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UNIVERSITY OF ALBERTA

DILUTION OF CIRCULAR WALL JET IN CROSSFLOW

BY

JOHN KIPKETER LANGAT



A THESIS

SUBMITTED TO THE FACULTY OF GRADUATE AND

RESEARCH IN PARTIAL FULFILLMENT OF THE

REQUIREMENT FOR THE DEGREE OF

OF

MASTER OF SCIENCE

IN

WATER RESOURCES ENGINEERING

DEPARTMENT OF CIVIL ENGINEERING

EDMONTON, ALBERTA SPRING, 1994.



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FACULTY OF GRADUATE STUDIES AND RESEARCH

The undersigned certify that they have read, and recommended to the Faculty of Graduate Studies and Research for acceptance, a thesis entitled Dilution of Circular Wall Jet in Crossflow submitted by John Kipketer Langat in partial fulfillment of the requirements for the degree of Master of Science in Water Resources Engineering.

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ABSTRACT

Dilution of turbulent circular wall jets discharging perpendicular to crossflow ambients has not received much attention compared to studies on free turbulent jets in crossflows. An effluent diffuser designed as a circular wall jet is an efficient means of enhancing dilution of waste discharges from both industrial and municipal weatment plants under river conditions. The outfall is located at / or near the bed in order to utilize the whole flow depth in mixing and therefore this increases the efficiency of effluent dilution within the mixing zone. The present study considers the dilution characteristics of a circular wall jet in river-like crossflows and its comparison with the free jet discharging perpendicularly from the bed. The location of the jet takes into account the limited depths available for mixing in most shallow rivers.

This thesis presents the results of an experimental study on the dilution produced in circular wall jets discharging perpendicular to a freestream. The deflected wall jets were explored for dilutions up to 100:1 for downstream distance x up to about 200 times the jet nozzle diameter. Dilution characteristics of the jet were studied by first performing a similarity analysis on the 3-dimensional concentration distributions. Non-dimensional concentration profiles were found to be similar both in the vertical and the transverse directions. Minimum dilutions were considered along the axial distance of the jet for the flow regimes defined by the momentum dominated near field, the far filed and the passive plume regions. For practical purposes, a mean equation was developed to describe the minimum dilution in the mixing region. The correlation was found to be of similar magnitude with the results of the free jets. The growth rate of the jet, both in the vertical and the transverse directions was also investigated. The results indicated that the fail with with the deflected jet is about twice that in the vertical direction for a saracteristic of wall jets takes advantage of the available mixing space since most rivers have widths several times larger than the flow depth, and conserver to increase the effluent dilution.

Effect of the flow depths on the trace delution was also studied. The results indicated significant reduction in tracer deletion for freestream flow depths less than 10 times the jet nozzle diameter. This problem prevails for effluent discharged into shallow rivers due to early surfacing.

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NOTATIONS

8	cross - section area of the sample delivery tubes
b	length scale in the velocity distribution for a simple jet
b _c	length scale in the concentration distribution for a simple jet
bz	length scale on the z - direction
b₊	length scale on the transverse η - direction
С	concentration at a section of the jet
C _o	initial jet concentration at the nozzle
C _m	maximum jet concentration at a section
d	jet nozzle diameter
D	depth of the freestream flow
k	diffusion coefficient ratio
k _i	empirical constant in a simple jet
l _m	momentum length scale [= $M^{1/2} / U$]
M	kinematic momentum flux at the jet nozzle
R	Reynolds number of the jet
r	radial distance to any point from the axis of a simple jet
•	velocity at any point of a simple jet in the axial direction
U	freestream velocity

- um maximum jet velocity at a section
- U₀ initial velocity of the jet at the nozzle
- x co ordinate direction along the freestream
- x. downstream x co ordinate of the deflected jet where momentum dominated near field ends
- x_† downstream x co ordinate of the deflected jet where momentum dominated far field ends
- y co ordinate direction along the original direction of the jet perpendicular to the freestream
- y_c centerline co ordinate on the y axis of the deflected jet
- z vertical (depth wise) co ordinate direction above the channel bed
- α the ratio of the jet velocity to the ambient freestream velocity
- δ elevation of the maximum concentration axis above the bed at a acction
- υ kinematic viscosity of the fluid
- θ angle of the jet deflection from the original orientation
- ξ distance along the centerline axis of the deflected jet
- ξ_m axial co ordinate of the deflected jet at the end of the potential core
- ξ_{\dagger} axial co ordinate of the deflected jet at the end of momentum dominated far field
- η transverse co ordinate direction normal to ξ
- ζ vertical co ordinate direction normal to ξ
- ρ density of the jet fluid

1.0 INTRODUCTION

1.1 General

A circular jet discharging at an angle into a moving free stream is of practical interest in the developments associated with waste disposal into rivers, lakes and into the atmosphere. The wastes which result from increased urbanization and industrialization have continuously increased the pollutant loadings of both the hydrosphere and the atmosphere. Examples of waste disposal issuing as a jet at an angle to the moving ambient fluid include the emission of industrial smoke into the wind, treated municipal waste and other industrial wastes into rivers. The aim of these jet discharges is to produce significant mixing in a relatively short distance.

The present study investigates the dilution of a circular wall jet in crossflow. The jet induced turbulence provides the initial mixing and the subacquent dilution of the pollutants and is greatly influenced by the design of the outfall. The interaction which occurs in a jet discharging normal to the streamflow enhances the mixing process and results in high dilutions compared to jets in stagnant or co-flowing ambients. Sufficient knowledge of pollutant concentration distributions within this reach of the river where the outfall is located is important. Outfall design would involve taking into considerations the shallow nature of natural streams and the pollutant control standards imposed to protect against indiscriminatory disposal of toxic effluents into rivers.

The advantages of locating the outfall near or at the stream bed are:

I. most industrial wastes are colored and would be aesthetically objectionable if discharged near the surface of the rivers. Discharges near the bed would

utilize the whole depth of flow in the river to diffuse the unpleasant color before surfacing.

II. effluent temperatures are normally slightly higher than that of the receiving waters (about 1° C - 5° C difference) due to the biological processes in the retention ponds before release into the river. Effluents therefore tend to float in a stratified manner in the river flow and this could be minimized by locating the jet diffuser near the bed.

A possible disadvantage is the possible erosion of the river bed in the vicinity of the outfall. To minimize the problem, provision of protection works would be necessary, such as constructing a concrete apron or stone pitching the sections that may be affected by the jet. A wall (or bed) jet diffuser similar to the present model and currently in use was designed by N. Rajaratnam and A. Charbannou for the Cariboo Pulp mill in Quesnel, in northern British Columbia which discharges into the Fraser river.

1.2 Turbulent Jet Mixing in Crossflows

Initial concentration of effluents at the outfall may be much higher than the allowable receiving stream concentration. A zone therefore exists where significant dilution occurs in a relatively short distance, normally referred to as the initial dilution or mixing zone. Gowda (1964) described this zone as a limited use zone because the effluent concentrations often exceed the set standards of the receiving stream. Circular turbulent jets discharging the effluent perpendicular to the free stream flow is one of the effective schemes of achieving this significant dilution. In some cases, the effluent is discharged upwards into the free stream from a submerged outfall embedded in the middle of the crossflow (Wright, 1977; Hodgson and Rajaratnam, 1992). To use the limited flow depth in crossflows such as shallow rivers, it is preferable to delay the effluents from surfacing by locating the jet on or near the bed and discharging normal to the freestream as a wall (bed) jet. It is assumed that the ratio of the stream flow depth (D) to the jet diameter (d) is large enough to minimize disturbance of the free surface in the stream by the jet. Therefore, the parameter (D/d) will be considered not important in this analysis of the circular wall jet in croasflows. (It should be mentioned that in our case, the wall jets should be called bed jets but following the general practice in Fluid Mechanics, they are referred to as wall jets, where all boundaries are referred to as walls.)

The work of Hodgson and Rajaratnam (1988) broadly classified deep water and shallow water jets based on the parameter $\alpha d / D$, where α is the ratio of the jet velocity, U₀ to the free stream velocity, U. A photographic study of circular jets discharging into river-like shallow flows showed that the critical value for the parameter $\alpha d / D$ is about 0.34. Larger values of this parameter correspond to shallow water jets and smaller values to deep water jets. In the present work, values of $\alpha d / D$ are as large as 1.2, and the jet showed no surface effects indicating that it behaved as a deep water jet. Use of the same parameter to categorize wall jets is therefore not applicable because the growth towards the surface is only through diffusion and the jet momentum is not in the direction of the freestream depth.

Simple axisymmetric circular turbulent jets discharging into a large quiescent ambient is first considered. Considering the region of developed flow, the dilution along the axis of a simple jet is approximated by the expression (Rajaratnam, 1983):

$$\frac{C_{n}}{C_{m}} = k_{1} \frac{x}{d}$$
(1.1)

where C_0 and C_m are respectively the concentrations at the jet nozzle of diameter d, and maximum concentration at an axial distance of x downstream of the nozzle and k_1 is an empirical constant determined through experiments. The work of Hodgson and Rajaratnam (1989) found that k_1 is approximately equal to 0.19.

Several studies have been conducted on the mixing of circular jets in crossflows of large extent (Rajaratnam, 1976). One of the more significant studies in this area is the work of Wright (1977). Recently, Hodgson and Rajaratnam studied the dilutions of circular jets in rivers, including shallow flows (Hodgson and Rajaratnam, 1992).

1.3 The Present Investigation

Most of the research work on circular turbulent jets in crossflow have emphasized the momentum characteristics compared to the concentration measurements. The present work emphasizes the practical importance of the jet dilution issuing as a wall jet in crossflows as applied in hydraulics.

The objective of the experimental investigation was to study the main dilution characteristics of circular wall jet in ambient flows normal to the jet discharge and to compare with the existing work for the momentum dominated regimes of the free jet. To understand the problem, general mixing characteristics of turbulent jets in crossflow were first considered, developing from the simple case of the axisymmetric jet. Review of the existing work on the free jet in crossflows formed the basis of studying dilution characteristics of circular wall jet in crossflow. The analysis of the dilution data was based on the flow regimes defined by the momentum dominated near field and far field regions of the jet. The results were then compared with those of the free jet conditions investigated by Wright (1977) and Hodgson and Rajaratnam (1992).

