of the effect of temperature on material properties curve fitting is utilized for  $E$  and  $\alpha$ . One-dimensional interpolation is used to find  $\sigma_{y_0}$ . As we have noted the tangent modulus  $E^t$  is a bivariate function of

temperature and strain, and two-dimensional interpolation is employed. Here we shall give some explanation.

For the initial yield limit of MN80 steel,  $\sigma_{Y0}$ , the interpolation nodes are given as in Table 5.1.

Table 5.1 Temperature Dependence of Yield Point for MN80 Steel

T°C	0°	21°	93°	204°	316°	427°
$\sigma_{Y0}$ (MPa)	622	610	570	500	420	355

The computational formula used in program is two-dimensional Lagrangian interpolation through three points at unequal intervals. That is

$$\sigma_Y(T) = \sum_{k=i}^{i+2} \left( \prod_{\substack{j=i \\ (j \neq k)}}^{i+2} \frac{T - T_j}{T_K - T_j} \right) \sigma(T_K) \quad (5.1)$$

in which

$$i = \begin{matrix} 1 & T < T_2 \\ S & T_s < T < T_{s+1}, T - T_s > T_{s+1} - T, s = 2, 3, \dots, n-2 \\ S-1 & T_s < T < T_{s+1}, T - T_s > T_{s+1} - T, s = 2, 3, \dots, n-2 \\ n-2 & T > T_{n-1} \end{matrix} \quad (5.2)$$

In the program we take  $n=6$ .

For the tangent modulus  $E^t$ , stress values provided by C-FER are tabulated in Table A.6.2.1 of their Phase 1 report on THERWELL, as a function of strain and temperature. The tangent modulus is defined as the slope of the  $\bar{\sigma}-\bar{\epsilon}$  diagram in the plastic region. So we can calculate  $E^t$  for each strain increment, under different temperatures, using Table A6.2.1, as

$$E^t = \frac{\Delta\sigma}{\Delta\varepsilon} \tag{5.3}$$

The interpolation values are listed in Table 5.2.

The corresponding formula for computation is two dimensional Lagrangian interpolation through three points at unequal intervals for bivariate functions, namely,

$$E^t(T, \varepsilon) = \sum_{r=i}^{i+2} \sum_{s=j}^{j+2} \left( \prod_{\substack{k=i \\ (k \neq r)}}^{i+2} \frac{T - T_k}{T_r - T_k} \right) \left( \prod_{\substack{\ell=j \\ (\ell \neq s)}}^{j+2} \frac{\varepsilon - \varepsilon_\ell}{\varepsilon_s - \varepsilon_\ell} \right) E^t(T_r, S_r) \tag{5.4}$$

in which  $r = 1, 2, \dots, N$   $N=6$

$s = 1, 2, \dots, M$   $M=8$

$N$  denotes the number of interpolation nodes of the first argument,  $T$ .  $M$  denotes the number of interpolation nodes of the second argument,  $\varepsilon$ . Values of  $i$  and  $j$  are the same as in (5.2).

The interpolation subroutines are named respectively LAG13 and LAG23. The latter is for two-dimensional interpolation of  $E^t$ .

Table 5.2 Stress-Strain Relation for MN80 Steel at Different Temperatures

$\epsilon$ %	0°		21°		93°		204°		316°		427°	
	$E^t$ (GPa)	$\sigma$ (MPa)	$E^t$ (GPa)	$\sigma$ (MPa)	$E^t$ (GPa)	$\sigma$ (MPa)	$E^t$ (GPa)	$\sigma$ (MPa)	$E^t$ (GPa)	$\sigma$ (MPa)	$E^t$ (GPa)	$\sigma$ (MPa)
0.3	2.5	622	5.0	610	12.5	570	25.0	500	45.0	420	45.0	355
0.5	3.0	627	3.3	620	5.0	595	10.0	550	15.0	510	9.3	445
0.8	2.58	636	2.5	630	2.08	610	4.17	580	4.58	555	3.92	473
2.0	1.55	667	1.75	660	2.5	635	3.10	630	3.0	610	0.75	520
4.0	1.0	698	1.0	695	1.0	685	1.25	690	1.75	670	-3.5	535
6.0	0.45	718	0.25	715	-0.5	705	-1.25	715	0.25	705	-3.15	471
10.0	-0.9	725	-1.6	710	-4.0	660	-6.5	640	-1.75	705	-3.15	471
12.0	707		678		580		510		670		408	

1  
5  
4  
1

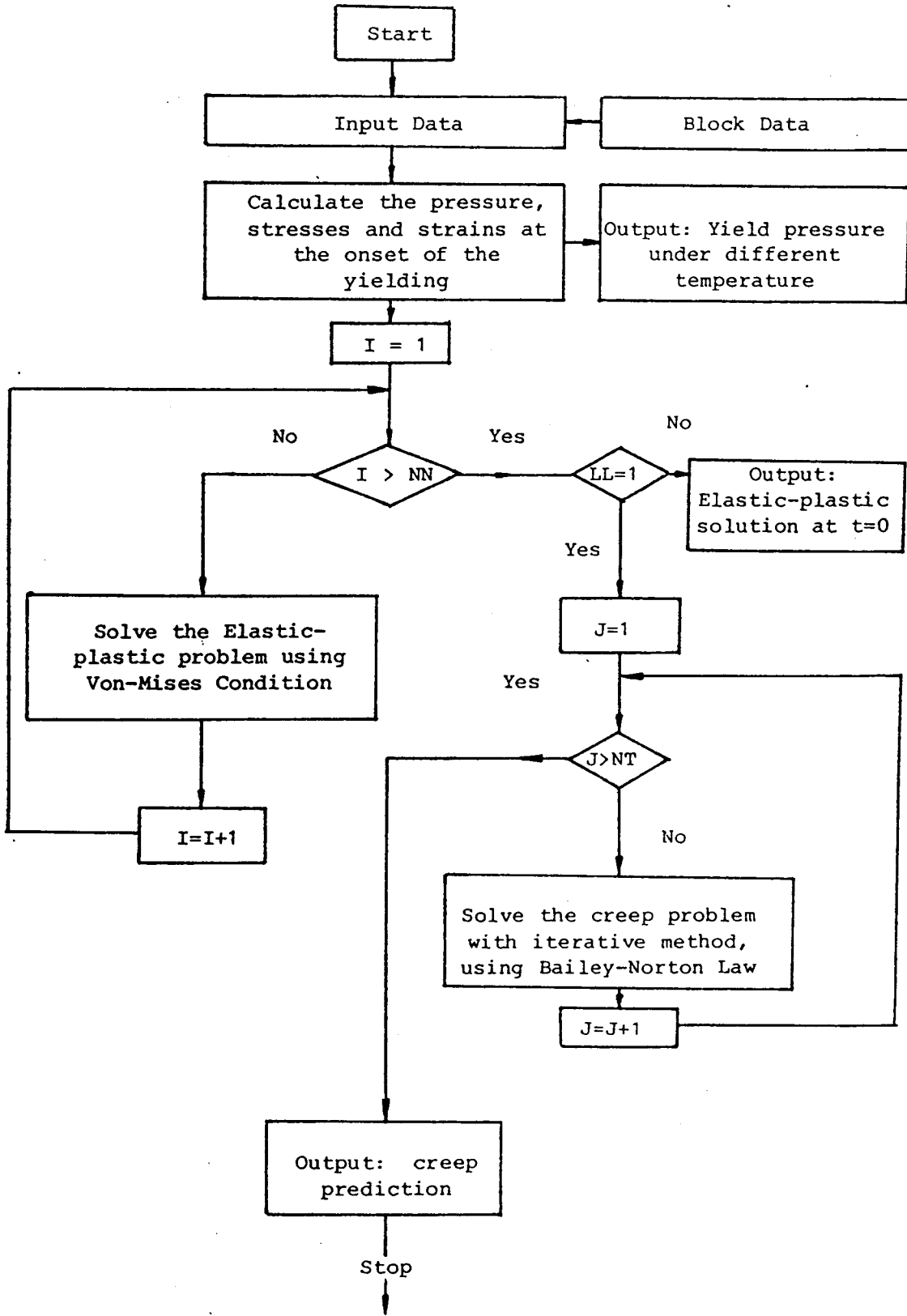


Fig. 5.1 Schematic of CASEPC

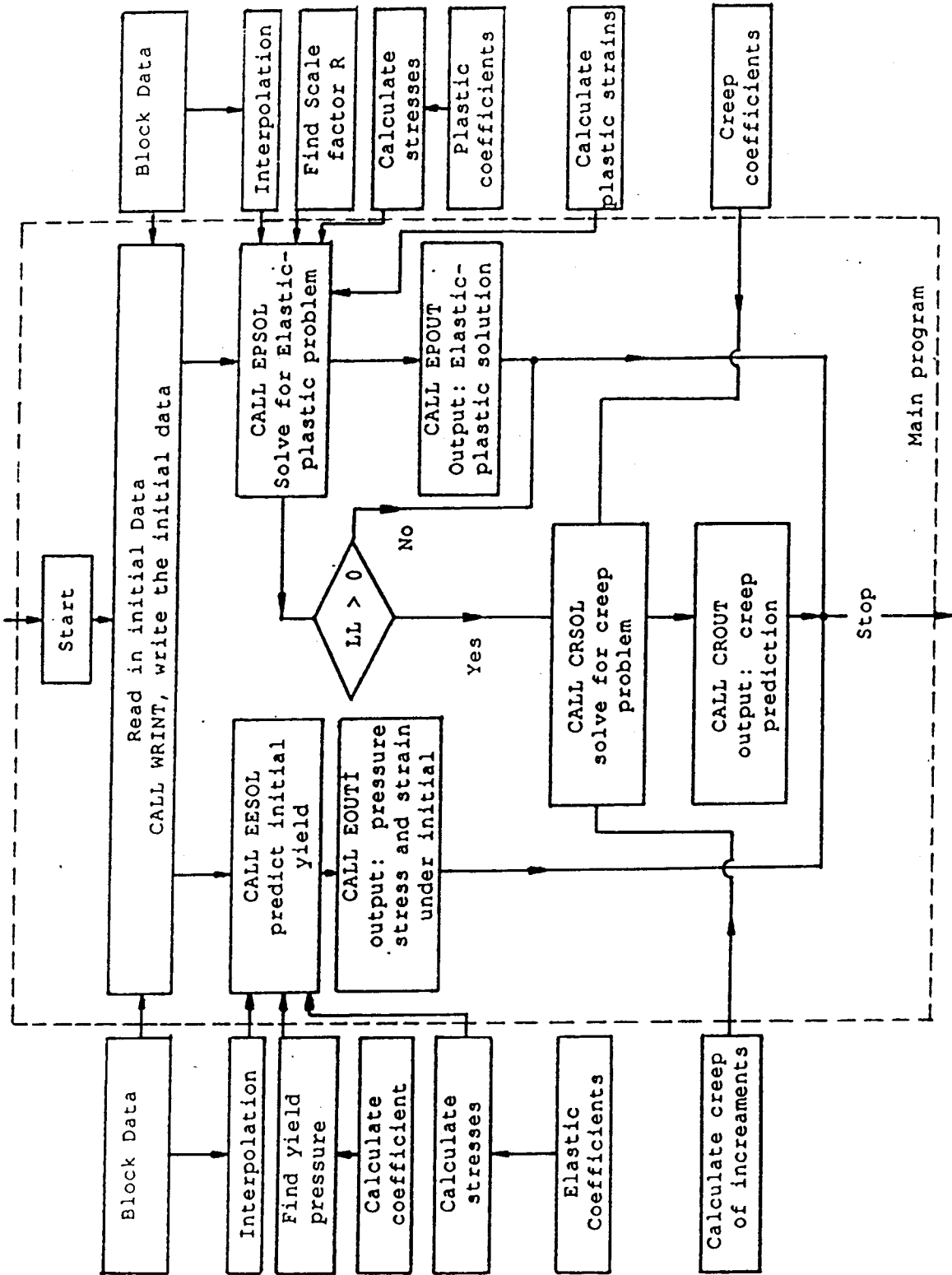


Fig. 5.2 Flow chart of interaction of subroutines

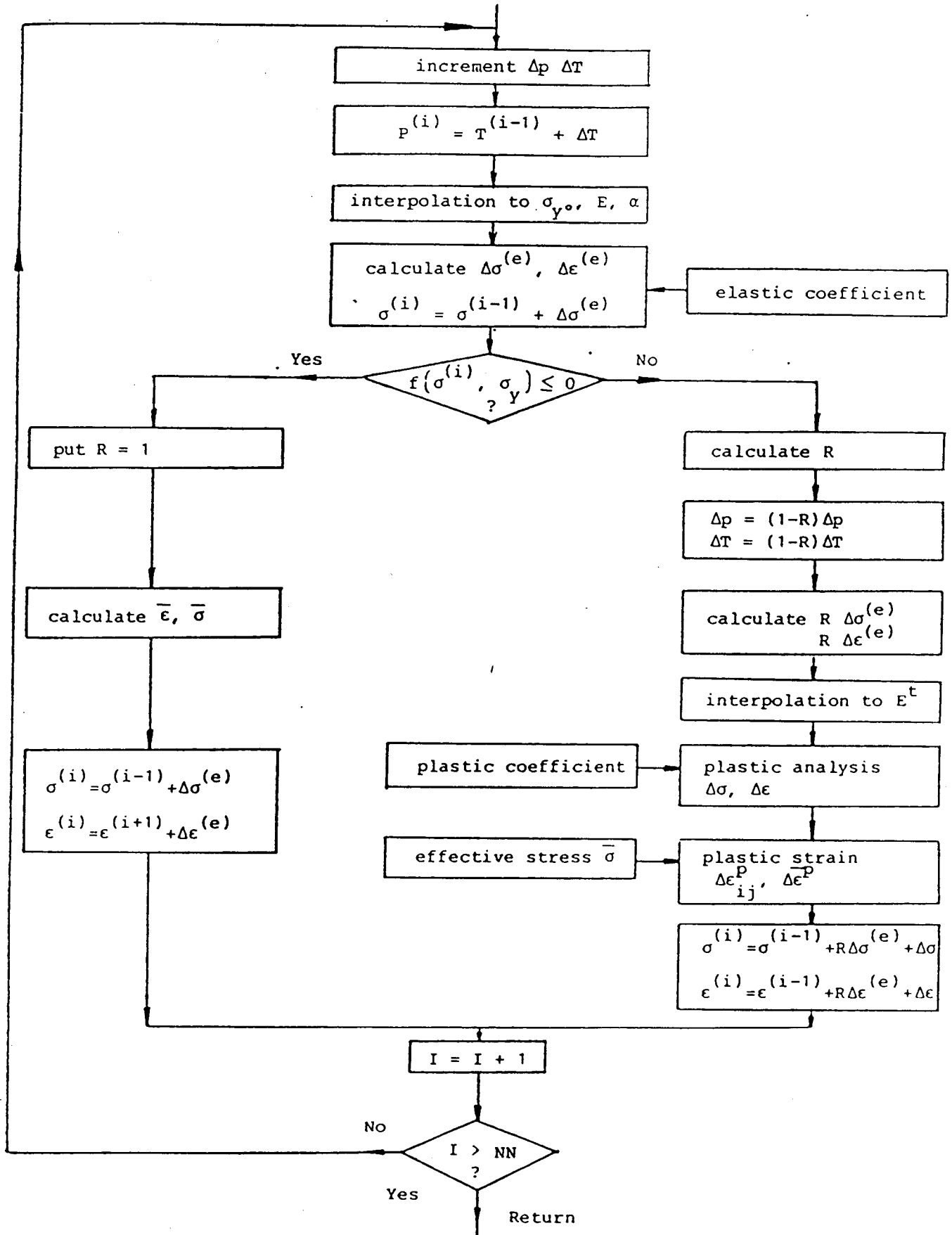


Fig. 5.3 Flow Chart of Subroutine EPSOL



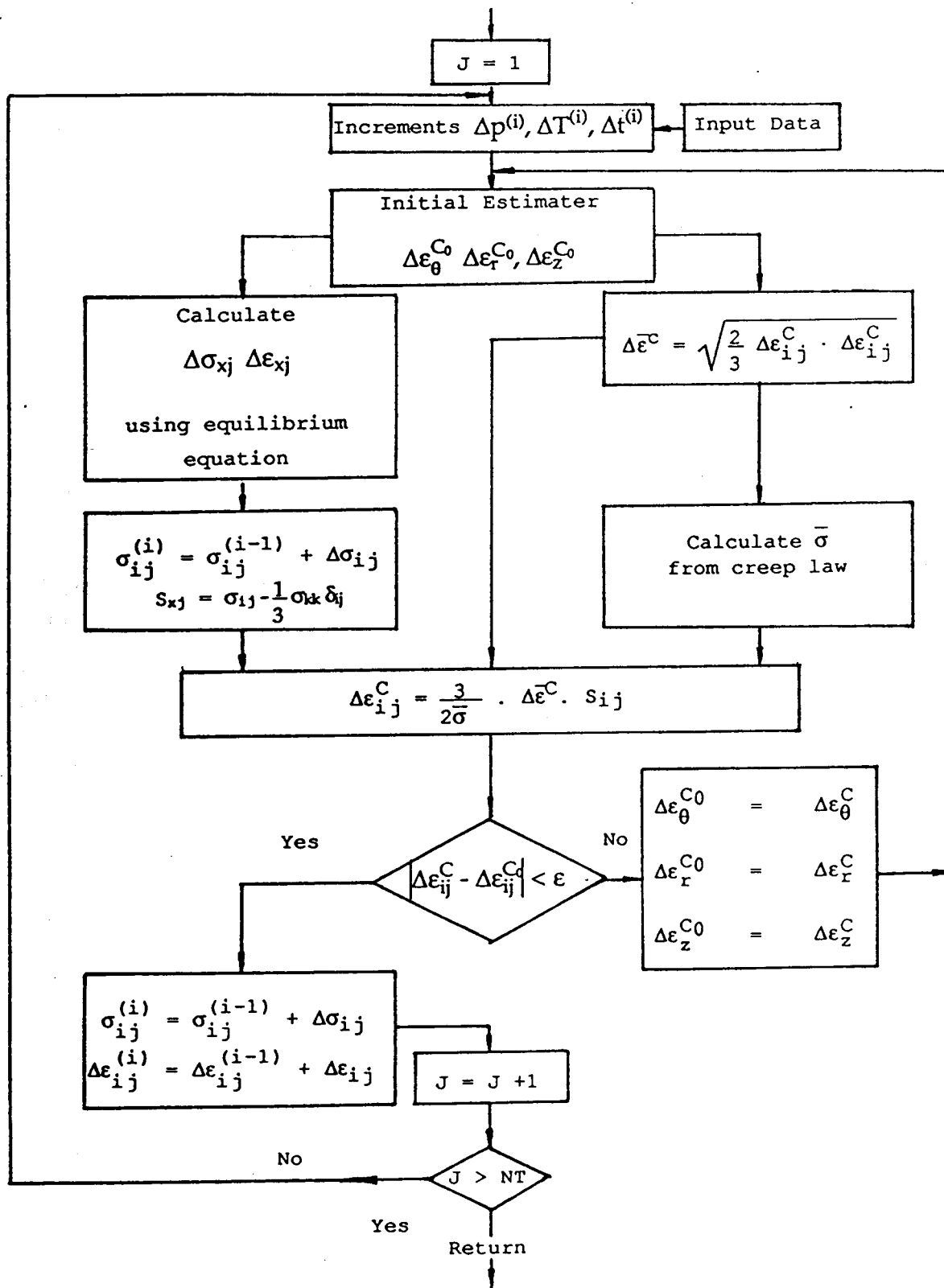


Fig. 5.4 Flow Chart of subroutine CRSOL

## CHAPTER 6 SOME NUMERICAL SOLUTIONS

### 6.1 Numerical Results

Prediction of initial yield using nondimensional ratios  $\tilde{p}_y$  and  $\tilde{T}_{y0}$  from (2.13a,b) is made for the following examples.

a. 4 1/2" casing

b. For oil sand, the data set is

$C_f = 0$  indicates no foundation

$(C_f)_{\max} = 10.9 \times 10^{-3}$  indicates  $\bar{E}_{\text{rock}} = 2,200$  MPa

$(C_f)_{\max} = 7.5 \times 10^{-3}$  indicates  $\bar{E}_{\text{rock}} = 1,600$  MPa

c. For shale the data set is

$C_f = 0$  indicates no foundation

$(C_f)_{\max} = 20.6 \times 10^{-4}$  indicates  $\bar{E}_{\text{rock}} = 400$  MPa

$(C_f)_{\max} = 4.5 \times 10^{-4}$  indicates  $\bar{E}_{\text{rock}} = 88$  MPa

Fig. 6.1 shows yield pressure ratio vs temperature ratio for oil sand and temperature independent properties. Fig. 6.2 is similar to Fig. 6.1 but for shale.

The prediction of initial yield for temperature dependent material properties has been carried out for the oil sand and shale properties summarized above. Fig. 6.3 shows the diagram of yield pressure vs temperature for a shale formation and 4 1/2" casing with temperature dependent properties. Fig. 6.4 is similar with an oil sand formation and 4 1/2" casing. These curves are lines of demarcation between the elastic and plastic regions.

### 6.2 Elastic-plastic solution with thermoplastic material

Data set:

4 1/2" casing                      outer diameter of pipe 114.3 mm

	thickness of wall	5.2 mm
Oil sand formation	elastic modulus	(temperature dependent)
	Poisson ratio	0.28

This example has been solved for two loading conditions:

- A: Keeping the pressure constant at  $p = 15$  MPa
- B: Allowing both pressure and temperature to increase simultaneously.

As this case is a temperature dependent problem, all properties of thermoplastic material are determined from Tables 4.1 and 4.2. The results are listed in Table 6.1.

The following for test problems were investigated.

1. A comparison between the results of an ADINA analysis by C-FER and the results which are computed by program CASEPC is made in Table 6.2 for the conditions where the loading path holds the pressure constant at  $p = 16.5$  MPa, while the temperature increases from  $0^{\circ}\text{C}$  to  $350^{\circ}\text{C}$ . The formation was an oil-sands formation.
2. A comparison between a temperature dependent and temperature independent solutions is shown in Fig. 6.5, using 4 1/2" casing and oil sand formation.
3. For loading consisting of temperature cycling and keeping the pressure constant at  $p = 15$  MPa the results are shown in Fig. 6.6.
4. The results of the elastic-plastic analysis with temperature dependent properties are shown as Fig. 6.7 and Fig. 6.8. In Fig. 6.7 the diagrams of effective stress vs pressure under different temperature are given and for Fig. 6.8 the diagram shows the results of effective stress vs temperature under different pressures.

Both of the above are taken from oil sand formation and 4.5" casing.

### **6.3 Creep Prediction**

The same data set as in the elastic-plastic computation is used as an example for a creep solution. The temperature and pressure were held constant (at 16.5 MPa and 350°C) for 1000 hours. The results are listed in Table 6.3 and the results of the first 300 hours are shown in Figs. 6.9 to 6.12.

For comparison, ADINA was used to compute creep effects for the first 12 hours, using the same data used in CASEPC. The solution contrasts between both are shown in Figs. 6.13 to 6.16.

### **6.4 Conclusion**

It is evident that the model we adopted and the program CASEPC are of many advantages, such as the reasonable simplified and theoretical basis, simple construct and convenient use for the program. Especially, the convincing results have been obtained through the comparison with the results from ADINA. So, the model along with the program CASEPC could be recommended to use in practice.

Table 6.1 Comparison of Effect of Variation in Sequence of Temperature Pressure Increments for Time Independent Problem

Loading	A				B	
Results	Temperature	3.5°	200°	300°	350°	
$\sigma_{\theta}$ (MPa)		134.5	82.9	50.18	34.2	50.58
$\sigma_r$		-15.0	-15.0	-15.0	-15.0	-15.0
$\sigma_z$		27.09	-434.8	-521.8	-531.3	-546.2
$\epsilon_{\theta} \times 10^{-2}$		0.668	3.48	5.236	6.278	6.335
$\epsilon_r \times 10^{-2}$		-0.2625	2.907	4.73	5.729	5.634
$\epsilon_z \times 10^{-2}$		0	0	0	0	0
$\epsilon_{\theta}^p \times 10^{-3}$		0	0	0.504	0.832	0.7633
$\epsilon_r^p \times 10^{-3}$		0	0	0.309	0.6519	0.5575
$\epsilon_z^p \times 10^{-3}$		0	0	-0.813	-1.482	-1.321

A = Constant pressure of 15 MPa.

B = Pressure increases linearly to 15 MPa while temperature increases linearly to 350°C.

