

Dynamics of spontaneous initial spreading and spreading of a hydrodynamically driven droplet  
under the influence of surrounding pressure

by

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## Abstract

We investigate experimentally the early time dynamics of spontaneous spreading of silicone oils with various viscosities at elevated surrounding pressures. The surrounding medium pressure is increased in a monotone fashion starting from atmospheric pressure to a maximum value of 30 megapascal (MPa) above atmospheric conditions in 10 MPa increments. We conduct our analysis using four grades of silicone oils namely: D10, N35, S60, and D500 and study their spreading behavior on Polytetrafluoroethylene (PTFE) substrates. Once contact with the substrate is established, the three-phase contact lines which form at the intersection of the solid, liquid, and gas phases advance rapidly to reduce the droplet's surface energy. Experimental observations which are conducted using a high-speed camera and a drop shape analyzer reveal three distinct regimes of spreading: an initial spreading regime, a short but intermediate transition regime followed by a late-stage viscous regime. The results from these experiments are analyzed in terms of the temporal evolution of the contact radius, contact line velocities, and droplet contact angle. In addition, we analyze the pressure effects on the onset of the transition regime and the empirically obtained spreading exponent. Our observations show that the early time spreading dynamics conform themselves to a power law where the empirically obtained exponent concomitant to spreading is a strong function of pressure. More specifically, a rise in pressure decreases the spreading exponent in the inertial regime. Additionally, an increase of pressure reduces the onset time of the transition regime as well as spreading velocities. This is due to increased viscous dissipation at the contact line which enables a quick conversion to the transition regime. We also propose a theoretical model for investigating the spreading behavior of hydrodynamically driven droplets on a solid substrate under various surrounding pressures. By combining the extended Overall Energy Balance (OEB) approach and the Lucas empirical model for estimating drop

viscosity at elevated pressures, we predict the advancement of a liquid droplet on a solid substrate as it undergoes constant mass flux addition while maintaining a spherical cap. For validation, we conduct an experimental investigation on a spreading droplet while considering the addition of mass over time in atmospheric pressure and surrounding gauge pressures of 10 and 20 MPa. We analyze the results of the pressure effects on the spreading droplet whose primary driving force is hydrodynamic in terms of the advancing contact angle, increase of volume over time, and spreading radius of water on four different substrates namely: Polytetrafluoroethylene (PTFE), Aluminium (Al), Copper (Cu), and Poly (methyl methacrylate) (PMMA). The results show that an increase in pressure during the advancing stage lowers the spreading radius and simultaneously increases the contact angle.

## **Preface**

All work summarized in this thesis is an original work by Sumaiya Farzana under the supervision of Dr. Prashant R Waghmare. Sumaiya Farzana was responsible for the experimental design, conducting experiments, data analysis, data interpretation, developing the theoretical framework and analysis, and writing the manuscripts. Dr. Prashant R Waghmare supervised all the works presented by Sumaiya Farzana and was involved with conceptualization and editing the manuscripts. A version of Chapter 2 of this thesis has been published to peer-reviewed journal: Journal of Experimental Thermal and Fluid science, as “Sumaiya Farzana, Ryan Baily, Prashant R. Waghmare, Study of Early Time Dynamics of Drop Spreading in Different Surrounding Pressure”. For the study of Chapter 2, Ryan Baily was involved with the necessary revisions of the manuscript and was involved in providing assistance in the experimental setup. Chapter 3 of this thesis has been submitted to peer-reviewed journal: Physics of Fluids, for publication as “Sumaiya Farzana, Prashant R. Waghmare, Dynamics of spreading of a hydrodynamically driven droplet under the influence of surrounding pressure”. Dr. Aleksey Baldygin was involved with the development of the dosing system (Syringe pump, 100DM, ISCO) used in conducting the experiments of Chapter 3 and was also involved with the necessary revisions of the manuscript.

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## 1. Introduction

The dynamics of droplet spreading on a solid surface has been studied extensively to characterize the wettability of a surface. Wetting and spreading play a significant role not only in oil recovery but also in the process of drainage water from highways, the efficiency of applying pesticides on plants, cooling of reactors, etc [1]–[3]. Whether a liquid will wet a surface or not is determined by the surface forces such as van der Waals and electrostatic forces [4]. These are important for determining the way fluids spread on solid surfaces and are also responsible for establishing the equilibrium thickness of wetting films.

For short ranges, wetting and thus the interaction between a fluid and a corresponding substrate plays a dynamically significant role, which impacts a wide variety of technological breakthroughs namely microfluidics, inkjet printing and nano printing [5]. These applications are largely controlled by interfacial forces being exerted at very small length scales usually which are in the order of a few nm such as the one for van der Waals force [4]. At present, these tiny length scales are being investigated experimentally using various approaches that permit a new look into physics previously unseen. Wetting phenomena involves the forward motion of the contact line as the droplet spreads and is primarily governed by two parameters which are surface roughness and the chemical composition of the substrate. The former causes friction or adhesive interactions between the substrate and the liquid.

In compliance with the hydrodynamic concept and molecular kinetic theory, the drag force associated with the adhesion interaction mentioned above serve to reduce the kinetic energy of the spreading droplet [6]. On a theoretical level, the drag can be thought of to be made of numerous energy barriers that hinders the contact line from advancing further. At these barricades, kinetic

energy is exhausted and when all the kinetic energy of the droplet is depleted, the contact line halts. In this state, the droplet is said to be pinned at the advancing interface. The only way to get unpinned from this position would be an influx of adequate momentum and kinetic energy and once all the energy barriers are prevailed over, the droplet will achieve a stable equilibrium state. [6].

Wetting, in general, can be subdivided into two types: spontaneous wetting which occurs without any external input to the system and forced spreading that involves a driving force. Considering the first type, spontaneous spreading during its later stages has been extensively researched and is now considered to be well understood as discussed in detail from the publications of [7], [8]. However, many aspects of the early time dynamics associated with full and particularly partial wetting has garnered little attention and is largely unexplored. The primary challenge that has plagued many researchers in the past was the limited frame rates of experimental imaging systems. With the advent of high-speed imaging and advanced image processing software packages, we are now able to look deeper into the nuanced dynamics during the first few milliseconds of wetting. In recent years, researchers have identified the presence of two distinct regimes in characterizing early spontaneous wetting. The first is an inertia dominated regime or ‘inviscid’ regime where the primary resistive force to capillary driven spreading is the drop inertia [9], [10]. We use the term inviscid to imply that the effect of drop viscosity on the initial dynamics is negligible. The second regime is a viscous dominated late regime that soon terminates in the drop achieving a thermodynamic equilibrium position [9], [10]. In the inviscid regime, the spreading dynamics follows a power law that scales in time. Although the duration of this regime has been investigated with respect to drop size [11] and surface chemistry [10], the impact of outer medium pressure on the spreading exponent and the interplay between capillary, inertia and viscous forces at high

pressure remains unanswered. This poses a rich experimental research challenge that has been systematically delved into in the current study.

Forced spreading on the other hand is more commonplace in industrial applications. Here, the primary driving force is usually hydrodynamic in nature although in some applications, the initial kinetic energy of a falling droplet can also serve as the unbalanced driving force for spreading [12]. The term hydrodynamic signifies the continual delivery of mass to the droplet in the form of mass influx. The spreading dynamics associated with this phenomena has been modeled previously by a few researchers [13]–[15]. The approach involves the use of the Overall Energy Balance (OEB) equation to predict the time varying radius of the advancing droplet as it evolves on a solid surface. However, the model does not take into account the impact of outer medium pressure on the long-term spreading behavior has not been investigated theoretically and experimentally and thus the study of this phenomena at elevated surrounding pressures requires careful consideration. One such application that bears similarity to this problem is the injection of pollutant CO<sub>2</sub> into deep saline aquifers in an effort to combat global climate change [16]. These large-scale geological sequestration projects result in post-injection dynamics that are complex and somewhat unknown. At reservoir depths, the outside temperatures and pressures are very high and as a result the CO<sub>2</sub> that is being pumped exists in its supercritical state and thus behaves like a liquid. The interfacial tension (IFT) and the contact angle (CA) between the sc-CO<sub>2</sub> and the porous rock surface impacts the trapping effectiveness and the storage security of the trapped injectate inside a brine filled aquifer. Further, the wettability of the rock surface changes with pressure. Because monitoring this activity visually at the reservoir level is very challenging, the need for simple yet robust analytical models that are experimentally validated to predict the long-term spreading mechanics of sequestered injectate becomes quite imperative. Although, the quantitative terms in a simplified

model might differ somewhat from the realistic scenario, the objective instead would be to gather insights into the underlying physics of the flow so that up scaled models can be built as a result.

In other words, experimental insights into the early time dynamics using high speed imaging coupled with a theoretical and experimental analysis on the long-term spreading behavior of advancing droplets at elevated pressures have been studied in this thesis. Pressure plays a major role not only in the initial dynamics of spontaneous droplets but also on the spreading phenomenon of droplets whose prime motive force is hydrodynamic in nature.

## **1.1 Aim of the present work**

Droplet spreading on a solid substrate has numerous applications. In high-pressure engineering such as refining, extraction, refrigeration, etc., it is important to understand the wetting behavior under the influence of surrounding pressure [17]. The droplet starts to spread after coming into contact with the solid surface and continues spreading until it reaches its equilibrium state. The later stages of spreading have been investigated extensively by many researchers but the different aspects of the early stages of wetting need to be investigated further. Also, to have a better understanding on the spreading behavior and wettability of the substrate, it is important to study the hydrodynamic spreading behavior by considering the advancing contact angle and advancement of drop spreading by adding volume at a constant rate. Considering all these aspects we have presented a detailed investigation of the role of pressure on the early stage of drop spreading and on the spreading of hydro-dynamically driven droplets under similar conditions from a theoretical and experimental standpoint. In summary, the aim of the present work is as follows:

- To characterize the influence of pressure on the early time spreading behaviors of sessile droplets
- To analyze the pressure dependence on the transition time & transition radius to intermediate and late-stage regimes.
- To establish a power law relationship of the spreading radius as a function of the empirically obtained spreading exponent and its consequent dependence on pressure.
- Using the extended OEB equation to develop a new model to predict hydrodynamically driven spreading in different surrounding pressures
- Perform experiments on liquid advancing to validate the effectiveness of the new approach using various substrates at different surrounding pressures.
- Conduct a parametric study investigating pressure effects of various non-dimensional numbers pertinent to spreading

The background of the investigations and their applications are discussed in the later sections of this chapter. Firstly, the idea about early time spreading, investigation procedure, the importance of investigating the early time spreading considering different pressure conditions, and the applications of this study are presented in detail. Afterwards, a theoretical and experimental study of hydrodynamically driven advancing is then discussed. The background investigation, theoretical model and the importance and applications of this study are also presented.

## **1.2 Influence of pressure on early time dynamics of drop spreading**

### **1.2.1 Literature review**

Early time spreading dynamics involve the first milliseconds of spreading after a droplet establishes contact on a solid surface. It is important to understand the whole spreading phenomenon. The later stage of spreading which is close to equilibrium has been studied

extensively [7], [8]. In [18]–[20], early stage of drop spreading has been investigated on a partially wettable surface and soft surface and under liquid medium respectively. Effects of pressure, temperature, etc on initial spreading dynamics yet need to be investigated. In our first investigation, we have conducted our analysis considering the effect of pressure on the early stage of spreading. The gradient of curvature and corresponding Laplace pressure gets balanced with resistive momentum which controls the spreading momentum [21]. Two temporal regime- inertial and viscous regime is considered in the entire spreading process. The later stage of spreading momentum is balanced by viscous friction. The visco-capillary balance near the moving triple line defines the shape of the droplet. On the other hand, the bulk droplet resistive inertia force at the early stage of spreading gets balanced with capillary force.