2.0 LITERATURE REVIEW

2.1 General

Circular jets in croasflows have attracted the attention of researchers because of their practical importance in several fields. List (1982) indicated that the earliest work on the subject of jets was by Young (1800) who recognized that the angle of expansion of turbulent jets was not a function of the jet velocity. A simple axisymmetric jet discharging into a stagnant ambient is well understood, but significant differences occur with complex turbulent shear flows when the ambient is changed to a crossflow. The earliest published work on jets in crossflow was carried out by Ruggeri et al, 1949 (see Rajaratnam, 1976) who investigated jet penetration in a wind tunnel. Hodgson (1991) tabulated a comprehensive summary of the previous experimental work on jet discharges in crossflows but none of the 33 references dealt with a wall jet.

To achieve a significant degree of mixing in the immediate vicinity of the outfall, effluent discharges into the environment, especially in rivers, need to be momentum dominated and because of the shallow nature of the rivers the outfalls need to be located near the bed. Such arrangements will maximize the mixing process within the depth of the river and delay the surfacing of the discharges. Dilutions up to 50 : 1 in the initial dilution zone, have been reported when a jet diffuser is used compared to an outfall which relies solely on the river turbulence as in the case of passive plumes (Hodgson, 1991).

To understand the main characteristics of a circular wall jet in crossflow, previous experimental investigations on the well understood free turbulent jets will be reviewed in this chapter. The simple axisymmetric turbulent jet discharging into a stagnant ambient will be described and compared with a wall jet of similar conditions. This will be followed by the description of the basic components of free circular jet discharges in crossflow. The knowledge on the free jet will be used as a basis for understanding the conditions of a wall jet with similar discharges in crossflow.

2.2 Description of Circular Turbulent Jets

The characteristics of the circular jet in crossflow can be better understood by first describing the simple free jet issuing into a stagnant ambient and then moving the ambient as a crossflow. In this section existing experimental observations on simple axisymmetric wall jets will be discussed. This will be later applied when discussing the effects of the wall on dilution of circular jets in crossflow. Free turbulent jets discharge into a stagnant ambient with boundaries sufficiently far away, while the wall jet discharges tangentially and grows along a wall (Abramovich, 1963). It is useful to note that if the Reynolds number R of the jet, defined as $R = U_0 d/v$ where v is the kinematic viscosity of the fluid, is greater than about 1000, the jet becomes turbulent fairly quickly (Fischer et al, 1979).

2.2.1 Simple Axisymmetric Jet

Consider a circular jet of diameter d and discharging with a uniform velocity U_0 into a stagnant ambient. The jet is divided into two distinct segions (Fig. 2.1). The region near the jet nozzle with undiminished discharge velocity U_0 is known as the potential core or flow development region. In the core, the flow is essentially potential with little or no turbulence and extends to about 6 d (Abraham, 1960;

Rajaratnam, 1976). The end of the potential flow occurs when the turbulence generated by the shear at the boundaries spreads to the axis of the jet.

The next zone, known as the fully developed flow region, occurs beyond the end of the potential core. The mean velocity u_m along the centerline decays with the axial distance x. Rajaratnam (1983) found that the velocity and concentration scales decay inversely with the distance along the jet axis and are described by the following equations:

$$\frac{u_m}{U_0} = \frac{6.13}{x/d}$$
(2.1)

$$\frac{C_{m}}{C_{0}} = \frac{5.34}{x/d}$$
(2.2)

where:

Uo is the initial velocity of the jet at the nozzle

um is the maximum jet velocity at any section

x is the axial distance from the jet nozzle to that section

d is the diameter of the jet nozzle

 C_0 is the initial concentration in the jet

Cm is the maximum concentration at the section considered.

The radial distributions of velocity and concentration are also known to be similar. These distributions are well described by the following equations:

$$\frac{u}{u_m} = \exp\left[-\ln 2\left(\frac{r}{b}\right)\right]$$
(2.3)

$$\frac{C}{C_{\rm m}} = \exp\left[-\ln 2\left(\frac{r}{b_{\rm c}}\right)\right] \qquad (2.4)$$

where:

u is the velocity at a point at a section

r is the radial distance to the point from the jet axis

b is the length scale (distance to the point where $u = 1/2 u_m$)

 b_c is the length scale (distance to the point where $C = 1/2 C_m$) and

C is the concentration at a point.

The variation of the length scale b used in the velocity distribution for the circular turbulent jets has been found to be a linear function of the axial distance x from the nozzle, given by:

Similarly, the growth of the concentration length scale b_c was found to be a linear function of the axial distance but with a greater slope and is given as:

$$b_c = 0.096 \text{ k x}$$
 (2.6)

where k is an empirical coefficient.

The work of Rajaratnam (1983) found the diffusion coefficient ratio k to have a constant value of about 1.17 in the whole region of the developed flow.

The work of Sforza and his associates (1966) and that of Rajaratnam and Pani (1974) considered three-dimensional turbulent wall jets. The shape of the nozzles used in the studies included a square, a circle and a rectangle with an aspect ratio close to unity. Experimental observations were carried out in the fully-developed region to find out the velocity distributions and the growth of the jet both in the vertical and the lateral directions. The results on the three-dimensional wall jet indicated that the center plane velocity distributions in the vertical direction were reasonably similar along the longitudinal distance greater than 10 times the nozzle height (or diameter). The classical wall jet similarity curve (plane turbulent wall jet issuing tangentially to a smooth wall, surrounded by a stagnant ambient of the same fluid of infinite extent) also described well the distributions of the three - dimensional jet. The results of the length scales indicated that both the vertical and the horizontal scales were linear functions of the longitudinal distance downstream of the nozzle. From the length scales, it was found that the lateral scale grows about 4 to 5 times faster than the vertical scale. The wall shear coefficient was also found to increase to a constant value beyond the downstream distance equal to 50 times the square root of the nozzle cross - sectional area. By neglecting the shear on the wall, these authors considered the problem could be analyzed in a manner similar to that of three-dimensional free jets.

2.2.2 Circular Jet in Cressflow: Preliminary discussion

When a moving ambient perpendicular to the jet discharge is introduced to replace the stagnant conditions in the axisymmetric jet, the set is described as a circular jet discharge into a crossflow. Free turbulent jet discharge in an infinite depth of the crossflow preva¹ when the walls (or boundaries) are sufficiently far away (Abramovich, 1963) and do not influence the growth of the jet in the momentum dominated region.

The main characteristics of jets in crossflow are defined by considering a circular jet of diameter d discharging with an initial velocity U_0 perpendicular to a moving free stream of velocity U (Figure 2.2). The jet would initially penetrate at

right angles to the ambient flow but eventually be deformed by the crossflow and deflected from its original course to approximately the ambient flow direction. Rajaratnam and Gangadhariah (1980) indicated the cause of the jet deflection as the result of the pressure field exerted near the jet nozzle by the free stream as well as the entrainment of the ambient fluid. Deflected jets experience three zones of flow, as opposed to two for the simple jet.

The cone-shaped potential core forms the central region of the jet near the nozzle characterized by the shear free flow and undiminished total pressure (Rajaratnam, 1976). Studies by Pratte and Baines (1967), Keffer and Baines (1971) and Rajaratnam and Gangadhariah (1980) have identified the length of the potential core in crossflows to be shorter than that of the simple jet because of the extra shear exerted by the crossflow. Accurate determination of the core length has not been achieved yet. Hodgson (1991) analyzed the measurements from the above mentioned studies and found that this length varies from 1 to 4 times the jet diameter for velocity ratios α greater than 10. Other approximations have considered extrapolating backwards the centerline velocity profile on a log-log plot to the point where $u_m = U_0$. This method was lound by Hodgson (1991) to overestimate the length when data from Keffer and Baines (1963) for velocity ratios $\alpha = 4$, 6 and 8 were used. According to Rajaratnam (1976), the end of the potential core is located on the jet centerline for $\alpha > 4$. Platten and Keffer (1971) noted a downwind displacement of the core length by the crossflow for weak jets with velocity ratios at < 2

The zone after the potential core where the jet undergoes major deflection is the second flow region known as the zone of maximum deflection and the jet axis have changes orientation from the original normal direction to almost parallel with the free stream flow. Turbulent mixing causes the outer layers of the jet to lose some of their momentum through entrainment of free stream fluid and become susceptible to the deflection of the crossflow. In this region therefore, the cross-section of the jet is deformed from the original circular to a kidney shape (Abramovich, 1963), which houses two counter-rotating vortices. As the jet continues interacting with the crossflow, it entrains more ambient fluid. This results in a growth in size and strength of the vortex pair and increases the total flow of the jet. Some decay of maximum velocity and hence the concentration in this region occurs along the curvilinear axis of the jet. According to Hodgson (1991) the maximum deflection zone extends to an axial distance (ξ) of about 15 to 20 times the jet diameter.

The third flow region is referred to as the vortex zone and occurs when the cross-section of the jet is almost completely occupied by the vortex pair. Further entraintment increases the size of the vortices and both the jet velocity and direction approach those of the ambient crossflow. Pratte and Baines (1967) and Fearn and Weston (1979) noted in their investigations the presence of the vortices at distances as near as x = 10 d and as far as x = 1000 d downstream of the jet nozzle. The strength of the vortices decrease with distance downstream of the jet.

2.2.3 Diffusion of Circular Jots in Cressilow

Several investigations have been carried out examining the structure of the velocity and pressure fields of the jets in crossflow (Sherif and Pletcher, 1989). Initial studies were conducted in air medium and using pitot tubes to measure the total and dynamic pressure heads in the jet. The axis of the jet was then located by the points of maximum pressure (Jordinson, 1956). Similar experiments were carried out by

Gordier (1959) using water instead of air, to measure the distribution of the total pressure in the jet in a water duct. Gordier found that the results were similar to those of the air jet.