Table 6.2 Comparison of Time Independent Solutions by CASEPC and ADINA\*

	CASEPC	ADINA	DIFFERENCE
$\sigma_{\theta}$ (MPa)	41.84	35.30	+18.5%
$\sigma_r$ (MPa)	-16.5	-16.5	
$\sigma_z$ (MPa)	-487.3	-415.2	+17.3%
$\epsilon_{\theta}$ (1.E-2)	0.6354	0.7252	-12.3%
$\epsilon_r$ (1.E-2)	0.5634	0.6413	-12.1%
$\epsilon_z$ (1.E-2)	0.0	0.0	
$\epsilon_{\theta}^p$ (1.E-3)	1.028	1.089	-5.6%
$\epsilon_r^p$ (1.E-3)	0.6893	0.7509	-8.2%
$\epsilon_z^p$ (1.E-3)	-1.717	-1.839	-6.8%
$\bar{\sigma}$ (MPa)	502.7	426.8	+17.7%
$\bar{\epsilon}$ (1.E-2)	0.4018	0.4581	-12.3%

\* p = constant = 16.5 MPa

T = increases from 0°C to 350°C

Table 6.3 Time Dependent Solutions for p = 16.5 MPa and T = 350°C

t	Solution								
(hr)	$\sigma_{\theta}$ (MPa)	$\sigma_r$ (MPa)	$\sigma_z$ (MPa)	$\epsilon_{\theta}$ (1.E-2)	$\epsilon_r$ (1.E-2)	$\epsilon_z$ (1.E-2)	$\bar{\sigma}$ (MPa)	$\bar{\epsilon}$ (1.E-2)	$\epsilon^c$
0	44.23	-16.5	-545.0	0.6235	0.5597	0.0	561.4	0.3961	
2	37.34		-393.9	0.6545	0.5757	0.0	407.0	0.4126	0.3367E-3
10	33.09		-305.5	0.6735	0.5840	0.0	316.7	0.4224	0.4391E-4
20	31.52		-274.2	0.6806	0.5867	0.0	284.7	0.4259	0.1818E-4
50	30.07		-246.1	0.6871	0.5883	0.0	256.1	0.4291	0.7135E-5
100	28.91		-224.4	0.6923	0.5903	0.0	233.9	0.4316	0.3301E-5
200	27.81		-204.5	0.7003	0.5922	0.0	201.2	0.4353	0.9299E-6
400	26.65		-183.8	0.7024	0.5926	0.0	192.5	0.4363	0.6503E-6
600	25.98		-172.3	0.7054	0.5931	0.0	180.8	0.4377	0.3916E-6
800	25.53		-164.5	0.7075	0.5935	0.0	172.9	0.4386	0.2741E-6
1000	25.18	-16.5	-158.7	0.7090	0.5937	0.0	167.0	0.4393	0.208E-6

Note: The results come from the program CASEPC under the following conditions.

- a. No pressure increment and temperature increment in the time period. That is to say, the pressure and temperature were kept constant, p = 16.5 MPa and  $\Delta T = 350^\circ\text{C}$ .
- b. For the tangent modulus  $E^t$ , two dimensional Lagrangian interpolation is employed. This differs from ADINA where just one dimensional interpolation is available.

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**FIGURES**

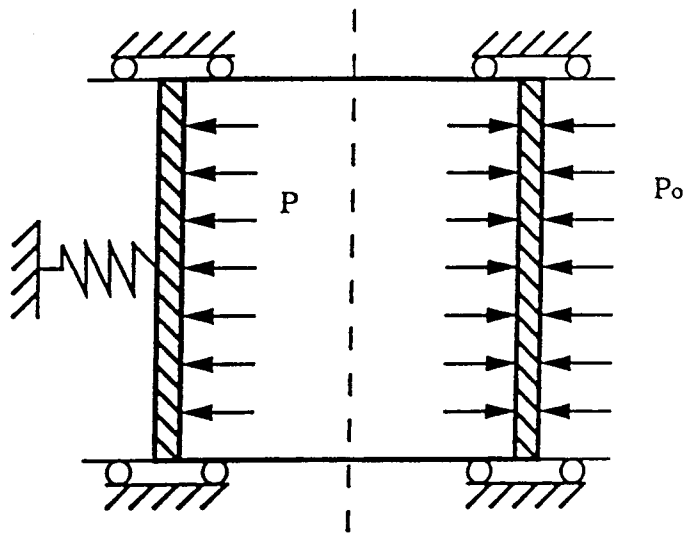


Fig. 1.1-- Casing System

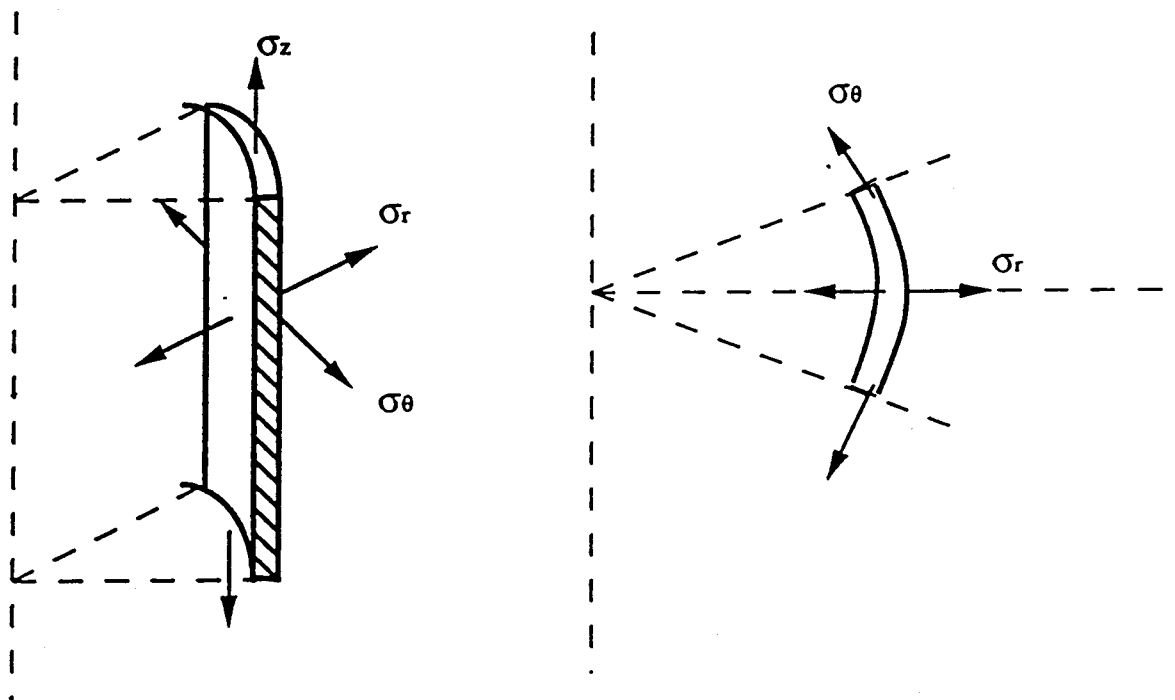


Fig. 1.2

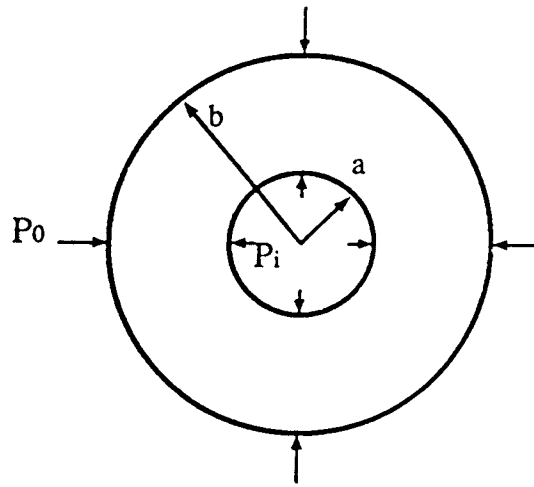


Fig. 1.3

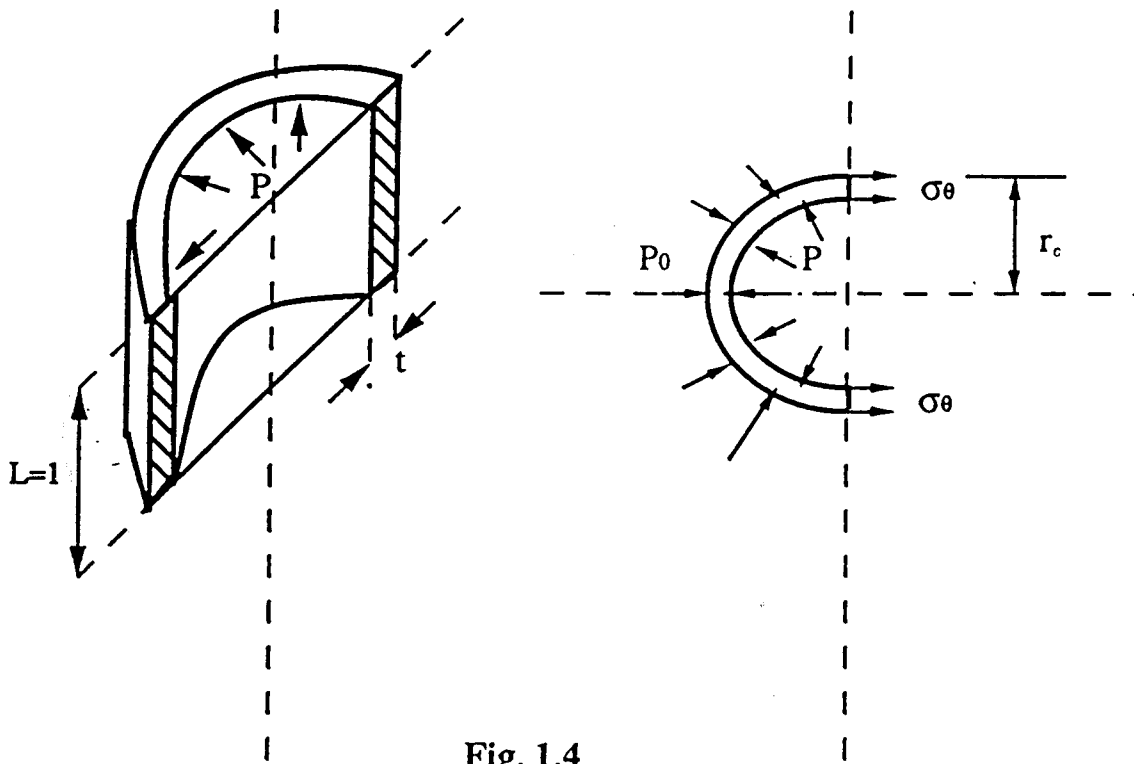


Fig. 1.4

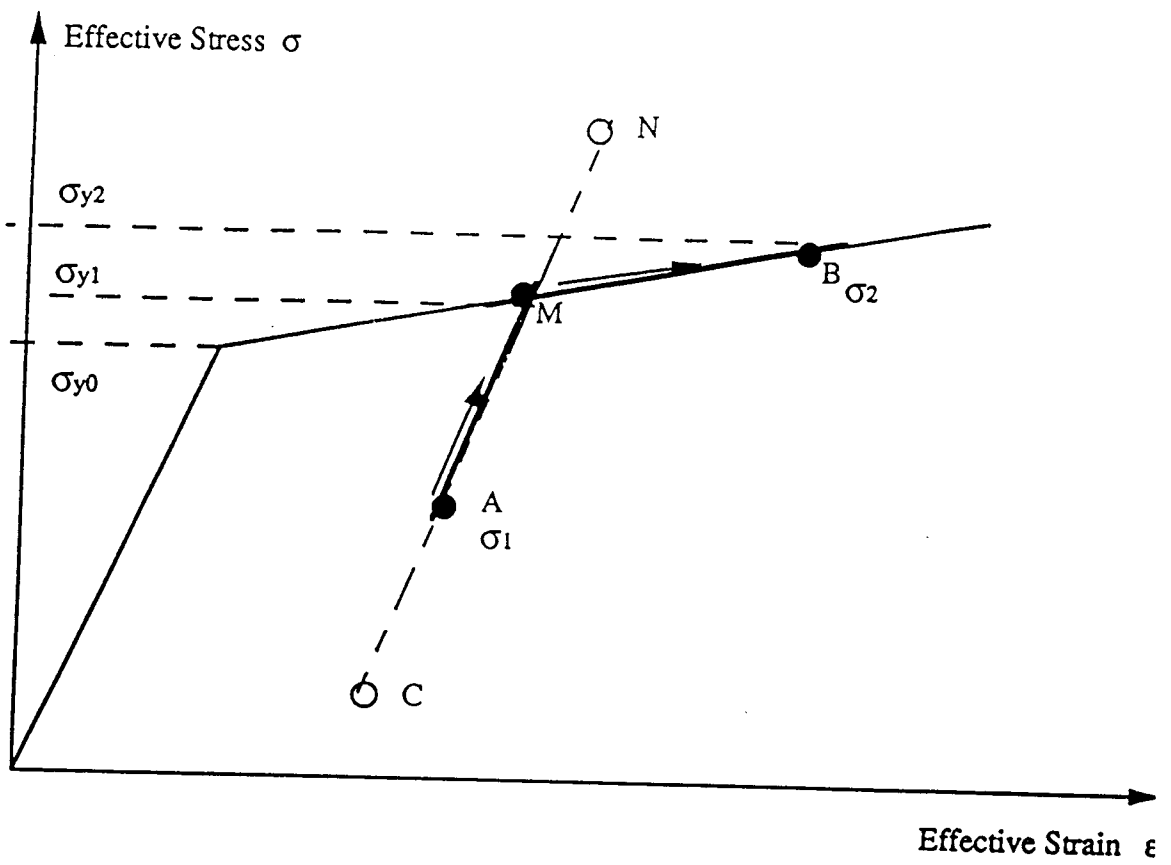


Fig. 2.1

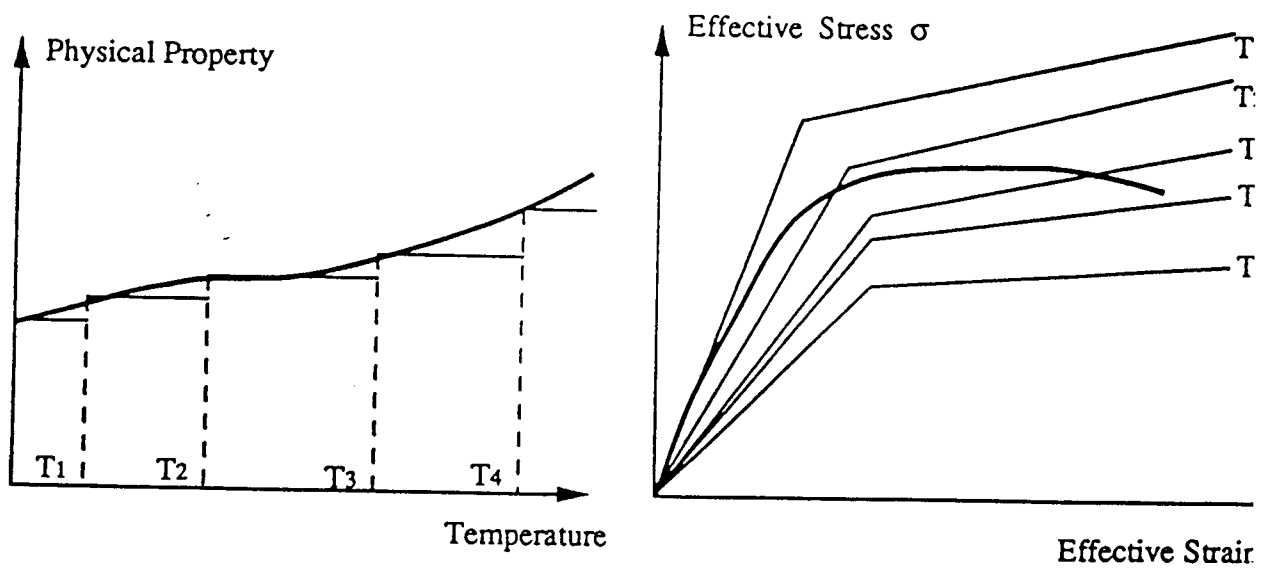
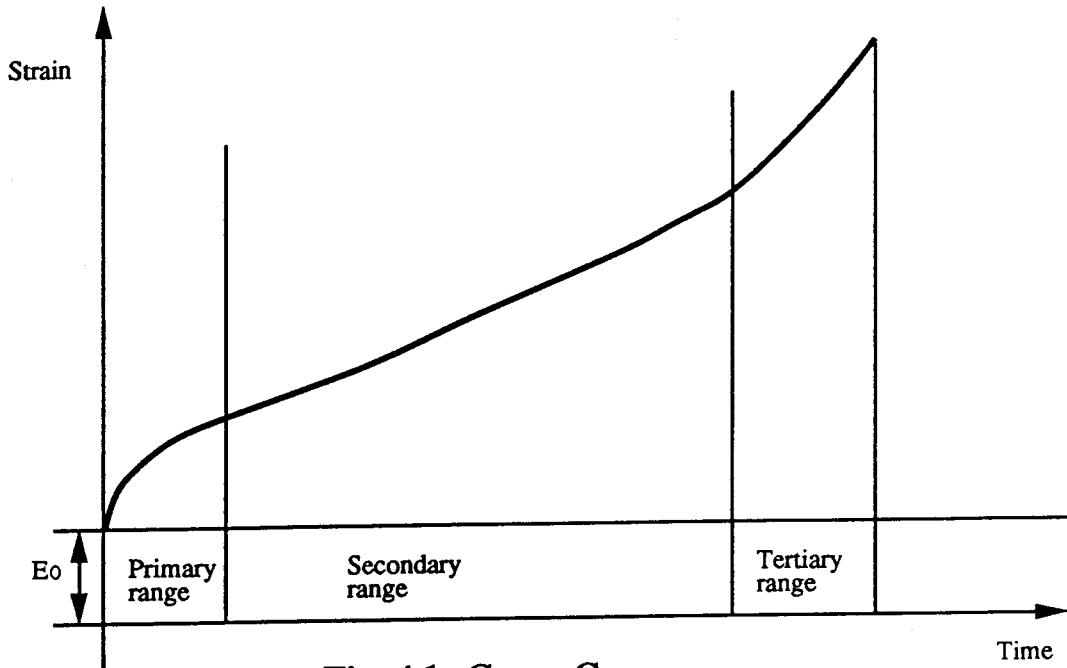
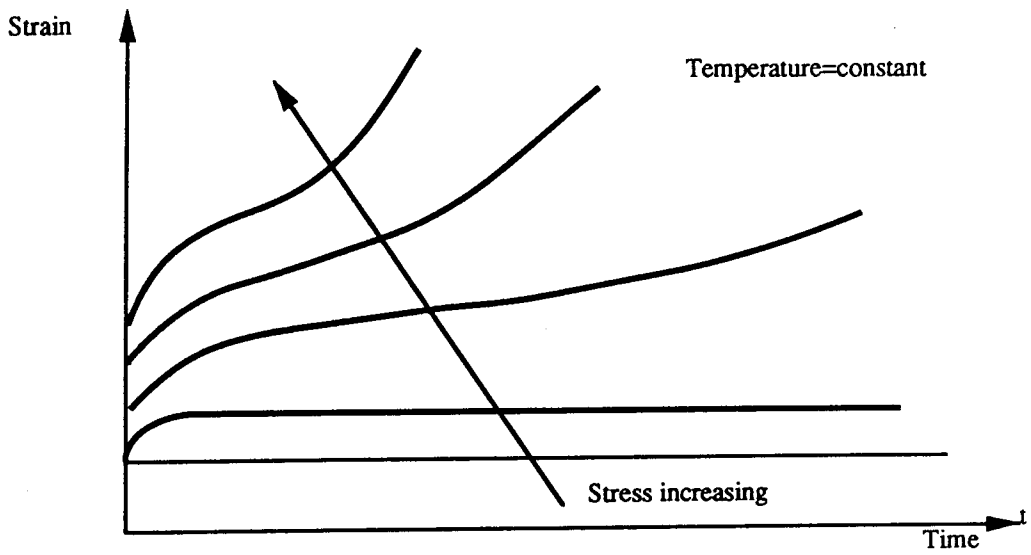


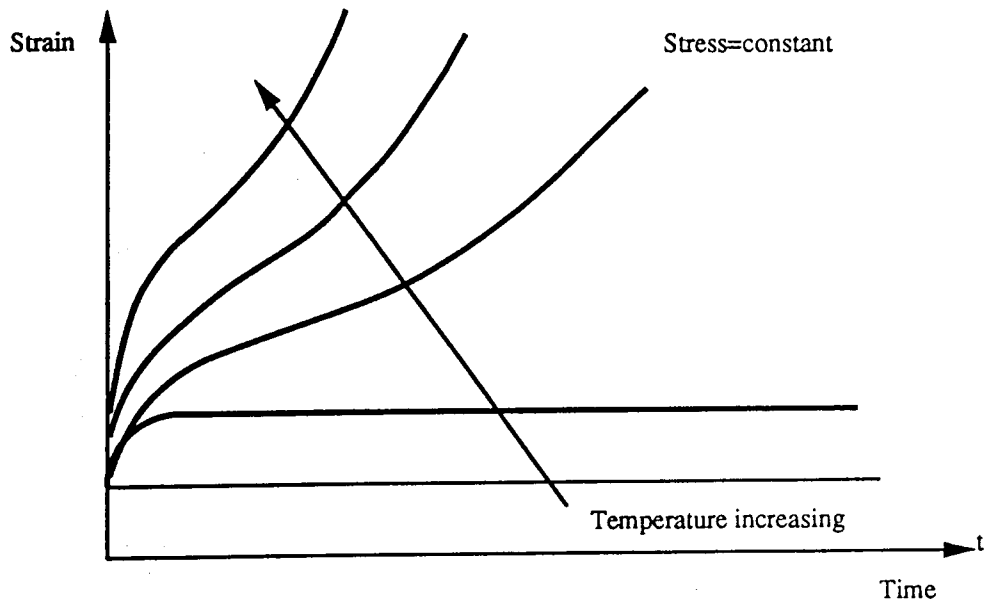
Fig. 3.1



**Fig. 4.1--Creep Curve**



**Fig. 4.2-Effect of stress on the creep curve at constant temperature**



**Fig. 4.3-** Effect of temperature on the creep curve at constant stress

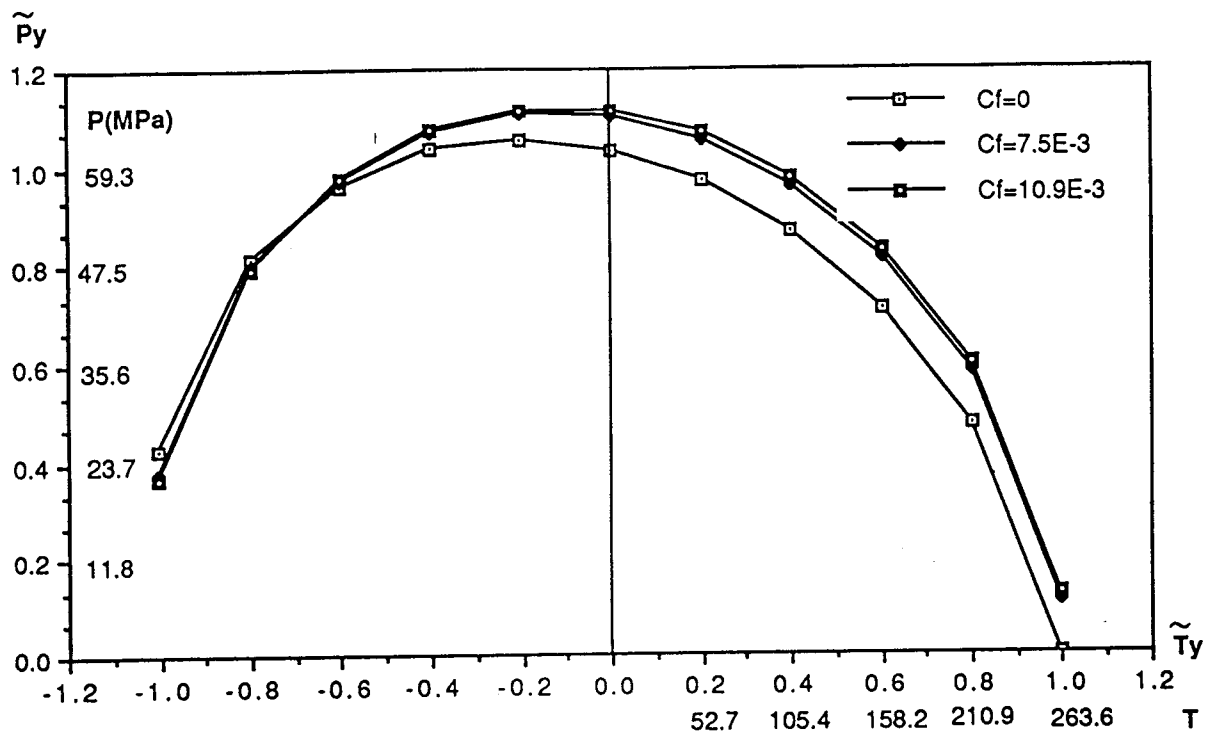


Fig. 6.1--Yield Pressure Ratio Vs Temperature Ratio  
( Oil sand, 4.5" Casing )