The total spreading procedure is dominated by initial inertia dominated regime and viscosity dominated regime [18]. For low viscosity liquid, early time spreading is dominated by inertia while at the later stage of spreading close to equilibrium is dominated by viscous dissipation. In the inertial regime of low viscosity liquid, the contact radius  $r$  follows a scaling,  $r \sim (\sigma_{DA} R / \rho_D)^{1/4}$  where  $\sigma_{DA}$  is the surface tension,  $R$  is the radius of the droplet and  $\rho_D$  is the density of the droplet. In the later stage of spreading, the drop spreads following Tanner's law  $r \sim (\sigma_{DA} R^9 / \mu_D)^{1/10} t^{1/10}$  (here  $\mu_D$  is the dynamic viscosity of the droplet) [8]. For high viscosity liquid, the whole spreading is viscosity dominated and follows a power law  $r \sim t^\alpha$ . Here  $\alpha$  is defined as  $\alpha = d \log r / d \log t$ . Initial spreading dynamics is similar to drop coalescing with a wet film for fully wetting substrate and spreading follows power law that depends on the type of force (inertia or viscous force) resisting drop deformation [10]. This scenario is different for partially wetting substrate where the spreading follows a power law  $r \sim t^\alpha$  with nonuniversal exponent  $\alpha$  [20].

In this project, we have studied the early time spreading dynamics of different grades of silicone oil on partially wetting PTFE substrate where the main dominated force is the inertial force. In the early stage of spreading viscous forces are negligible. The balance between the capillary force and the inertial force leads the drop to spread at an early stage. The relation can be written as follows [22]:

$$\sigma R \sim \frac{d}{dt}(\rho l R^2 u_0) \quad (1)$$

Here,  $l$  is the filling length,  $\sigma$  is the surface tension and  $u_0$  is the spreading velocity where  $u_0 = l/t$ . From the relationship between the filling length and filling, time will be [22]:

$$l \sim t \sqrt{\sigma / \rho R} \quad (2)$$

Considering  $l \sim R$ , we obtain the time scale which characterizes the early regime as:

$$t \sim l / u_0 \sim \sqrt{\rho R^3 / \sigma} \quad (3)$$

This time scale indicates the “initial” time where viscous force is negligible. Using this characteristic scale, we nondimensionalize our experimental time. As inertia plays an important role as a resistive force during the early time of spreading and spreading law takes the form of  $r/R = C(t/\tau_C)^\alpha$  where  $\tau_C = (\rho R^3 / \sigma)^{1/2}$  is the inertial timescale.

In this project, we have considered the influence of pressure on the spreading radius, contact angle as well as spreading exponent  $\alpha$ . We have discussed the regimes of spreading and the forces which dominate the spreading. In addition, we have calculated the transition time,  $\tau_t$ , and transition radius,  $r_t$ , and the influence of viscosity and pressure on it. This transition time,  $\tau_t$  and transition radius,  $r_t$  defines the regime change from inertial to the viscous stage. Finally, contact line velocity and contact angle in the early stage of spreading are analyzed.



### 1.2.2 Applications

Drop spreading on a smooth and dry surface has a wide range of applications [4], [23]–[26]. In icing of aircraft wings, spray painting, fuel injection, surface cooling, and so forth, the result of the spreading procedure defines its effectiveness [27][10][19]. The efficiency of this kind of operation depends on the size of the droplet, spreading behavior, and the interaction between the droplet and the surface. The impact behavior of a droplet on the cylinder and piston surface associates with the formation of combustion chamber deposits (CCD) in internal combustion engines [27]. The cooling of a heated slab depends on the rate and extent of spreading of the water droplet hitting the heated slab [27]. The spreading of a droplet depends on the nature and roughness of the substrate, liquid density, viscosity, and surface tension. Temperature, surrounding air pressure, and wettability have a significant effect on the spreading behavior as well. In a reservoir system, the efficiency of trapping  $CO_2$  depends on the wettability that changes with pressure. To have a better understanding of these, it is important to investigate the spreading of a droplet. More specifically the initial stage of spreading at different surrounding pressure need more attention. The knowledge of this study is useful in the spreading mechanics of supercritical  $CO_2$  in deep porous reservoirs for curbing atmospheric emissions [16][28]. The study of the early time dynamics reveals the time droplet when the droplet switches to the viscous dominated spreading regime from the inertial dominated spreading regime. If the drop transitions to the viscous dominated regime faster, further spreading will be curtailed. Studying the influence of pressure on this transition time will give valuable insight as to how fast the drop will spread and the extent to which the transition time and radius is pressure dependent.

### 1.3 Influence of surrounding pressure on the dynamics of spreading of a hydrodynamically driven droplet

#### 1.3.1 Literature review

Understanding the spreading of a droplet is important to know about the wetting characteristics of a liquid droplet on a solid substrate. The spreading phenomenon involves a three-phase contact line. The angle measured at the three-phase contact line is called the equilibrium contact angle and is also widely termed as “Young’s angle” [6]. Young’s contact angle,  $\theta$ , of a liquid drop on a solid substrate can be presented as:

$$\sigma_{lv} \cos \theta = \sigma_{sv} - \sigma_{sl} \quad (4)$$

where  $\sigma_{lv}$  is the liquid-vapor surface tension,  $\sigma_{sv}$  is the solid-vapor surface tension, and  $\sigma_{sl}$  is the solid-liquid surface tension.

According to Young’s equation, the solid substrate is assumed to be smooth, homogeneous, and rigid. It should also be chemically and physically inert concerning the liquids to be employed [29]. But in a real scenario, solid surface cannot be found in an ideal condition and because of that liquid drop on it behave differently, and it cannot be determined by Young’s equation. Though in ideal conditions, a unique contact angle is expected for a given system according to Young’s equation, but in a real scenario, a range of contact angles is observed. The upper limit of the range (or the angle measured during the advancement of a contact line or the wetting of the surface) is referred to as the advancing contact angle  $\theta_a$ , and the lower limit (or the angle measured during the de-wetting of the surface) is referred as the receding contact angle  $\theta_r$  [29]. The difference between  $\theta_a$  and  $\theta_r$  is known as contact angle hysteresis  $\theta_{hyst}$  which is presented as follows:

$$\theta_{hyst} = \theta_a - \theta_r \quad (5)$$

In [30], it is stated that contact angle on a rough surfaces cannot be considered as Young's contact angle. In [31]–[33], the authors gives this same argument for the heterogeneous and porous surfaces. The concept of metastable state was introduced in 1946 by Derjaguin and developed by other researchers [29]. In an ideal system, the free energy of the system is global minimum and only one contact angle can be obtained according to Young's equation which represents the stable equilibrium state. According to metastable states, the free energy of the system is a local minimum and it exists on heterogeneous and rough surfaces. The imperfections of the solid surface, mechanically more than one stable contact angle can be observed. So, contact angle hysteresis can be understood in respect of metastable states [30]. Among those studies, in [34] the authors discussed contact angle hysteresis more in the sense of a true hysteresis, but, not in terms of metastable states. The advancing and receding contact angle values to decrease with the time of contact between solid and liquid [34]. Detail explanation on measuring advancing and receding contact angles was first provided in [29]. During the advancement of drop volume in advancing contact angle measurement, the drop spreads gradually. This hydrodynamic spreading of a drop depends on several factors which were discussed by some other researchers in their research studies [13], [14]. The spreading of a droplet can be investigated by fluid mechanics approach or through a surface physics approach [13]. In fluid mechanics approach, the main difficulty is to model the slip at a three-phase contact line. Also, determining the relationship between the dynamic contact angle and the velocity of the advancing contact radius is difficult [35]. According to the surface physics approach, the spreading process is dependent largely on the interfacial tensions and contact angles. But various chemical, physical and quantum mechanical parameters

may or may not be possible to determine [36]. The combination of these two approaches is more accepted as they depend on empiricism to quantify the solution.

Many researchers have studied the relationship between dynamic contact angle and the advancing speed of the three-phase line. By relating the dynamic contact angle to the capillary number plus a 'shift factor' simple and semi-empirical equations were developed [37]. The first general theoretical model on the dynamic spreading of droplets was proposed in [15] which is called the overall energy balance model (OEB). In [13][12], the authors adopted the approach of [15] to model the spontaneous and low-speed impact drop spreading. The initial kinetic energy of a fixed mass droplet was considered as a primary driving force. But for industrial coating processes, polymer processing technologies, and welding, drop spreading by adding liquid to the system is essential. In these cases, the primary driving force is considered to be hydrodynamic in nature. In [14], a model to study drop spreading driven by hydrodynamic forces was successfully developed by extending the OEB model where adding mass and energy to the system was considered. The equation is a non-linear first-order ordinary differential equation. Here, the drop is considered to be spherical and having a constant dynamic contact angle.

In this project, experiments on spreading of droplet considering the addition of mass is observed in different surrounding pressures ( $P_{\text{atm}}$ , 10 MPa and 20 MPa). Experiments were performed on four different substrates - PTFE (Polytetrafluoroethylene), 6061-grade Aluminium, Copper, UV-Resistant Acrylic Sheet (PMMA) substrates. To explain the hydrodynamic spreading phenomenon, we used the OEB model [14] and combine it with the equation of viscosity proposed in [38]. The OEB approach in [14], describes the spreading of the drop with the addition of mass until it attains the equilibrium. In the system of drop spreading, the energy entered is balanced with the change in internal energy of the system, increase or decrease in surface and gravitational

potential energies, boundary movement work, and the viscous dissipation work. Thus, the OEB equation can be presented as follows:

$$\frac{dE_{in}}{dt} = \frac{d}{dt} (E_{sys} + E_s + E_g) + \frac{d}{dt} (W_v + W_b) \quad (6)$$

where  $E_{in}$ ,  $E_{sys}$ ,  $E_g$ ,  $E_s$ ,  $W_v$ ,  $W_b$  are the energy entered in the system, internal energy of the system, gravitational potential energy, surface energy, viscous dissipation work, boundary movement work, respectively. The final governing equation was obtained by relating the speed of the advancing contact radius to the material properties and the dynamic contact angle. The equation will be as follows by considering  $\theta_d$  constant and  $\frac{dm}{dt} = \dot{m}$ :

$$6\pi \frac{R\mu}{\theta_d} \ln(\varepsilon^{-1}) \left(\frac{dR}{dt}\right)^2 + [2\pi R\sigma_{lv}[2h(\theta_d) - \cos\theta_e] + \frac{(m_o + \dot{m}t)g}{4} f(\theta_d)] \frac{dR}{dt} + \left[\frac{Rg}{4} f(\theta_d) - \frac{2\sigma_{lv}}{\rho R} \sin(\theta_d) - \frac{v^2}{2}\right] \dot{m} = 0 \quad (7)$$