The introduction of the Hot - Wire Anemometry (HWA) in the early 1960's and the Laser Doppler Anemometry (LDA) in the mid 1970's improved the understanding of the jet discharges. Both instruments were able to measure the velocity profiles, jet axis trajectory, the turbulence and the vortex structures of the deformed jet.

Rajaratnam and Gangadhariah (1980) studied experimentally the behaviour of circular jet in crossflow for four different discharges of velocity ratios $\alpha = 2.73$, 4.52, 7.05 and 23, in a water flume. The velocity and piezometric pressure distribution in the full cross-section of the deflected jet were measured using the pitot tube. The results predicted the shape of the deformed jet, the centerline trajectory, the velocity distribution, and the mass and momentum fluxes of the jet discharges.

Photographic investigations were also used in a number of studies to locate the jet axis trajectory and jet boundaries. Gordier (1959) located the jet penetration (outer boundary) and the width from photographs of a deformed jet in a water flume. Pratte and Baines (1967) used photographs to locate the jet axis trajectory, the width and thickness of a jet marked by oil aerosol discharging in an air crossflow. Similar photographic studies were carried out by Crowe and Riesebieter (1967) using amoke, while Margason (1968) used water vapour for the observations. Wright (1977) photographed extensively 60 different deflected jet discharges of dye for the velocity ratio α between 0.8 and 116 for a downstream distance extending to a maximum of x/d = 113. Wright considered both the buoyant and non-buoyant jets in uniform and stratified crossflow and used his photographic trajectory plots to define the distinct

flow regimes which occurred in the deflected jet. Wright also defined a momentum length scale $l_m = M^{1/2}/U$ as a criteria for separating the flow regimes.

2.2.4 Dilution of Circular Jets in Crossflow

Investigations of dilution characteristics of circular jets in crossflow are not as numerous as the studies of the diffusion. Studies using temperature and other conservative tracers have added significant knowledge to the understanding of dilution of turbulent jet discharges. The following section summarizes the published experimental studies on dilution of jets in crossflow.

Patrick (1967) investigated both the velocity and concentration distributions of jet discharges in crossflow using nitrous oxide (NO2) as a tracer. The experiments were carried out in an air medium for 23 different jet nozzle discharges and velocity ratio α between 8.5 and 54. Sampling was carried out using the pitot tube. The results indicated that the jet centerline dilution was a function of the axial distance (ξ) of the jet. Centerline dilutions of up to 100:1 for the downstream distance of $\xi/d = 50$ were reported.

The work of Fan (1967) considered the dilution of buoyant jet discharges in crossflow. The investigation was carried out in a water medium for 10 different discharges of ask solution through a jet orifice. The velocity ratios α were between 4 and 16, and the densimetric Froude number F₀ between 10 and 80. Crossflow simulation was done by towing the orifice. Concentration distributions within the flow field were determined using a conductivity meter or probe for downstream distances upto x/d = 250. Fan concluded from his results that the enhanced mixing was due to the solf-generated turbulence of the jet.

Ramsey and Goldstein (1971) investigated the distribution of temperature for weak air jet discharges in crossflow, and velocity ratios $\alpha = 0.1$, 0.5, 1.0 and 2.0. They used both the hot-wire anemometry and a thermocouple to measure the temperature profiles. Their measurements covered a downstream distance of x / d = 10. The results indicated that the weak jets ($\alpha = 0.1$ and 0.5) remained attached to the wall, while the stronger jets ($\alpha = 1$ and 2) were clearly acparated from the wall. Similar investigations were carried out by Kamotani and Greber (1972) for high velocity ratios, $\alpha = 3.9$ and 7.7. Their results indicated that the vortex structure was not established for the weaker jet discharge ($\alpha = 3.9$), while the stronger jet discharge ($\alpha = 7.7$) had a developed twin vortex structure typical of jet discharges in crossflow. The decay of the centreline temperature was found to be a function of the axial distance (ξ). Initially the temperature decay was alower for the stronger jet but the dilution was found to be about the same beyond an axial distance of $\xi / d = 70$.

Chu and Goldberg (1974) carried out both the theoretical and experimental investigation on the location and dilution of buoyant jet discharges in crossflow. The experiments were carried out in a water medium for 8 different jet discharges of velocity ratios α between 6.3 and 44. Dyed salt solution was used as a tracer and both the measurements of salinity and the photography were used to interpret the results. Their conclusion indicated that the jet dilution was a function of the downstream distance and described by the two-thirds power law. It will be realized later the two-third power normally describes the dilution of the momentum dominated region of pure jets. The relatively high velocity ratios ($\alpha > 3.5$) used by the authors in most of their tests explains the similarity of their results to those expected for the non-buoyant jets.

Wright (1977) investigated extensively the whole range of buoyant and nobuoyant jets discharging into uniform and stratified crossflows. Wright used dimensional considerations, photographic analysis and dilution measurements using Rhodamine B Extra dyc as a tracer. The jet source was towed at a constant velocity discharging into a stagnant ambient in a flume. Concentration measurements were carried out for 20 non-buoyant jet discharges into unstratified crossflows and velocity ratios or between 20 and 35. Relative concentration of samples along the jet trajectory were analyzed using a G.K. Turner Associates Model 111 fluorometer. Dilution analysis were carried out from measurements of the maximum concentrations along the jet centerline. The experimental results agreed with the theoretical formulations and indicated that distinct transitions occur between the various flow regimes of the jet. For the non-buoyant jets discharging into a uniform crossflow, Wright used both the dilution and trajectory plots to define the two slopes of the momentum dominated regions of the jet. The region described by the one-half slope near the jet nozzle was defined as the momentum dominated near field (MDNF) and the next region described by the two-thirds slope as the momentum dominated far field (MDFF). The data also indicated that the concentration centerline was located about 20 % higher than the centerline derived from the photographic analysis.

Hodgson and Rajaratnam (1992) conducted dilution studies in the laboratory with a similar approach to that of Wright (1977) and compared these results with some investigations in rivers. He considered flow velocity ratios α between 1.46 and 10.55, and depth ratios between 15.7 and 41.3. The investigations were meant to simulate more closely the actual field situations for downstream distances in the range $1.1 < \alpha x / d < 990$. The results were well represented by a power law relation.


Fig. 2.1 Turbulent Circular Jet Discharge into a Stagnant Ambient



int Circular Jet Diacharge in Croadion (after Rajaratuan, 1976) Pg 22 8da

3.0 LABORATORY INVESTIGATIONS

3.1 General

The laboratory study on the dilution of a wall jet in crossflows was carried out to simulate the prototype river conditions by introducing a freestream flow in the channel (here referred to as crossflow) with a flow depth (D) to jet diameter (d) ratio of similar magnitude to the actual field situation. The bed of the channel would not represent the true conditions prevailing in the river but assumed the effect is minimal on the model experimental results. The channel flow was from the laboratory sump water refilled from the regular city water. Temperatures remained constant at the ambient laboratory conditions of about 20 (± 0.5) degrees Celsius throughout the experimental work.

Detailed discussion of the experimental procedure is presented in the following sections under three categories, namely: the experimental arrangement, calibration of the fluorometer and the sampling procedure.

3.2 Experimental Arrangement

3.2.1 The Flume

The experiments were carried out in a rectangular channel 0.91 m wide, 0.77 m deep and 36.5 m long (Plate 3.1) in the University of Alberta's T. Blench Hydraulics Laboratory. The channel is fitted with an adjustable slope mechanism, and the freestream flow was provided by a pump located in the laboratory sump. The discharge was measured using an inline magnetic flow meter with a digital measuring

scale (reading to 2 decimal points) and calibrated for 1.0 VC unit to be equivalent to 0.015 m³/s. The mean velocity in the channel section of the experiment was calculated from the measured discharge in the flume and the section flow area. The average velocity was also measured for 3 locations across the flume at distances of 80 mm, 220 mm and 450 mm (center of the flume) from the side wall using Prandtl-type pitot-static tube and transducers. The velocity profiles shown in Figure 3.1 clearly indicate that the wall effect extends as far as 80 mm into the freestream flow.

3.2.2 Jet Arrangement

Well-designed nozzles of diameters equal to 6.35 mm and 12.7 mm were used to produce the circular wall jets and both were located at a distance of about (22.0 m) downstream of the flume entrance. The nozzles were installed flush with the side wall of the flume with the bottom of the internal diameter at about the same level as the channel bed (Plate 3.2). The jet flows were generated by a 1/3 horsepower Jacuzzi model SP125/B pump fitted with a bypass line and connected to a constant head tank by a nalgene 800 tubing of 25.4 mm (1-inch ID) internal diameter and 3 mm wall thickness. The Rhodamine dye used as a tracer in the present study was mixed in a 900 litres tank connected to the pumping unit near its base, and received both the bypass line flow and the overflow from the head tank. Water used for mixing the dye was pumped from the same sump which provided the freestream flow.

The flow rate through the jet nozzle was measured by a calibrated Fischer rotameter rated in litres per minute (Plate 3.3) and controlled by two gate valves, one on the bypass line and the other on the line connecting to the constant head tank and

the jet nozz.'e. The calibration of the rotameter was also checked by volumetric measurement.

The velocity at the jet nozzle was calculated from the rotameter discharge reading and the jet nozzle diameter. Velocity at the nozzle was maintained constant by the provision of the constant head tank on the flow line with an overflow arrangement.