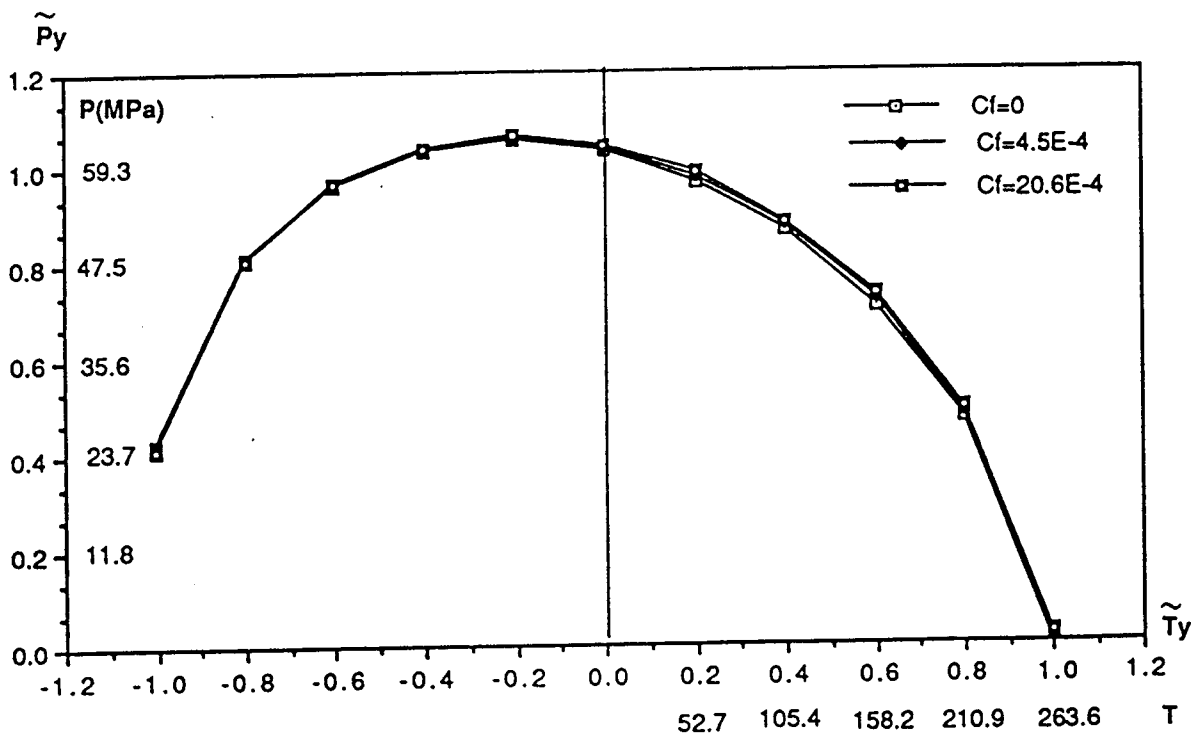
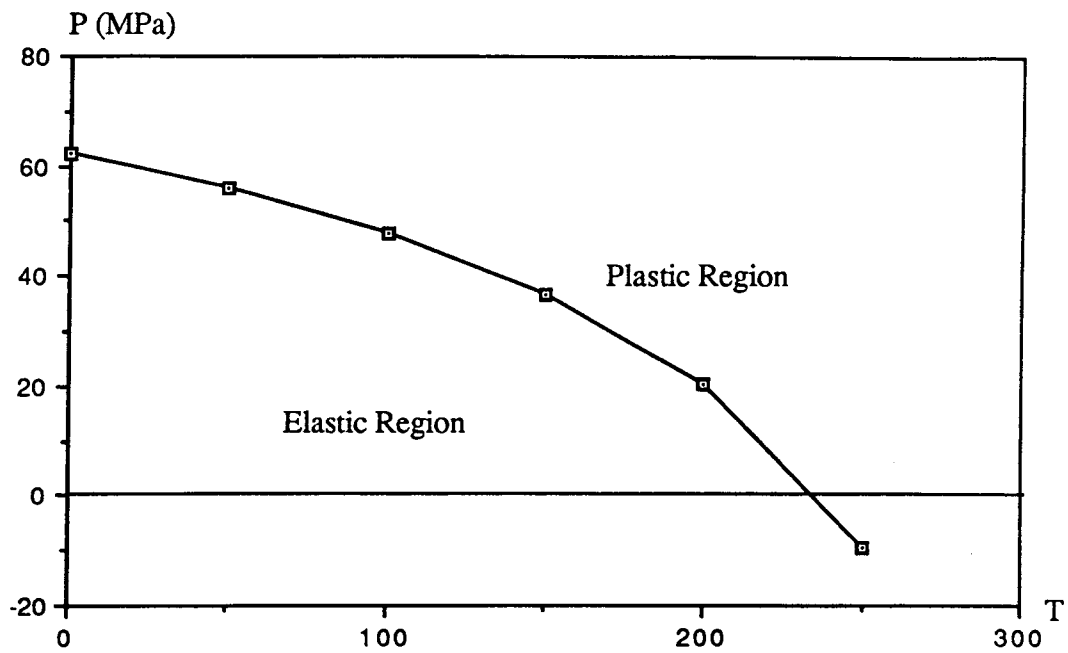
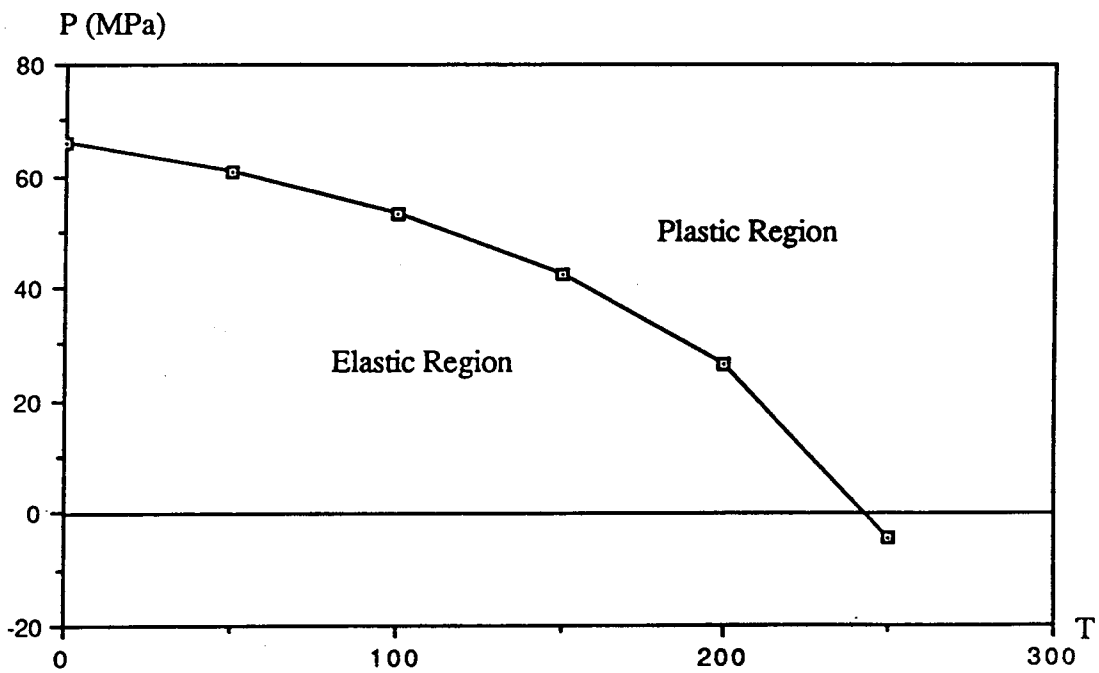


Fig. 6.2--Yield Pressure Ratio Vs Temperature Ratio  
( Shale Formation, 4.5" Casing )

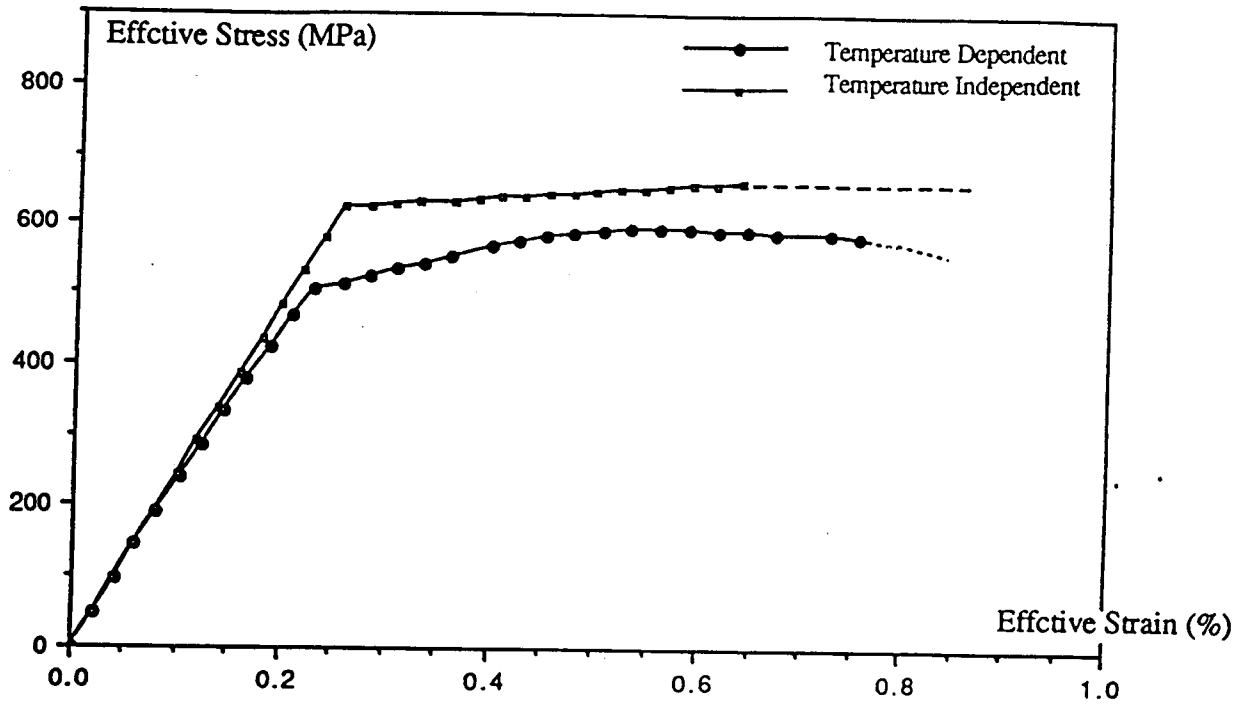


**Fig. 6.3--Yield Pressure Vs Temperature  
( Shale Formation, 4.5" Casing )**

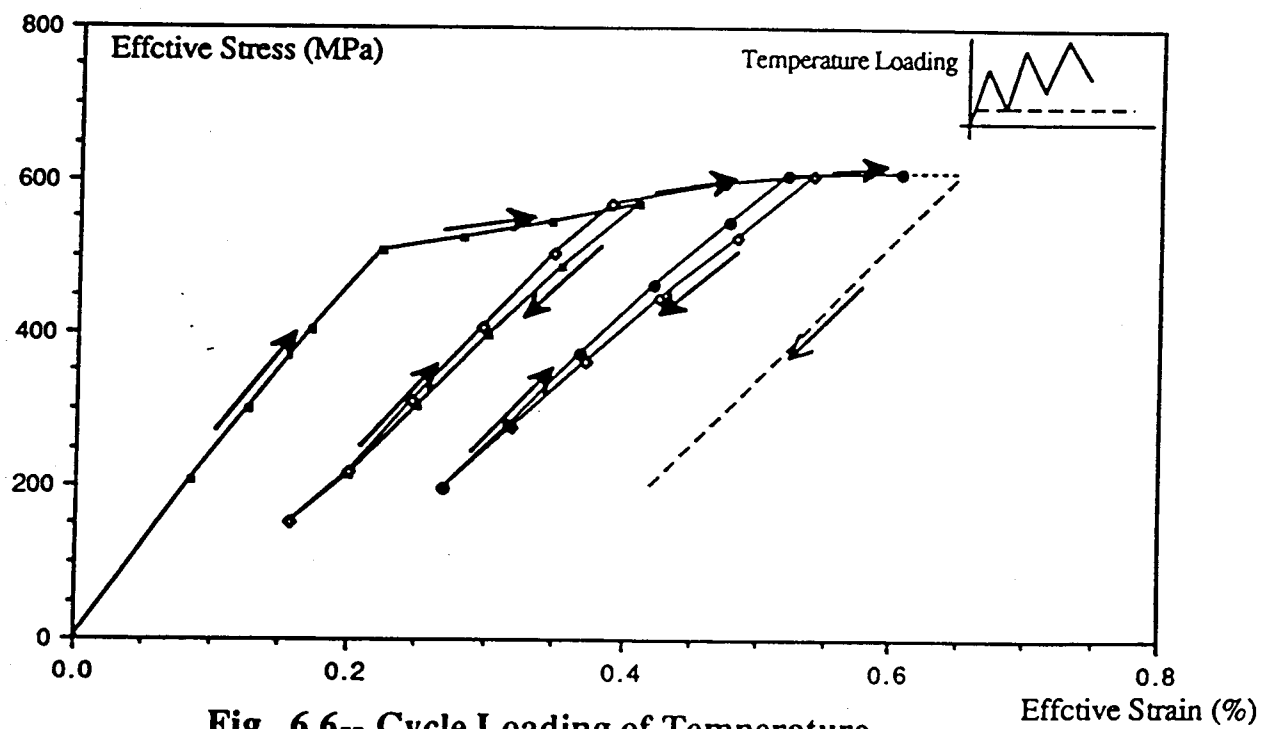


**Fig. 6.4-- Yield Pressure Vs Temperature  
( Oil Sand, 4.5" Casing )**

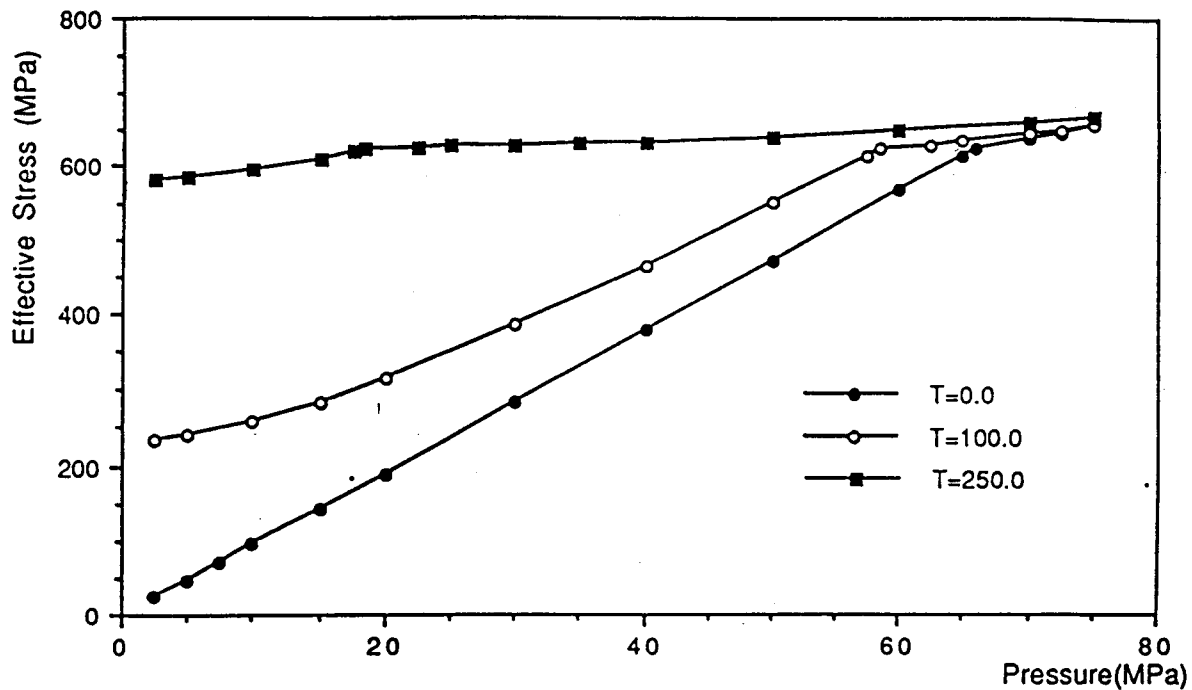




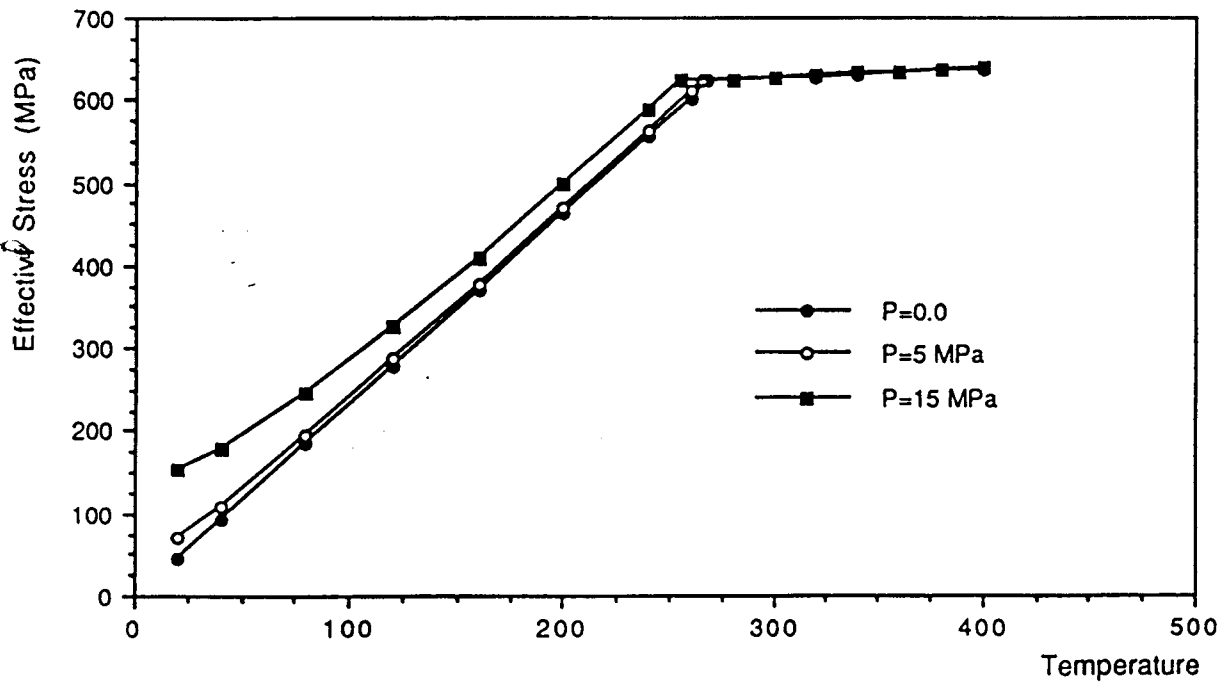
**Fig. 6.5--Temperature Dependent  
( Oil Sand, 4.5" Casing )**



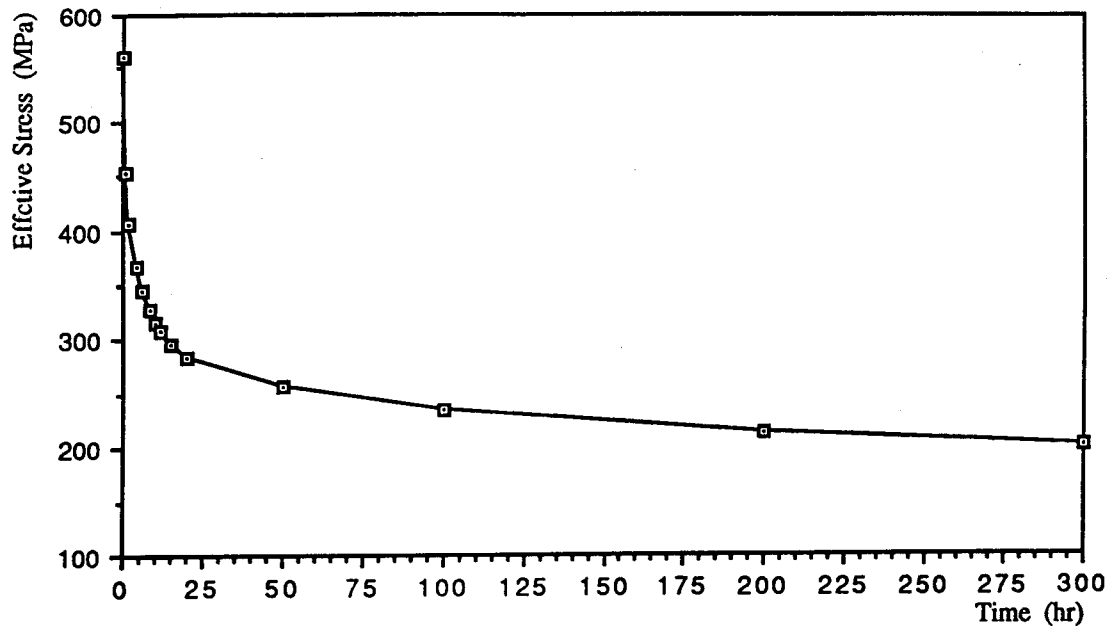
**Fig. 6.6-- Cycle Loading of Temperature  
( Oil Sand, 4.5" Casing, P=15 MPa )**



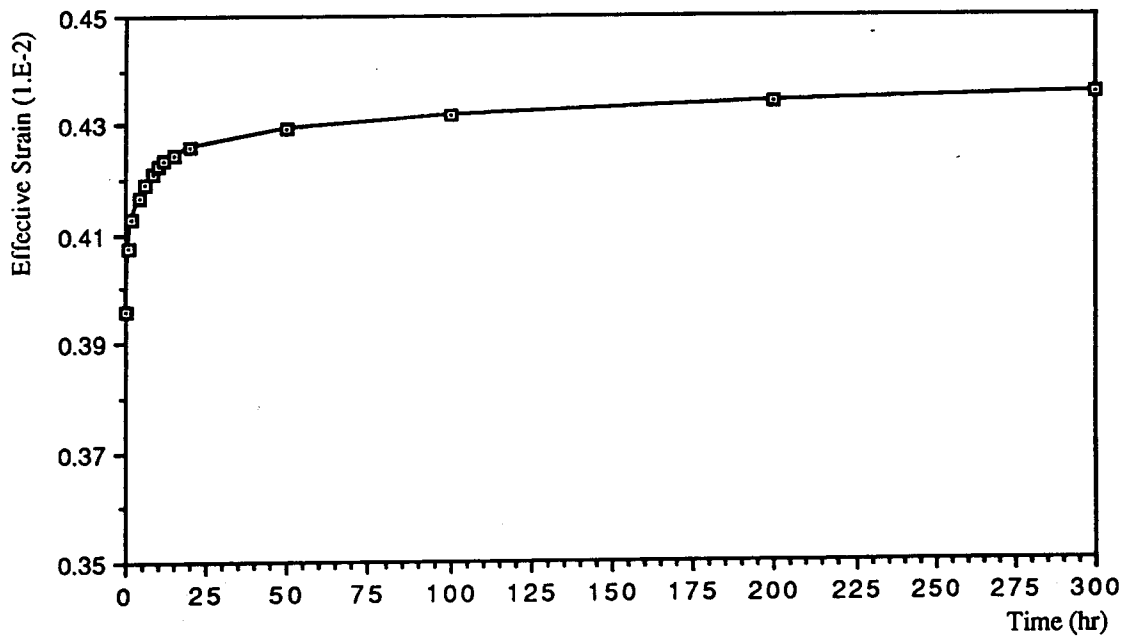
**Fig. 6.7--Effective Stress Vs Pressure  
(Oil Sand,4.5" Casing, Under Different Temperature)**