The non-dimensional form of Equation (7) will be used to predict the spreading of the droplet. In order to study the variation of pressure, viscosity equation proposed by Lucas will be used instead of using a fixed viscosity value. In [38], a test with 55 polar and nonpolar liquids were conducted and suggested the following equation that can estimate the change of viscosity with pressure:

$$\frac{\mu}{\mu_{SL}} = \frac{1 + D(\Delta P_r / 2.118)^A}{1 + C\omega\Delta P_r} \quad (8)$$

where  $\mu$  = liquid viscosity at the pressure of interest  $P$ ,  $\mu_{SL}$  = liquid viscosity of the saturated liquid at the temperature of interest  $T$ ,  $\Delta P_r = [(P - P_{VP})/P_C]$ ,  $P_{VP}$  = saturated vapor pressure at the temperature of interest  $T$ ,  $P_C$  = critical pressure,  $\omega$  = is the acentric factor,  $A = 0.9991 -$

$$\left[\frac{4.674 \times 10^{-4}}{(1.0523 T_R - 0.3877 - 1.0513)}\right], D = \left[\frac{0.3257}{(1.0039 - T_R^{2.573})^{0.2906}}\right] - 0.2086, C = -0.07921 + 2.1616 T_R -$$

$13.4040T_R^2 + 44.1706T_R^3 - 84.8291T_R^4 + 96.1209 T_R^5 - 59.8127 T_R^6 + 15.6719 T_R^7$  and  $T_R =$  reduced temperature ( $=T/T_C$ ). The author found out that with the increase of pressure, the viscosity of the liquid increases. To validate we choose o-cresol and m-cresol which have the viscosity close to the viscosity of our experimental liquid D10 (12.09 mPa.s) and calculate the viscosity using the equation at  $P = 10$  MPa, 20 MPa, and 30 MPa in room temperature. We observe an increase in the viscosity with the increase of surrounding pressure. Results are listed in the following table:

Table 1-1: Viscosity dependency on surrounding pressure

Name of fluid/Pressure	$P_{atm}$	10 MPa	20 MPa	30 MPa
<b>Viscosities (mPa.s)</b>				
o-Cresol ( $C_7H_8O$ )	9.56	10.1	10.7	11.2
m- Cresol ( $C_7H_8O$ )	16.4	17.5	18.4	19.3

### 1.3.2 Applications

Carbon geo-sequestration (CGS) is a process where  $CO_2$  is captured from the source and then transferred to the deep saline aquifers or hydrocarbon reservoir to store it for a long period to ensure a safe environment by reducing anthropogenic  $CO_2$  [39]. For enhancing oil recovery to accelerate oil production,  $CO_2$  is injected into the reservoir [40]. During these procedures due to the lighter density of  $CO_2$  compared to water, a continuous buoyant  $CO_2$  rises upward and tends to leak through cap rock. Structural trapping, residual trapping, dissolution trapping, and mineral trapping are the four most popular  $CO_2$  trapping mechanisms. Structural trapping which is the main storage mechanism requires minimum leakage of  $CO_2$ . Capillary failure through cap rock is one possible reason for rapid leakage [16] which involves the measurement of interfacial tension and contact angle in different surrounding conditions. When the surrounding pressure reduces, the

wettability of cap rock changes and this affect the design and planning of  $CO_2$  injection and storage project. On the other hand, for capillary trapping which governs by relative permeability and capillary pressure, careful measurement of interfacial tension and contact angle in reservoir pressure, temperature, and fluid composition help to increase the storage efficiency. As a result, the measurement of contact angle and wettability of a solid surface has become important. In this project, we have measured the advancing contact angle by adding volume with constant flow rate and spreading radius to observe the changes of wettability with pressure to understand the phenomenon that can be seen in reservoir conditions. We use  $N_2$  as a surrounding medium to avoid contamination at interfaces.

#### **1.4 Thesis outline**

The background knowledge and the opportunities of the topics in the succeeding chapters are discussed briefly in this section. The investigation is focused on analyzing the influence of pressure on the spreading dynamics of a droplet on a solid substrate. We tried to explain the influence of pressure on the early time spreading dynamics using four different grades of silicone oil (D10, N35, S60, D500) on PTFE (Polytetrafluoroethylene) substrate (chapter 2) and after that, we did a detailed investigation on the hydrodynamic spreading of a water droplet on four different substrates - PTFE (Polytetrafluoroethylene), 6061 grade Aluminium, Copper, UV-Resistant Acrylic Sheet (PMMA) substrates experimentally (chapter 3). Detail of the theoretical model is also presented in chapter 3 which can predict the hydrodynamic spreading of a droplet in different surrounding pressure. Summary of the analysis presented in chapter 2 and chapter 3 of the thesis are as follows:

- In chapter 2, an experimental investigation is outlined for early time spreading dynamics of oil droplets on a partially wettable surface under various surrounding pressures. We

observe the change of droplet shape of intermediate viscosity liquid with the increase of pressure. Also, instantaneous spreading radius and equilibrium radius is investigated with the rise in viscosity. We have identified two regimes in this study- a slow starting regime (Phase I) that is found after the drop makes contact with the substrate and the fast-paced dynamics which materialize before the transition regime (Phase I,I). Exponent,  $\alpha$  has been investigated for each case and change of  $\alpha$  is presented with rising pressure. We have also discussed the effect of pressure and viscosity on the transition time and transition radius. In the later section we investigated the effect of pressure on contact line velocities and contact angle as well.

- In chapter 3, a theoretical model is developed for the hydrodynamic spreading of water droplets under the influence of pressure. In addition, the experimental result of spreading of droplet and advancing contact angle of water on four different substrates is presented in this study. Comparison between the advancing to equilibrium contact angle at different surrounding pressure conditions is also shown. Moreover, effects of pressure on the Reynold number and Weber number have also been investigated. Later on, the theoretical model is validated with an experimental investigation of a droplet with the addition of mass overtime for different surrounding pressure at the end of this chapter.

A brief discussion of the whole thesis work is given in the final chapter. The major findings are presented shortly in this chapter with the future scopes of this work. Also, in this chapter, we have mentioned the areas which can be further investigated in the future.



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## 2 Study of Early Time Dynamics of Drop Spreading in Different Surrounding Pressure

### 2.1 Abstract

The initial spreading phenomena of oil droplets on a partially wettable surface under various surrounding pressures was conducted through an experimental investigation outlined in this study. The gauge pressure of the surrounding medium was varied stepwise in the range atmospheric pressure,  $P_{\text{atm}}$  to 30 MPa. Upon contact with the surface, spreading alters the total energy of the system. The timescale for this process is very fast and the spreading involves the motion of the three-phase contact line established at the junction of the droplet, the substrate and the surrounding medium. In this study, the early time dynamics of different grades of oil drops spreading on PTFE substrates were examined utilizing high-speed imaging and drop shape analysis. Results show that the temporal evolution of the spreading radius closely resembles a power law where the empirically obtained exponent,  $\alpha$  was calculated for different surrounding pressures. Observations of instantaneous drop spreading reveal that spreading occurs through three distinct regimes: an initial spreading regime, a short transition regime and a viscous regime. In the inertial regime of the early time dynamics, spreading radius scales as  $r \sim t^\alpha$  where the spreading exponent,  $\alpha$  decreases with the increase of pressure. In addition, pressure influences the onset of the transition regime, contact line speed and, as well, the contact angle during the initial wetting process.

### 2.2 Introduction

The spreading dynamics of a droplet on a solid surface has been studied for more than two centuries [1]–[5]. Applications of this phenomenon can be seen from fuel injection, spray painting, surface cooling, coatings on surfaces, adhesion, and ink jet printing to some common natural phenomenon such as oil spreading on fish scales, raindrops falling on leaves, and more [1]–[8].

Drops of different sizes, speeds and materials have an impact on the surfaces which defines the effectiveness and efficiency of a particular operation. For example, heat transfer in spray cooling process of a steel slab depends on the amount and speed of the liquid droplets spreading on a solid surface. For this reason, the entire spreading procedure, from start to finish, is very important to understand. Several factors have an impact on the spreading of a droplet. To have a better understanding the role of these factors, it is important to segregate each parameter of interest and study its effect. The effect of one such factor is pressure surrounding the droplet. In order to estimate the effects of pressure on wetting dynamics, several studies have been carried out in relation to oil recovery systems to evaluate the geo-storage of CO<sub>2</sub> in porous aquifers. A number of studies [9], [10], [11] have been conducted to investigate the effects of surrounding pressure on the wettability and surface free energy of surface coatings. The wettability of water was found to be poor on Polytetrafluoroethylene (PTFE), intermediate on stainless steel and good for glass substrates in a pressure range of 0.1-27 MPa and 40°C [11]. However, these studies do not consider pressure effects on the initial spreading and the associated transition to later regimes. Initial spreading dictates the final contour of the drop as well as the equilibrium contact angle. In this paper, we investigate experientially the early time spreading dynamics on a solid substrate while considering a change in surrounding pressure, the results from which can be used to improve high pressure engineering processes such as refining, extraction, refrigeration, polymer foaming, polymerisation , pressing of oil seeds, etc [11].

The entire spreading phenomena is broadly made up of three distinct regimes namely, an inertia dominated early regime, a short transition regime, and a viscous dominated late regime. Extensive studies have been conducted on the later stage of spreading droplets [12]–[14]. The later stage of spreading dynamics, whose onset time varies between different fluids, is slow and the wetting

radius grows as  $r \sim t^{1/10}$  [13] close to the contact line where the surface tension and viscous forces balance; in this relation,  $r$  is the wetting radius and  $t$  is the evolution of time. Compared to the later stage, the dynamics of the early stage of drop spreading is much faster. With new technical developments in recent years, such as high-speed video cameras for capturing the first milliseconds of drop spreading, researchers are now able to study the early stage of droplet spreading.

In droplet spreading, a drop with a specific radius is gently brought into contact with a substrate. After the deposition of the drop on a solid substrate, the unbalanced horizontal force  $\sigma (\cos \theta - \cos \theta_{eq})$  drives the droplet to spread until it reaches a final equilibrium contact angle ( $\theta_{eq}$ ). Here,  $\sigma$  is the surface tension at the liquid-vapor interface and  $\theta$  is the contact angle at a particular instant. At the instant of contact, the curvature of the interface is infinite which leads to a diverging Laplace pressure jump. This generates a rapid flow in the drop that refills the liquid neck causing the wetted area to grow over time [15]. This Laplace pressure is the surface tension, which balances the bulk droplet resistive inertia motion in the early stage of spreading.

The entire spreading regime can be characterized by an early inertia dominated regime followed by a viscosity dominated regime close to the equilibrium condition for low viscosity liquids [16]–[19]. For high viscosity liquids, the entire process is viscosity dominated [12]–[14], [16], [18]–[21]. For the intermediate class of liquids, both the early viscous and the inertial regimes have the possibility to exist [22]; In this paper, liquids of intermediate viscosity to high viscosity (i.e. 12.09, 73.16, 136.1 and 713.4 mPa.s) were chosen to observe the early stage of spreading in different surrounding pressure conditions.