3.3 Calibration of the Fluorometer

The analysis of the fluid samples was carried out in a Turner model 10 portable fluorometer (Plate 3.4) by manually filling the 35 ml cuvette or a clear test tube, with the sample from each bottle container at a time. The fluorometer output reading is on a double scale from 0 to 10 on the upper scale or 0 to 3.16 on the lower scale and both scales are related to each other by a factor of 3.16. For convenience with the decimal points, the upper scale was used throughout the present work. The fluorometer was calibrated using diluted standards of the Rhodamine dye (original market concentration of 20 % by weight of the dye and specific gravity of 1.19) solution corresponding to its range of 0.01 ppb to 55 ppb in order to derive the relation for interpreting the relative concentrations of the fluorescent dye in the sampled fluid. Standardized dye concentrations were plotted against the fluorometer output scale reading and the calibration indicated that the dye fluorescence is a linear function of the tracer concentration (Figure 3.2). The calibration best fit line ($r^2 = 0.999$) was expressed by the following relation:

$$C(ppb) = \frac{5.56 + Scale Reading (0 - 10)}{Multiplier + Range}$$
(3.1)

The multiplier was either 1 or 100, while the range of sensitivity was 1, 3.16, 10 or 31.6 and the larger figures indicate low concentration levels of the tracer in the sampled fluid. The multiplier of 1 was constantly used throughout the analysis but the range was varied through all the sensitivities depending on the tracer concentration. The concentrations higher than the maximum reading of the fluorometer could be analyzed by making standard solutions of the dye. The initial concentration (C_0) at the jet nozzle ranged from about 15 ppb to 20 ppb depending on the initial mixing of the dye in the 900 litres tank for the different jet discharges.

Consistency of the fluorometer calibration was checked by taking the readings of the standard solutions before and after the analysis of the samples. It was also necessary to turn on the fluorometer for about 20 minutes before carrying out the analysis of the fluid samples. The fluorometer is capable of measuring the fluorescence intensity of any chemical solution by simply installing the right kind of filter necessary for the particular solution and carrying out similar calibration procedure for the concentrations. The fluorometer could also be equipped with a continuous flow - through sampling procedure when using a single sampling tube.

3.4 Sampling Procedure

3.4.1 Sampling Rake Arrangement

Assessment of the jet dilution in the crossflow was done by carrying out the concentration measurements using Rhodamine WT as a tracer in the jet fluid. Fluid samples were withdrawn by means of a rake comprising a minimum of 7 sampling

probes, each of 3.175 mm (1/8-inch OD) and a wall thickness of 0.9 mm, aligned with the ξ -axis (centerline trajectory) of the jet. The ξ -axis at a section was determined from the photographic trajectory plots of the centerline (Appendix III) and by the injection of the food color dye prior to sampling the tracer dye. The angle of the jet deflection from its original perpendicular direction was measured by a protractor, attached to the top of the sampling rake.

The sampling rake shown in Plate 3.5 was made up of L-shaped brass tubes act at a spacing of 30 mm and held in position by a wooden bracket which was kept out of the water surface at all times during the experiments. The spacing between the tubes was adjustable depending on the position of the sampling and the width of the jet. A minimum of 10 mm spacing was used near the jet nozzle and a maximum of 40 mm was used in the region away from the nozzle, where the lateral concentration gradients were small. The probes were set to withdraw the samples at the same level in the transverse η -axis simultaneously, starting from the bed level and the first tube was sampling near the inner boundary of the jet.

The tracer samples were collected for different levels at a spacing of 5 mm in the z-axis until the level which marks the outer boundary of the jet growth in the vertical elevation. To be able to sample across the jet in the η -axis, a second act of sampling was carried out with the last probe sampling the outer boundary (penetration axis) of the jet and the other tubes located between the positions of the first setting. The procedure followed in the first setting is repeated during the second setting for all the levels. It was assumed that steady state flow conditions prevailed both in the jet nozzle and the channel discharges and therefore sampling at any particular location was not time dependent. The samples for the background channel flow ware withdrawn concurrently by a similar probe located upstream but near the jet nozzle. A siphoning system of vinyl nalgene premium tubings of 3.175 mm (1/8-inch ID) internal diameter and 2.0 m long was used to discharge the tracer samples and collect in 60 ml opaque plastic bottles marked with the positions of the sampling probes. The collection bottles were set up and contained in a wooden sample rack (see Plate 3.6) positioned at an appropriate level to create a sampling velocity at the probe nozzle approximately equal to the main channel flow velocity and a continuous free flow in the tubings. All sample bottles were filled within the same amount of time but sufficient duration was allowed to eliminate the adsorption or desorption of the dye onto the delivery tubings. To minimize the loss of the dye fluorescence due to light, the samples were kept in a dark sample box until the concentration analysis of all the samples was completed.

3.4.2 Proliminary Investigations

The preliminary runs were carried out while developing the operational procedures of the experimental arrangements by visual observations of the jet trajectories and measuring the local velocity profiles of the flow in the flume for comparison with the mean value used in the present study. Figure (3.2) gives the profiles for the 3 locations across the flume at the section of the jet nozzle (mean channel flow velocity U = 0.60 m/s) which indicate that the mean velocity used is satisfactory in the region of the flow where the jet would be growing.

The centerline trajectory profiles of the various jet discharges were visually observed with the help of the colored dye and compared with the co-ordinate locations carlier obtained by Diebel and Rajaratnam (1980) for a similar circular wall jet (Appendix III). Adjustments of the centerline axis were made independently for each jet discharge where necessary during the actual investigations.

In addition to the observations in the flume, the jet discharge was timed to empty the 900 litres tank in order to evaluate the length of the tracer sampling time before refilling the tank. During this process the fluctuation of the dye concentration in the tank was observed to be negligible and assumed constant (as the initial nozzle concentration, C_0) during the actual sampling. It was found that the dye solution in the tank was only enough to carry out sampling for one section of the jet at a time (running for about 3 hours). For every set of sampling it was found that the initial jet nozzle concentration depends on the amount of the dye mixed in the tank and therefore not necessarily constant for the different sections of the jet. To unify a steady state condition of the jet for all the sections investigated, the concentrations were considered with reference to the initial nozzle value (i.e. C/C_0).

3.4.3 Test Series

The present study involved a total of 9 experiments, including five jet discharges ($\alpha = 2, 4, 6, 8$ and 12), two jet nozzles of diameters (d) 6.35 mm and 12.7 mm, and three flow depth ratios (D/d) of 20, 10 and 5. The summary of the conditions investigated are given in Table 3.1.

Sampling procedure of the tracer concentration began with making standard solutions of the Rhodamine dye (original market concentration is 20 % by weight). A standardized working solution of $1.0 + 10^5$ ppb (parts-per-billion) was made from the original solution and kept in a dark 4 litre bottle for use throughout the experiments. For every experiment, about 20 - 30 ml of the standard solution was

withdrawn with the help of a pipette and mixed thoroughly in the 900 litres tank. Tracer mixture in the tank was checked for uniformity by measuring the concentrations at different levels (surface, center and bottom) of the tank. A constant value of the concentration passing through the bypass line of the pumping unit and the overflow line of the constant head tank was used as the initial jet nozzle concentration (C_0) because these flow lines were connected to the nozzle. The nozzle concentration was also checked after every sampling for any fluctuations which might have occurred. Also the temperature of the dye mixture was recorded before and after completing sampling each section of the jet. While the dye mixing was being carried out, a constant discharge was established in the flume.

The movement of the sampling rake perpendicular to the jet axis along the η -axis was restricted because errors could be introduced in the angular shift. The acting therefore was limited to two positions aligned with the centerline axis located by the jet deflection angle (θ) measured from the normal direction to the channel flow. For every setting the parameters noted were; the centerline axis co-ordinates (x, y_c) at the section, the jet deflection angle (θ) and the sampling probe number which locates the centerline of the jet.

The sampling rack containing the bottles for collecting the tracer samples was positioned to determine the discharge velocity of the samples. This was carried out by calculating the discharge velocity (u_s) from the time taken to fill the 60 ml bottles and dividing the discharge by the cross - sectional area (a) of the siphoning delivery tube (i.e. $u_s = 60 \text{ ml} + 1 / \text{time} + 1 / a$). Repositioning was carried out until the velocity u_s was approximately in the range of the crossflow velocity U and the siphon system was free flowing without the interference of the bubbles in the tubings. This procedure was necessary to avoid the sampling probe acting as a sink in the

streamflow or causing obstruction to the tracer discharge. Flushing of the bubbles was carried out whenever they were found.

The jet discharge was allowed to stabilize once all the settings were done before collecting the samples. Sampling was carried out at elevation spacing of 5 mm (z - axis) over the whole jet region in order to minimize the chances of missing the peak concentration (a characteristic maximum) of the jet. The samples were analyzed manually in the fluorometer and the concentration results were corrected for the ambient flow fluorescence which fluctuated between 0.05 - 0.10 ppb, depending on the amount of fresh water fed into the sump.

3.4.4 Error Estimation

The most critical error evaluation was carried out for the concentration measurements using the fluorometer. Significant errors could occur, especially for low values of concentration because the lowest possible reading of the fluorometer scale was 0.05 units when manually operated. The analysis of the standard error introduced by the calibration was carried out using 13 observations of the standard solutions of Rhodamine (diluted to known concentrations with distilled water). The calibration data were compared with the scale reading and the data given in Table 3.2 indicate that the fluorometer scale reading introduces a standard error of 0.01 units (equivalent to 0.002 ppb). This result compared to the lowest recorded reading of 0.5 units of the fluorometer's upper scale gave an error of about 2 % of the tracer concentration. For a 50 : 1 dilution of the tracer, the error introduced in the concentration measurements was less than 1 %. Manual operation of the fluorometer was used and hence resulted in minimal errors compared to those obtained when the

automatic mode was in operation. Concentration errors due to desorption, adsorption, the bubbles in the tubing and the mixing process of making the standard solutions were found difficult to quantify, but may add or lower the actual concentration readings.

Other experimental errors were introduced during the measurements of the discharges, the crossflow and jet velocities and the angular measurements of the sampling probe co-ordinates. The error in the jet velocity resulted from the nozzle diameter machined to a precision of 0.025 mm and the rotameter discharge measurements with acale precision of 0.05 L/min. Linear measurements of the flume flow depth and the sampling spacing were carried out using the scale with a precision of 0.5 mm. Table 3.3 gives a summary of the experimental errors estimated for the parameters considered.