**Fig. 6.8--Effective Stress Vs Temperature  
(Oil Sand,4.5" Casing, Under Different Pressure)**



**Fig. 6.9--Time Dependent Solution: Effective Stress  
(Oil Sand, 4.5" Casing, P=16.5 MPa, T=350)**



**Fig. 6.10--Time Dependent Solution: Effective Strain  
(Oil sand, 4.5" Casing, P=16.5 MPa, T=350)**

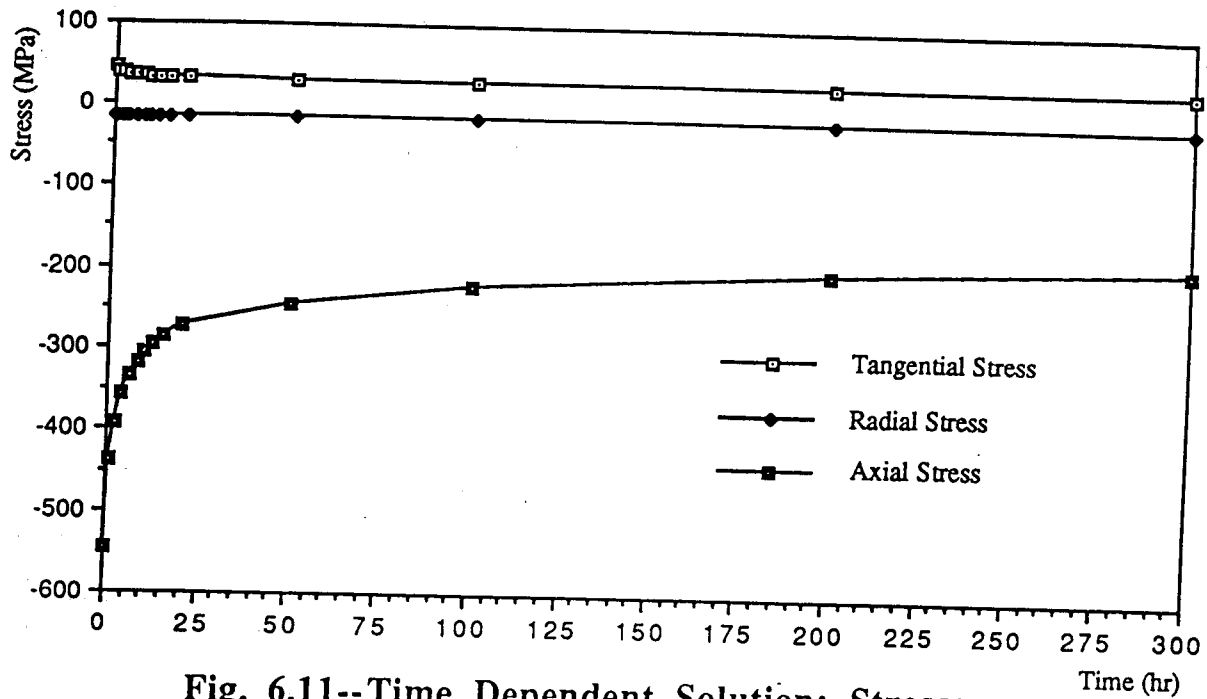


Fig. 6.11--Time Dependent Solution: Stresses  
(Oil Sand, 4.5" Casing, P=16.5MPa, T=350)

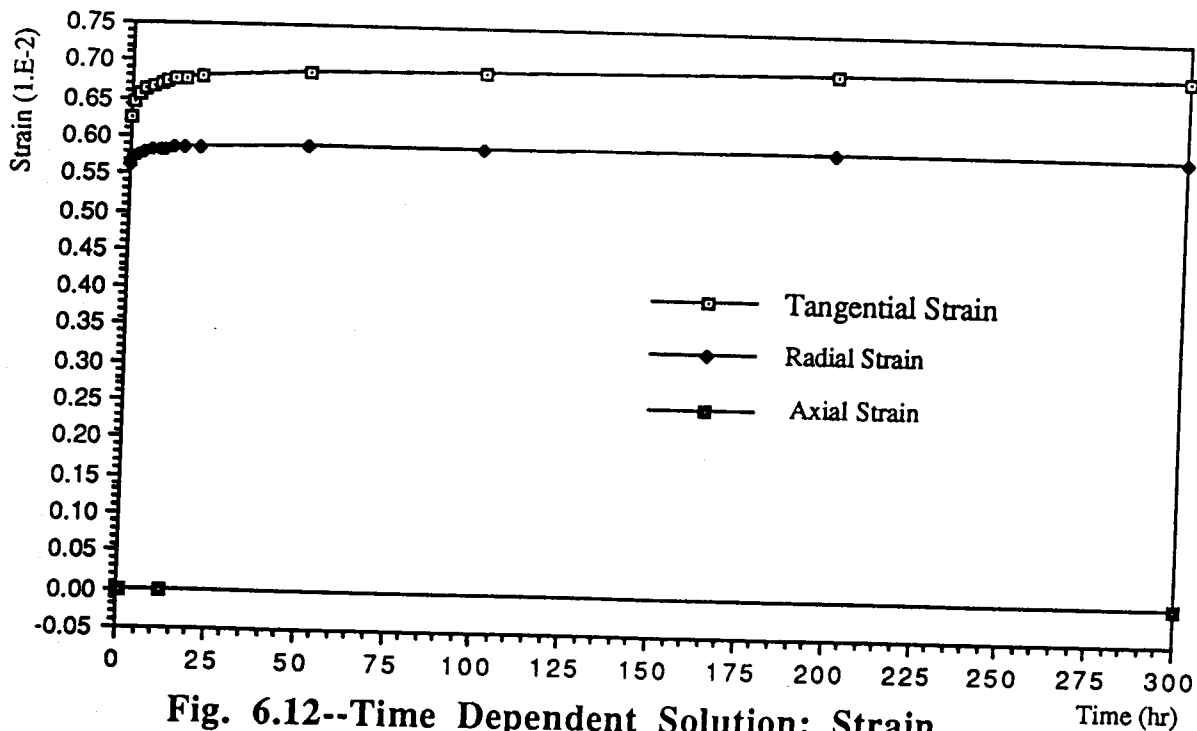
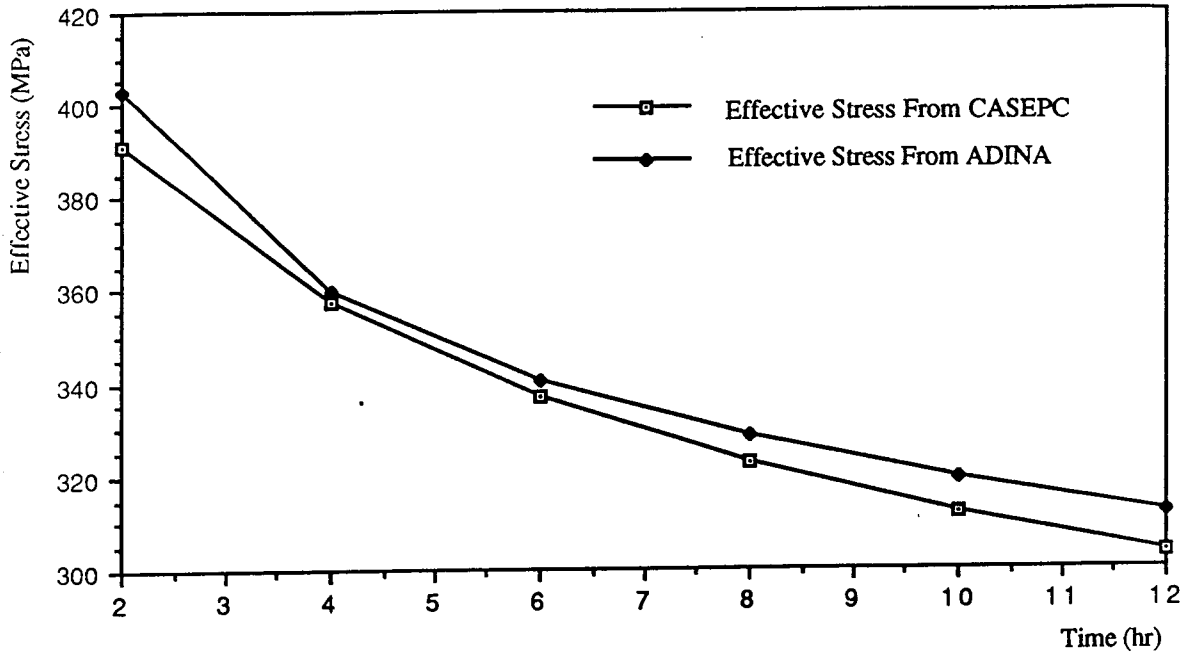
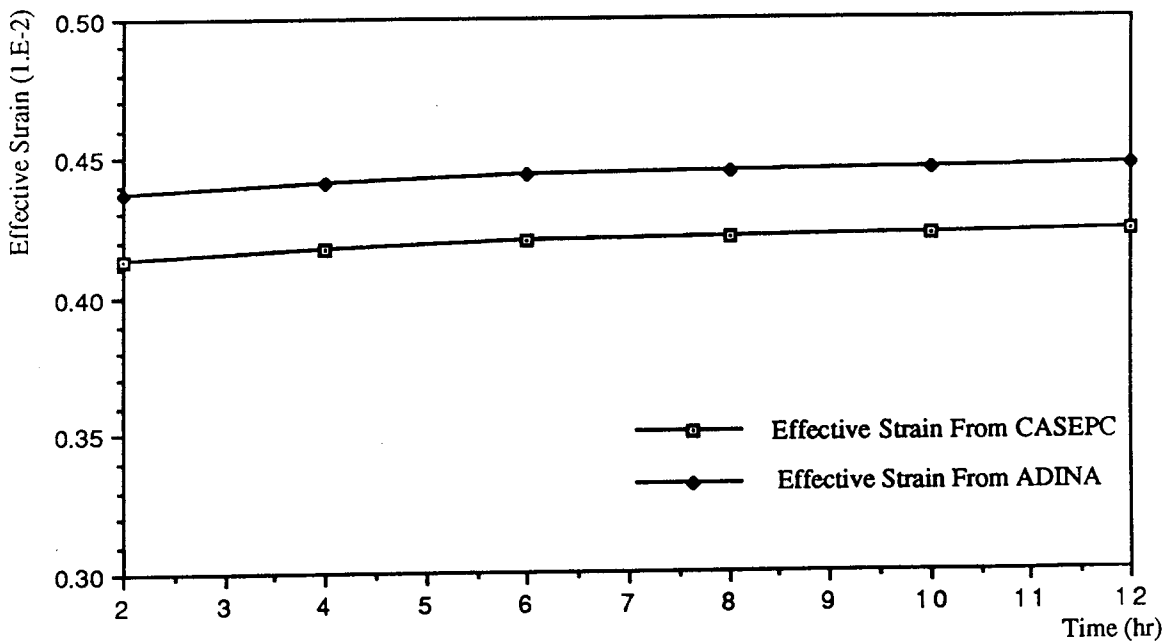


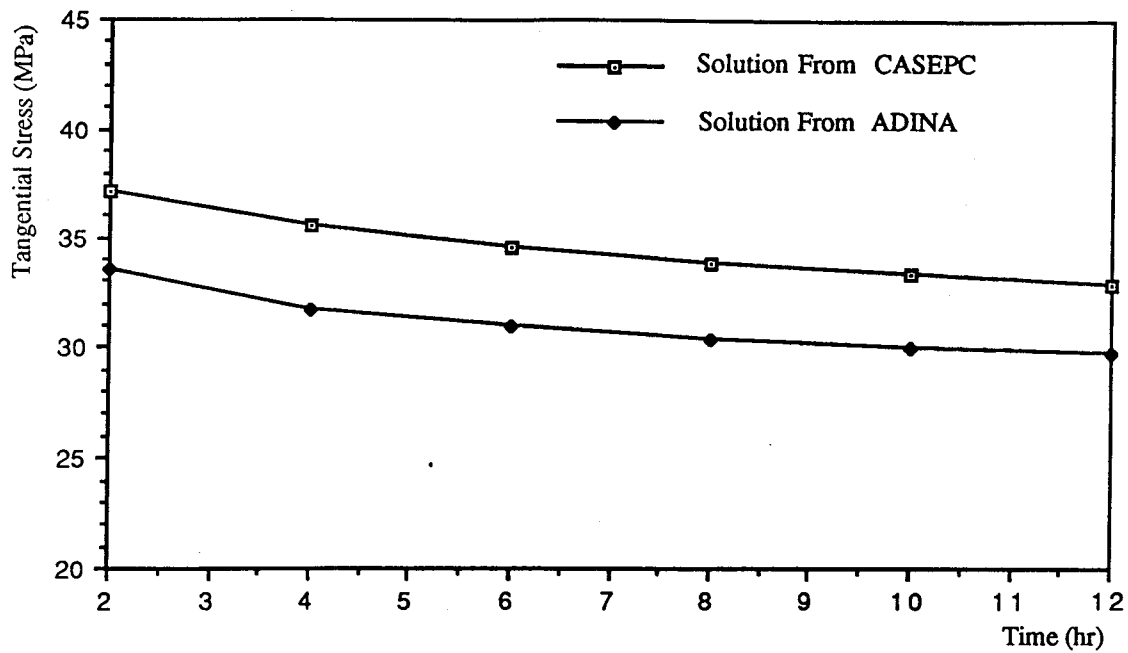
Fig. 6.12--Time Dependent Solution: Strain  
(Oil Sand, 4.5" Casing, P=16.5MPa, T=350)



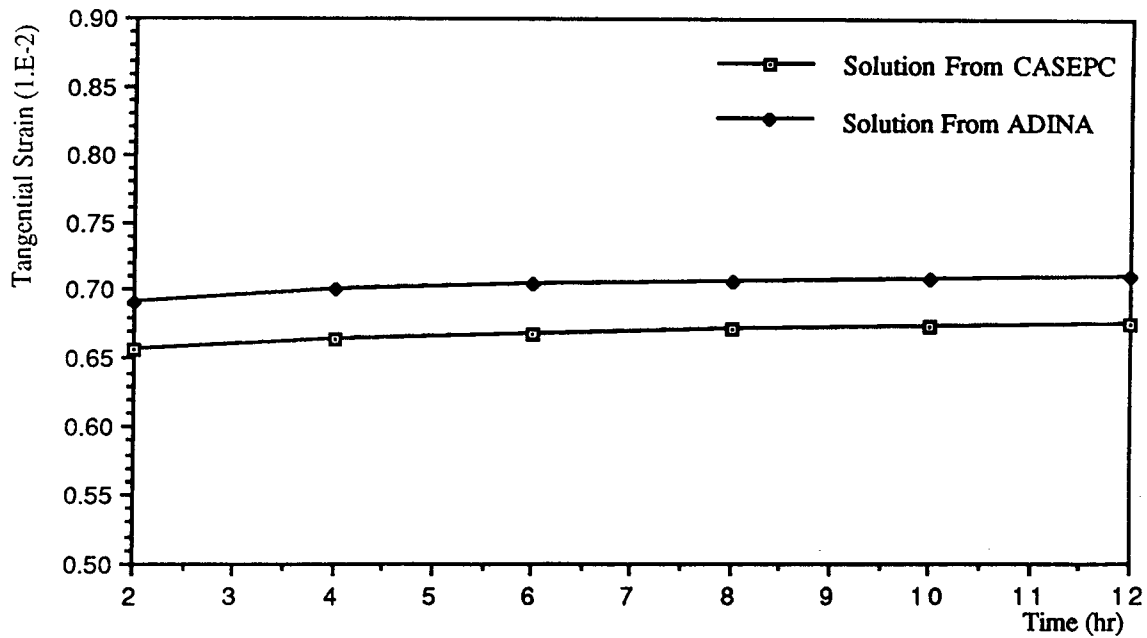
**Fig. 6.13--Creep Solution Contrast Between CASEPC and ADINA (Oil Sand, 4.5" Casing, P=16.5MPa, T=350)**



**Fig. 6.14--Creep Solution Contrast Between CASEPC and ADINA (Oil Sand, 4.5" Casing, P=16.5MPa, T=350)**



**Fig. 6.15--Creep Solution Contrast: Tangential Stress  
(Oil Sand, 4.5" Casing, P=16.5MPa, T=350)**



**Fig. 6.16--Creep Solution Contrast: Tangential Strain  
(Oil Sand, 4.5" Casing, P=16.5MPa, T=350)**

**APPENDIX A**

**USER'S MANUAL**

## USER'S MANUAL FOR PROGRAM CASEPC

Program CASEPC is written in Fortran 77 language. The program uses an incremental isotropic elastic-plastic temperature dependent material model based on the Von-Mises yield condition to analyze a horizontal slice of the casing for which the vertical strain is assumed to be zero, and the supporting rock is assumed linear elastic. It can be used for both elastic-plastic loading and unloading to solve the temperature dependent and time dependent problems in which the material properties of the steel are temperature dependent. The current file name for the object deck is WEACC.

### A1. LISTING OF SUBROUTINES

#### A1.1 Main Program

The main program calls eight subroutines: INPUT, WRINT, EESOL, EOUT1, EPSOL, EPOUT1, CRSOL, CROUT. The function of each is discussed in the following.

#### A1.2 Subroutine INPUT

Function: Reads the initial data, which includes the control parameters, casing size, properties of materials and loading history, which consists of increments of pressure, temperature and time.

#### A1.3 Block Data WDAT (See also Sect. A6)

Function: Prescribes initial data which includes the following:

XX1(2,9), XX2(2,9): data points for determining E and  $\alpha$  of the casing as a function of temperature.

XO(6), YO(6): data points for one-dimensional interpolation



of  $\sigma_y^o$  as a function of temperature.

XO(6), Y1(8), ZO(6,8,1): data points for two-dimensional interpolation of  $E^t$  as a function of temperature and time.

TM(6): temperatures for the calculation of the initial yield under different temperature.

#### **A1.4 Subroutine WRINT**

Function: Prints (i.e., echo checks) all initial data.

#### **A1.5 Subroutine EESOL**

Function: Calculates the pressure, stresses and strains at the onset of yield under different temperatures. To do so, some supplementary subroutines which perform operations, such as, curve fitting and interpolation, are used to determine the yield pressure and the elastic response.

Input: N, the number of temperatures at which the yield value is to be computed. In the program the maximum value of N is prescribed as 6.

Output: SC(N), SR(N), SZ(N) denote tangential, radial and axial stress respectively.

EC(N), ER(N), EZ(N) denote tangential, radial and axial strain respectively.

ED(N) and EE(N) are effective stress and strain respectively.

BY(N) are the yield values.

### **A1.6 Subroutine YEN**

Function: Calculates the yield pressure at the onset of initial yield under different temperatures.

Input: N

Output: PY(N)

### **A1.7 Subroutine ECOEF**

Function: Finds elastic response. The coefficients of (2.34) are computed in this subroutine.

### **A1.8 Subroutine EOUT1**

Function: Outputs elastic stresses and strains at the onset of yield.

### **A1.9 Subroutine EPSOL**

All arrays in this subroutine are adjustable. NNC = 50 is maximum upper limit.

Function: Finds the elastic-plastic solution for thermally dependent properties using Von-Mises condition.

Input: NN, the number of computing steps which, in turn, determines the number of increments of  $\Delta P$  or  $\Delta T$ .  $NN \leq NNC$

Output:

- stresses PSC(J), PSR(J), PSZ(J)
- total strains PEC(J), PER(J), PEZ(J)
- plastic strains ECP(J), ERP(J), EZP(J)
- effective stress PED(J)
- effective strain PEE(J)
- effective plastic strain EEP(J)

Here the number of J varies from 1 to NN.

**A1.10 Subroutine ELASE**

Function: Calculates elastic stresses and strains using (1.16) of this report. The subroutine ELASE is called by the subroutine EPSOL.

Input: Pressure  $P$  (or  $\Delta P$ ), temperature  $T$  (or  $\Delta T$ ) and material properties of casing  $E$ ,  $\alpha$  (ALPHA),  $\sigma_Y^o$  (BYO).

Output: Elastic response.

**A1.11 Subroutine QUAD**

Function: Calculates the real roots of second degree algebraic equations. The subroutine is commonly used.

**A1.12 Subroutine EDSTS**

Function: Calculates effective stress.

**A1.13 Subroutine EESTN**

Function: Calculates effective strain.

**A1.14 Subroutine PCOEF**

Function: Presents the coefficients, as shown in (2.34) of this report, for computing plastic response.

**A1.15 Subroutine PLASE**

Function: Finds the increment of plastic response using (2.32) and (2.33) of this report. The subroutine PCOEF is called.

Input: Pressure increment  $\Delta P$ , temperature increment  $\Delta T$ , material properties  $E$ ,  $\alpha$ ,  $\sigma_Y^o$ ,  $E^P$ .

- Output:
- Increment of stresses DPSC, DPSR, DPSZ (i.e.,  $\Delta\sigma_\theta$ ,  $\Delta\sigma_r$ ,  $\Delta\sigma_z$ ).
  - Increment of strains DPEC, DPER, DPEZ (i.e.,  $\Delta\varepsilon_\theta$ ,  $\Delta\varepsilon_r$ ,  $\Delta\varepsilon_z$ ).

#### A1.16 Subroutine RCOEF

Function: Calculates the scale factor for elastic response.

#### A1.17 Subroutine FPOUT

Function: Outputs the elastic-plastic solution. In this subroutine some checking formulas, as shown in Eq. (2.48), are used.

Input: N1, N2, the integer numbers which control the range of load increments for the output.

- Output:
- CTE(J) the temperature before current step.
  - CDP(J), CDT(J) the increment of pressure and temperature in current step.
  - BYE(J), EEE(J), ALE(J), EPE(J): material properties  $\sigma_y$ , E,  $\alpha$ ,  $E^t$  for each step.
  - RC(J), scale factor.
  - SC1, SR, SZ, SC2 are check values for  $\sigma_\theta$ ,  $\sigma_v$ ,  $\sigma_z$ . Others are all elastic-plastic response.
  - Here the number of J varies from N1 to N2.

#### A1.18 Subroutine PSRAN

Function: Calculates plastic strains using (2.44).

#### A1.19 Subroutine LAG13

Function: One-dimensional Lagrangian interpolation for  $\sigma_y^o$ . This subroutine is commonly used.

**A1.20 Subroutine LAG23**

Function: Two-dimensional Lagrangian interpolation for  $E^t$ .

**A1.21 Subroutine FIT13**

Function: Calculates the coefficients of fitting a curve with a 5th-degree polynomial.

Input: Data points of the fitting  $X(2,N)$ .

Output: - A(M1) the coefficients of the fitting polynomial  
- M1 the order of fitting polynomial  
- M1=M+1, M2=M1+1  
- S(M1,M2) working array

**A1.22 Subroutine GS**

Function: Solves for the system of linear algebra equations using the direct method.

**A1.23 Subroutine FF**

Function: Fits a 5th-degree polynomial to a set of points.

**A1.24 Subroutine CRSOL**

All array sizes in this subroutine are adjustable. The integer NTC is maximum upper limit of the adjustable array. (NTC = 50)

Function: Solves the creep problem for thermally dependent properties with the Bailoy-Norton Law using iteration.

Input: NT, the number of computing steps which determines the number of increments of  $\Delta t$ ,  $\Delta p$  and  $\Delta T$ .  $NT < NTC$ . The results at  $t=0$ , that come from subroutine EPSOL, are the initial point for creep

computation.

Output: Total stresses, total strains, creep strains and creep strain rate for each time increment.

#### **A1.25 Subroutine CROUT**

Function: Outputs the final results for the creep problem.

Input: NT1, NT2, the integer which is to control the amount of output.

Output: Final results for creep.

#### **A1.26 Subroutine CCDEF**

Function: Determines the coefficients as shown in Eqs. (4.16) and (4.17) of this report for computing creep.

#### **A1.27 Subroutine CSTRS**

Function: Calculates the increments of total stress and strain within one time step.

### **A2. INPUT INSTRUCTIONS**

Program CASEPC reads its primary input on unit 5 and prints the results on units 9 and 10. When the user runs the program the input will be read from an input file called INDATA which should be assigned to connect with unit 5.

The data are input through the following data cards (lines) and each card (line) must start with Column 1.

#### **A2.1. Card 1 Format (\*)**

---

| LL, CDPM, CDTM |

---

LL: Integer. Control parameter indicating solution mode required.  
EQ.0, Elastic-plastic solution only.  
EQ.1, Includes elastic-plastic and creep behavior.

CDPM: If LL=0 put CDPM=0.0.  
If LL=1 enter maximum pressure for the loading condition (in MPa).

CDTM: If LL=0 put CDTM=0.0.  
If LL=1 enter maximum temperature for the loading condition.

**A2.2 Card 2 Format (\*)**

BM, CNEFO, CNBMF

BM: Poisson ratio of casing  
CNEFO: Modulus of the supporting rock  
CNBMF: Poisson ratio of the supporting rock

**A2.3 Card 3 Format (\*)**

DI, DO

DI: Internal diameter of casing (in meters)  
DO: Outer diameter of casing (in meters)

**A2.4 Card 4      Format (\*)**

      
  | NN, N1, N2 |  
    

NN:      Integer number of increments of pressure and temperature for elastic-plastic solution. Since the upper limit of the adjustable array is 50,  $NN \leq 50$ .

N1:      Integer to indicate the load step at which to begin output of the results.

N2:      Integer to indicate load step at which to end output of the results.

Notes:   a.  $N1 \leq N2 \leq NN$ .

         b. N1 and N2 control the output mode for solution results. If you don't want to print the elastic-plastic results, just put  $N1=N2=0$ . In that case the following message will appear on the screen:

          "(NO OUTPUT FOR ELASTIC-PLASTIC SOLUTION. IF YOU WANT,  
          PUT:  $N2 > N1 > 0$ )".