For completely wetting surfaces (equilibrium contact angle,  $\theta_{eq} = 0$ ), the presence of a precursor film caused the initial dynamics to be nearly identical to the coalescence of droplets with a wet



film [19]. Initial dynamic wetting is driven by capillarity and is averse to viscous dissipation of the liquid for completely wetting surfaces [3], [13], [23], [24]. For spreading on completely wetting surfaces, droplets grow as  $r \sim t^{(1/2)}$  [19] and for partially wetting surfaces the spreading radius,  $r$ , follows a power law,  $r = Ct^\alpha$  (where  $C$  is a constant value and  $\alpha$  is the spreading exponent). The power law depends on the magnitude of force that resists drop deformation. To identify whether our study displays power-law regimes, we determine the exponent as  $\alpha = d \ln r / d \ln t$  [19]. For partially- and completely-wetting scenarios, inertia dominates spreading in the early stage, but the duration of spreading for a particular drop depends on the characteristics of the surface and the surrounding medium. In addition, theoretical and experimental studies have shown that an inertial regime occurs before the viscous regime [19] [25]. Although the inertial regime is short lived as compared to the viscous regime, inertia plays a significant role as a resistive force to spreading during its early stages. As a result, the non-dimensional spreading law takes the form  $r/R = C(t/\tau_c)^\alpha$  where  $\tau_c = (\rho R^3/\sigma)^{1/2}$  is the inertial timescale. In the early stage of spreading, inertia resists drop deformation, and the spreading radius scales as  $(R\sigma/\rho)^{1/2}t^{1/2}$ , where  $\rho$  is the density of drop,  $R$  is the drop radius and  $\sigma$  is the surface tension. In the later stage, where viscosity dominates spreading, the radius follows a scale,  $r \sim \sigma t/\mu$ , where  $\mu$  is the dynamic viscosity [20]. In addition, it was also shown that the spreading of a droplet is independent of drop size and surface wettability in the first regime of spreading [15].

As discussed in this paper, we conducted the experimental investigation of early time dynamics of microscale droplet spreading using high-speed imaging and drop shape analysis at elevated pressure environments. Specifically, we observe the initial spreading phenomena of spreading droplets on a partially wetting surface under various surrounding pressures using nitrogen gas. The results from our experiments have shed light on interesting phenomena occurring at early

timescales that was previously unknown to researchers. Here, we have varied the pressure of the surrounding medium stepwise in 10 MPa increments within the range between atmospheric pressure,  $P_{\text{atm}}$  to 30 MPa (gauge). In the following sections, the experimental setup to capture the dynamics; the pressure effects on the droplet radius; the spreading velocities; the transition times; and the contact angles are discussed in detail.

Results show that the temporal evolution of the spreading radius closely resembles a power law where we calculate the empirically obtained exponent at different surrounding pressures. Observations of spontaneous drop spreading reveal that spreading occurs through three distinct regimes, namely: initial spreading regime, a short transition regime and a viscous regime. In this study, we show that the spreading radius in the inertia-dominated early time dynamics scales as  $r \sim t^\alpha$  where the exponent varies moderately with pressure. In addition, high pressure affects contact line velocities and contact angle at the three-phase contact line. This work shows that high values of spreading velocity and contact angle characterize the early regime of spreading.

### **2.3 Materials and experimental configuration**

Experiments were carried out using a DSA100HP (KRÜSS GmbH - Germany) and a Phantom V711 camera. Figure 2.1 shows the schematic of the experimental setup used. Information of the substrate and experimental liquids used, experimental setup and the imaging system and also the procedure of the experiment have been described in detail in the following three subsections.

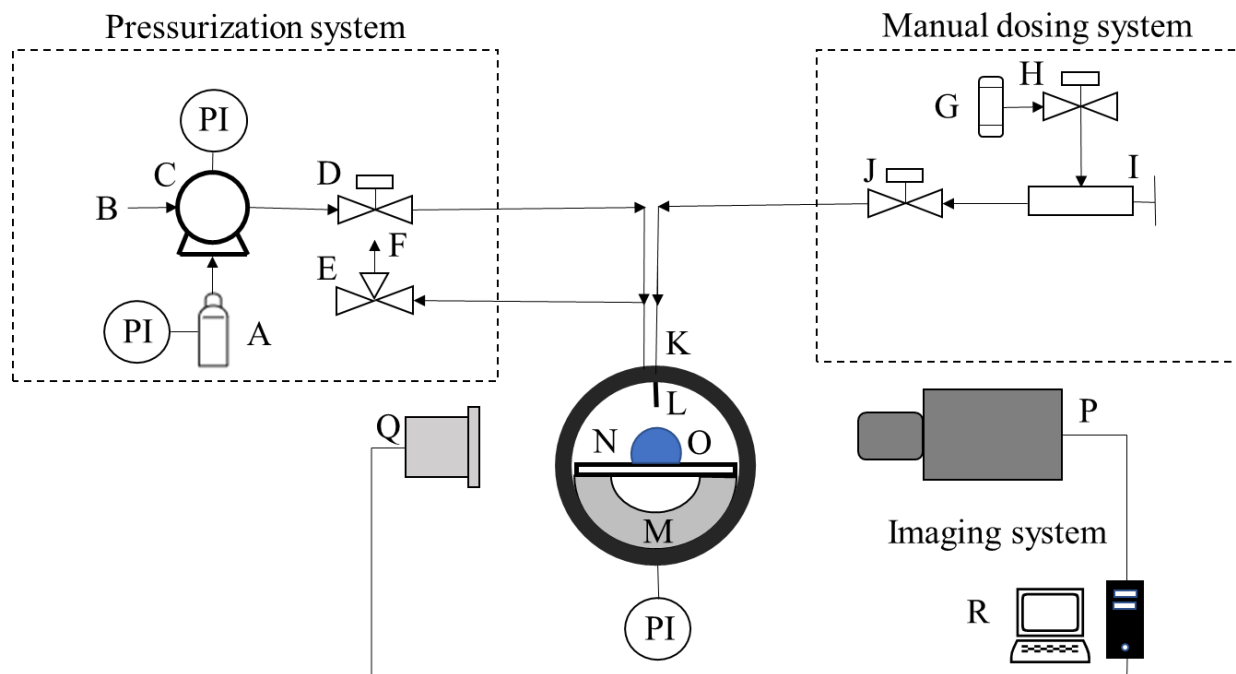


Figure 2.1 Schematic of the experimental setup (pressurization system, manual dosing system and imaging system) for investigating drop spreading in elevated pressure environments. A: nitrogen gas cylinder, B: building nitrogen gas supply, C: booster pump, D: inlet gas valve, E: outlet gas valve, F: fumehood, G: liquid reservoir, H: liquid inlet valve, I: manually controlled piston accumulator, J: discharge valve connected to capillary, K: pressurized chamber, L: capillary, M: substrate holder, N: solid substrate, O: droplet of liquid, P: high speed camera, Q: LED light source, R: desktop computer for controlling the image system and drop shape analysis, PI: pressure indicator. Items Q (high-speed camera) and R (LED light source) are oriented in such a way they view the chamber perpendicularly to the page. Figure is not drawn to scale.

### 2.3.1 Substrate and experimental liquids

A hydrophobic, non-stick, temperature-resistant substrate - PTFE (Polytetrafluoroethylene – purchased from McMaster-Carr), measuring 37mm W × 25.5 mm L × 1 mm D, was used in the experiments. The cleaning procedure for the substrate employed isopropanol (IPA), deionized water and compressed air for drying purposes. At first, the substrate was submerged in isopropanol before being sonicated for 15 minutes. We then repeated the same sonication while submerging in deionized water for another 15 min. Finally, we dried the PTFE substrates using compressed air.

For the test liquids, silicone oils of four different grades, purchased from Millipore Sigma were used. We have determined experimentally the surface tension of the different grades of oil named as D10, N35, S60, D500 to be  $28.64 \pm 0.10 \text{ mN/m}$ ,  $30.97 \pm 0.17 \text{ mN/m}$ ,  $31.54 \pm 0.05 \text{ mN/m}$  &  $32.28 \pm 0.19 \text{ mN/m}$  respectively at atmospheric conditions using the pendant drop method.

The properties of the silicone test liquids, as given by Millipore Sigma, are listed in Table 2-1.

Table 2-1 Physical properties of the different test liquids under investigation

Type of oil	Temperature (°C)	Viscosity (mPa.s)	Density (g/ml)
D10	20	12.09	0.8299
N35	20	73.16	0.8562
S60	20	136.1	0.8601
D500	20	713.4	0.8723

### 2.3.2 Experimental setup (pressurization and manual dosing system)

The experiments were performed at  $P_{atm}$  as well as 10, 20 and 30 MPa gauge pressure. We performed three trials in each condition to calculate the average value and standard deviation of the spreading radius and contact angle. To avoid bubbles, several drops were allowed to form and detach with the liquid line valve open on the high-pressure vessel. In this state, the pressure inside the chamber remained at standard atmospheric conditions. Afterwards the pressurization chamber was cleaned with isopropanol. Then, the substrate was placed on top of a custom-made 3D-printed substrate holder, and both were then placed horizontally inside the chamber. The chamber was then sealed using two sapphire viewing windows. By opening valve D, we pressurized the unit to the desired level using nitrogen gas from cylinder A (purchased from Praxair Inc., Product number: NI 5. OUH-T) and nitrogen gas from the building supply, B supplied at 100 psi (0.68 MPa). Both lines of nitrogen gas merge at the booster pump, which amplified the pressure of the cylinder

nitrogen to a desired value before the gas was sent to the chamber. Afterwards, the system was left to stabilize for 10 minutes.

A sequence of steps was followed to deposit the drop. Initially, the liquid reservoir, G, is filled with the test liquid. Valve H was then opened, and piston I was moved to the opening position to allow liquid to advance from the reservoir. After valve H is closed, and piston I is moved to the closed position, valve J is opened to allow the test liquid to flow to the 1.1 mm diameter capillary/needle for drop formation. Test liquids were discharged at a minimum outflow rate and the drop is created using a quasi-static process for growth at the tip of the needle, which is always kept at a fixed height above the substrate. Once the pendent drop was formed, valve J was closed, and 2 minutes were then allowed for the interface of the droplet inside the pressurized chamber to stabilize. Later, valve J was opened slightly and by moving piston I to the left, the drop detaches from the needle and at the same time, the high-speed camera was triggered in order to record the video of the initial spreading dynamics. After a trial was completed, the system was depressurized by opening valve E by releasing pressurized gas to the fume hood. Finally, the substrate and substrate holder were removed. The experiments were repeated after cleaning the needle, substrate holder, pressurization chamber and by using a new PTFE substrate in each case.

### **2.3.3 Imaging system of the setup**

We utilized a high-speed camera (Vision research, Phantom V711) for observing the initial dynamics of the droplet. A tripod mount which consists of a tripod base (Manfrotto model 475B Pro Geared Tripod) and a tripod head (Manfrotto 229 3D Super-Pro Head) was used to keep the camera in position, and the camera was coupled to a macro lens to capture the front of the droplet undergoing spreading. An LED light source (Nila Zaila - Daylight v2) was used as the backlight to enhance image quality and contrast. For the camera settings, a frame rate of 20,000 fps and

exposure time of 40  $\mu\text{s}$  were used with a  $512 \text{ px} \times 384 \text{ px}$  resolution for all the experiments. The precision in measuring the spreading radius is  $\pm 0.01 \text{ mm}$  and contact angle is  $\pm 0.1^\circ$  in all cases. The magnification of the lens was adjusted until two-thirds of the sensor's field of view is covered by the droplet.

Image calibration was performed before each repetition of the experiment using the capillary outer diameter. Contact angles were measured using the ADVANCE software (KRÜSS GmbH) coupled with the DSA (Drop Shape Analyser) 100 HP unit (KRÜSS GmbH); The base radius is, in turn, measured from the raw video of the experiments processed in a separate image processing software package (Image-Pro 10 Media Cybernetics). At high pressures, the surface tension was measured from the experiment utilizing the Pendant Drop Technique.