		8	Jet Velecky Ue (m / s)	Cheered Vehecity U (m/s)	a (UeV)	Reymith 76 Life A
A100	6.36	20	0.53	0.26	2	3145
A200	6.35	20	2.4	0.59	•	14243
A300	6.35	20	2.6	0.43	6	15430
84	6.35	20	2.9	0.36	••	17210
A500	6.35	20	3.42	0.285	12	20296
8100	12.7	20	0.96	0.143	¢	10207
8200	12.7	10	0.78	0.13	÷	9258
8300	12.7	v	1.57	0.26	•	18635
8400	12.7	20	0.06	0.07	12	10207

Table 3.1 Details of Experiments of Turbulant Circular Wall Jet in Creation

Standard Solution	Scale Reading	Calibration	Difference
(ppb)	(Upper)	(Equation)	(2-3)
•		3	۲
50	8.9	8.962	-0.062
45	8.1	8.064	0.036
40	7.2	7.166	0.034
35	6.3	6.268	0.032
30	5.35	5.370	-0.020
25	4.5	4.472	0.028
20	3.5	3.574	-0.074
15	2.7	2.676	0.024
10	1.8	1.778	0.022
8	1.4	1.419	-0.019
6	1.05	1.060	-0.010
4	0.7	0.70 0	0.000
2	0.35	0.341	0.009
ileie:			
	0.036 and Standa		
(of the	e essie reading diff		

Table 3.2 Fluoromotor Scale Roading Error Estimation

ļ	Produce	Error Estimation (± %)
Proconcer (Upper Scale)	0.05	< 2
Jet Discharge (L/min)	0.05	< 5 S
let Nozzle Diancter (nnn)	0.025	~
Chemel Discharge (L/scc)	0.05	د ج ک
Row Depth (mm)	0.5	4
Sumpling Spacing (mm)	0.5	s S
Angular Alignment (degree)	0.5	د م ۲

Table 3.3 Brve Butantian







Fig. 3.2 Calibration Curve for the Fluorometer



(b) Sampling Peakton



Plate 3.1 Flume Arrangement

(a) Location of the Nozzies



(b) Diffusion of the Jot: $\alpha = 12$, $\frac{D}{d} = 20$ and d = 6.35 mm



Plate 3.3 Jet Nemie Arrangement



Plate 3.3 Retempter



Plate 3.4 Threase Madel 30 Planeters



(b) Sampling Probes Arrangement



Plate 3.5 Sampling Rake



Pate 3.6 Sampling Rack (showing @ mi batter and defivery takes)

4.0 EXPERIMENTAL RESULTS AND ANALYSIS

4.1 General

Interpretation of the concentration results in the following sections follows the objectives of identifying the main characteristics of dilution of circular wall jets in crossflow and comparing with those of the free jets.

The dilution of circular wall jet in crossflow is a complex three-dimensional problem and the present analysis considers the maximum concentration (or minimum dilution) to characterize the dilution problem. Since no published investigations exist on this problem, the present results are compared with those on dilution of the free circular jet (Wright, 1977; and Hodgson and Rajaratnam, 1992). A generalized form of the results is suggested for practical purposes for predicting the minimum dilution (C_0 / C_m) along the non-dimensional distance $\alpha \times / d$ for each of the flow regimes earlier defined as the momentum dominated near and far fields, as well as the passive plume region at the end.

The growth of the jet both in the vertical ($z - \xi$ plane) and the transverse ($\eta - \xi$ plane) directions would be assessed from the concentration profiles, by comparing the trends of the length scales along the curvilinear distance (ξ) of the jet. In addition, the effect of changing the jet nozzle diameter (d) and the flow depth ratio (D/d) would be considered.

The boundary limits of the momentum dominated flow regimes identified from the earlier results of Wright (1977) and Rajaratnam and Gangadhariah (1980) would be defined. The results of the present investigation would then be categorized graphically into the momentum dominated near field, the far field and the passive plume regions.

4.2 Boundary Limits of the Momentum Dominated Flow Regimes

The analysis presented in this section defines the momentum dominated flow regimes of the deflected jet in crossflows. Based on existing works, the different mixing regions would be identified first for the free circular jet and then applied to the wall jet in crossflows.

The mixing process experienced by the effluents discharged into a receiving stream, from the review of the existing literature is considered to occur in two momentum dominated regimes; the near field region (MDNF) where $y_c \ll l_m$ and the far field (MDFF) where $y_c \gg l_m$. In the MDNF, the jet is displaced by the crossflow but retains its similarity to jets in a stagnant ambient whereas in the MDFF, the diffusion of the jet is affected by the moving ambient. These two regimes are generally followed by what can be termed the passive plume region (PPR) where the velocity excess in the jet has decayed and the mixing and dilution is due to the turbulence in the ambient. However, the boundary limits of the regions were not clearly defined and now form part of the present analysis. The work of Wright (1977) on the momentum dominated trajectories and that of Rajaratnam and Gangadhariah (1960) on circular jets in croasflow are used in the present investigation to derive the limits of the three regimes.

4.2.1 Momentum Dominated Near Field Regime

The photographic observations of Wright (1977) indicate that the intersection of the 1/2 and the 1/3 slopes on the data plot of y_c / l_m against x / l_m defines the co-

ordinates of the separation point between these two regimes. The point is given by $y_c/l_m = 1.5$ and $x/l_m = 1.0$, where l_m is the momentum length scale [= $M^{1/2}/U$] as had earlier been defined. The trajectory relation for the momentum dominated near field flow regime, from dimensional analysis is given by (Wright, 1977):

$$\frac{y_c}{l_m} = C_1 \left(\frac{x}{l_m}\right)^{1/2}$$
(4.1)

where C₁ is the proportionality constant and a function of α .

Let x \cdot to be the x co-ordinate where MDNF ends. Rewriting the above Equation 4.1 and substituting for y_c/l_m becomes:

$$\frac{x_{*}}{l_{m}} = \frac{2.25}{C_{1}^{2}}$$
(4.2)

The distance downstream of the freestream flow from the jet nozzle is given in a nondimensional form as $\alpha x / d$ and, therefore, Equation 4.2 becomes:

$$x_{+} = \frac{2 \alpha d}{C_{1}^{2}}$$
(4.3)

The coefficient C_1 is derived from the experimental data using the general Equation 4.1 and plotted against the velocity ratio α . The data are described by the equation (Hodgson and Rajaratnam, 1988):

$$C_1 = 2.0 - \frac{2.2}{\alpha^{2/3}}$$
(4.4)

Therefore, a plot of $\alpha x_* / d$ against α is given in Fig. 4.1 as the end limit of the MDNF or the lower limit of the MDFF.

4.2.2 Momentum Dominated Far Field Regime

Similar analysis was followed to define the upper limit of the momentum dominated far field region. The centerline trajectory relation derived by Wright (1977) for this region is given by:

$$\frac{y_c}{l_m} = C_2 \left(\frac{x}{l_m}\right)^{1/3}$$
(4.5)

The proportionality constant C_2 is again found to be a function of α . The logarithmic plot of the coefficient determined from the experimental data was given by the equation (Hodgson and Rajaratnam, 1988):

$$C_2 = 0.89 \alpha \frac{1/6}{2}$$
 (4.6)

The upper boundary of the MDFF could not be determined from Wright's trajectory plot and therefore a new basis was explored to define the limit. Based on the velocity field of the flow, the end of the MDFF was assumed to occur where the excess velocity in the jet above that of the ambient falls to about 1% of the initial velocity excess given as $(U_m - U) = 0.01 (U_0 - U)$. From the work of Rajaratnam and Gangadhariah (1980), $(U_m - U)/(U_0 - U) = 0.01$ occurs approximately when $\xi/\xi_0 = 10$ where ξ_0 is the length scale in the velocity correlation and defined by the axial distance of the deflected jet from the virtual origin to the end of the potential core. The length scale is a function of α and fitted by the logarithmic function given by:

$$\xi_{+} / d = 1.5 \alpha \ 0.43 \tag{4.7}$$

Pratte and Baines (1967) plotted the centerline penetration of the deflected jet against the distances along the ξ - axis and the data were described by a straight line with the equation:

$$\frac{y_c}{\alpha d} = 1.63 \left(\frac{\xi}{\alpha d}\right)^{1/3}$$
(4.8)

for all the data points beyond the potential core.

Let ξ_{\uparrow} and x_{\uparrow} be the axial distance and the downstream longitudinal distance of the deflected jet respectively at the end of the MDFF. Therefore, incorporating the above assumption into Equation 4.8 and combining with Equation 4.5 results in the following relation, for the end limit of the MDFF regime;

$$x_{\uparrow} = 5.47 \frac{\xi_{\uparrow}}{C_2^3}$$
 (4.9)

A plot of $\alpha x_{\dagger}/d$ against α is given in Figure 4.1. The figure indicates the three distinct regions delineated by the Equations 4.3 and 4.9 as the momentum dominated near field regime, momentum dominated far field regime and the passive plume regime which occurs beyond the upper boundary of the far field regime.

4.3 Experimental Results

4.3.1 Parameters Considered

Concentrations across the section of the jet were sampled on a minimum grid of 5 mm by 5 mm near the nozzle to a maximum grid of 10 mm by 10 mm on the wider section of the jet to ensure that the peak concentration point was sampled. The grid system resulted in a minimum of about 80 co-ordinate points near the jet nozzle and a maximum of about 200 co-ordinate points for the fully grown jet, of concentration measurements for one section. An average of 10 sections were investigated for each of the 5 jet discharges covering downstream distances as far as x = 4000 mm (or a maximum of $\alpha x/d = 5000$).

Two jet nozzles of diameters of 6.35 mm and 12.7 mm were used for the range of velocity ratios α of 2, 4, 6, 8 and 12. The ratio of the freestream flow depth (D) and the jet nozzle diameter (d) was maintained constant at 20 throughout the measurements, except for $\alpha = 6$ where cases with D / d = 10 and 5 were also investigated.

4.3..2 Presentation of the Results

The concentration data were corrected to eliminate the ambient fluorescence and tabulated in Appendix II. Concentration measurements in all the sections were taken perpendicular to the jet axis. A similar procedure was used by Fan (1967) while Wright (1977) and Hodgson (1991) considered vertical cross-sections (parallel to y-axis) for the free jet. The data for the different transverse ($\eta - \xi$ plane) levels are presented, with the lowest level fixed at an elevation of 1.5 mm above the bed at the axis of the sampling probe centerline while touching the bed. The schematic sketch given in Figure 4.2 defines the planes considered in the presentation of the results. The unit for the tracer concentration is parts per billion (ppb).