**A2.5 Card 5      Format (\*)**

      
  | CDP(1), CDP(2), ... CDP(NN) |  
    

CDP(J), (J=1,NN):    Pressure increments (in MPa)

Notes:   a. For  $LL=0$ , that is, for elastic-plastic solution only, the input CDP(J) is necessary.

         If  $LL=1$  skip this card. The program generates automatically



all required pressure increments in the following way.

$$CDP(J) = CDPM/NN \quad J=1, NN.$$

- b. The number of values specified must be equal to NN, when LL=0. If less than NN values are input, the unspecified fields should be set zero.

**A2.6 Card 6      Format (\*)**

CDT(1), CDT(2), ... CDT(NN) |

CDT(J), (J=1, NN): Temperature increments

Notes: a. For LL=0, that is for elastic-plastic solution only, the input CDT(J) is necessary.

If LL=1 skip the card. The program generates automatically all required temperature increments in the following way.

$$CDT(J) = CDTM/NN \quad J=1, NN$$

- b. See the previous note b. for Card 5.

Note for cards 5 and 6:

A special strategy is designed in this program to guarantee that after passing the transition from elasticity to plasticity the output values of temperature and pressure still remain integers. However, to do so the increments of pressure and temperature should be taken in a definite way. Both must be either increasing or decreasing at the same time. However, one of them may be kept constant, but if one is increased and the other is decreased it will bring confusion. In fact, it is impossible for the latter case to arise in engineering practice. So the above restriction on the input

of pressure and temperature increments is reasonable.

Example: If NN=30, the user might input the following

$\Delta P$ : 20 \* 0.8E6, 10 \* 0.0

$\Delta T$ : 15 \* 20, 5 \* 10, 10 \* 0.0

or  $\Delta P$ : 16.E6, 19 \* 0.0, 10 \* -0.8E6

$\Delta T$ : 20 \* 20, 10 \* 0.0

But what is shown below is not permissible

$\Delta P$ : 10 \* 0.8E6, 10 \* -0.8E6, 10 \* 0.0

$\Delta T$ : 20 \* 20, 10 \* 0.0

**A2.7 Card 7      Format (\*)** If LL=0, skip this card.

NT, NT1, NT2

NT:            Integer = the number of increments of the time step-length for creep computation. NT  $\leq$  50.

NT1:           Integer = beginning point for output of the results.

NT2:           Integer = ending point for output of the results.

Notes:        a. NT1  $\leq$  NT2  $\leq$  NT

b. NT1 and NT2 control the output mode for solution results. If you don't want to print the creep results, just put NT1=NT2=0. In this case the following message will appear on the screen:

"(NO OUTPUT FOR CREEP RESULTS. IF YOU WANT, PUT: NT2  $>$  NT1  $>$  0)".

**A2.8 Card 8**      **Format (\*)** If LL=0, skip this card.

CTI(1), CTI(2), ... CTI(NT)

CTI(J), (J=1,NT): Time increments (in hours)

Note: It is important that the value specified for each time increment should be more than zero. A zero increment is absolutely forbidden. Otherwise the computation will fail to proceed and the following message will appear on the screen:

"TIME INCREMENT SHOULD BE GREATER ZERO, IMPORTANT!"

**A2.9 Card 9**      **Format (\*)** If LL=0, skip this card.

CCDP(1), CCDP(2), ... CCDP(NT)

CCDP(J), (J=1,NT): Pressure increments within the correspondent time interval as  $t > 0$ .

**A2.10 Card 10**      **Format (\*)** If LL=0, skip this card.

CCDT(1), CCDT(2), ... CCDT(NT)

CCDT(J), (J=1,NT): Temperature increments within the correspondent time interval as  $t > 0$ .

Note: For cards 9 and 10, the number of values specified must be equal to NT, when LL=1. If less than NT values are input, or if no increment



predictions. Unit 10 includes the prediction of initial yield only. To obtain the results the user needs to create two files, for instance, RESULT1 and RESULT2, which are to be connected with unit 9 and unit 10, respectively.

#### A4. Operation

In MTS the user first compiles the program using the Fortran compiler (if needed):

```
$RUN *FORTRANVS SCARDS=WEACC SPUNCH=OBJ
```

Then the program is invoked with a \$RUN command:

```
$RUN OBJ 5=INDATA 9=RESULT1 10=RESULT2 T=5S
```

Then the command LIST RESULT1 or LIST RESULT2 presents the results of the solution.

#### A5. Examples

Four examples of input and output forms are listed below. All examples assume a 4.5" casing, Poisson ratio = 0.28, and an oil sand geological formation under the loading conditions  $P = 16.5$  MPa and  $T = 350^{\circ}\text{C}$ .

##### Example Case 1

Elastic-plastic solution only.

<Input>

Card 1 LL, CDPM, CDTM : 0, 0.0, 0.0

Card 2 BM, CNEF, CNBMF : 0.28, 2200.E6, 0.25

Card 3 DI, DO : 0.1039, 0.1143  
Card 4 NN, N1, N2 : 3.0, 14, 21  
Card 5 CDP(I), I=1,30 : 1.6 \* 1.E6, 14 \* 0.5E6  
Card 6 CDT(I), I=1,30 : 16 \* 20.0, 30.0, 13 \* 0.0

Card 7 - Card 12 are omitted.

<Output> RESULT1 : Elastic-plastic solution from Steps 14-21. No  
creep output.

RESULT2 : Prediction of initial yield.

### Example Case 2

Elastic-plastic solution and creep prediction.

<Input>

Card 1 LL, CDPM, CDTM : 1, 16.5E6, 350.  
Card 2 BM, CNEF, CNBMF : 0.28, 2200.E6, 0.25  
Card 3 DI, DO : 0.1039, 0.1143  
Card 4 NN, N1, N2 : 20, 18, 20  
Card 5 Omitted  
Card 6 Omitted  
Card 7 NT, NT1, NT2 : 25, 1, 10  
Card 8 CTI(I), I=1,25 : 2\*1.0, 4\*2.0, 10., 30., 7\*50., 10\*100.  
Card 9 CCDP(I), I=1,25 : 25 \* 0.0  
Card 10 CC DT(I), I=1,25 : 25 \* 0.0  
Card 11 EPS, CO(I), I=1,3 : 1.E-7, 1.E-3, 5.E-4, -2.E-3  
Card 12 CAC, CMC, CNC : 1.5E-32, 6.0, 0.667

<Output> RESULT1 : Elastic-plastic solution from Steps 18-20.  
Creep prediction from time Steps 1-10.  
(Maximum time interval is 1400 hours, no

increments of pressure and temperature as  
t>0).

RESULT2 : No change from Case 1.

### Example Case 3

The solution is the same as the Case 2, except that no elastic-plastic results are printed.

In this case only Card 4 needs to be changed to the following.

Card 4 NN, N1, N2 : 20, 0, 0

<Output> RESULT1 : Only creep results are printed.

RESULT2 : No change.

### Example Case 4

The same as Case 2 except that neither elastic-plastic nor creep solution is printed.

In this case Card 4 and Card 7 need to be changed.

Card 4 NN, N1, N2 : 20, 0, 0

Card 7 NT, NT1, NT2 : 25, 0, 0

<Output> RESULT1 : Only original input is printed.

RESULT2 : No change.

The output forms for the four cases listed above are shown sequentially on the following pages.

## A6. TEMPERATURE DEPENDENT PROPERTIES

Note that temperature dependent properties for NM80 steel, as supplied by C-FER and displayed in Table 4.2, are contained in the BLOCK DATA subroutine.

Reference to the BLOCK DATA subroutine in the program listing of Appendix B shows how the data of Table 4.2 has been assigned to the arrays defined in Sect. A1.3. If different temperature dependent material property data is supplied, the data in the BLOCK DATA subroutine can be altered and the program recompiled.



&lt;CASE1 OUTPUT&gt;

## CASEPC RESULTS

\*\*\*\*\*

( FILE NAME : WEACC )

## &lt;INPUT DATA&gt;

OUTER DIAMETER	WALL THICKNESS	POISSON RATIO
0.1143000	0.0052000	0.2800

MODULUS OF ROCK	POISSON RATIO OF ROCK
0.2200E+10	0.2500

INCREMENT OF PRESSURE AT  $t=0$ :

0.1000E+07	0.1000E+07	0.1000E+07	0.1000E+07	0.1000E+07	0.1000E+07
0.1000E+07	0.1000E+07	0.1000E+07	0.1000E+07	0.1000E+07	0.1000E+07
0.1000E+07	0.1000E+07	0.1000E+07	0.1000E+07	0.5000E+06	0.5000E+06
0.5000E+06	0.5000E+06	0.5000E+06	0.5000E+06	0.5000E+06	0.5000E+06
0.5000E+06	0.5000E+06	0.5000E+06	0.5000E+06	0.5000E+06	0.5000E+06

INCREMENT OF TEMPERATURE AT  $t=0$ :

20.0000	20.0000	20.0000	20.0000	20.0000	20.0000
20.0000	20.0000	20.0000	20.0000	20.0000	20.0000
20.0000	20.0000	20.0000	20.0000	30.0000	0.0000
0.0000	0.0000	0.0000	0.0000	0.0000	0.0000
0.0000	0.0000	0.0000	0.0000	0.0000	0.0000

## INCREMENT OF TIME:

0.0000	0.0000	0.0000	0.0000	0.0000	0.0000
--------	--------	--------	--------	--------	--------

INCREMENT OF PRESSURE AS  $t>0$ :

0.0000E+00	0.0000E+00	0.0000E+00	0.0000E+00	0.0000E+00	0.0000E+00
------------	------------	------------	------------	------------	------------

INCREMENT OF TEMPERATURE AS  $t>0$ :

0.0000	0.0000	0.0000	0.0000	0.0000	0.0000
--------	--------	--------	--------	--------	--------

## &lt;PHYSICAL PROPERTIES OF THE CASING UNDER DIFFERENTE TEMPERATURE&gt;:

POINT- 1:	0.00	BYO= 0.6220E+09	E= 0.2075E+12	ALFA= 0.1138E-04
POINT- 2:	50.00	BYO= 0.5937E+09	E= 0.1995E+12	ALFA= 0.1173E-04
POINT- 3:	100.00	BYO= 0.5659E+09	E= 0.1885E+12	ALFA= 0.1205E-04
POINT- 4:	150.00	BYO= 0.5352E+09	E= 0.1780E+12	ALFA= 0.1236E-04
POINT- 5:	200.00	BYO= 0.5027E+09	E= 0.1680E+12	ALFA= 0.1267E-04
POINT- 6:	250.00	BYO= 0.4683E+09	E= 0.1572E+12	ALFA= 0.1299E-04

## &lt;ELASTIC-PLASTIC SOLUTION&gt;

\*\*\*\*\*

```

=====
CT = 280.00
BY = 0.4441E+09 E = 0.1497E+12 ALFA = 0.1318E-04 EP = 0.3907E+10

STEP ( 15 ): DP = 0.1000E+07 DT = 20.00 FACTOR--RR= .000

PRES-R= 0.1500E+08 TEMP-T= 300.00 SGMA--Y= 0.5068E+09
SGMA-C= 0.4725E+08 SGMA-R=-0.1500E+08 SGMA-Z=-0.4878E+09 SGMA-BR= 0.5068E+09
EPS--C= 0.5304E-02 EPS--R= 0.4693E-02 EPS--Z= 0.0000E+00 EPS--BR= 0.3351E-02
EPS-CP= 0.6086E-03 EPS-RP= 0.4362E-03 EPS-ZP=-0.1045E-02 EPS-BRP= 0.1050E-02
=====

```

&lt;CHECK THE NUMBER VALUE OF STRESS&gt;:

```

SGMC-1= 0.4725E+08 SGMR-1=-0.1500E+08 SGMZ-1=-0.4878E+09
SGMC-2= 0.4725E+08
=====

```

```

=====
CT = 300.00
BY = 0.4305E+09 E = 0.1444E+12 ALFA = 0.1330E-04 EP = 0.3864E+10

STEP ( 16 ): DP = 0.1000E+07 DT = 20.00 FACTOR--RR= .000

PRES-R= 0.1600E+08 TEMP-T= 320.00 SGMA--Y= 0.5078E+09
SGMA-C= 0.4883E+08 SGMA-R=-0.1600E+08 SGMA-Z=-0.4883E+09 SGMA-BR= 0.5078E+09
EPS--C= 0.5739E-02 EPS--R= 0.5057E-02 EPS--Z= 0.0000E+00 EPS--BR= 0.3620E-02
EPS-CP= 0.7634E-03 EPS-RP= 0.5426E-03 EPS-ZP=-0.1306E-02 EPS-BRP= 0.1312E-02
=====

```

&lt;CHECK THE NUMBER VALUE OF STRESS&gt;:

```

SGMC-1= 0.4883E+08 SGMR-1=-0.1600E+08 SGMZ-1=-0.4883E+09
SGMC-2= 0.4884E+08
=====

```

```

=====
CT = 320.00
BY = 0.4174E+09 E = 0.1388E+12 ALFA = 0.1342E-04 EP = 0.3766E+10

STEP ( 17 ): DP = 0.5000E+06 DT = 30.00 FACTOR--RR= 0.000

PRES-R= 0.1650E+08 TEMP-T= 350.00 SGMA--Y= 0.5093E+09
SGMA-C= 0.4236E+08 SGMA-R=-0.1650E+08 SGMA-Z=-0.4938E+09 SGMA-BR= 0.5093E+09
EPS--C= 0.6332E-02 EPS--R= 0.5632E-02 EPS--Z= 0.0000E+00 EPS--BR= 0.4008E-02
EPS-CP= 0.9882E-03 EPS-RP= 0.6948E-03 EPS-ZP=-0.1683E-02 EPS-BRP= 0.1692E-02
=====

```

&lt;CHECK THE NUMBER VALUE OF STRESS&gt;:

```

SGMC-1= 0.4236E+08 SGMR-1=-0.1650E+08 SGMZ-1=-0.4938E+09
SGMC-2= 0.4236E+08
=====

```

CT = 350.00  
 BY = 0.3986E+09 E = 0.1304E+12 ALFA = 0.1360E-04 EP = 0.3519E+10  
 STEP ( 18 ): DP = 0.5000E+06 DT = 0.00 FACTOR--RR= 0.000  
 PRES-R= 0.1700E+08 TEMP-T= 350.00 SGMA--Y= 0.5093E+09  
 SGMA-C= 0.4668E+08 SGMA-R=-0.1700E+08 SGMA-Z=-0.4915E+09 SGMA-BR= 0.5094E+09  
 EPS--C= 0.6367E-02 EPS--R= 0.5618E-02 EPS--Z= 0.0000E+00 EPS--BR= 0.4018E-02  
 EPS-CP= 0.9937E-03 EPS-RP= 0.6986E-03 EPS-ZP=-0.1692E-02 EPS-BRP= 0.1701E-02

<CHECK THE NUMBER VALUE OF STRESS>:  
 SGMC-1= 0.4668E+08 SGMR-1=-0.1700E+08 SGMZ-1=-0.4915E+09  
 SGMC-2= 0.4668E+08

CT = 350.00  
 BY = 0.3986E+09 E = 0.1304E+12 ALFA = 0.1360E-04 EP = 0.3519E+10  
 STEP ( 19 ): DP = 0.5000E+06 DT = 0.00 FACTOR--RR= .000  
 PRES-R= 0.1750E+08 TEMP-T= 350.00 SGMA--Y= 0.5093E+09  
 SGMA-C= 0.5100E+08 SGMA-R=-0.1750E+08 SGMA-Z=-0.4892E+09 SGMA-BR= 0.5094E+09  
 EPS--C= 0.6402E-02 EPS--R= 0.5604E-02 EPS--Z= 0.0000E+00 EPS--BR= 0.4028E-02  
 EPS-CP= 0.9995E-03 EPS-RP= 0.7026E-03 EPS-ZP=-0.1702E-02 EPS-BRP= 0.1711E-02

<CHECK THE NUMBER VALUE OF STRESS>:  
 SGMC-1= 0.5100E+08 SGMR-1=-0.1750E+08 SGMZ-1=-0.4892E+09  
 SGMC-2= 0.5100E+08

CT = 350.00  
 BY = 0.3986E+09 E = 0.1304E+12 ALFA = 0.1360E-04 EP = 0.3519E+10  
 STEP ( 20 ): DP = 0.5000E+06 DT = 0.00 FACTOR--RR= .000  
 PRES-R= 0.1800E+08 TEMP-T= 350.00 SGMA--Y= 0.5094E+09  
 SGMA-C= 0.5532E+08 SGMA-R=-0.1800E+08 SGMA-Z=-0.4869E+09 SGMA-BR= 0.5095E+09  
 EPS--C= 0.6437E-02 EPS--R= 0.5590E-02 EPS--Z= 0.0000E+00 EPS--BR= 0.4039E-02  
 EPS-CP= 0.1005E-02 EPS-RP= 0.7065E-03 EPS-ZP=-0.1712E-02 EPS-BRP= 0.1721E-02

<CHECK THE NUMBER VALUE OF STRESS>:  
 SGMC-1= 0.5532E+08 SGMR-1=-0.1800E+08 SGMZ-1=-0.4869E+09  
 SGMC-2= 0.5532E+08

CT = 350.00  
 BY = 0.3986E+09 E = 0.1304E+12 ALFA = 0.1360E-04 EP = 0.3519E+10  
 STEP ( 21 ): DP = 0.5000E+06 DT = 0.00 FACTOR--RR= .000  
 PRES-R= 0.1850E+08 TEMP-T= 350.00 SGMA--Y= 0.5094E+09  
 SGMA-C= 0.5963E+08 SGMA-R=-0.1850E+08 SGMA-Z=-0.4845E+09 SGMA-BR= 0.5095E+09  
 EPS--C= 0.6472E-02 EPS--R= 0.5575E-02 EPS--Z= 0.0000E+00 EPS--BR= 0.4049E-02  
 EPS-CP= 0.1012E-02 EPS-RP= 0.7105E-03 EPS-ZP=-0.1722E-02 EPS-BRP= 0.1731E-02

<CHECK THE NUMBER VALUE OF STRESS>:  
 SGMC-1= 0.5963E+08 SGMR-1=-0.1850E+08 SGMZ-1=-0.4845E+09  
 SGMC-2= 0.5963E+08

( NO OUTPUT FOR CREEP SOLUTION. IF YOU WANT, PUT: LL=1 )

C  
C  
C  
C

&lt;CASE2 OUTPUT&gt;

## CASEPC RESULTS

\*\*\*\*\*

( FILE NAME : WEACC )

## =====

&lt;INPUT DATA&gt;

-----

OUTER DIAMETER	WALL THICKNESS	POISSON RATIO
0.1143000	0.0052000	0.2800

MODULUS OF ROCK	POISSON RATIO OF ROCK
0.2200E+10	0.2500

INCREMENT OF PRESSURE AT  $t=0$ :

0.8250E+06	0.8250E+06	0.8250E+06	0.8250E+06	0.8250E+06	0.8250E+06
0.8250E+06	0.8250E+06	0.8250E+06	0.8250E+06	0.8250E+06	0.8250E+06
0.8250E+06	0.8250E+06	0.8250E+06	0.8250E+06	0.8250E+06	0.8250E+06
0.8250E+06	0.8250E+06				

INCREMENT OF TEMPERATURE AT  $t=0$ :

17.5000	17.5000	17.5000	17.5000	17.5000	17.5000
17.5000	17.5000	17.5000	17.5000	17.5000	17.5000
17.5000	17.5000	17.5000	17.5000	17.5000	17.5000
17.5000	17.5000				

## INCREMENT OF TIME:

1.0000	1.0000	2.0000	2.0000	2.0000	2.0000
10.0000	30.0000	50.0000	50.0000	50.0000	50.0000
50.0000	50.0000	50.0000	100.0000	100.0000	100.0000
100.0000	100.0000	100.0000	100.0000	100.0000	100.0000
100.0000					

INCREMENT OF PRESSURE AS  $t>0$ :

0.0000E+00	0.0000E+00	0.0000E+00	0.0000E+00	0.0000E+00	0.0000E+00
0.0000E+00	0.0000E+00	0.0000E+00	0.0000E+00	0.0000E+00	0.0000E+00
0.0000E+00	0.0000E+00	0.0000E+00	0.0000E+00	0.0000E+00	0.0000E+00
0.0000E+00	0.0000E+00	0.0000E+00	0.0000E+00	0.0000E+00	0.0000E+00
0.0000E+00					

INCREMENT OF TEMPERATURE AS  $t>0$ :

0.0000	0.0000	0.0000	0.0000	0.0000	0.0000
0.0000	0.0000	0.0000	0.0000	0.0000	0.0000
0.0000	0.0000	0.0000	0.0000	0.0000	0.0000
0.0000	0.0000	0.0000	0.0000	0.0000	0.0000
0.0000					

## =====

&lt;PHYSICAL PROPERTIES OF THE CASING UNDER DIFFERENTE TEMPERATURE&gt;:

POINT- 1:	0.00	BY0= 0.6220E+09	E= 0.2075E+12	ALFA= 0.1138E-04
POINT- 2:	50.00	BY0= 0.5937E+09	E= 0.1995E+12	ALFA= 0.1173E-04
POINT- 3:	100.00	BY0= 0.5659E+09	E= 0.1885E+12	ALFA= 0.1205E-04
POINT- 4:	150.00	BY0= 0.5352E+09	E= 0.1780E+12	ALFA= 0.1236E-04
POINT- 5:	200.00	BY0= 0.5027E+09	E= 0.1680E+12	ALFA= 0.1267E-04
POINT- 6:	250.00	BY0= 0.4683E+09	E= 0.1572E+12	ALFA= 0.1299E-04

=====

## &lt;ELASTIC-PLASTIC SOLUTION&gt;

\*\*\*\*\*

```

=====
CT = 297.50
BY = 0.4322E+09 E = 0.1451E+12 ALFA = 0.1329E-04 EP = 0.3872E+10

STEP ( 18 ): DP = 0.8250E+06 DT = 17.50 FACTOR--RR= -.000

PRES-R= 0.1485E+08 TEMP-T= 315.00 SGMA--Y= 0.5009E+09
SGMA-C= 0.4038E+08 SGMA-R=-0.1485E+08 SGMA-Z=-0.4859E+09 SGMA-BR= 0.5009E+09
EPS--C= 0.5582E-02 EPS--R= 0.5005E-02 EPS--Z= 0.0000E+00 EPS--BR= 0.3545E-02
EPS-CP= 0.7248E-03 EPS-RP= 0.5390E-03 EPS-ZP=-0.1264E-02 EPS-BRP= 0.1268E-02
=====

```

&lt;CHECK THE NUMBER VALUE OF STRESS&gt;:

```

SGMC-1= 0.4038E+08 SGMR-1=-0.1485E+08 SGMZ-1=-0.4859E+09
SGMC-2= 0.4038E+08
=====

```

```

=====
CT = 315.00
BY = 0.4206E+09 E = 0.1402E+12 ALFA = 0.1339E-04 EP = 0.3796E+10

STEP ( 19 ): DP = 0.8250E+06 DT = 17.50 FACTOR--RR= -.000

PRES-R= 0.1567E+08 TEMP-T= 332.50 SGMA--Y= 0.5018E+09
SGMA-C= 0.4131E+08 SGMA-R=-0.1567E+08 SGMA-Z=-0.4866E+09 SGMA-BR= 0.5018E+09
EPS--C= 0.5960E-02 EPS--R= 0.5329E-02 EPS--Z= 0.0000E+00 EPS--BR= 0.3781E-02
EPS-CP= 0.8585E-03 EPS-RP= 0.6346E-03 EPS-ZP=-0.1493E-02 EPS-BRP= 0.1499E-02
=====

```

&lt;CHECK THE NUMBER VALUE OF STRESS&gt;:

```

SGMC-1= 0.4131E+08 SGMR-1=-0.1567E+08 SGMZ-1=-0.4866E+09
SGMC-2= 0.4131E+08
=====

```

```

=====
CT = 332.50
BY = 0.4094E+09 E = 0.1352E+12 ALFA = 0.1350E-04 EP = 0.3678E+10

STEP ( 20 ): DP = 0.8250E+06 DT = 17.50 FACTOR--RR= -.000

PRES-R= 0.1650E+08 TEMP-T= 350.00 SGMA--Y= 0.5027E+09
SGMA-C= 0.4218E+08 SGMA-R=-0.1650E+08 SGMA-Z=-0.4873E+09 SGMA-BR= 0.5027E+09
EPS--C= 0.6341E-02 EPS--R= 0.5654E-02 EPS--Z= 0.0000E+00 EPS--BR= 0.4018E-02
EPS-CP= 0.9937E-03 EPS-RP= 0.7302E-03 EPS-ZP=-0.1724E-02 EPS-BRP= 0.1731E-02
=====

```

&lt;CHECK THE NUMBER VALUE OF STRESS&gt;:

```

SGMC-1= 0.4218E+08 SGMR-1=-0.1650E+08 SGMZ-1=-0.4873E+09
SGMC-2= 0.4218E+08
=====