## 2.4 Results and discussion

The effect of pressure on the initial spreading of droplet was investigated in this study. A pendent drop, of volume 10  $\mu\text{L}$ , was generated and was allowed to detach completely from the needle. When the drop touches the PTFE substrate, it starts to spread. The capillary force works as the driving force and the inertia force works as the resistive force during the initial stage of spreading [26]. But if the substrate has wettability gradient we have to consider the instantaneous difference in the contact angle that causes the spreading due to the unbalanced interfacial tension along the three phase contact line [22]. Important parameters which affects the spreading behavior are viscosity of the liquid,  $\mu_L$  and interfacial tension of the liquid,  $\sigma_{LG}$  [15], [27]. In our study we have used the Nitrogen gas ( $N_2$ ) as surrounding medium and by increasing the surrounding medium pressure, the cohesive energy of the nitrogen gas ( $N_2$ ) increases and thus the density of the medium increases [28]. Unlike gas molecules, the molecules of a liquid are packed together tightly with fixed intermolecular distance and can be marginally compressed with surrounding pressure [29].

Therefore, the density of the liquid ( $\rho_L$ ) is assumed to be constant with respect to pressures applied in these experiments [30]. As the surrounding pressure is increased, the density difference between the liquid phase and gas phase also decreases. By increasing the surrounding pressure, the viscosity of the test liquids increases. Lucas [31], [32] performed tests with 55 polar and nonpolar liquids. The author suggested an empirical equation that can estimate the change of viscosity with pressure with errors less than 10%. The reduced fluidity or increased viscosity at the macro level can be explained by an increased degree of intermolecular entanglement when the surrounding pressure is raised [33].

Snapshots of the silicone oil N35 (73.16 mPa.s) droplet on the PTFE substrate, at different time steps, in  $P_{atm}$  and in 30 MPa pressure conditions, are shown in Figure 2.2. At  $t = 796.85$  ms droplets from both cases reach the equilibrium state. We observe that the drop has a more spherical shape at 30 MPa pressure compared to the drop at  $P_{atm}$ ; thus, an increase in surrounding pressure appears to produce a more pronounced spherical shape. This clearly suggests that at a given time step, the N35 (73.16 mPa.s) droplet spreads more at  $P_{atm}$  compared to a higher-pressure surrounding condition (e.g. 30 MPa) and this difference is more prominent in the later stage of spreading. The surrounding pressure generates a resistive force against the spreading of droplet which causes the drop to spread less in higher surrounding pressures compared to atmospheric conditions. This indicates that the wetting area and wetting radius is higher in atmospheric pressure conditions than at elevated chamber pressures and it reduces gradually with the increase of pressure. We also observe that in case of low viscous liquids such as D10 (12.09 mPa.s), the drop completely detaches from the needle before touching the substrate. However for highly viscous liquids such as D500 (713.4 mPa.s) a neck is observed to form between the droplet and the needle during drop deposition. In addition, the viscosity of the test liquid increases with

the increase of pressure. For this reason, at  $P_{atm}$  pressure there is no neck formation but in 30 MPa pressure the neck formation phenomenon is observed for N35 (73.16 mPa.s) (Figure 2.2).

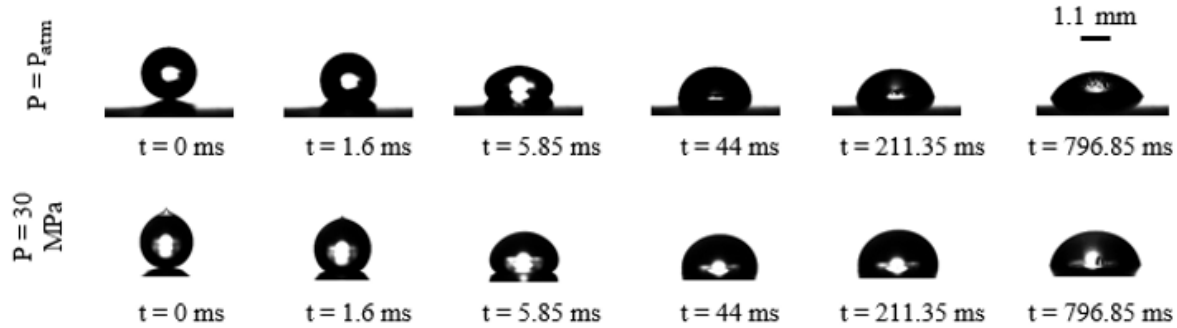


Figure 2.2 Snapshots of droplet spreading on PTFE substrate at  $P_{atm}$  and 30 MPa: Depicting the temporal evolution of the N35 (73.16 mPa.s) droplet spreading until it attains equilibrium.

In the case of droplet spreading on partially wetting surfaces, there exists two different regimes, with a transition regime in-between, of contact line movement. High spreading velocities characterize the early time dynamics, which corresponds to the first case. The driving force of drop motion is the capillary force and the factor that predominantly contributes to the resistive force of the droplet is inertia. Thus, the motion of the droplet following immediate contact with the PTFE is inertia dominated. Low contact line velocities characterize the later stages of spreading and, in this region, the movement is mainly governed by viscous forces resisting the advancement of the oil droplets. The bridge connecting both of these stages, namely the inertia dominated spreading and the viscous dominated spreading, is brought about from a transition regime which differs in size and scale depending on the experimental parameters used. It is widely understood from literature [15] that the size of this crossover regime depends on the initial radius of the droplet. However, the onset time of the transition regime is controlled by not only drop radius but also drop viscosity attributed to the various grades of silicone oil used, and the surrounding pressure under which the drops spread. This will be described in detail in section 2.4.3. We investigate the effect



of these experimental parameters in detail. Furthermore, the dynamics of the droplets, for the four different grades of oil used, is characterized by the instantaneous spreading velocity in both the early regime, as well as the late regime, of spreading. In addition to the spreading velocity plots, the temporal evolution of the contact angle was used to characterize the spreading dynamics in these regimes with special emphasis on the early time dynamics that appear before the transition regime.

#### **2.4.1 Surrounding pressure effects on the spreading radius**

Figure 2.3 illustrates the time evolution of the spreading radius experimental values during the early stage of spreading, as well as the later stages of wetting, for different grades of silicone oils. For a select silicone oil, the surrounding ambient chamber pressure surrounding the droplet distinguishes the four curves from one another. Specifically, we plot, on a cartesian axis, the results for four different gauge pressure conditions:  $P_{\text{atm}}$ , 10 MPa, 20 MPa and 30 MPa. Here, we observe that the curves splinter into four distinct sets quickly after making contact with the PTFE substrate. In Figure 2.3, four different viscosity liquids were used to characterize the spreading of droplets with the variation of pressure. Considering D10 at atmospheric conditions, the starting radius is  $0.5 \pm 0.1$  mm. It is  $0.25 \pm 0.1$  mm for N35 and for S60 and D500 it is  $0.1 \pm 0.1$  mm. This variation is due to the effect of viscosity of the liquid. With the increase of viscosity, a smaller wetting perimeter of the droplet is generated at the point when the drop touches the substrate. The effect of viscosity on the wetting perimeter remains for elevated pressures as well. Furthermore, pressure has a significant influence on the spreading radius for intermediate and high viscosity silicone oil in the later stages of wetting. Although, in the later stages of wetting, pressure has larger effect on the spreading radius for intermediate viscosity liquids than for high viscosity liquids; In the latter case, pressure effects are significantly less pronounced. The inset plot of Figure 2.3 depicts values

of the equilibrium radius as a function of the chamber pressure for the four different grades of silicone oils. A rise in chamber pressure notably lowers the wetting radius for D10, N35 and S60. However, the same pressure effects are remarkably less distinct for D500, which bears a higher viscosity in comparison with the other oil grades. A decrease of equilibrium radius as a function of the chamber pressure can be observed with the increase of the viscosity of the liquid. In addition, we note that for each grade of silicone oil, the drop achieves equilibrium faster in elevated pressure environments. In Figure 2.3 for D10, it takes approximately 0.25 s, 0.2 s, 0.15 s and 0.1 s respectively for the drop radius to plateau at atmospheric pressure,  $P = 10$  MPa,  $P = 20$  MPa and  $P = 30$  MPa. Similar scenarios are also observed for N35 and S60 where it takes less amount of time to level off when the surrounding pressure is raised. But for D500 which has a relatively higher viscosity, no noticeable trend is observed. Change of the equilibrium contact angle (CA) with pressure for four different grades of oils are listed in

Table 2-2. We observe a rise in equilibrium contact angle for D10, S60 and D500 with the increase of pressure. For N35, contact angle increases by  $18^\circ$  from atmospheric pressure to  $P = 10$  MPa and by  $13^\circ$  from  $P = 20$  MPa to  $P = 30$  MPa. The difference was negligible for  $P = 10$  MPa and 20 MPa.

Table 2-2 Change of the equilibrium contact angle with pressure

Type of oil	CA at $P = P_{atm}$ ( $^\circ$ )	CA at $P = 10$ MPa ( $^\circ$ )	CA at $P = 20$ MPa ( $^\circ$ )	CA at $P = 30$ MPa ( $^\circ$ )
D10	49±2	61±1	70±1	73±2
N35	60±3	78±1	78±2	91±2
S60	61±2	81±1	90±2	91±2
D500	63±1	89±2	92±2	98±3