4.4 Analysis

4.4.1 Concentration Profiles

Transverse concentration profiles ($\eta - \xi$ plane) were plotted for all the sections investigated. Sample plots for the velocity ratio $\alpha = 4$ at locations in the near field (x = 5 mm) and the far field region (x = 100 mm and 285 mm) are given in Fig. 4.3 (a) - (c). Observation of the plots indicates no symmetry between the profiles near the bed and those on the upper half of the jet because of the boundary layer effect. The maximum concentration (C_m) occurs at a distance δ measured from the bed to the elevation of maximum concentration, then decreases to a constant value of the ambient fluorescence. It was also interesting to note the shift of the profiles above the level $z = \delta$ show a shift of their peaks towards the inside boundary layer velocity which allows greater jet penetration. The maximum concentration is not symmetrically located on the visual centerline of the jet trajectory as observed in axisymmetric jets but occurs generally on the outer half-section of the jet width.

Double peaked concentration profiles appeared for the sections investigated in the maximum deflection zone. The possible explanation could be as a result of the vortex formation which could only be further investigated by plotting the contour profiles. This has not been considered in the present study. Concentration profiles for the locations far downstream indicated complete mixing and little change occurs in the concentration gradient throughout the flow depth where the jet momentum has diminished to ambient flow conditions.

4.4.2 Minimum Dilution

Dilution of the tracer concentration at a section is explored by considering its characteristic minimum value which occurs at the location of maximum concentration in the jet. Minimum dilution is therefore given by the ratio of the initial concentration (C_0) at the jet nozzle to that of the characteristic maximum concentration (C_m) . Both the vertical and the transverse planes containing the characteristic value are considered for the similarity test and the growth of the jet half-width (where the instantaneous concentration is one half the maximum value, or $C = 1/2 C_m$) length scales. The main features of the various flow regimes would be identified by presenting the analysis based on the momentum dominated near field, momentum dominated far field and the passive plume regions.

The vertical $(z - \xi plane)$ concentration profiles were considered by plotting the maximum value of C/C_0 against the elevation z (mm) for all the sections and for each of the five jet discharges as given in Figures 4.4 - 4.8. The results clearly indicate the distributions assume the wall jet shape quickly and the profiles are not symmetrical along the maximum value as expected for the simple jet. This is because the jet growth is restricted near the bed. The plots also indicate that little dilution (less than 2 : 1) occurs near the jet nozzle where the jet penetrates the ambient flow with its undiminished momentum. Most of the tracer accumulated in the lower one-third of the flow depth.

Similarity of the concentration profiles was tested by plotting non-dimensional values of C/C_m against z/b_z (where b_z is the elevation measured from the bod where $C = 1/2 C_m$), for all the flow discharges. Plots for the different flow regimes are given for each of the five jet discharges in Figures 4.9 - 4.14. The plots indicate reasonably similar concentration distributions on the outer edge of the jet but

noticeable scatter exists near the bed due to the boundary layer effects. Considering the data points away from the wall (bed), the concentration distributions for the sections in the momentum dominated near field all collapse into a single curve with minimal scatter. The jet dominates the flow in this region and is least affected by the crossflow ambient, and therefore the mixing processes are similar to those in the simple axisymmetric jet. As similarity trend is also shown by the data in the momentum dominated far field. However, some scatter is displayed by the data points on the outside periphery of the jet. This might be due to the effects of non-uniform velocity distribution of the crossflow.

The distributions for the passive plume region show a lot of scatter over the whole depth of the flow. This indicates that the characteristics of the flow regime are different from those of the momentum dominated regimes. Some trend, however, exists for most of the data points neglecting those near the wall. The jet mixing in the region has grown to the free surface and consequently the dilution is constrained.

Concentration distributions in the transverse ($\eta - \xi$ plane) direction were investigated by plotting C/C₀ against the lateral distance η measured on both sides from the axis of the maximum concentration as the origin. The plots are given in Figures 4.15 - 4.19 for each of the five jet discharges. The profiles are fairly symmetrical and can be approximated to the Gaussian distribution, especially beyond the maximum deflection zone of the jet when the profiles become more uniform. Concentration profiles for the sections at downstream distance of about $\xi = 50$ d show minimum dilution of about 20:1 for each of the jet discharges. The spread of the jet growth to the inner wall occurred faster for the weaker jet discharge of $\alpha = 2$ because of its smaller penetration and susceptibility to deformation by the crossflow ambient compared to the relatively stronger jets of $\alpha = 4$, 6, 8 and 12. Similarity plots of the transverse concentration profiles were obtained by plotting C/C_m against η/b_* (where the jet half-width, b_* is the transverse distance measured from the axis of the maximum tracer concentration to the position where concentration is one-half the maximum value, or $C = 1/2 C_m$). The plots for the momentum dominated regime are presented in Figures 4.19 - 4.24 for each of the five jet discharges and the combined distributions given in Figure 4.25. The distributions indicate similarity for the momentum dominated flow regimes. Limited scatter occurs for data points sampled near the inner wall boundary, which is expected because of the uneven distribution of the freestream velocity near the walls (see cross-section velocity distributions, Figure 3.1).

Concentration distributions for the passive plume region were not included because the jet half-width, b_{*} could not be obtained due to a near uniform concentration profile across the jet flow in the region. This further supports the differences of the passive plume region from those dominated by the jet.

Next we study the decay of the maximum concentration (C_m) along the downstream distance x by considering the minimum dilution ratio (C_0 / C_m). Patrick (1967) expressed the dilution ratio empirically as a function of the curvilinear distance (ξ) of the deflected jet. Wright (1977) used dimensional considerations to define the dilution as proportional to the vertical distance (y_c) of the centerline of the jet. For practical purposes and the convenience of measuring the downstream distance x. Hodgson and Rajaratnam (1992) reduced the dilution expression given by Wright to be a function of $\alpha \times / d$. Similar consideration is adopted for the present analysis and the results compared to those of the free jet (Wright, 1977; Hodgson and Rajaratnam, 1992) for the momentum dominated regions.

The power law form of minimum dilution given by Patrick (1967) is reduced with the help of the logarithms into a simple correlation with the downstream distance x. The choice of using the downstream distance x is however limited in the field application where river flow goes through a meandering reach and therefore prior knowledge of the river reach is necessary to ensure sufficient length of the initial dilution zone is available before locating the outfall. In the present investigation plots of the minimum dilution (C_0 / C_m) against the downstream non-dimensional distance ($\alpha \times / d$) are given on the double-logarithmic scale for each of the jet discharges investigated in Figures 4.26 - 32 which cover the investigated range of $\alpha \times / d$ from 2.0 to 5000. The data indicate a general trend for the characteristic dilution and is fitted by two slopes of 1/2 and 2/3, respectively. For a free jet in a crossflow, according to Wright (1977), the two flow regimes represent the momentum dominated near field (MDNF) and the momentum dominated far field (MDFF) regions of a circular jet in crossflow. The present consideration of a circular wall jet in crossflow indicates similar slope trends and, therefore, similar flow regimes prevail. The effect of the wall (bed) on the level of dilution would be assessed by comparing the two forms of the jet. The most significant observation of the two slopes is the transition limit. Based on the criteria earlier defined for the limits of the MDNF, MDFF and the PPR regions given in Figure 4.1, the transition for the slopes occur within 2 orders of magnitude ($\alpha x/d < 100$) for the jet discharges with the velocity ratios $\alpha = 2$ and 4 and fairly close to the criteria used. However, the transitions for the stronger jet discharges of velocity ratios $\alpha = 6$, 8 and 12 are shifted more downstream and occur between the distance range of $100 < \alpha \times / d < 1000$. This results suggest stronger wall jet flows behave as though the near field regimes are longer than those of the corresponding free jets. The extended length of the near field regime may be due to the underestimated crossflow velocity near the bod which is less than the mean

value used. For all the discharges considered, the plots indicate that dilutions less than 2:1 are generally non-linear and occur in the vicinity of the outfall $(\alpha \times / d < 10)$.

The weakest jet, ($\alpha = 2$) shows a more superior dilution as compared to the other jet discharges and also higher than Wright's near field (1/2 slope). Dilutions of between 10:1 and 20:1, normally recommended within the vicinity of a jet diffuser is achieved by the jet discharge for $\alpha \times / d < 100$ and dilution of upto 100:1 occurs for $\alpha \times / d < 1000$. The model for Hodgson (1991) achieved a maximum dilution of 50:1 at a downstream distance of $\alpha \times / d = 900$, which extended only through the range for the initial dilution (or mixing) zone.

The jet discharge with velocity ratio $\alpha = 4$ had similar range of dilutions with those of $\alpha = 2$. Dilutions of 10 : 1 upto 20 : 1 were achieved within the downstream distances of 100 < $\alpha \times /d$ < 200. The initial dilution zone requirement of dilution of 50 : 1 was achieved by the jet discharge at a distance of about $\alpha \times /d = 700$. The other jet discharges ($\alpha = 6$, 8 and 12) indicated low dilutions ranging generally between 30 : 1 and 35 : 1 within downstream distances $\alpha \times /d < 1000$.

Effect of the flow depth ratios (D/d) on the dilution was also investigated by considering D/d = 10 and 5 for $\alpha = 6$. The results are given in Figures 4.28 (h) and (c). For D/d = 10, the results for the momentum dominated near field compares well with those of Wright's equation and extends to a downstream distance of about $\alpha x/d = 400$ for dilutions of approximately 20:1. The same jet discharge and that of D/d = 20 fitted similarly well for the near field data for a downstream distance of about $\alpha x/d = 300$ and dilutions of up about 20:1. Further document of the flow depth ratio to D/d = 5, resulted in significant effects on the dilution because of the surfacing of the jet. The dilutions presented graphically in

Figure 4.28 (c) assumed an almost constant value of about $C_0 / C_m = 15$ for downstream distances ($\alpha x / d$) above 100.