```

<CREEP-TIME DEPENDENT SOLUTION>  
 \*\*\*\*\*

```

=====
<ORIGINAL STATE>:  INITIAL TIME t= 0.0
                    PRES-R= 0.1650E+08  TEMP-T= 350.00
SGMA-C= 0.4218E+08  SGMA-R=-0.1650E+08  SGMA-Z=-0.4873E+09  SGMA-BR= 0.5027E+09
EPS--C= 0.6341E-02  EPS--R= 0.5654E-02  EPS--Z= 0.0000E+00  EPS--BR= 0.4018E-02
=====

STEP( 1 ):  CUMULATIVE TIME=    1.00  C-RATE= 0.5615E-03  CGMA=BR= 0.4266E+09

SGMA-C= 0.3875E+08  SGMA-R=-0.1650E+08  SGMA-Z=-0.4128E+09  SGMA-BR= 0.4266E+09
EPS--C= 0.6495E-02  EPS--R= 0.5732E-02  EPS--Z= 0.0000E+00  EPS--BR= 0.4099E-02
C-EPSC= 0.3335E-03  C-EPSR= 0.2244E-03  C-EPSZ=-0.5579E-03  C-EPSBR= 0.5615E-03
-----

STEP( 2 ):  CUMULATIVE TIME=    2.00  C-RATE= 0.2640E-03  CGMA=BR= 0.3910E+09

SGMA-C= 0.3711E+08  SGMA-R=-0.1650E+08  SGMA-Z=-0.3778E+09  SGMA-BR= 0.3909E+09
EPS--C= 0.6569E-02  EPS--R= 0.5766E-02  EPS--Z= 0.0000E+00  EPS--BR= 0.4138E-02
C-EPSC= 0.4917E-03  C-EPSR= 0.3283E-03  C-EPSZ=-0.8200E-03  C-EPSBR= 0.8254E-03
-----

STEP( 3 ):  CUMULATIVE TIME=    4.00  C-RATE= 0.1230E-03  CGMA=BR= 0.3577E+09

SGMA-C= 0.3555E+08  SGMA-R=-0.1650E+08  SGMA-Z=-0.3452E+09  SGMA-BR= 0.3576E+09
EPS--C= 0.6639E-02  EPS--R= 0.5797E-02  EPS--Z= 0.0000E+00  EPS--BR= 0.4174E-02
C-EPSC= 0.6405E-03  C-EPSR= 0.4234E-03  C-EPSZ=-0.1064E-02  C-EPSBR= 0.1071E-02
-----

STEP( 4 ):  CUMULATIVE TIME=    6.00  C-RATE= 0.7559E-04  CGMA=BR= 0.3374E+09

SGMA-C= 0.3458E+08  SGMA-R=-0.1650E+08  SGMA-Z=-0.3252E+09  SGMA-BR= 0.3372E+09
EPS--C= 0.6682E-02  EPS--R= 0.5816E-02  EPS--Z= 0.0000E+00  EPS--BR= 0.4196E-02
C-EPSC= 0.7325E-03  C-EPSR= 0.4811E-03  C-EPSZ=-0.1214E-02  C-EPSBR= 0.1222E-02
-----

STEP( 5 ):  CUMULATIVE TIME=    8.00  C-RATE= 0.5302E-04  CGMA=BR= 0.3231E+09

SGMA-C= 0.3390E+08  SGMA-R=-0.1650E+08  SGMA-Z=-0.3112E+09  SGMA-BR= 0.3229E+09
EPS--C= 0.6713E-02  EPS--R= 0.5828E-02  EPS--Z= 0.0000E+00  EPS--BR= 0.4211E-02
C-EPSC= 0.7974E-03  C-EPSR= 0.5212E-03  C-EPSZ=-0.1319E-02  C-EPSBR= 0.1328E-02
-----

STEP( 6 ):  CUMULATIVE TIME=   10.00  C-RATE= 0.4020E-04  CGMA=BR= 0.3124E+09

SGMA-C= 0.3337E+08  SGMA-R=-0.1650E+08  SGMA-Z=-0.3006E+09  SGMA-BR= 0.3120E+09
EPS--C= 0.6737E-02  EPS--R= 0.5838E-02  EPS--Z= 0.0000E+00  EPS--BR= 0.4223E-02
C-EPSC= 0.8468E-03  C-EPSR= 0.5514E-03  C-EPSZ=-0.1398E-02  C-EPSBR= 0.1409E-02
=====

```

STEP( 7 ): CUMULATIVE TIME= 20.00 C-RATE= 0.1894E-04 CGMA=BR= 0.2864E+09  
 SGMA-C= 0.3211E+08 SGMA-R=-0.1650E+08 SGMA-Z=-0.2756E+09 SGMA-BR= 0.2865E+09  
 EPS--C= 0.6793E-02 EPS--R= 0.5858E-02 EPS--Z= 0.0000E+00 EPS--BR= 0.4252E-02  
 C-EPSC= 0.9647E-03 C-EPSR= 0.6210E-03 C-EPSZ=-0.1586E-02 C-EPSBR= 0.1598E-02  
 -----

STEP( 8 ): CUMULATIVE TIME= 50.00 C-RATE= 0.7291E-05 CGMA=BR= 0.2570E+09  
 SGMA-C= 0.3061E+08 SGMA-R=-0.1650E+08 SGMA-Z=-0.2468E+09 SGMA-BR= 0.2571E+09  
 EPS--C= 0.6861E-02 EPS--R= 0.5879E-02 EPS--Z= 0.0000E+00 EPS--BR= 0.4284E-02  
 C-EPSC= 0.1103E-02 C-EPSR= 0.6989E-03 C-EPSZ=-0.1802E-02 C-EPSBR= 0.1817E-02  
 -----

STEP( 9 ): CUMULATIVE TIME= 100.00 C-RATE= 0.3359E-05 CGMA=BR= 0.2347E+09  
 SGMA-C= 0.2943E+08 SGMA-R=-0.1650E+08 SGMA-Z=-0.2247E+09 SGMA-BR= 0.2346E+09  
 EPS--C= 0.6914E-02 EPS--R= 0.5894E-02 EPS--Z= 0.0000E+00 EPS--BR= 0.4310E-02  
 C-EPSC= 0.1210E-02 C-EPSR= 0.7570E-03 C-EPSZ=-0.1967E-02 C-EPSBR= 0.1984E-02  
 -----

STEP( 10 ): CUMULATIVE TIME= 150.00 C-RATE= 0.2051E-05 CGMA=BR= 0.2211E+09  
 SGMA-C= 0.2869E+08 SGMA-R=-0.1650E+08 SGMA-Z=-0.2113E+09 SGMA-BR= 0.2209E+09  
 EPS--C= 0.6947E-02 EPS--R= 0.5903E-02 EPS--Z= 0.0000E+00 EPS--BR= 0.4325E-02  
 C-EPSC= 0.1276E-02 C-EPSR= 0.7917E-03 C-EPSZ=-0.2068E-02 C-EPSBR= 0.2087E-02  
 -----

C  
C  
C  
C

&lt;CASE3 OUTPUT&gt;

## CASEPC RESULTS

\*\*\*\*\*

( FILE NAME : WEACC )

-----  
<INPUT DATA>

OUTER DIAMETER	WALL THICKNESS	POISSON RATIO
0.1143000	0.0052000	0.2800

MODULUS OF ROCK	POISSON RATIO OF ROCK
0.2200E+10	0.2500

INCREMENT OF PRESSURE AT  $t=0$ :

0.8250E+06	0.8250E+06	0.8250E+06	0.8250E+06	0.8250E+06	0.8250E+06
0.8250E+06	0.8250E+06	0.8250E+06	0.8250E+06	0.8250E+06	0.8250E+06
0.8250E+06	0.8250E+06	0.8250E+06	0.8250E+06	0.8250E+06	0.8250E+06
0.8250E+06	0.8250E+06				

INCREMENT OF TEMPERATURE AT  $t=0$ :

17.5000	17.5000	17.5000	17.5000	17.5000	17.5000
17.5000	17.5000	17.5000	17.5000	17.5000	17.5000
17.5000	17.5000	17.5000	17.5000	17.5000	17.5000
17.5000	17.5000				

## INCREMENT OF TIME:

1.0000	1.0000	2.0000	2.0000	2.0000	2.0000
10.0000	30.0000	50.0000	50.0000	50.0000	50.0000
50.0000	50.0000	50.0000	100.0000	100.0000	100.0000
100.0000	100.0000	100.0000	100.0000	100.0000	100.0000
100.0000					

INCREMENT OF PRESSURE AS  $t>0$ :

0.0000E+00	0.0000E+00	0.0000E+00	0.0000E+00	0.0000E+00	0.0000E+00
0.0000E+00	0.0000E+00	0.0000E+00	0.0000E+00	0.0000E+00	0.0000E+00
0.0000E+00	0.0000E+00	0.0000E+00	0.0000E+00	0.0000E+00	0.0000E+00
0.0000E+00	0.0000E+00	0.0000E+00	0.0000E+00	0.0000E+00	0.0000E+00
0.0000E+00					

INCREMENT OF TEMPERATURE AS  $t>0$ :

0.0000	0.0000	0.0000	0.0000	0.0000	0.0000
0.0000	0.0000	0.0000	0.0000	0.0000	0.0000
0.0000	0.0000	0.0000	0.0000	0.0000	0.0000
0.0000	0.0000	0.0000	0.0000	0.0000	0.0000
0.0000					

-----  
<PHYSICAL PROPERTIES OF THE CASING UNDER DIFFERENTE TEMPERATURE>:

POINT- 1:	0.00	BYO= 0.6220E+09	E= 0.2075E+12	ALFA= 0.1138E-04
POINT- 2:	50.00	BYO= 0.5937E+09	E= 0.1995E+12	ALFA= 0.1173E-04
POINT- 3:	100.00	BYO= 0.5659E+09	E= 0.1885E+12	ALFA= 0.1205E-04
POINT- 4:	150.00	BYO= 0.5352E+09	E= 0.1780E+12	ALFA= 0.1236E-04
POINT- 5:	200.00	BYO= 0.5027E+09	E= 0.1680E+12	ALFA= 0.1267E-04
POINT- 6:	250.00	BYO= 0.4683E+09	E= 0.1572E+12	ALFA= 0.1299E-04

-----  
( NO OUTPUT FOR ELASTIC-PLASTIC SOLUTION. IF YOU WANT, PUT: N2 > N1 > 0 )



## &lt;CREEP-TIME DEPENDENT SOLUTION&gt;

\*\*\*\*\*

```

=====
<ORIGINAL STATE>:  INITIAL TIME t= 0.0
                   PRES-R= 0.1650E+08  TEMP-T= 350.00
SGMA-C= 0.4218E+08  SGMA-R=-0.1650E+08  SGMA-Z=-0.4873E+09  SGMA-BR= 0.5027E+09
EPS--C= 0.6341E-02  EPS--R= 0.5654E-02  EPS--Z= 0.0000E+00  EPS--BR= 0.4018E-02
=====

STEP( 1 ):  CUMULATIVE TIME=    1.00  C-RATE= 0.5615E-03  CGMA=BR= 0.4266E+09

SGMA-C= 0.3875E+08  SGMA-R=-0.1650E+08  SGMA-Z=-0.4128E+09  SGMA-BR= 0.4266E+09
EPS--C= 0.6495E-02  EPS--R= 0.5732E-02  EPS--Z= 0.0000E+00  EPS--BR= 0.4099E-02
C-EPSC= 0.3335E-03  C-EPSR= 0.2244E-03  C-EPSZ=-0.5579E-03  C-EPSBR= 0.5615E-03
-----

STEP( 2 ):  CUMULATIVE TIME=    2.00  C-RATE= 0.2640E-03  CGMA=BR= 0.3910E+09

SGMA-C= 0.3711E+08  SGMA-R=-0.1650E+08  SGMA-Z=-0.3778E+09  SGMA-BR= 0.3909E+09
EPS--C= 0.6569E-02  EPS--R= 0.5766E-02  EPS--Z= 0.0000E+00  EPS--BR= 0.4138E-02
C-EPSC= 0.4917E-03  C-EPSR= 0.3283E-03  C-EPSZ=-0.8200E-03  C-EPSBR= 0.8254E-03
-----

STEP( 3 ):  CUMULATIVE TIME=    4.00  C-RATE= 0.1230E-03  CGMA=BR= 0.3577E+09

SGMA-C= 0.3555E+08  SGMA-R=-0.1650E+08  SGMA-Z=-0.3452E+09  SGMA-BR= 0.3576E+09
EPS--C= 0.6639E-02  EPS--R= 0.5797E-02  EPS--Z= 0.0000E+00  EPS--BR= 0.4174E-02
C-EPSC= 0.6405E-03  C-EPSR= 0.4234E-03  C-EPSZ=-0.1064E-02  C-EPSBR= 0.1071E-02
-----

STEP( 4 ):  CUMULATIVE TIME=    6.00  C-RATE= 0.7559E-04  CGMA=BR= 0.3374E+09

SGMA-C= 0.3458E+08  SGMA-R=-0.1650E+08  SGMA-Z=-0.3252E+09  SGMA-BR= 0.3372E+09
EPS--C= 0.6682E-02  EPS--R= 0.5816E-02  EPS--Z= 0.0000E+00  EPS--BR= 0.4196E-02
C-EPSC= 0.7325E-03  C-EPSR= 0.4811E-03  C-EPSZ=-0.1214E-02  C-EPSBR= 0.1222E-02
-----

STEP( 5 ):  CUMULATIVE TIME=    8.00  C-RATE= 0.5302E-04  CGMA=BR= 0.3231E+09

SGMA-C= 0.3390E+08  SGMA-R=-0.1650E+08  SGMA-Z=-0.3112E+09  SGMA-BR= 0.3229E+09
EPS--C= 0.6713E-02  EPS--R= 0.5828E-02  EPS--Z= 0.0000E+00  EPS--BR= 0.4211E-02
C-EPSC= 0.7974E-03  C-EPSR= 0.5212E-03  C-EPSZ=-0.1319E-02  C-EPSBR= 0.1328E-02
-----

STEP( 6 ):  CUMULATIVE TIME=   10.00  C-RATE= 0.4020E-04  CGMA=BR= 0.3124E+09

SGMA-C= 0.3337E+08  SGMA-R=-0.1650E+08  SGMA-Z=-0.3006E+09  SGMA-BR= 0.3120E+09
EPS--C= 0.6737E-02  EPS--R= 0.5838E-02  EPS--Z= 0.0000E+00  EPS--BR= 0.4223E-02
C-EPSC= 0.8468E-03  C-EPSR= 0.5514E-03  C-EPSZ=-0.1398E-02  C-EPSBR= 0.1409E-02
=====

```

STEP( 7 ): CUMULATIVE TIME= 20.00 C-RATE= 0.1894E-04 CGMA=BR= 0.2864E+09  
 SGMA-C= 0.3211E+08 SGMA-R=-0.1650E+08 SGMA-Z=-0.2756E+09 SGMA-BR= 0.2865E+09  
 EPS--C= 0.6793E-02 EPS--R= 0.5858E-02 EPS--Z= 0.0000E+00 EPS--BR= 0.4252E-02  
 C-EPSC= 0.9647E-03 C-EPSR= 0.6210E-03 C-EPSZ=-0.1586E-02 C-EPSBR= 0.1598E-02

---

STEP( 8 ): CUMULATIVE TIME= 50.00 C-RATE= 0.7291E-05 CGMA=BR= 0.2570E+09  
 SGMA-C= 0.3061E+08 SGMA-R=-0.1650E+08 SGMA-Z=-0.2468E+09 SGMA-BR= 0.2571E+09  
 EPS--C= 0.6861E-02 EPS--R= 0.5879E-02 EPS--Z= 0.0000E+00 EPS--BR= 0.4284E-02  
 C-EPSC= 0.1103E-02 C-EPSR= 0.6989E-03 C-EPSZ=-0.1802E-02 C-EPSBR= 0.1817E-02

---

STEP( 9 ): CUMULATIVE TIME= 100.00 C-RATE= 0.3359E-05 CGMA=BR= 0.2347E+09  
 SGMA-C= 0.2943E+08 SGMA-R=-0.1650E+08 SGMA-Z=-0.2247E+09 SGMA-BR= 0.2346E+09  
 EPS--C= 0.6914E-02 EPS--R= 0.5894E-02 EPS--Z= 0.0000E+00 EPS--BR= 0.4310E-02  
 C-EPSC= 0.1210E-02 C-EPSR= 0.7570E-03 C-EPSZ=-0.1967E-02 C-EPSBR= 0.1984E-02

---

STEP( 10 ): CUMULATIVE TIME= 150.00 C-RATE= 0.2051E-05 CGMA=BR= 0.2211E+09  
 SGMA-C= 0.2869E+08 SGMA-R=-0.1650E+08 SGMA-Z=-0.2113E+09 SGMA-BR= 0.2209E+09  
 EPS--C= 0.6947E-02 EPS--R= 0.5903E-02 EPS--Z= 0.0000E+00 EPS--BR= 0.4325E-02  
 C-EPSC= 0.1276E-02 C-EPSR= 0.7917E-03 C-EPSZ=-0.2068E-02 C-EPSBR= 0.2087E-02

---

&lt;CASE4 OUTPUT&gt;

## CASEPC RESULTS

\*\*\*\*\*

( FILE NAME : WEACC )

## &lt;INPUT DATA&gt;

OUTER DIAMETER	WALL THICKNESS	POISSON RATIO
0.1143000	0.0052000	0.2800

MODULUS OF ROCK	POISSON RATIO OF ROCK
0.2200E+10	0.2500

INCREMENT OF PRESSURE AT  $t=0$ :

0.8250E+06	0.8250E+06	0.8250E+06	0.8250E+06	0.8250E+06	0.8250E+06
0.8250E+06	0.8250E+06	0.8250E+06	0.8250E+06	0.8250E+06	0.8250E+06
0.8250E+06	0.8250E+06	0.8250E+06	0.8250E+06	0.8250E+06	0.8250E+06
0.8250E+06	0.8250E+06				

INCREMENT OF TEMPERATURE AT  $t=0$ :

17.5000	17.5000	17.5000	17.5000	17.5000	17.5000
17.5000	17.5000	17.5000	17.5000	17.5000	17.5000
17.5000	17.5000	17.5000	17.5000	17.5000	17.5000
17.5000	17.5000				

## INCREMENT OF TIME:

1.0000	1.0000	2.0000	2.0000	2.0000	2.0000
10.0000	30.0000	50.0000	50.0000	50.0000	50.0000
50.0000	50.0000	50.0000	100.0000	100.0000	100.0000
100.0000	100.0000	100.0000	100.0000	100.0000	100.0000
100.0000					

INCREMENT OF PRESSURE AS  $t>0$ :

0.0000E+00	0.0000E+00	0.0000E+00	0.0000E+00	0.0000E+00	0.0000E+00
0.0000E+00	0.0000E+00	0.0000E+00	0.0000E+00	0.0000E+00	0.0000E+00
0.0000E+00	0.0000E+00	0.0000E+00	0.0000E+00	0.0000E+00	0.0000E+00
0.0000E+00	0.0000E+00	0.0000E+00	0.0000E+00	0.0000E+00	0.0000E+00
0.0000E+00					

INCREMENT OF TEMPERATURE AS  $t>0$ :

0.0000	0.0000	0.0000	0.0000	0.0000	0.0000
0.0000	0.0000	0.0000	0.0000	0.0000	0.0000
0.0000	0.0000	0.0000	0.0000	0.0000	0.0000
0.0000	0.0000	0.0000	0.0000	0.0000	0.0000
0.0000					

## &lt;PHYSICAL PROPERTIES OF THE CASING UNDER DIFFERENTE TEMPERATURE&gt;:

POINT- 1:	0.00	BYO= 0.6220E+09	E= 0.2075E+12	ALFA= 0.1138E-04
POINT- 2:	50.00	BYO= 0.5937E+09	E= 0.1995E+12	ALFA= 0.1173E-04
POINT- 3:	100.00	BYO= 0.5659E+09	E= 0.1885E+12	ALFA= 0.1205E-04
POINT- 4:	150.00	BYO= 0.5352E+09	E= 0.1780E+12	ALFA= 0.1236E-04
POINT- 5:	200.00	BYO= 0.5027E+09	E= 0.1680E+12	ALFA= 0.1267E-04
POINT- 6:	250.00	BYO= 0.4683E+09	E= 0.1572E+12	ALFA= 0.1299E-04

( NO OUTPUT FOR ELASTIC-PLASTIC SOLUTION. IF YOU WANT, PUT: N2 > N1 > 0 )  
 ( NO OUTPUT FOR CREEP RESULTS. IF YOU WANT, PUT: NT2 > NT1 > 0 )

C  
C  
C  
C  
C  
C  
C  
C

&lt;OUTPUT RESULT2&gt;

&lt;YIELD PRESSURE UNDER DIFFERENT TEMPERATURES&gt;

POINT	TEMPERATURE	YIELD PRESSURE
1	0.00	0.6920E+08
2	50.00	0.6374E+08
3	100.00	0.5602E+08
4	150.00	0.4498E+08
5	200.00	0.2829E+08
6	250.00	-0.4219E+07

YIELD PRESSURE AND INITIAL STRESS AND STRAIN

POINT	SGMA-C	SGMA-R	SGMA-Z	SGMA-BR	SGMA-Y
1	0.6345E+09	-0.6920E+08	0.1583E+09	0.6220E+09	0.6220E+09
2	0.5691E+09	-0.6374E+08	0.2458E+08	0.5937E+09	0.5937E+09
3	0.4822E+09	-0.5602E+08	-0.1077E+09	0.5659E+09	0.5659E+09
4	0.3651E+09	-0.4498E+08	-0.2403E+09	0.5352E+09	0.5352E+09
5	0.1977E+09	-0.2829E+08	-0.3783E+09	0.5027E+09	0.5027E+09
6	-0.1099E+09	0.4219E+07	-0.5399E+09	0.4970E+09	0.4683E+09

POINT	EPS-C	EPS-R	EPS-Z	EPS-BR
1	0.2938E-02	-0.1403E-02	0.0000E+00	0.2558E-02
2	0.3495E-02	-0.5667E-03	0.0000E+00	0.2540E-02
3	0.4006E-02	0.3512E-03	0.0000E+00	0.2561E-02
4	0.4354E-02	0.1405E-02	0.0000E+00	0.2566E-02
5	0.4389E-02	0.2667E-02	0.0000E+00	0.2553E-02
6	0.3502E-02	0.4431E-02	0.0000E+00	0.2698E-02

**APPENDIX B**

**PROGRAM LISTING FOR CASEPC**

CASEPC  
 \*\*\*\*\*  
 (FILE NAME: WEACC)

---

THIS PROGRAM IS DESIGNED TO MAKE THE CREEP PREDICTION AS WELL AS TO FIND OUT THE ELASTIC-PLASTIC SOLUTION WITH THERMOPLASTIC MATERIALS FOR CASING PIPE IN THE OIL-WELL BOTTOM OF CANADA THERMAL RECOVERY WELL.

IT IS BASED ON VON-MISES CONDITION USING A SIMPLIFIED MODEL AND THE BAILOY-NORTON LAW IS ADOPTED AS THE CURRENT CREEP LAW IN THE PROGRAM, SO THIS PROGRAM CAN BE VERY EASILY EMPLOYED TO PRESENT THE CREEP STRAINS AND ELASTIC -PLASTIC RESPONSES OF BOTH LOADING AND UNLOADING.