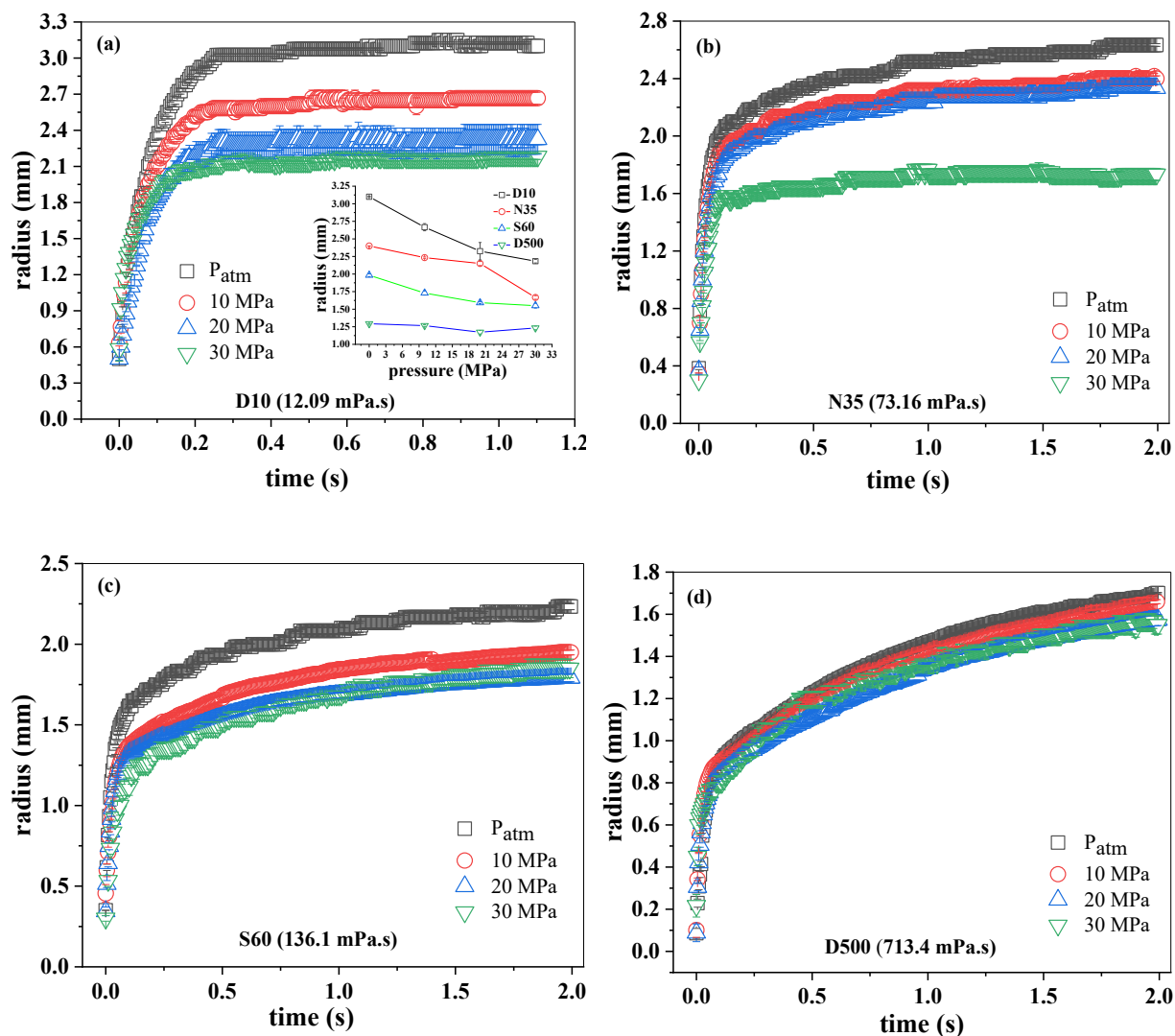


Figure 2.3 Effect of surrounding pressure on spreading dynamics: (a) D10 (12.09 mPa.s) and the inset depicts the equilibrium radius as a function of pressure for four different viscosities (i.e., D10, N35, S60 and D500) (b) N35 (73.16 mPa.s) (c) S60 (136.1 mPa.s) and (d) D500 (713.4 mPa.s).

In the initial stage of droplet spreading, the driving force that acts is the capillary force. Initially, the mass of the droplet is large at the advancing interface. Considering this mass, inertia acts as the resistive force. With the evolution of time, the drop spreads and the mass of the droplet becomes distributed over a larger advancing interface. As a result, the inertia force drops and the dominant resistive force that acts on the droplet becomes the viscous force. As the early time

dynamics are inertia dominated, we normalize the time axis by the inertial time scale and radius by initial radius,  $R$ . This is shown in the logarithmic plots of Figure 2.4. Here, log-log plots were generated to show the effect of pressure at the initial stage of spreading, which effectively emphasizes the early time dynamics. Evolution of dimensionless spreading radius  $r/R$  as a function of dimensionless time  $t^* = t/(\rho R^3/\sigma)^{0.5}$  for D10, N35, S60 and D500 droplets at gauge pressure of  $P_{atm}$ , 10 MPa, 20 MPa and 30 MPa has been shown.

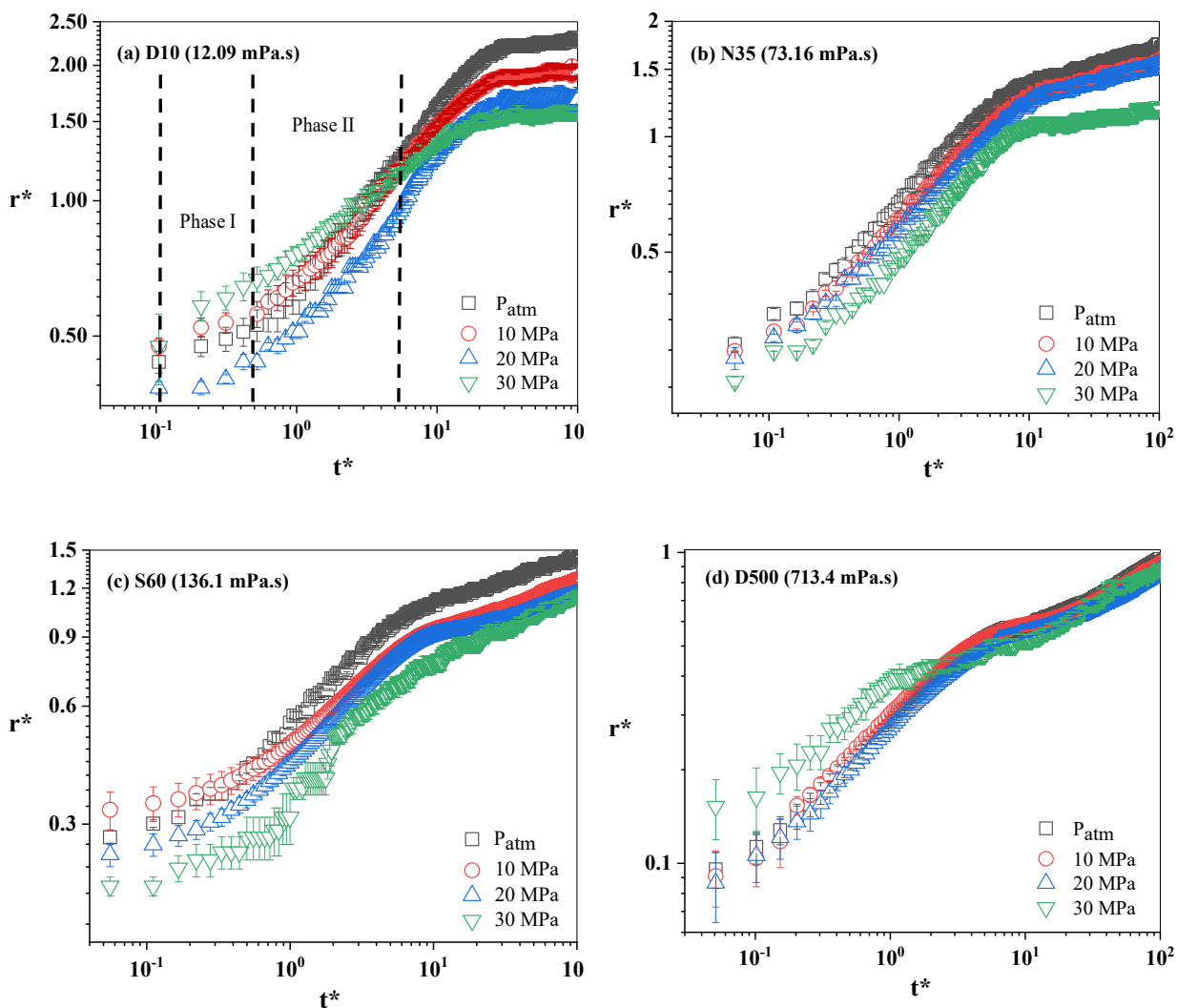


Figure 2.4 Dynamics of droplet spreading under various surrounding chamber pressures: Evolution of dimensionless spreading radius,  $r^* = r/R$  as a function of non-dimensional time,  $t^* = t/(\rho R^3/\sigma)^{0.5}$  for (a) D10 (12.09 mPa.s) (b) N35 (73.16 mPa.s) (c) S60 (136.1 mPa.s) and (d) D500 (713.4 mPa.s) droplets at gauge pressure of  $P_{atm}$ , 10 MPa, 20 MPa and 30 MPa.

As evident from Figure 2.4, pressure has a strong influence on the early time dynamics for D10.

Two parts of initial spreading are identified based on the spreading radius, namely: Phase I – which takes place at very early times immediately following contact of the drop with the substrate, and Phase II – faster paced dynamics before the onset of transition (Figure 2.4). Both of these regimes are influenced by pressure. Although few data points exist because of the fast dynamics involved, the conventional inertial regime is preceded by another regime characterized

by slightly slower dynamics for intermediate viscosity oils such as D10, N35 and S60. This Phase I spreading regime is heavily influenced by pressure. However, this new early regime of dynamics is non-existent for high viscosity oil such as D500 as evident from its monotone climb in spreading radius immediately following contact with the substrate. The viscosity ratio of D500 to air is very high ( $\mu_{D500}/\mu_{air} = 38835.05$ ). This high viscosity ratio gives evidence that inertial spreading for D500 is viscosity dominated which is similar to the study found in [16] for high viscosity liquids. Moreover, this early regime splintering is observable for all cases with varying degree of intensity. N35 and D500 shows little influence of pressure on the early stage of spreading except when the pressure rose to 30 MPa for D500.

#### **2.4.2 Spreading coefficient, $\alpha$**

The dimensionless spreading radius scales as  $r/R \propto (t/\tau_c)^\alpha$  where  $\tau_c$  is the inertial time scale and  $\alpha$  is the experimentally derived spreading exponent. The spreading exponent,  $\alpha$ , for D10, N35, S60 and D500 at various gauge chamber pressures ( $P_{atm}$ , 10, 20, 30 MPa) has been calculated and is plotted in Figure 2.5 We observe that with the increase of pressure, exponent  $\alpha$  decreases for the four different grades of liquid with D10 showing the highest decrease in  $\alpha$  between 20 and 30 MPa. In addition, exponent  $\alpha$  increases with the increase of the viscosity of the liquids.

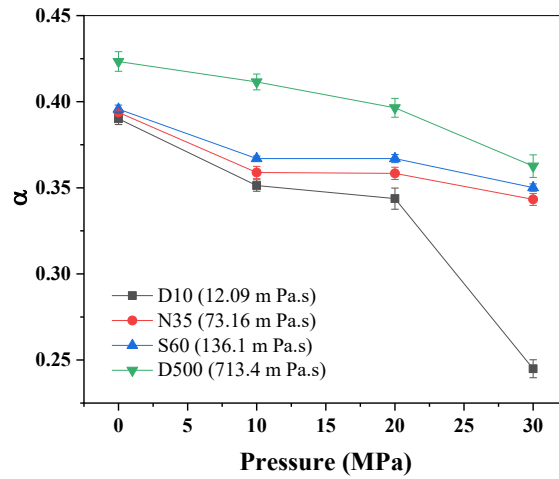


Figure 2.5 Spreading exponent  $\alpha$  for D10, N35, S60 and D500 at various gauge chamber pressures:  $P_{atm}$ , 10, 20, 30 MPa. The dimensionless spreading radius  $r/R \propto (t/\tau)^\alpha$  where  $\tau$  is the inertial time scale.

### 2.4.3 Transition time, $\tau_t$ , and transition radius, $r_t$

In Figure 2.6, transition time,  $\tau_t/\tau_c$  and corresponding transition radius,  $r_t/R$  for D10, N35, S60 and D500 at various gauge pressures is plotted. Here the transition signifies regime change from the inertial to the viscous stages and is marked by a slow and gradual transformation from fast to slow dynamics. Pressure influences the onset of the transition regime in early spreading: particularly, the starting time of transition and transition radius. To be more precise, an increase in pressure subsequently lowers the transition time and radius. This particularly prominent in the case of D10, whose viscosity ranks in the lower range when compared to the other grades of silicone oils. For instance, a change in pressure from  $P_{atm}$  to 30 MPa for D10 results in a 43% reduction in the dimensionless transition time and a 27% fall in the transition radius, which marks the end of the inertial regime. In comparison D500, experiences an 11.9% and a 16.7% decrease in the non-dimensional transition time and transition radius respectively for the same pressure difference. Thus, low viscosity oils experience the highest fall in transition time and radius because of chamber

pressure elevation. However, for highly viscous oils such as D500, pressure effects on the onset of the transition regime are quite negligible compared to other liquids.