The combined results given in Figures 4.31 and 4.32 indicate the lower jet discharge of $\alpha = 2$ compares well with the results of Hodgson (1991) for the near field dilutions but more superior for downstream distances above $\alpha \times / d = 100$. This may be explained by the change of the flow regime from the momentum dominated to the passive plume condition which show higher dilutions. Hodgson's results compare well with the data for $\alpha = 4$ above $\alpha \times / d = 100$. The higher jet discharges of $\alpha = 6$, 8 and 12 maintained the momentum dominated near field characteristics similar to those of Wright's for distances up to three orders of magnitude ($\alpha \times / d = 1000$). The dilutions in the far field regime compare approximately well to those of Hodgson. Therefore, the results in the present investigation suggest that circular wall jet dilutions in the near field regime are represented well by the Wright's (1/2 slope) solution , while for the far field regime are predicted by the results of Hodgson (0.56 slope). However, the use of Wright's far field solution could also be applicable for the wall jet but the coefficient C4 used and hence the constant in the dilution equation will change and be higher than the given value of 0.223.

For convenience in estimating the dilutions in a practical situation, the results of the present study given in Figure 4.31 are described by a mean equation of the form:

$$C_o / C_m = 0.67 (\alpha x / d)^{0.63}$$
 (4.10)

The above equation describes fairly well the data points in the momentum dominated near field because all the jet discharges behave similar in the regime. Distinct scatter of the data is shown in the far field because dilutions are more dependent on the
strength of the jet discharge. Dilutions of upto 100 : 1 are represented by the correlation.

4.4.3 Length Scale Analysis

The concentration profiles for circular wall jet in crossflow have been shown to be similar both in the vertical, $z - \xi$ plane (depth-wise) and the transverse, $\eta - \xi$ planes. The following analysis considers the variation of the respective characteristic length scales along the curvilinear (ξ) distance. The jet nozzle diameter d is used as a length scale to make the results non-dimensional.

4.4.3.1 Vertical Scale, b_x

The vertical length scale b_z is measured from the bed (z = 0) to the point where the concentration is one-half of the maximum value at the section ($C = 1/2 C_m$). Figure 4.33 graphically shows the variation of b_z/d along the axial distance, ξ/d (measured along the trajectory of maximum concentrations) for the five jet discharges. The results indicate that the momentum dominated near field flow regime ($\xi/d < 30$) is fitted by a unifying length scale described by the following relation:

$$\frac{b_{\rm r}}{d} = 1.0 + 0.3 \,(\xi/d) \tag{4.11}$$

The variation of the scale in the momentum dominated far field appears to be a function of the flow velocity ratio α . The weakest growth is shown by the jet discharge of velocity ratio $\alpha = 2$, while for $\alpha = 4$ and 6, b_z grows at rates higher than that of $\alpha = 2$. Jet discharges of velocity ratios $\alpha = 8$ and 12, grow at rates higher than

those of the weaker jets. However, all the jet flows grow with similar slopes and are described by the following relations:

$$\frac{b_z}{d} = 3.0 + 0.02 (\xi/d) \quad (\alpha = 2) \quad (4.12)$$

$$\frac{b_{\pi}}{d} = 6.0 + 0.03 \,(\xi/d) \quad (\alpha = 4 \text{ and } 6) \qquad (4.13)$$

$$\frac{b_z}{d} = 10.0 + 0.03 (\xi/d) \quad (\alpha = 8 \text{ and } 12) \quad (4.14)$$

The dependence of α of the scale in the momentum dominated far field region is shown by plotting ξ/ξ_m against $b_z/\alpha d$ (where ξ_m is the axial distance where $x = \alpha d$) and given in Figures 4.34 (a) and (b). For velocity ratios α greater than 4, the plot indicate the dimensionless distance ξ_m/d is a linear function of α .

The variation of the vertical scale, b_z indicates that the wall jet grows at about 10 times faster in the momentum dominated near field compared to the momentum dominated far weld regime.

4.4.3.2 Transverse Scale, ba

Transverse characteristic length b_0 at a section is defined as the distance measured from the jet axis (defined by maximum tracer concentration) to the location where the instantaneous concentration is one - half the maximum value on either sides of the axis. The outer scale is conventionally considered positive and the inner acale negative. The transverse growth characteristics of the wall jet are derived graphically by plotting ξ/d against the absolute sum of the scale $(+b_0 + 1-b_0 1)/d$ and given in Figure 4.35. The results for all the five jet discharges fall into two distinct flow regimes. For the first region defined by the momentum dominated near field, the variation is described by the linear relation:

$$\frac{(+b_{+}+1-b_{+}1)}{d} = 0.53 \frac{\xi}{d}$$
(4.15)

For the second flow region known as the momentum dominated far field, the corresponding relation is:

$$\frac{(+b_{+}+l-b_{+}l)}{d} = 7.5 + 0.075 \frac{\xi}{d} \qquad (4.16)$$

Unique trends were observed when the transverse length scales for the growth on the outside (positive) and the inside (negative) regions of the deflected jet were considered separately. The results given in Figure 4.36 (a) and (b) indicate that the transition between the momentum dominated near field and far field regions for the outside scale does not occur gradually whereas for the inner scale, there is a smooth transition.

Comparing these results to those obtained for the vertical scale easily, we can conclude that for circular wall jets in crossflow, the transverse growth is about two times that of the corresponding growth in the vertical (depth - wise) direction. These results support the fact that the wall (bed) restricts the vertical growth of the jet but favors the horizontal transverse growth and it is an advantage for dilution in shallow rivers where normally the widths are several times more than the flow depth.



Pig. 4.1 Decemberry Limits for the Momentum Diminuted Regions



(b) Three-Dimensional Concentration Profiles of a Wall Jot

Fig. 4.2 Definition Skotches of a Circular Wall Jot in Creatilow



Pig. 43(a) Transverse Concentration Profiles







Pig. 43(c) Tranverse Concentration Prefiles



Fig. 4.4 Concentration Profiles along the Controllers of the Jet



Fig. 4.5 Concentration Profiles along the Controllers of the Jet



Pig. 4.6 Concentration Profiles along the Centraline of the Jet



Pig. 4.7 Concentration Profiles along the Controller of the Jet



Fig. 4.3 Concentration Profiles along the Centreline of the Jet







Pig. 4.9 (b) Non - Dimensional Profiles of Concentration Distribution in the Vertical Direction



Fig. 4.9 (c) Nen - Dimensional Profiles of Concentration Distribution in the Vertical Direction













Pig. 4.10 (d) Nen - Dismatenal Profiles of Concentration Distribution in the Vertical Direction





Pig. 4.11 (b) Non - Dimensional Profiles of Concentration Distribution in the Vertical Direction







Fig. 4.11 (d) Nen - Dimensional Profiles of Concentration Distribution in the Vertical Direction



Pig. 4.12 (a) Non - Dimensional Profiles of Concentration Distribution in the Vertical Direction



Pig. 4.12 (b) Nen - Dimensional Profiles of Concentration Distribution in the Vertical Direction



Pig. 4.12 (c) Nen - Dimensional Profiles of Concentration Distribution in the Vertical Direction



Fig. 4.12 (d) Nen - Dimensional Profiles of the Concentration Distribution in the Vertical Direction



Fig. 4.13 (a) Nen - Dimensional Profiles of Concentration Distribution in the Vertical Direction







Fig. 4.13 (c) Non - Dimensional Profiles of Concentration Distribution in the Vertical Direction



Pig. 4.13 (d) Non - Dimensional Profiles of Concentration Distribution in the Vertical Direction



Pig. 4.14 (a) Similarity Profiles of Concentration Distribution in the Vertical Direction



Pig. 4.14 (b) Similarity Profiles of Concentration Distribution in the Vertical Direction



Pig. 4.14 (c) Statisticity Profiles of Concentration Distribution in the Vertical Direction


Fig. 4.14(d) Similarity Profiles of Concentration Distribution in the Vertical Direction



Fig. 4.15 Transverse Concentration Profiles Perpendicular to the Jut Axis









P.S. 4.17 Theoremse Concentration Profiles Perpendicular to the Jet Auts



Pg. 4.15 Transvers Concentration Profiles Perpendicular to the jat Auto









Pig. 4.20 Nen - Dimensional Profiles of Concentration Distribution in the Transverse Direction



Pis. 4.21 Nen - Dimensional Profiles of Concentration Distribution in the Transvere Direction



Fig. 4.22 Nen - Dimensional Profiles of Concentration Distribution in the Transverse Direction



Pig. 4.23 Nen - Dimensional Profiles of Concentration Distribution in the Transverse Direction



Pig. 4.24 Nen - Dimensional Profiles of Concentration Distribution in the Transverse Direction



Fig. 4.25 Similarity Profiles of Concentration Distribution in the Transverse Direction

















Pip. 4.39 Characteristic Minimum Dilution of the Deficeted Jet

























Fig. 4.35 Generalized Scales for the Transverse Concentration Profiles of the Deflected Jet



Fig. 4.36 Generalized Pesitive and Negative Transverse Scales

5.0 CONCLUSIONS AND RECOMMENDATIONS

5.1 Conclusions

The objective of the present study was to investigate the dilution characteristics of circular wall jets in crossflows. These include the similarity analysis of the concentration profiles both in the vertical and transverse directions, minimum dilution and the growth of the jet half - width. The maximum concentrations (C_m) was used as a characteristic scale parameter. It is of practical importance to have knowledge of the maximum value of tracer concentration at any section of the jet because it is useful in assessing the extend of pollutant dilutions released into the environment.

The conclusions presented below are applicable for the region where the jet generated turbulence dominates the ambient flow conditions. The characteristics in the region where the jet strength has decayed to the ambient flow turbulence and referred to in the text as a passive plume condition, was not considered in detail but could be investigated under a separate study.