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 DEPARTMENT OF CIVIL ENGINEERING  
 UNIVERSITY OF ALBERTA, EDMONTON

DECEMBER 1989

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< NOTATIONS IN THE PROGRAM >

PSC AND SC: TANGENTIAL STRESS  
 PSR AND SR: RADIAL STRESS  
 PSZ AND SZ: AXIAL STRESS  
 PED AND ED: EFFECTIVE STRESS  
 PEC AND EC: TANGENTIAL STRAIN  
 PER AND ER: RADIAL STRAIN  
 PEZ AND EZ: AXIAL STRAIN  
 PEE AND EE: EFFECTIVE STRAIN  
 BYO AND BY: YIELD LIMITATION OF CASING UNDER DIFFERENT TEMPERATURE

PBY: YIELD SURFACE  
 ECP: PLASTIC OR CREEP TANGENTIAL STRAIN  
 ERP: PLASTIC OR CREEP RADIAL STRAIN  
 EZP: PLASTIC OR CREEP AXIAL STRAIN  
 EEP: PLASTIC OR CREEP EFFECTIVE STRAIN

TM: TEMPERATURE  
 CT: CUMULATIVE TEMPERATURE  
 CTE: CUMULATIVE TEMPERATURE  
 CDP: INCREMENT OF PRESSURE AT  $t=0$   
 CDT: INCREMENT OF TEMPERATURE AT  $t=0$   
 CCT: CUMULATIVE TIME  
 CCTE: CUMULATIVE TIME  
 CCDP: INCREMENT OF PRESSURE AS  $t>0$   
 CCDT: INCREMENT OF TEMPERATURE AS  $t>0$



```

C
C
C
C
C
C      ( 1 ) ---- MAIN PROGRAM
C
C      PROGRAM MAIN
C
C      DIMENSION PSZ(50), PED(50), PEC(50), PER(50), PEZ(50), PEE(50), BY(10)
C      DIMENSION ECP(50), ERP(50), EZP(50), EEP(50), BYE(50), EEE(50), ED(10)
C      DIMENSION SC(10), SR(10), SZ(10), EC(10), ER(10), EZ(10), EE(10), CO(3)
C      DIMENSION ALE(50), EPE(50), CTE(50), CCTE(50), CCKE(50), PPED(50)
C      DIMENSION PPY(50), PTM(50), PBYP(50), RC(50), PSC(50), PSR(50)
C      DIMENSION CDP(50), CDT(50), CCDP(50), CCDT(50), CTI(50)
C      COMMON/CN1/NN, NT, N1, N2, NT1, NT2/CN2/LL/A7/CO
C
C      ****          ****          ****          ****          ****
C
C      **** INPUT DATA
C      CALL INPUT(CDP, CDT, CTI, CCDP, CCDT)
C
C      **** PRINT THE ORIGINAL DATA
C      CALL WRINT(50, NN, CDP, CDT, CCDP, CCDT, 40, NT, CTI)
C
C      **** CALCULATE THE PRESSURE ,STRESS AND STRAIN ON THE YIELDT LIMIT-
C      ATION UNDER DIFFERENT TEMPERATURE
C      CALL EESOL(6, SC, SR, SZ, EC, ER, EZ, ED, EE, BY)
C
C      **** OUTPUT OF THE STRESSES AND THE STRAINS AT THE PLASTIC ONSET ****
C      CALL EOUT1(6, SC, SR, SZ, EC, ER, EZ, ED, EE, BY)
C
C      **** FIND OUT THE ELASTIC-PLASTIC SOLUTION WITH THERMOPLASTIC MATER-
C      IALS USING VON-MISES CONDITION
C      CALL EPSOL(50, NN, CDP, CDT, CTE, PPY, PTM, PBYP, RC, PSC, PSR, PSZ, PED, PEC, PE
C      $R, PEZ, PEE, ECP, ERP, EZP, EEP, BYE, EEE, ALE, EPE)
C
C      **** OUTPUT OF THE ELASTIC-PLASTIC SOLUTION
C      CALL EPOUT(50, NN, CDP, CDT, CTE, N1, N2, PPY, PTM, PBYP, RC, PSC, PSR, PSZ, PED,
C      $PEC, PER, PEZ, PEE, ECP, ERP, EZP, EEP, BYE, EEE, ALE, EPE)
C
C      IF(LL.GT.0) THEN
C      **** SOLVE THE CREEP PROBLEM
C      CALL CRSOL(50, NT, CCDP, CCDT, CTI, CCTE, PSC, PSR, PSZ, PED, PEC, PER, PEZ, PE
C      $E, ECP, ERP, EZP, EEP, CCKE, PPED)
C
C      **** OUTPUT OF THE RESULTS OF TIME DEPENDENT PROBLEM
C      CALL CROUT(50, NT, CCTE, NT1, NT2, PSC, PSR, PSZ, PEC, PER, PEZ, ECP, ERP, EZP,
C      $PED, PEE, EEP, CCKE, PPED)
C      ELSE
C      WRITE(6,*) ' ( NO OUTPUT FOR CREEP SOLUTION. IF YOU WANT, PUT:
C      $LL=1 ) '
C      WRITE(9,*) ' ( NO OUTPUT FOR CREEP SOLUTION. IF YOU WANT, PUT:
C      $LL=1 ) '
C      END IF
C
C      STOP 'THE END OF THE JOB'
C      END
C
C -----

```



```

C
C
C
C      ( 2 ) ----- INPUT DATA
C
C      SUBROUTINE INPUT(CDP,CDT,CTI,CCDP,CCDT)
C
C      DIMENSION CDP(50),CDT(50),CTI(50),CCDP(50),CCDT(50),CO(3)
C      COMMON/C1/CNEF,CNBMF,CNR,CNT/D1/BM/C3/DO,CNEFO/C4/EPS
C      COMMON/CN1/NN,NT,N1,N2,NT1,NT2/C2/CAC,CMC,CNC/CN2/LL/A7/CO
C
C          ****          ****          ****          ****          ****
C
C      **** PRESENT SOME INITIAL DATA ****
C
C      ENTER CONTROL PARAMETERS
C
C          LL:      LL=0, ONLY ELASTIC-PLASTIC SOLUTION
C                  LL=1, INCLUDE CREEP PREDICTION
C          CDPM:    =MAXIMUM PRESSURE WHEN LL=1
C                  =0.0 WHEN LL=0
C          CDTM:    =MAXIMUM TEMPERATURE WHEN LL=1
C                  =0.0 WHEN LL=0
C
C      READ(5,*) LL,CDPM,CDTM
C
C      ENTER THE ORIGINAL PARAMETERS
C
C          BM:      POISSON RATIO OF CASING
C          CNEFO:   MODULUS OF THE SUPPORTING ROCK
C          CNBMF:   POISSON RATIO OF THE SUPPORTING ROCK
C
C      READ(5,*) BM,CNEFO,CNBMF
C
C      ENTER THE CASING SIZE
C
C          DI:      INTERNAL DIAMETER OF CASING
C          DO:      OUTER DIAMETER OF CASING
C
C      READ(5,*) DI,DO
C
C      IF(LL.LT.1) THEN
C      ENTER THE SOME INTEGER NUMBERS
C
C          NN:      THE NUMBER OF INCREMENTS OF PRESSURE
C                  AND TEMPERATURE
C          N1:      BIGINING POINT OF THE RESULT OUTPUT
C          N2:      ENDING POINT OF THE RESULT OUTPUT
C
C      READ(5,*) NN,N1,N2
C
C      ENTER THE PRESSURE INCREMENT
C      READ(5,*) (CDP(I),I=1,NN)
C
C      ENTER THE TEMPERATURE INCREMENT
C      READ(5,*) (CDT(I),I=1,NN)
C      ELSE
C      READ(5,*) NN,N1,N2
C          PN=CDPM/NN
C          TN=CDTM/NN
C          DO 2 I=1,NN
C              CDP(I)=PN
C              CDT(I)=TN
C
C      2      CONTINUE
C      END IF

```

C  
C  
C  
C

```

      IF(LL.LT.1) THEN
        NT=6
        DO 4 I=1,NT
          CTI(I)=0.0
          CCDP(I)=0.0
          CCDT(I)=0.0
4      CONTINUE
        GO TO 5
C
      ELSE
C      ENTER THE FOLLOWING INTEGER NUMBERS
C          NT: THE NUMBER OF INCREMENTS OF TIME STEP-LENGTH
C          NT1: BIGINIG POINT OF THE RESULT OUTPUT
C          NT2: ENDING POINT OF THE RESULT OUTPUT
      READ(5,*) NT,NT1,NT2
C
C      ENTER THE TIME INCREMENT
      READ(5,*) (CTI(I),I=1,NT)
C      CHECK IF THE VALUE OF TIME INCREMENT IS MORE THAN ZERO
      DO 1 I= 1,NT
        IF(CTI(I).EQ.0.0) THEN
          STOP 'TIME INCREMENT MUST BE GREATER ZERO, IMPORTANT!'
        END IF
1      CONTINUE
C      ENTER THE PRESSURE INCREMENT AS t>0
      READ(5,*) (CCDP(I),I=1,NT)
C      ENTER THE TEMPERATURE INCREMENT AS t>0
      READ(5,*) (CCDT(I),I=1,NT)
C
C      ENTER THE ERROR LIMIT AND INITIAL ESTIMATES FOR ITERATION
      READ(5,*) EPS,CO
C
C      ENTER THE COEFFICIENTS OF CREEP LAW
      READ(5,*) CAC,CMC,CNC
      END IF
C
C          ****          ****          ****          ****          ****
C
5      CNR=0.5*DI
      CNT=0.5*(DO-DI)
      CNEF=DO*CNEFO/DI
C
      RETURN
      END
C -----

```

( 3 ) ---- CREATE DATA BLOCK  
 THIS SEGMENT INITIALIZES SOME VARIABLES

XX1: FITTING POINTS OF THE ELASTIC MODULUS OF CASING  
 XX2: FITTING POINTS OF THE THERMAL EXPANSION COEFFICIENT OF CASING  
 X0: INTERPOLATION POINTS OF TEMPERATURE  
 Y0: INTERPOLATION POINTS OF YIELD LIMITATION  
 Y1: INTERPOLATION POINTS OF EFFECTIVE STRAIN  
 Z0: INTERPOLATION POINTS OF PLASTIC MODULUS OF CASING

\*\*\*\*            \*\*\*\*            \*\*\*\*            \*\*\*\*            \*\*\*\*

BLOCK DATA WDAT

DIMENSION TM(6),XX1(2,9),XX2(2,9),X0(6),Y0(6),Y1(8),Z0(6,8,1)  
 COMMON/A1/XX1/A2/XX2/A3/X0/A4/Y0/A5/Y1/A6/Z0/DM/TM

DATA TM/0.,50.,100.,150.,200.,250./  
 DATA XX1/0.,207.E9,10.,206.5E9,21.,206.E9,50.,198.5E9,93.,190.E9,  
 \$150.,178.7E9,204.,166.7E9,316.,140.E9,427.,118.33E9/  
 DATA XX2/0.,11.4E-6,10.,11.45E-6,21.,11.5E-6,50.,11.75E-6,93.,  
 \$12.E-6,150.,12.35E-6,204.,12.7E-6,316.,13.4E-6,427.,14.E-6/  
 DATA X0/0.,21.,93.,204.,316.,427./  
 DATA Y0/622.E6,610.E6,570.E6,500.E6,420.E6,355.E6/  
 DATA Y1/0.3E-2,0.5E-2,0.8E-2,2.0E-2,4.0E-2,6.0E-2,8.0E-2,10.0E-2/  
 DATA Z0/2.5E9,5.E9,12.5E9,25.E9,45.E9,45.E9,3.E9,3.3E9  
 \$,5.E9,10.E9,15.E9,9.3E9,2.58E9,2.5E9,2.08E9,4.17E9,4.58E9,3.92E9,  
 \$1.55E9,1.75E9,2.5E9,3.E9,3.E9,.75E9,1.E9,1.E9,1.E9,1.25E9,1.75E9,  
 \$-3.5E9,.45E9,.25E9,-.5E9,-1.25E9,.25E9,-1.2E9,-0.1E9,-0.5E9,-1.75  
 \$E9,-2.5E9,-.25E9,-1.65E9,-0.9E9,-1.6E9,-4.0E9,-6.5E9,-1.75E9,-3.15  
 \$E9/  
 DATA Z0/2.06E9,2.12E9,2.29E9,3.58E9,3.79E9,2.33E9,  
 \$2.06E9,2.12E9,2.29E9,3.58E9,3.79E9,2.33E9,  
 \$2.06E9,2.12E9,2.29E9,3.58E9,3.79E9,2.33E9,  
 \$2.06E9,2.12E9,2.29E9,3.58E9,3.79E9,2.33E9,  
 \$2.06E9,2.12E9,2.29E9,3.58E9,3.79E9,2.33E9,  
 \$2.06E9,2.12E9,2.29E9,3.58E9,3.79E9,2.33E9,  
 \$2.06E9,2.12E9,2.29E9,3.58E9,3.79E9,2.33E9/  
 END

-----

C  
C  
C  
C  
C

( 4 ) ---- WRITE ORIGINAL DATA

C

SUBROUTINE WRINT(NNC, NN, CDP, CDT, CCDP, CCDT, NTC, NT, CTI)

C

DIMENSION CDP(NNC), CDT(NNC), CTI(NTC), CCDP(NTC), CCDT(NTC)  
COMMON /D1/BM/C1/CNEF, CNBMF, CNR, CNT/C3/DO, CNEFO

```

WRITE(6,90)
WRITE(9,90)
WRITE(6,100) DO, CNT, BM
WRITE(9,100) DO, CNT, BM
WRITE(6,120) CNEFO, CNBMF
WRITE(9,120) CNEFO, CNBMF
WRITE(6,130) (CDP(I), I=1, NN)
WRITE(9,130) (CDP(I), I=1, NN)
WRITE(6,140) (CDT(I), I=1, NN)
WRITE(9,140) (CDT(I), I=1, NN)
WRITE(6,141) (CTI(I), I=1, NT)
WRITE(9,141) (CTI(I), I=1, NT)
WRITE(6,142) (CCDP(I), I=1, NT)
WRITE(9,142) (CCDP(I), I=1, NT)
WRITE(6,143) (CCDT(I), I=1, NT)
WRITE(9,143) (CCDT(I), I=1, NT)
WRITE(6,145)
WRITE(9,145)

```

C

```

90 FORMAT(30X, 'CASEPC RESULTS', /, 25X, 26('*'), /, 28X, '( FILE NAME : W
  $EACC )', /, 74('='), /, 4X, '<INPUT DATA>', /, 3X, 15('-'), /)
100 FORMAT(5X, 'OUTER DIAMETER', 4X, 'WALL THICKNESS', 3X, 'POISSON RATIO'
  $, /, 7X, F10.7, 8X, F10.7, 7X, F7.4, /)
120 FORMAT(5X, 'MODULUS OF ROCK', 3X, 'POISSON RATIO OF ROCK', /,
  $7X, E11.4, 7X, F7.4, /)
130 FORMAT(2X, 'INCREMENT OF PRESSURE AT t=0: ', /, 6(E12.4))
140 FORMAT(/, 2X, 'INCREMENT OF TEMPERATURE AT t=0: ', /, 6(F12.4))
141 FORMAT(/, 2X, 'INCREMENT OF TIME: ', /, 6(F12.4))
142 FORMAT(/, 2X, 'INCREMENT OF PRESSURE AS t>0: ', /, 6(E12.4))
143 FORMAT(/, 2X, 'INCREMENT OF TEMPERATURE AS t>0: ', /, 6(F12.4))
145 FORMAT(1X, 73('='))
RETURN
END

```

C

-----

```

C
C
C
C      ( 5 ) ---- CALCULATE THE PRESSURE, STRESSES AND STRAINS
C                AT THE INITIAL YIELDT LIMITATION UNDER DIFFERE
C                -NT TEMPERATURE
C
C      SUBROUTINE EESOL(N, SC, SR, SZ, EC, ER, EZ, ED, EE, BY)
C
C      DIMENSION TM(6), PY(10), X0(6), Y0(6), XX1(2, 9)
C      DIMENSION SC(10), SR(10), SZ(10), BY(10), EC(10), ER(10), EZ(10), EE(10)
C      DIMENSION ED(10), A(6), CK1(6), CK2(6), AX(6), S(6, 7), XX2(2, 9)
C      COMMON /A1/XX1/A2/XX2/A3/X0/A4/Y0/B1/CK1, CK2/D1/BM/DM/TM
C
C      DO 13 I=1, N
13  A(I)=0.0
C
C      **** CALCULATE THE COEFICIENTS OF CURVE FITTING FOR ELASTIC MODULUS
C           OF CASING
C      CALL FIT13(9, 6, 7, XX1, A, S, FATL)
C
C      DO 14 I=1, N
14  CK1(I)=A(I)
C
C      **** CALCULATE THE COEFICIENTS OF CURVE FITTING FOR THERMAL EXPANSION
C           OF CASING
C      CALL FIT13(9, 6, 7, XX2, A, S, FATL)
C
C      DO 16 I=1, N
16  CK2(I)=A(I)
C
C      **** FIND OUT THE YIELD PRESSURE UNDER DIFFERENT TEMPERATURE
C           CALL YEP(N, PY)
C
C      DO 20 I=1, N
C
C      E=FF(TM(I), CK1)
C      ALFA=FF(TM(I), CK2)
C
C      **** TAKE LAGRANGE INTERPOLATION FOR YIELD LIMIT OF CASING
C      CALL LAG13(X0, Y0, 6, TM(I), BY(I))
C
C      **** CALCULATE THE STRESSES AND THE STRAINS AT THE PLASTIC ONSET
C      CALL ELASE(PY(I), TM(I), SC(I), SR(I), SZ(I), EC(I), ER(I), EZ(I), E, ALFA,
C      $BY(I))
C
C      **** CALCULATE THE EFFECTIVE STRESS
C      CALL EDSTS(SC(I), SR(I), SZ(I), ED(I))
C
C      **** CALCULATE THE EFFECTIVE STRAIN
C      CALL EESTN(EC(I), ER(I), EZ(I), EE(I))
C
C      20  CONTINUE
C           RETURN
C           END
C -----

```

C  
C  
C  
C  
C  
C

( 6 ) ---- CALCULATE THE YIELD PRESSURE UNDER DIFFERENT  
TEMPERATURE

C  
C  
C  
C

SUBROUTINE YEP(N,PY)

DIMENSION PY(10),TM(6),CK1(6),CK2(6),X0(6),YO(6)  
COMMON /D1/BM/B1/CK1,CK2/DM/TM  
COMMON/C1/CNEF,CNBMF,CNR,CNT/A3/X0/A4/YO

C  
C  
C

WRITE(6,116)

WRITE(9,116)

116 FORMAT(2X,'<PHYSICAL PROPERTIES OF THE CASING UNDER DIFFERENTE TEM  
\$PERATURE>:',/)

C  
C  
C  
C

DO 10 I=1,N

CALL LAG13(X0,YO,6,TM(I),BYO)

E=FF(TM(I),CK1)

ALFA=FF(TM(I),CK2)

WRITE(6,123) I,TM(I),BYO,E,ALFA

WRITE(9,123) I,TM(I),BYO,E,ALFA

123 FORMAT(1X,'POINT-',I2,', ':',F7.2,4X,'BYO=',E11.4,4X,'E=',E11.4,4X,  
\$'ALFA=',E11.4)

C  
C  
C  
C  
C

CALL ECOEF(AA,BB,CC,DD,AAO,BBO,CCO,DDO,E,ALFA,BYO)

BM1=BM-1.

BB1=BB-1.

TO=(BM\*BBO-E\*ALFA)\*TM(I)

X=BB\*(BB+1.)+BM\*BM1\*BB1\*BB1+1.

Y=TO\*(2.\*BM\*BB1+1.-BB)+BBO\*(2.\*BB+1.-BM\*BB1)\*TM(I)

Z=BBO\*BBO\*TM(I)\*TM(I)+TO\*(TO-BBO\*TM(I))-BYO\*BYO

IF((ABS(X)+ABS(Y)+ABS(Z)).EQ.0.0) STOP

C  
C  
C  
C  
C

CALL QUAD(X,Y,Z,C1,C2,D,KEY)

IF(KEY) 22,23,23

22 WRITE(6,\*) 'DATA ERROR'

WRITE(9,\*) 'DATA ERROR'

GO TO 10

23 PY(I)=C1

10 CONTINUE

C  
C  
C  
C  
C

WRITE(6,101)

WRITE(9,101)

WRITE(6,100)

WRITE(10,100)

WRITE(6,200) (I,TM(I),PY(I),I=1,N)

WRITE(10,200) (I,TM(I),PY(I),I=1,N)

WRITE(6,101)

WRITE(10,101)

C  
C  
C  
C  
C

100 FORMAT(8X,'<YIELD PRESSURE UNDER DIFFERENT TEMPERATURES>',/,1X,  
\$73('='),//,1X,'POINT',12X,'TEMPERATURE',12X,'YIELD PRESSURE',/)

200 FORMAT(1X,I3,14X,F7.2,14X,E12.4)

101 FORMAT(1X,73('='))

RETURN

END

C



( 9 ) ---- CALCULATE THE INITIAL VALUES OF STRESS AND STRAIN

SUBROUTINE ELASE(DP,DT,DSC,DSR,DSZ,DEC,DER,DEZ,E,ALFA,BYO)

COMMON/C1/CNEF,CNBMF,CNR,CNT/D1/BM

CALL ECOEF(AA,BB,CC,DD,AAO,BBO,CCO,DDO,E,ALFA,BYO)

DU=DP\*AA+AAO\*DT

DSC=DP\*BB+BBO\*DT

DSR=-DP

DSZ=DP\*BM\*(BB-1.)+(BM\*BBO-E\*ALFA)\*DT

DEC=DP\*CC+CCO\*DT

DER=- (DP\*DD+DDO\*DT)

DEZ=0.0

RETURN

END

( 10 ) ---- FIND OUT THE ELASTIC-PLASTIC SOLUTION WITH  
THERMOPLASTIC MATERIALS USING VON-MISES CONDITION

SUBROUTINE EPSOL(NNC,NN,CDP,CDT,CTE,PPY,PTM,PBY,RC,PSC,PSR,PSZ,PED  
\$,PEC,PER,PEZ,PEE,ECP,ERP,EZP,EEP,BYE,EEE,ALE,EPE)

DIMENSION CDP(NNC),CDT(NNC),CTE(NNC)

DIMENSION XO(6),YO(6),Y1(8),XX1(2,9),XX2(2,9),Z(1),BYE(NNC)

DIMENSION A(6),CK1(6),CK2(6),AX(6),S(6,7),ZO(6,8,1),RC(NNC)

DIMENSION PSC(NNC),PSZ(NNC),PSR(NNC),PED(NNC),PBY(NNC),EEE(NNC)

DIMENSION PEC(NNC),PER(NNC),PEZ(NNC),PEE(NNC),PPY(NNC),ALE(NNC)

DIMENSION ECP(NNC),ERP(NNC),EZP(NNC),EEP(NNC),PTM(NNC),EPE(NNC)

COMMON/CD1/DPSC,DPSR,DPSZ,DPEC,DPER,DPEZ/DC/CNSC,CNSR,CNSZ

COMMON/AC3/AE,AL/AC4/ATM,APP,AED,APE/AC5/ASC,ASR,ASZ,AEC,AER,AEZ

COMMON/A1/XX1/A2/XX2/A3/XO/A4/YO/A5/Y1/A6/ZO/B1/CK1,CK2

DATA CNEE,CNPY,CNTM,CNECP,CNERP,CNEZP,CNEEP/7\*0.0/

DATA KK,RR,CT,DDP,DDT/1,1.,3\*0.0/

DATA CNEC,CNER,CNEZ,DDP1,DDT1/5\*0.0/

CNSC=0.

CNSR=0.

CNSZ=0.