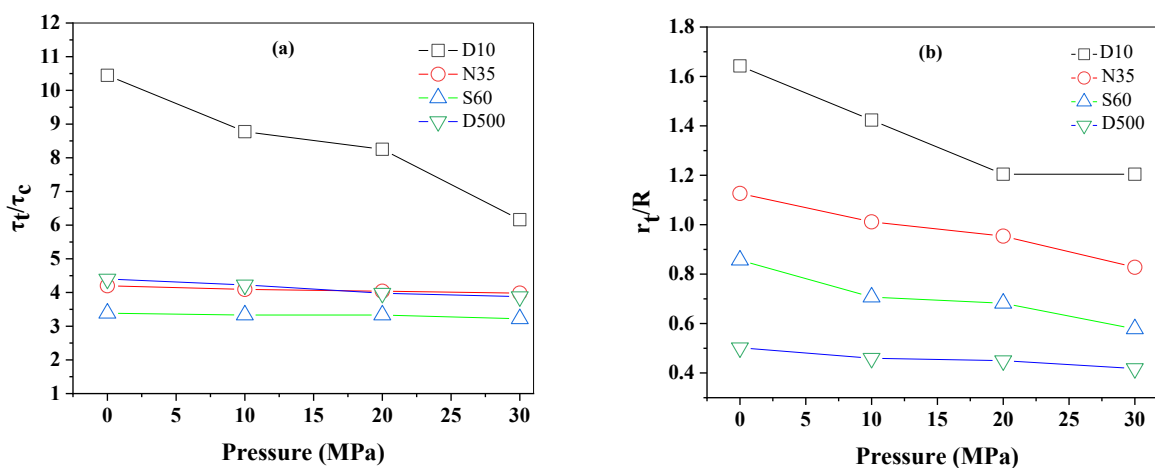


Figure 2.6 Dynamics of spreading: (a) transition time,  $\tau_t/\tau_c$  and (b) transition radius,  $r_t/R$  for D10, N35, S60 and D500 at various gauge pressures:  $P_{atm}$ , 10, 20, and 30 MPa.

#### 2.4.4 Role of viscosity on spreading kinetics

We now study the dependence of viscosity on spreading kinetics. Figure 2.7 depicts the temporal evolution of the spreading radius and compares the results between drops of four different viscosities at different surrounding pressure conditions. The viscosities, range from 12.09 mPa.s for D10 to 713.4 mPa.s for D500 droplets at standard atmospheric conditions. Between each set of results that we report, we increase the pressure by 10 MPa to study the pressure effects as well. At the early stages of spreading, viscosity plays a minor role on the spreading dynamics. Here, the curves of the various grades of silicone oils collapse on each other. However, the gaps between the curves widen during the transition as well as the viscous regime. This is because viscous dissipation begins to play an increasingly prominent role in arresting the advancement of the contact line. As the contact line advances further, the total wetting perimeter rises, which increases the area on upon which viscous drag can act. For low viscosity drops such as D10, the hindrance to spreading from viscous drag is smaller in comparison to the high viscosity oils such as D500.



As a result, drops of D500 spread less during, and beyond, the transition regime until the spreading radius plateaus near the equilibrium position. Thus, a rise in liquid viscosity lowers the spreading radius most notably during the later stages. In addition, the onset and duration of the transition regime also depends on the viscosity.

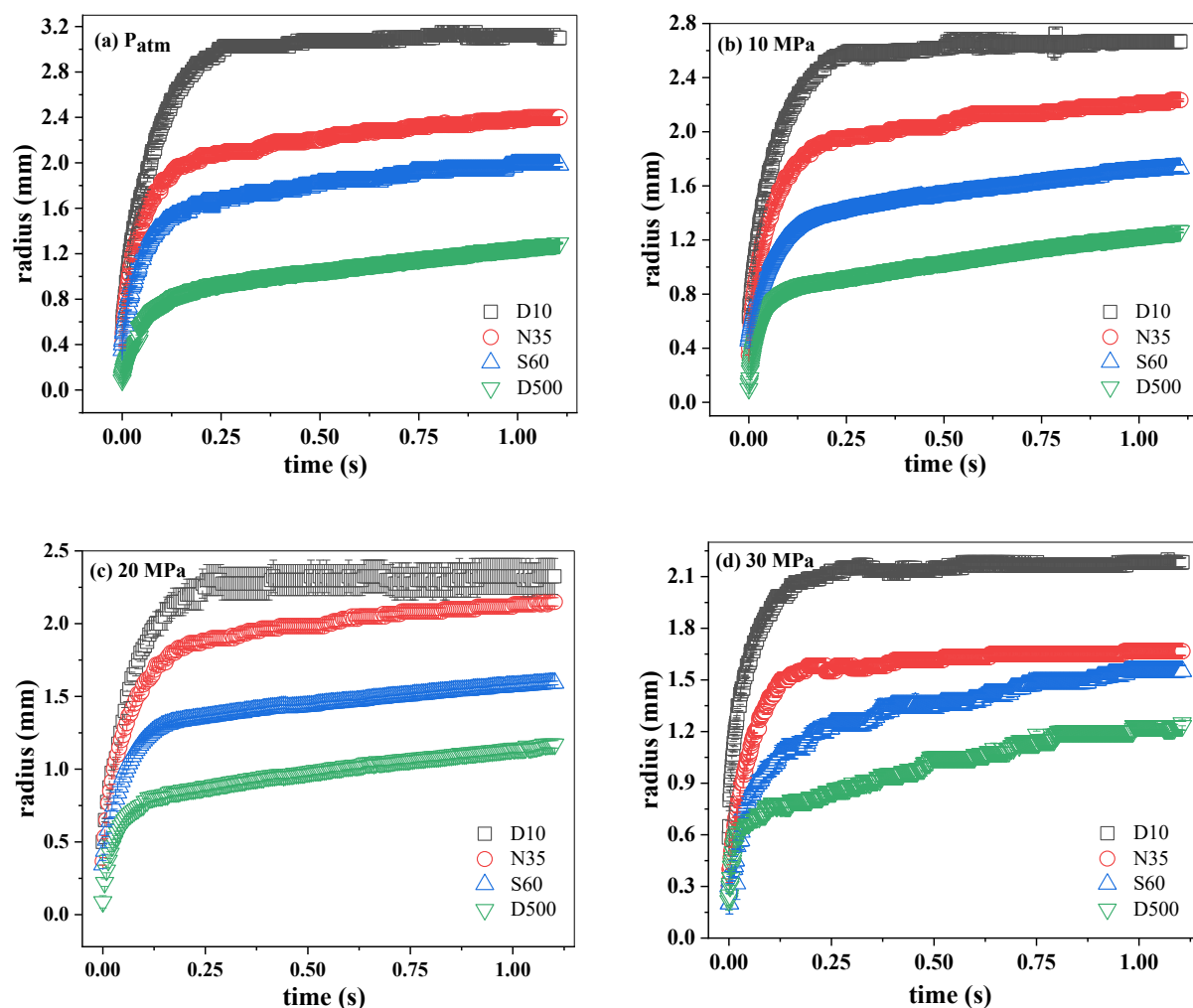


Figure 2.7 Dynamics of spreading of the droplet in elevated pressure environments: evolution of dimensional spreading radius as a function of time for D10 (12.09 mPa.s), N35 (73.16 mPa.s), S60 (136.1 mPa.s) and D500 (713.4 mPa.s) droplets at (a)  $P_{atm}$ , (b) 10 MPa, (c) 20 MPa and (d) 30 MPa.

### 2.4.5 Contact line velocity

We analyze the effects of pressure on the spreading velocity of the liquids (Figure 2.8). The velocity plots were derived by curve fitting. To obtain the velocity plots, the radius values were poly fitted (6th order) and a first order derivative of the polynomial equation with respect to time was then calculated. The  $R^2$  (Coefficient of determination) for all the curve fitted equations is greater than 0.95. Early time dynamics are characterized by two distinct regimes of rapid spreading bridged together by a transition state. These are stage 1, high-velocity early time dynamics immediately following drop deposition and stage 2, slow-paced moderate velocity regime. In stage 1, the capillary force act as the driving force which dominates significantly over two resistive forces, inertia and viscous forces, that, in the later stage, slow down the speed of advancement. From Figure 2.8 we observe that approximately 10 ms following the contact with the substrate, the dynamics after sustaining a gradual crossover region switches to stage 2. Here, the drop velocity incurs a stage of aggressive deceleration as a consequence of the increased dominance of viscous drag over a dwindling starting inertia. The decreased velocities in this region are also attributed to a growing wetted area which increases the extent of the adhesive forces from the substrate, thus preventing the drop from spreading further. For each grade of silicone oil, the curves of velocities gradually collapse in this region signifying pressure invariance with the evolution of time. In all cases, the data gathered are distinguished by four different curves pertaining to four different chamber pressures. The axes are marked with dimensional units.

For D10 and S60 the rapid dynamics pertaining to stage 1 are heavily influenced by surrounding pressure. In this stage, an increase in pressure is associated with a fall in drop velocities – albeit in small variations. This is attributed to high drop viscosities listed in Table 2-1 Physical properties of the different test liquids under investigation. Higher viscosities in turn increase drag and viscous

dissipation on the droplets. In effect, this phenomenon changes the balance between inertia and viscous forces leading to a diminishing difference between the two forces. Thus a greater influence of viscous drag serves to reduce contact line speeds in this region. In the case of N35, the increase of pressure from 20 to 30 MPa has a profound impact on spreading velocity. In contrast, a steady rise of pressure from atmospheric up to 20 MPa apparently has a negligible impact on the spreading rate. Finally, for D500, a drastic fall in drop velocities is observed following pressure increase from atmospheric to 10MPa. Successive pressure rises to 10, 20 and 30 MPa at stage 1 has a marginal decline in velocities in this case.