5.1.1 Interpretation of the Experimental Results

The similarity profiles of concentration distributions both in the vertical $(z - \xi \text{ plane})$ and the transverse $(\eta - \xi \text{ plane})$ are presented in Figures 4.14 and 4.25. The plots indicate that normalized concentration distributions for a circular wall jet in crossflow are preserved from section to section. This behavior is assumed to hold true under the boundary conditions of zero jet velocity at the wall where some scatter on the dilution data were observed, and the upper boundary of the free mixing region is marked by the jet decay to the ambient flow velocity (U). The dimensionless plots of

both the momentum dominated near field and the far field regimes showed no observable distinction because the jet momentum dominates the mixing and hence controls the dilution in the regions. The concentration distributions in the passive plume region were not similar to those of the momentum dominated region.

The wall effect was shown by the scatter of dilution data for all the jet discharges. The generalized distributions in the vertical direction indicated that the boundary layer thickness δ could be derived approximately by the elevation z = 0.5 bz. Transverse concentration distributions were approximately symmetrical along the jet axis and could be described by the Gaussian distribution.

The minimum dilution (C_0 / C_m) plots along the downstream distance $\alpha \ge 1/d$ indicated a general trend of increase in tracer dilution shown by the data slope transition from 1/2 to 2/3. The trends imply that the dilution along the axis of the jet are governed by the inter - dependent cross - sections with similar rates, or constant slope. Beyond the momentum dominated region, the stream turbulence influences mixing. The passive plume flow regime is shown by the data with steeper slope than the 2/3 of the momentum dominated far field. This feature was clearly distinct for the weaker jet with the velocity ratio $\alpha = 2$ beyond the downstream distance $\alpha \ge 100$.

The results have shown that dilutions of 50:1 were achieved by all the jet discharges within the momentum dominated far field regime. Dilutions in the range of 3:1 to 20:1 were attained in the momentum dominated near field depending on the strength of the jet discharge. These results indicate that the momentum dominated far field is a very important region in achieving significant dilutions within the initial dilution zone in river discharges.

The dilution data were correlated by a unifying equation which describes the whole region dominated by the jet. The relation (Equation 4.10) is sufficient for practical purposes in estimating tracer dilution in rivers.

The length scale analysis for both the vertical and the transverse directions indicated that the jet grows faster in the momentum dominated near field than in the far field regime. The result is attributed to the dominance of the jet strength in the MDNF; while in the MDFF, the growth behavior is similar to that of a jet in a co-flowing stream. It was also found that the jet growth was about twice in the transverse direction compared to that in the vertical direction.

The transition between the momentum dominated near and the far fields occurred approximately at $x/\alpha d = 1$ for $\alpha = 2$ and 4. The stronger jets with velocity ratios greater than 4 had their transitions at distances $(x/\alpha d) > 2$.

The effect of the freestream flow depth was also considered and found that significant reduction in the tracer dilution occurred for depths less than 10 times the jet nozzle diameter. This result indicates the restriction which occurs in the vertical mixing in shallow rivers.

5.1.2 Comparison with Previous Studies

Dilution of circular wall jet in crossflows has not been studied previously. The present results therefore, are compared with the investigations of Wright (1977) and Hodgson and Rajaratnam (1992) on circular free jets in crossflows.

The results plotted in Figure 4.32 show that the dilution characteristics of the wall jet discharges are fitted well by the results of Wright (1977) for the MDNF and

those of Hodgson and Rajaratnam (1992) approximates the MDFF. When each of the jet discharges was considered separately, the dilution of the wall jet in the MDFF were fitted well by a 2/3 - slope of the power law equation but with a higher constant coefficient than that given for the Wright's equation.

The transition from the MDNF to the MDFF, based on the trajectory plots of Wright, occurs approximately where $l_m = x$, and therefore reduces $\alpha \times / d$ to approximately α^2 . For α varying between 2 and 12, corresponds to α^2 in the range of 4 to 144. This criteria was only met by the wall jet discharges of $\alpha = 2$ and 4 For α greater than 4, the transition values were greater than the α^2 values. Higher values may have been caused by the smaller resistance of low velocity of the crossflow near the bed.

5.2 Recommendations

The present investigation considered only the dilution characteristics of circular wall jet in crossflow. To complete the study of the jet requires an integration of all the other related hydraulic aspects. Some of the important characteristics recommended for further study include:

I. detailed study of the concentration contours across the section of the jet. The study would identify the differences in the vortex structure of both the wall and free jets in crossflow.

II. measurements of the bed shear as an indicator of the extent of the jet erosion of the river bed.

III. detailed photographic study of the jet growth to compare with the known conditions of the free jet in crossflow.

IV. dilution study of buoyant jet discharges for comparison with the non-buoyant cases.

V. to carry out the dilution study under the prototype river conditions similar to the investigations of Hodgson and Rajaratnam (1992).

VI. to study the turbulence characteristics of the wall jet in crossflow.

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APPENDIX I

Photographic Representation of the Circular Wall Jet Diffusion in Crossflow



Plate I (a) Experiment No. A200: Velocity I

Velocity Ratio $(U_o/U) = 2$ D = 127 mm, d = 6.36 mm



Plate I (b) Experiment No. A200:

Velocity Radio $(U_e/U) = 4$ D = 127 mm, d = 6.35 mm



Plate I (c) Experiment No. A300:

Velocity Radio $(U_0/U) = 6$ D = 127 mm, d = 6.35 mm

Plate I (d) Experiment No. B100:

Velocity Ratio $(U_0/U) = 6$ D = 254 mm, d = 12.7 mm





Plate I (e) Experiment No. A400:

Velocity Radio $(U_0/U) = 8$ D = 127 mm, d = 6.36 mm



Plate I (f) Experiment No. A500:

Velocity Ratio $(U_0/U) = 12$ D = 127 mm, d = 6.36 mm

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APPENDIX II

Experimental Dilution Data

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APPENDIX III

Centerline Wall Jet Trajectories for the Momentum Dominated Flow Regimes

(Figures(a)-(u))

Appendix III

Diffusion of Circular Wall Jet in Crossflows

General

The analysis on this section is based on the photographic trajectory data from an experimental study by M. Diebel and N. Rajaratnam which was carried out in the T. Blench hydraulics laboratory, University of Alberta. The results predict both the path and the growth rate of the deflected wall jet and were used to approximate the location of the jet during the dilution investigations.

Diebel's experiments were conducted in the same rectangular channel, 0.91m wide, 0.76 m deep and 36.6 m long, used in the present study. Well-designed nozzles of diameters equal to 6.35 mm and 19.05 mm, located at a distance of about 21 m from the flume entrance were used to generate the wall jet discharges from one side of the flume. Diluted solution of cochineal food color dye used was introduced into the piping system near the jet nozzle by gravity to produce a colored jet. The specific gravity of the dye was 1.020 at a temperature of 16.5 °C and with significant dilution, its density was assumed to approach the freestream water density.

Photographs of the coloured jets with exposure time of a 1/2 to 1 accord were taken from above the flume to provide the plan view of the deflected wall jet. Measurements of the jet centerline and its total width B at different locations along the ξ - axis were carried out using a 2 cm x 2 cm grid painted on the bed of the flume at the study section. The mid point between the inner and the outer jet penetration boundaries at each location was taken as the jet centerline. Photographs taken for the side elevation were used to obtain the vertical thickness h of the deflected jet at locations along the axial distance.

Growth of the Deflected Wall Jet

The deflected jet centerline in the work of Diebel was defined by the x and y coordinates where x represents the longitudinal distance along the stream and y represents the perpendicular distance to the stream with the origin at the jet nozzle. The axial distance ξ was measured from the nozzle along the centerline of the deflected jet for which the y co-ordinate was written as y_c.

The analysis of the jet trajectory showed no sensitivity to the magnitude of D/d, the ratio of freestream depth to the jet nozzle diameter which was generally in the range of 20-37, in the lateral growth. The present study on deeply submerged jets therefore, would show minimal dependence on the parameter. However the jet growth in the vertical (depth wise) direction would be influenced by the surface effects for relatively low values of D/d.

The Present Investigation

The objectives of the study were to define the flow regimes of the deflected circular wall jet using the photographic data of Diebel and to compare the trajectory relations with those established by Wright (1977). Diebel's data give the co-ordinates of the jet centerline.

The investigation of the flow regimes were carried out by plotting the dimensionless centerline co - ordinates, x / l_m against y_c / l_m and then superimposing the known trajectory solutions for the momentum dominated near and far fields. Based on the trajectory analysis of Wright (1977), the path of a free deflected circular jet in crossflow was fitted by a 1/2 - slope and 1/3 - slope for the MDNF and the MDFF respectively. Diebel's wall jet data are presented graphically in Figures III (a) - (s) for the two jet nozzle diameters used. The plots indicate that most of the data fall on the far field flow regime. The transition point was found to vary in the range of $y_c / l_m = 1.0$ to 2.0 and approximated the value obtained from the work of Wright (1977) of $y_c / l_m = 1.5$.

The dependence of the proportionality coefficients C_1 and C_2 on the velocity ratio α was also analyzed based on the two slopes for the MDNF and the MDFF respectively. The variation of each of these coefficients against α is presented in Figures III (t) and (u). The coefficients are fitted by the equations:

$$C_1 = 1.11 \alpha^{0.29}$$
 (III-1)

$$C_2 = 0.98 \alpha^{0.28}$$
 (III-2)

The results indicate that both these coefficients are not constants but are approximately linear functions of α . The increase of C₁ tends to an asymptotic value of 3, while C₂ approaches a constant value of about 2.5 for α above 20. The results given by the work of Wright showed similar trends but both the coefficients asymptotically approached a constant value of 2 for very high velocity ratios of about $\alpha = 100$. Hodgson (1991) carried out similar analysis for river - like shallow crossflows for α less than 15. The present results from Diebel's data closely approximated Hodgson's results for α less than 10 but about 30 % greater than Wright's results. The comparisons in the momentum dominated far field regime indicates that the wall jet coefficient C_2 tends to increase faster than values given by the work of Hodgson. These results indicate that the wall jet penetration is greater than that of the corresponding free deflected jet. Low velocity of freestream flow near the bed might be responsible for this difference.







