DO 1 J=1,NN

IF(CDT(J).LT.0.0) THEN

    CNDT=CDT(J)-DDT1

    CNDP=CDP(J)-DDP1

ELSE IF(CDP(J).LT.0.0) THEN

    CNDP=CDP(J)-DDP1

    CNDT=CDT(J)-DDT1

ELSE

    CNDP=CDP(J)+DDP

    CNDT=CDT(J)+DDT

END IF





```
C
C
C
C WHEN THE LOADING POINT EXCEEDS THE YIELD SURFACE, CALCULATE THE SCALE
C FACTOR OF ELASTIC PART
C     CALL RCOEF(CNSC, CNSR, CNSZ, DSC, DSR, DSZ, CNED, RR)
C
C     END IF
C
C     RC(J)=RR
C     IF(RR.LT.0.99) KK=KK+1
C
C     DSC=RR*DSC
C     DSR=RR*DSR
C     DSZ=RR*DSZ
C     DEC=RR*DEC
C     DER=RR*DER
C     DEZ=RR*DEZ
C
C     IF(RR.LT.1.E-2) THEN
C         DDP=0.0
C         DDT=0.0
C         DDP1=0.0
C         DDT1=0.0
C     ELSE
C         DDP=(1.-RR)*CNDP
C         DDT=(1.-RR)*CNDT
C         DDP1=RR*CNDP
C         DDT1=RR*CNDT
C     END IF
C
C     CNSC=CNSC+DSC
C     CNSR=CNSR+DSR
C     CNSZ=CNSZ+DSZ
C     CNEC=CNEC+DEC
C     CNER=CNER+DER
C     CNEZ=CNEZ+DEZ
C
C CALCULATE EFFECTIVE STRAIN
C     CALL EESTN(CNEC, CNER, CNEZ, CNEE)
C
C     IF(RR.GT.1.E-2) THEN
C         PSG(J)=CNSC
C         PSR(J)=CNSR
C         PSZ(J)=CNSZ
C         PEC(J)=CNEC
C         PER(J)=CNER
C         PEZ(J)=CNEZ
C
C         CALL EDSTS(PSG(J), PSR(J), PSZ(J), CNED)
C
C         CNDP=RR*CNDP
C         CNDT=RR*CNDT
C
C         GO TO 801
C     END IF
C
```

```
C
C
C
C TAKE TWO-DIMENSIONAL INTERPOLATION FOR PLASTIC MODULUS AGAIST DIFFERENT
C TEMPERATURE AND TOTAL STRAIN
  CALL LAG23(X0,Y1,Z0,6,8,1,CT,CNEE,EPE(J))
  EP=EPE(J)
C
C CALCULATE THE STRESSES AND TOTAL STRAINS UNDER PLASTIC CONDITION
  CALL PLASE(CNDP,CNDT,E,ALFA,BYO,EP)
C
  PSC(J)=CNSC+DPSC
  PSR(J)=CNSR+DPSR
  PSZ(J)=CNSZ+DPSZ
  PEC(J)=CNEC+DPEC
  PER(J)=CNER+DPER
  PEZ(J)=CNEZ+DPEZ
C
C CALCULATE THE PLASTIC STRAINS
  CALL PSRAN(CNDT,DDPEE,DDPEC,DDPER,DDPEZ,E,ALFA,BYO,EP)
C
  IF(ABS(E-EP).LT.1.E-2) THEN
    PCNBY=CNBY
  ELSE
    PCNBY=CNBY+(((E*EP)/(E-EP))*DDPEE)
  END IF
C
  CALL EDSTS(PSC(J),PSR(J),PSZ(J),CNED)
C
  CALL EESTN(PEC(J),PER(J),PEZ(J),CNPEE)
C
  PED(J)=CNED
  PBY(J)=PCNBY
  PEE(J)=CNPEE
  ECP(J)=CNECP+DDPEC
  ERP(J)=CNERP+DDPER
  EZP(J)=CNEZP+DDPEZ
  EEP(J)=CNEEP+DDPEE
  GO TO 400
C
800 RR=1.
  RC(J)=RR
801 EPE(J)=0.0
  CALL EESTN(PEC(J),PER(J),PEZ(J),CNEE)
C
  PED(J)=DCNED
  IF(DCNED.GT.CNED) PED(J)=CNED
  PBY(J)=CNED
  PEE(J)=CNEE
  ECP(J)=CNECP
  ERP(J)=CNERP
  EZP(J)=CNEZP
  EEP(J)=CNEEP
C
```

C  
C  
C

```

400  PPY(J)=CNPY+CNDP
      PTM(J)=CNTM+CNDT
      CNECP=ECP(J)
      CNERP=ERP(J)
      CNEZP=EZP(J)
      CNEEP=EEP(J)
      CNSC=PSC(J)
      CNSR=PSR(J)
      CNSZ=PSZ(J)
      CNEC=PEC(J)
      CNER=PER(J)
      CNEZ=PEZ(J)
      CNEE=PEE(J)
      CNBY=PBYP(J)
      CNPY=PPY(J)
      CNTM=PTM(J)

```

C

```

      IF(RR.LT.1.E-2) THEN
        CDT(J)=CNDT
        CDP(J)=CNDP
      ELSE IF(CDT(J).LT.0.0) THEN
        CDT(J)=CNDT
        CDP(J)=CNDP
      ELSE IF(CDP(J).LT.0.0) THEN
        CDT(J)=CNDT
        CDP(J)=CNDP
      ELSE
        CDP(J)=RR*CDP(J)
        CDT(J)=RR*CDT(J)

```

```

      END IF

```

C

```

1  CONTINUE

```

C

```

C  PUT THE ELASTIC-PLASTIC SOLUTION AT THE WORKING POINT INTO THE INITIAL
C  POINT FOR CREEP COMPUTE

```

```

      ASC=PSC(NN)
      ASR=PSR(NN)
      ASZ=PSZ(NN)
      AEC=PEC(NN)
      AER=PER(NN)
      AEZ=PEZ(NN)
      ATM=PTM(NN)
      APP=PPY(NN)
      AED=PED(NN)
      APE=PEE(NN)
      AE=EEE(NN)
      AL=ALE(NN)

```

C

```

      RETURN
      END

```

C

-----



C  
C  
C  
C  
C

( 12 ) ---- CALCULATE THE EFFECT STRESS

C

SUBROUTINE EDSTS(A,B,C,CNED)

```
AA=(A-B)*(A-B)
BB=(B-C)*(B-C)
CC=(C-A)*(C-A)
ABC=(AA+BB+CC)/2.
CNED=SQRT(ABC)
RETURN
END
```

C  
C  
C  
C  
C

( 13 ) ---- CALCULATE THE EFFECT STRAIN

C

SUBROUTINE EESTN(A,B,C,CNEE)

```
AB=(A-B)*(A-B)
BC=(B-C)*(B-C)
CA=(C-A)*(C-A)
ABC=AB+BC+CA
CNEE=(SQRT(ABC))*1.4142/3.
RETURN
END
```

C

( 14 ) ----- PRESENTE THE COEFFCINTS FOR PLASTIC COMPUTATION

SUBROUTINE PCOEF(E,ALFA,BY0,EP)

COMMON/DA/A10,A20,ATO,B10,B20,BT0,C11,C21,CT1  
COMMON/DC/CNSC,CNSR,CNSZ/D1/BM

ALAM=E\*BM/((1.+BM)\*(1.-2.\*BM))  
ALAM1=ALAM\*(1.-BM)/BM

CALL EDSTS(CNSC,CNSR,CNSZ,ED)

A1=3.\*E/(2.\*(1.+BM))

IF(ABS(E-EP).LT.1.E-2) THEN

A2=0.0

AE=0.0

ELSE

A2=E\*EP/(E-EP)

AE=(A1\*A1)/((A1+A2)\*ED\*ED)

END IF

ASC=(2.\*CNSC-CNSR-CNSZ)/3.

ASR=(2.\*CNSR-CNSC-CNSZ)/3.

ASZ=(2.\*CNSZ-CNSC-CNSR)/3.

CCR=AE\*ASC\*ASR

CCZ=AE\*ASC\*ASZ

CRZ=AE\*ASR\*ASZ

CCC=AE\*ASC\*ASC

CRR=AE\*ASR\*ASR

CZZ=AE\*ASZ\*ASZ

AA1=ALAM1-CCC

AA2=ALAM-CRR

BB1=AA2

BB2=ALAM1-CRR

CC1=ALAM-CCZ

CC2=ALAM-CRZ

AB12=(AA1\*BB2)-(BB1\*AA2)

A10=BB2/AB12

A20=-AA2/AB12

B10=-BB1/AB12

B20=AA1/AB12

C11=CC1\*A10+CC2\*B10

C21=CC1\*A20+CC2\*B20

A2A=2.\*ALAM+ALAM1

AAT=CCC+CCR+CCZ-A2A

BBT=CCR+CRR+CRZ-A2A

CCT=CCZ+CRZ+CZZ-A2A

ATO=A10\*AAT+A20\*BBT

BTO=B10\*AAT+B20\*BBT

CT1=CCT-CC1\*ATO-CC2\*BTO

RETURN

END

C  
C  
C  
C  
C

( 15 ) ----- CALCULATE PLASTIC STRESS AND TOTAL STRAIN

C  
C  
C  
C  
C

SUBROUTINE PLASE(DP,DT,E,ALFA,BY0,EP)

COMMON/DA/A10,A20,AT0,B10,B20,BT0,C11,C21,CT1  
COMMON/C1/CNEF,CNBMF,CNR,CNT/CK/AKK  
COMMON /CD1/DPSC,DPSR,DPSZ,DPEC,DPER,DPEZ

CALL PCOEF(E,ALFA,BY0,EP)

RCT=CNR/CNT  
RCK=1.+AKK\*A10\*RCT  
DPSC=((1.+AKK\*A20)/RCK)\*RCT\*DP)+((AT0\*AKK\*RCT/RCK)\*ALFA\*DT)  
DPSR=-DP  
ALT=ALFA\*DT  
DPSZ=C11\*DPSC+C21\*DPSR+CT1\*ALT  
DPEC=A10\*DPSC+A20\*DPSR-AT0\*ALT  
DPER=B10\*DPSC+B20\*DPSR-BT0\*ALT  
DPEZ=0.  
RETURN  
END

C  
C  
C  
C

( 16 ) ----- CALCULATE THE SCALE FCTOR OF ELASTIC PART: <R>

C  
C  
C  
C  
C

SUBROUTINE RCOEF(AC,AR,AZ,DSC,DSR,DSZ,BY,RR)

A0=(AC+AR+AZ)/3.  
EAO=(DSC+DSR+DSZ)/3.  
AA=(DSC-EAO)\*(DSC-EAO)  
BB=(DSR-EAO)\*(DSR-EAO)  
CC=(DSZ-EAO)\*(DSZ-EAO)  
A=AA+BB+CC  
AA=(AC-A0)\*(DSC-EAO)  
BB=(AR-A0)\*(DSR-EAO)  
CC=(AZ-A0)\*(DSZ-EAO)  
B=(AA+BB+CC)\*2.  
AA=(AC-A0)\*(AC-A0)  
BB=(AR-A0)\*(AR-A0)  
CC=(AZ-A0)\*(AZ-A0)  
C=(AA+BB+CC)-(BY\*BY\*2./3.)

C  
C

CALL QUAD(A,B,C,C1,C2,D,KEY)

IF(KEY) 2,3,3  
2 WRITE(6,\*) 'DATA ERROR'  
WRITE(9,\*) 'DATA ERROR'  
GO TO 5  
3 RR=C1  
5 RETURN  
END

C





```

C
C
C
C EXPORT THE ELASTIC-PLASTIC SOLUTION
  WRITE(6,30) CTE(J)
  WRITE(9,30) CTE(J)
  WRITE(6,50) BYE(J),EEE(J),ALE(J),EPE(J)
  WRITE(9,50) BYE(J),EEE(J),ALE(J),EPE(J)
  WRITE(6,100) J,CDP(J),CDT(J),RG(J)
  WRITE(9,100) J,CDP(J),CDT(J),RG(J)
  WRITE(6,120) PPY(J),PTM(J),PBY(J)
  WRITE(9,120) PPY(J),PTM(J),PBY(J)
  WRITE(6,140) PSC(J),PSR(J),PSZ(J),PED(J)
  WRITE(9,140) PSC(J),PSR(J),PSZ(J),PED(J)
  WRITE(6,160) PEC(J),PER(J),PEZ(J),PEE(J)
  WRITE(9,160) PEC(J),PER(J),PEZ(J),PEE(J)
  WRITE(6,180) ECP(J),ERP(J),EZP(J),EEP(J)
  WRITE(9,180) ECP(J),ERP(J),EZP(J),EEP(J)
  WRITE(6,190) SC1,SR,SZ,SC2
  WRITE(9,190) SC1,SR,SZ,SC2
10 CONTINUE
C
  ELSE
  WRITE(6,200)
  WRITE(9,200)
  END IF
C
30 FORMAT(1X,'CT =',F7.2)
50 FORMAT(1X,'BY =',E13.4,2X,'E =',E12.4,3X,'ALFA =',E12.4,1X,
$'EP =',E12.4)
100 FORMAT(/,3X,'STEP (' ,I3,' ):',6X,'DP =',E12.4E2,3X,'DT =',F7.2
$,6X,'FACTOR--RR= ',F5.3,/)
120 FORMAT(1X,'PRES-R=',E11.4,2X,'TEMP-T=',F7.2,26X,'SGMA--Y=',E11.4)
140 FORMAT(1X,'SGMA-C=',E11.4,2X,'SGMA-R=',E11.4,2X,'SGMA-Z=',E11.4,
$2X,'SGMA-BR=',E11.4)
160 FORMAT(1X,'EPS--C=',E11.4,2X,'EPS--R=',E11.4,2X,'EPS--Z=',E11.4,
$2X,'EPS--BR=',E11.4)
180 FORMAT(1X,'EPS-CP=',E11.4,2X,'EPS-RP=',E11.4,2X,'EPS-ZP=',E11.4,
$2X,'EPS-BRP=',E11.4,/,83(' - '))
190 FORMAT(4X,'<CHECK THE NUMBER VALUE OF STRESS>:',/,1X,
$'SGMC-1=',E11.4,2X,'SGMR-1=',E11.4,2X,'SGMZ-1=',E11.4,/,1X,
$'SGMC-2=',E11.4,/,83(' - '))
200 FORMAT(/,2X,'( NO OUTPUT FOR ELASTIC-PLASTIC SOLUTION. IF YOU WANT
$, PUT: N2 > N1 > 0 )',/)
C
301 RETURN
  END
C
-----

```

( 18 ) ----- CALCULATE PLASTIC STRAIN

SUBROUTINE PSRAN(DDT, CNEE, DDPEC, DDPER, DDPEZ, E, ALFA, BYO, EP)

COMMON /DC/CNSC, CNSR, CNSZ/D1/BM  
COMMON /CD1/DPSC, DPSR, DPSZ, DPEC, DPER, DPEZ

CNEE=0.0  
DDPEC=0.0  
DDPER=0.0  
DDPEZ=0.0

CALL EDSTS(CNSC, CNSR, CNSZ, PD)

IF(ABS(E-EP).LT.1.E-2) THEN  
    HH=0.0  
ELSE  
    HH=E\*EP/(E-EP)  
END IF

GG=.5\*E/(1.+BM)  
AT=PD\*(1.+(HH/(3.\*GG)))  
AC=1./AT  
AS=(3.\*AC)/(2.\*PD)  
EA=(CNSC+CNSR+CNSZ)/3.  
SC=CNSC - EA  
SR=CNSR - EA  
SZ=CNSZ - EA  
ASD=SC\*DPEC+SR\*DPER+SZ\*DPEZ  
XE=AC\*(ASD-ALFA\*DDT\*(SC+SR+SZ))  
XC=AS\*ASD\*SC  
XR=AS\*ASD\*SR  
XZ=AS\*ASD\*SZ

CNEE=CNEE+XE  
DDPEC=DDPEC+XC  
DDPER=DDPER+XR  
DDPEZ=DDPEZ+XZ  
RETURN  
END

-----

( 19 ) ----- ONE-DIMENSIONAL LAGRANGE INTERPOLATION FOR YIELD  
LIMITATION

SUBROUTINE LAG13(XX0, YX0, N, X, YX)

DIMENSION XX0(N), YX0(N)

I=1

10 IF(X.LT.0.5\*(XX0(I+1)+XX0(I+2))) GO TO 30  
IF(X.GE.0.5\*(XX0(N-2)+XX0(N-1))) GO TO 20

I=I+1

GO TO 10

20 I=N-2

30 M=I+2

YX=0.0

DO 60 J=I, M

P=1.0

DO 50 K=I, M

IF(J-K) 40, 50, 40

40 P=P\*(X-XX0(K))/(XX0(J)-XX0(K))

50 CONTINUE

60 YX=YX+P\*YX0(J)

RETURN

END

C -----

C  
C  
C  
C  
C  
C  
C  
C  
C  
C  
C

( 20 ) ---- TWO-DIMENSIONAL LAGRENGE INTERPOLATION FOR  
PLASTIC MODULUS

SUBROUTINE LAG23(XX0,YY0,ZZ0,N,M,L,X,Y,Z)

DIMENSION XX0(N),YY0(M),ZZ0(N,M,L),Z(L),V(3),U(3)

N1=N-2  
M1=M-2  
DO 1, I=1, N1  
IF(X.LE.XX0(I+1)) GO TO 2  
1 CONTINUE  
I=N-2  
2 DO 3 J=1, M1  
IF(Y.LE.YY0(J+1)) GO TO 4  
3 CONTINUE  
J=M-2  
4 IF(I.EQ.1) GO TO 5  
IF((X-XX0(I)).GE.(XX0(I+1)-X)) GO TO 5  
I=I-1  
5 IF(J.EQ.1) GO TO 6  
IF((Y-YY0(J)).GE.(YY0(J+1)-Y)) GO TO 6  
J=J-1  
6 A1=XX0(I)  
A2=XX0(I+1)  
A3=XX0(I+2)  
B1=YY0(J)  
B2=YY0(J+1)  
B3=YY0(J+2)  
U(1)=(X-A2)\*(X-A3)/((A1-A2)\*(A1-A3))  
U(2)=(X-A1)\*(X-A3)/((A2-A1)\*(A2-A3))  
U(3)=(X-A1)\*(X-A2)/((A3-A1)\*(A3-A2))  
V(1)=(Y-B2)\*(Y-B3)/((B1-B2)\*(B1-B3))  
V(2)=(Y-B1)\*(Y-B3)/((B2-B1)\*(B2-B3))  
V(3)=(Y-B1)\*(Y-B2)/((B3-B1)\*(B3-B2))  
DO 8 K=1, L  
W=0.0  
DO 7 II=1, 3  
DO 7 JJ=1, 3  
I1=I+II-1  
J1=J+JJ-1  
7 W=W+U(II)\*V(JJ)\*ZZ0(I1, J1, K)  
8 Z(K)=W  
RETURN  
END

C -----





C  
C  
C  
C  
C  
C

( 24 ) ----SOLVE THE CREEP PROBLEM WITH ITERATIVE METHOD  
USING THE BAILOY-NORTON LAW

SUBROUTINE CRSOL(NTC,NT,CCDP,CCDT,CTI,CCTE,PSC,PSR,PSZ,PED,PEC,PER,  
\$,PEZ,PEE,ECP,ERP,EZP,ECP,CCKE,PPED)

C

DIMENSION PSC(NTC),PSR(NTC),PSZ(NTC),PEC(NTC),PER(NTC),PEZ(NTC)  
DIMENSION ECP(NTC),ERP(NTC),EZP(NTC),PED(NTC),PEE(NTC),EEP(NTC)  
DIMENSION CTI(NTC),CCTE(NTC),CCKE(NTC),PPED(NTC),CO(3)  
DIMENSION CCDP(NTC),CCDT(NTC)  
COMMON /A7/CO/C2/CAC,CMC,CNC/C4/EPS/AC5/AASC,AASR,AASZ,AAEC,AAER,A  
\$AEZ  
DATA CTE,CEC,CER,CZE,DPEC,DPER,DPEZ/7\*0.0/

C

C OFFER THE INITIAL VALUES AT t=0

ASC=AASC  
ASR=AASR  
ASZ=AASZ  
AEC=AAEC  
AER=AAER  
AEZ=AAEZ

C

C OFFER THE INITIAL ESTIMATES FOR ITERATION

AA=CO(1)  
BB=CO(2)  
CC=CO(3)

C

C SOLVE THE CREEP PROBLEM FOR EACH TIME INTERVAL

DO 11 I=1,NT  
CNDP=CCDP(I)  
CNDT=CCDT(I)  
CTE=CTE+CTI(I)  
CCTE(I)=CTE  
M=0

C

C ITERATIVE PROCESS IN ONE TIME INTERVAL

10 M=M+1  
IF(M.GT.100) THEN  
STOP 'FAIL IN DIVERGENCE'  
END IF

C

C CALCULATE THE COEFFICIENTS FOR COMPUTING CREEP

CALL CCOEF(CNDT,DPEC,DPER,DPEZ,AA,BB,CC)

C

C CALCULATE THE INCREMENTS OF TOTAL STRESSES AND STRAINS

CALL CSTRS(CNDP,DSC,DSR,DSZ,DEC,DER,DEZ)

C

PSCN=ASC+DSC  
PSRN=ASR+DSR  
PSZN=ASZ+DSZ

C

C CALCULATE THE DEVIATORIC STRESS TENSOR

SSC=(2.\*PSCN-PSRN-PSZN)/3.  
SSR=(2.\*PSRN-PSCN-PSZN)/3.  
SSZ=(2.\*PSZN-PSCN-PSRN)/3.



C  
C  
C

```

CALL EESTN(AA, BB, CC, CDEE)
CMCD=1./CMC
CKE=CAC*CNC*(CTE**(CNC-1.))*CTI(I)
C COMPUTE THE EFFECTIVE STRESS FROM CREEP LAW
  CPED=((CDEE/CKE)**CMCD)*(6.895*1.E3)
C COMPUTE THE CREEP STRAIN INCREMENTS
  CCS=1.5*CDEE/CPED
  CDEC=CCS*SSC
  CDER=CCS*SSR
  CDEZ=CCS*SSZ
C CHECK THE DIFFERENCE BETWEEN TWO COMPUTATIONS
  CR1=ABS(AA-CDEC)
  CR2=ABS(BB-CDER)
  CR3=ABS(CC-CDEZ)
  CR=AMAX1(CR1, CR2, CR3)
C CHECK THE CONVERGENCE USING THE DESIGNED CRITERION EPS
  IF(CR.GT.EPS) THEN
    AA=CDEC
    BB=CDER
    CC=CDEZ
    GO TO 10
  END IF
C CALCULATE THE CREEP STRAIN RATE
  CCKE(I)=CAC*CNC*(CTE**(CNC-1.))*((CPED/(6.895*1.E3))**CMC)
C
  PEC(I)=AEC+DEC
  PER(I)=AER+DER
  PEZ(I)=AEZ+DEZ
  ECP(I)=CEC+CDEC
  ERP(I)=CER+CDER
  EZP(I)=CEZ+CDEZ
  PPED(I)=CPED
  PSC(I)=PSCN
  PSR(I)=PSRN
  PSZ(I)=PSZN
C
  CALL EDSTS(PSC(I), PSR(I), PSZ(I), PED(I))
  CALL EESTN(PEC(I), PER(I), PEZ(I), PEE(I))
  CALL EESTN(ECP(I), ERP(I), EZP(I), EEP(I))
  AA=CDEC
  BB=CDER
  CC=CDEZ
  ASC=PSC(I)
  ASR=PSR(I)
  ASZ=PSZ(I)
  AEC=PEC(I)
  AER=PER(I)
  AEZ=PEZ(I)
  CEC=ECP(I)
  CER=ERP(I)
  CEZ=EZP(I)
11 CONTINUE
  RETURN
  END
C -----

```

( 25 ) ---- OUTPUT OF THE FINAL RESULTS FOR CREEP PROBLEM

SUBROUTINE CROUT(NTC,NT,CCTE,NT1,NT2,PSC,PSR,PSZ,PEC,PER,PEZ,ECP,E  
\$SRP,EZP,PED,PEE,ECP,PPED)

DIMENSION PSC(NTC),PSR(NTC),PSZ(NTC),PEC(NTC),PER(NTC),PEZ(NTC)  
DIMENSION ECP(NTC),ERP(NTC),EZP(NTC),PED(NTC),PEE(NTC),EEP(NTC)  
DIMENSION CCTE(NTC),CCKE(NTC),PPED(NTC)  
COMMON /AC4/ATM,APP,AED,APE/AC5/ASC,ASR,ASZ,AEC,AER,AEZ

IF(NT1.GT.0) THEN

INDICATE THE ORIGINAL STATE WITHOUT CREEP OCCURANCE AT t=0

CTT=0.0

WRITE(6,5) CTT

WRITE(9,5) CTT

5 FORMAT(///,22X,'<CREEP-TIME DEPENDENT SOLUTION>',/,20X,35('\*'),//,  
\$80('='),//,2X,'<ORIGINAL STATE>:',2X,'INITIAL TIME t=',F4.1)

WRITE(6,20) APP,ATM

WRITE(9,20) APP,ATM

WRITE(6,100) ASC,ASR,ASZ,AED

WRITE(9,100) ASC,ASR,ASZ,AED

WRITE(6,120) AEC,AER,AEZ,APE

WRITE(9,120) AEC,AER,AEZ,APE

20 FORMAT(21X,'PRES-R=',E11.4,2X,'TEMP-T=',F7.2)

WRITE(6,25)

WRITE(9,25)

25 FORMAT(80('='))

EXPORT THE FINAL RESULTS INCLUDING CREEP

DO 10 I=NT1,NT2

WRITE(6,30) I,CCTE(I),CCKE(I),PPED(I)

WRITE(9,30) I,CCTE(I),CCKE(I),PPED(I)

30 FORMAT(/,2X,'STEP(',I3,'):',2X,'CUMULATIVE TIME-',F8.2,2X,'C-RATE  
\$=',E11.4,2X,'CGMA-BR=',E11.4,/) )

WRITE(6,100) PSC(I),PSR(I),PSZ(I),PED(I)

WRITE(9,100) PSC(I),PSR(I),PSZ(I),PED(I)

WRITE(6,120) PEC(I),PER(I),PEZ(I),PEE(I)

WRITE(9,120) PEC(I),PER(I),PEZ(I),PEE(I)

WRITE(6,140) ECP(I),ERP(I),EZP(I),EEP(I)

WRITE(9,140) ECP(I),ERP(I),EZP(I),EEP(I)

10 CONTINUE

ELSE

WRITE(6,150)

WRITE(9,150)

END IF

100 FORMAT(1X,'SGMA-C=',E11.4,2X,'SGMA-R=',E11.4,2X,'SGMA-Z=',E11.4,  
\$2X,'SGMA-BR=',E11.4)

120 FORMAT(1X,'EPS--C=',E11.4,2X,'EPS--R=',E11.4,2X,'EPS--Z=',E11.4,  
\$2X,'EPS--BR=',E11.4)

140 FORMAT(1X,'C-EPSC=',E11.4,2X,'C-EPSR=',E11.4,2X,'C-EPSZ=',E11.4,  
\$2X,'C-EPSBR=',E11.4,/,80('-'))

150 FORMAT(/,2X,'( NO OUTPUT FOR CREEP RESULTS. IF YOU WANT, PUT:  
\$ NT2 > NT1 > 0 )',/)

RETURN

END



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