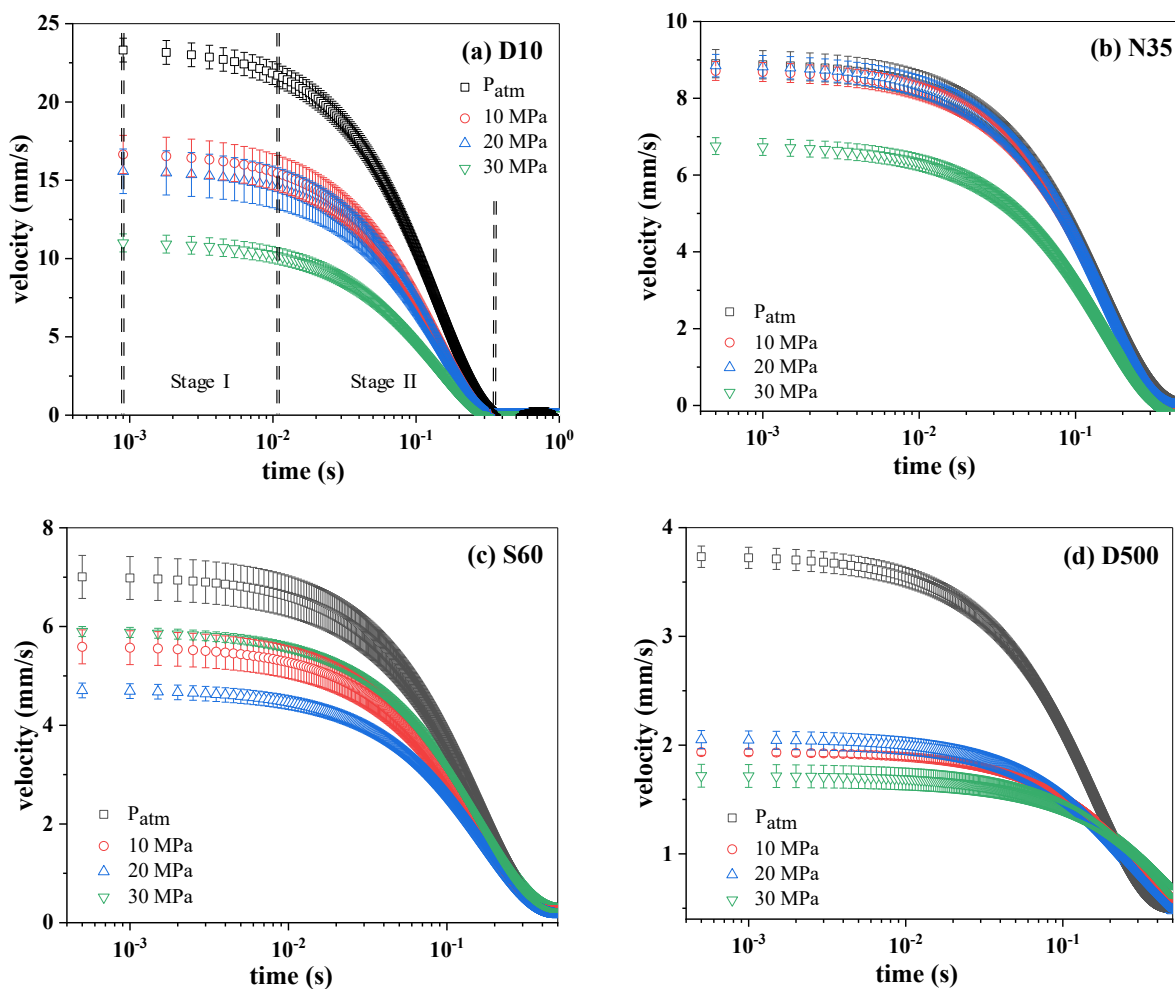


Figure 2.8 Spreading dynamics for different grades of silicone oil (a) D10 (b) N35 (c) S60 and (d) D500 at various chamber pressures.

### 2.4.6 Contact angle during the early stage of spreading

For the four different grades of liquids, the change of initial contact angle with pressure is negligible as all the curves almost overlap on one another (Figure 2.9). The effect of pressure can be seen in the later stage. Initial contact angle at early stage reaches a quasi equilibrium state after which it demonstrates a period of rapid decline to a low contact angle state. The onset time of the switchover is a strong function of the pressure. Prior to the switchover, the drop assumes an almost spherical shape with a high contact angle. This in turn creates a small wetting perimeter which is consistent with the spreading radius plots. A smaller wetting area incurs less resistance to the

advancement of the contact line which is in line with lower retardation and higher velocities in this regime as opposed to the slower dynamics post switchover. Thus, pressure has a greater influence on the start of the switchover than on the contact angle values themselves which seems to be pressure invariant on account of the fact that the curves collapse on one another. Hence, the Phase I dynamics, which depend largely on pressure, are characterized by high contact angle and high velocities.

In contrast, the influence of pressure is heightened during the rapid transition period and this is especially apparent for N35, S60 and D500. The splintering due to high pressure is further amplified in the post switchover regime with curves of N35 and S60 increasingly digressing from one another showing a strong pressure dependence in this region. For D10 however the curves gradually converge with the evolution of time. This shows that for liquids with low viscosity, the post switchover to slower dynamics is largely independent of pressure.

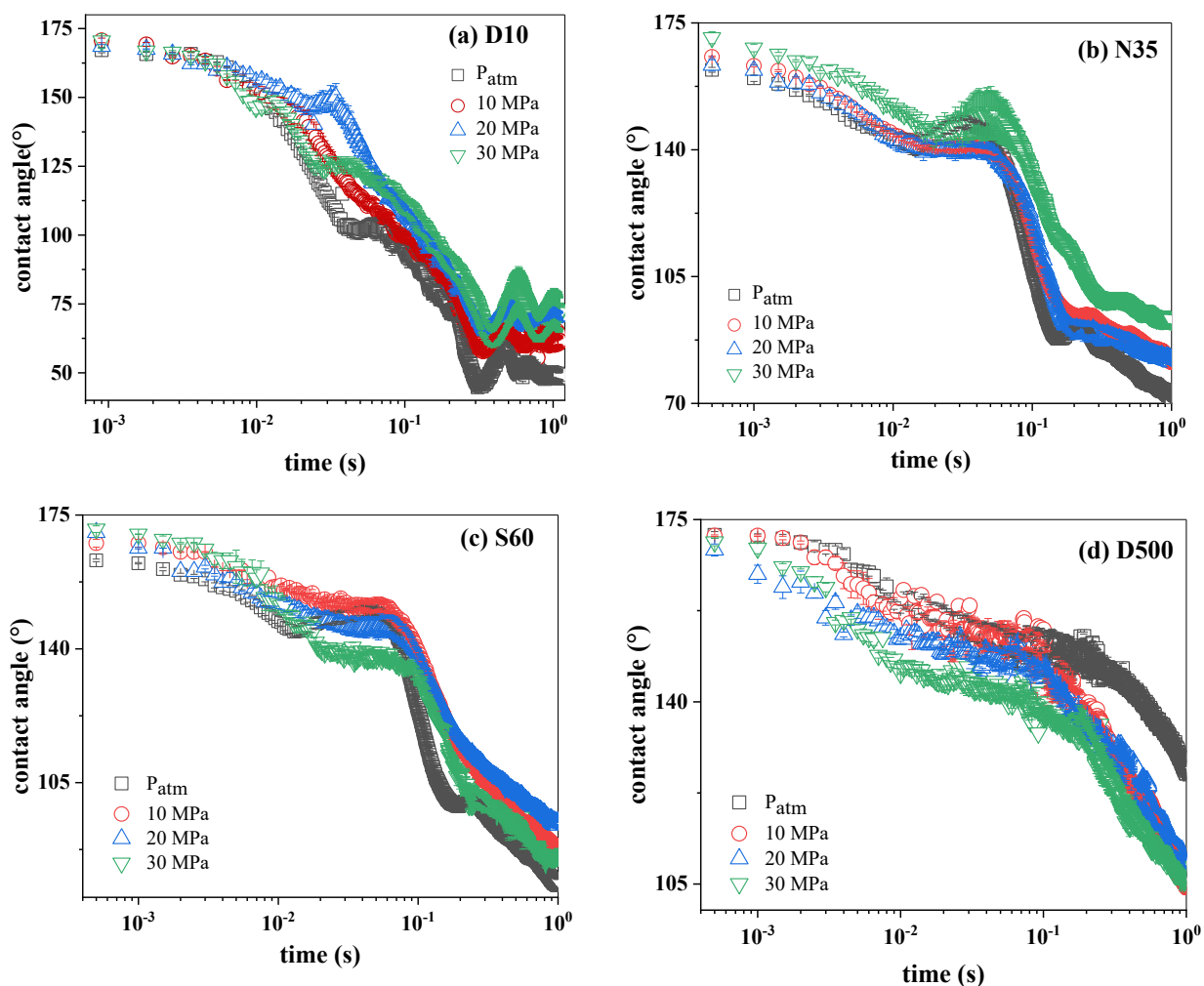


Figure 2.9 Initial contact angle as a function of time for different grades of silicone oil droplets at various chamber pressures. (a) D10 (b) N35 (c) S60 (d) D500

The findings of this study can be applied to the spreading mechanics of supercritical CO<sub>2</sub> in deep porous reservoirs used for curbing atmospheric emissions [34]. At these reservoirs depths, the CO<sub>2</sub> in its supercritical state behaves like a liquid with the laws of fluid mechanics governing its motion. In addition, the spreading phenomenon takes place with an exceedingly high pressure being applied by the surrounding brine, which is typically found in depleted reservoirs. Since capillarity effects play a large role at the pore scale of these aquifers, our research findings can thus be used in estimating the timescales for both lateral and vertical spreading of the injectate at high pressure.

It would be interesting to further investigate the initial spreading using CO<sub>2</sub> or in a viscous outer medium at different surrounding pressure condition.

## 2.5 Conclusion

In this study, we investigate the early time spreading dynamics of different grades of silicone oils on PTFE at various surrounding pressures. Experiments show that the drops of intermediate viscosity cover a smaller wetting area and attains a pronounced spherical shape with the increase of ambient pressure although pressure effects are less prominent on the advancement of the contact line in spreading for high viscosity oils. In addition, a rise in viscosity produces similar effects on the spreading dynamics: the instantaneous spreading radius and equilibrium radius both fall on account of higher viscous dissipation. We observe that the inertia dominated early-stage spreading consists of two regions: a slow starting regime that occurs at Phase I after the drops make contact with the substrate and Phase II is the fast-paced dynamics which materializes before the transition regime. In the first region, the curves of spreading radius are splintered and the degree of this splintering is largely dependent on pressure. As time passes, the splintered curves collapse onto a master curve with varying levels of magnitude. High velocity and a large empirically obtained exponent characterize the cross over to the second regime at the early stages. The exponent,  $\alpha$ , falls with rising pressure and exhibits the largest change for low viscosity oils. With increasing pressure and viscosity, the transition time and transition radius both fall, again with noteworthy significance for the lowest viscosity silicone oil used. Similar to the results of  $\alpha$ , pressure effects on the transition time, are negligible for highest viscous liquid (D500) whose entire spreading regime is viscosity dominated. We also conclude that contact line velocity is highly influenced by pressure, particularly in the first stage of the early time dynamics where a rise in pressure significantly lowers the spreading velocity. In the second stage of the early inertial

regime, the curves of velocities collapse upon each other. The fall in velocities is due to increased viscous drag stemming from contact line dissipation. Finally, the contact line dynamics at the early stages are distinguished by high values of the contact angle and spreading velocities. For future work, initial spreading studies can be further investigated to include a high viscosity liquid surrounding medium and can be conducted at elevated substrate or surrounding temperatures.

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### **3 Dynamics of spreading of a hydrodynamically driven droplet under the influence of surrounding pressure**

#### **3.1 Abstract**

The dynamics of hydrodynamically driven water droplet is studied on four different substrates at elevated surrounding pressures experimentally and theoretically. By combining the extended Overall Energy Balance (OEB) approach and the Lucas empirical model for estimating drop viscosity at elevated pressures, the advancement of a liquid droplet on a solid substrate is predicted as it undergoes constant mass flux addition while maintaining a spherical cap. Theoretical model is validated with an experimental investigation of a droplet with the addition of mass overtime at atmospheric pressure and elevated pressures of 10 and 20 megapascal (MPa) above atmospheric pressure. The results of the pressure effects on the spreading droplet in terms of the advancing contact angle and spreading radius of water droplet are analyzed on four different substrates namely: Polytetrafluoroethylene (PTFE), Aluminium (Al), Copper (Cu), and Poly (methyl methacrylate) (PMMA). The results show that an increase in surrounding pressure lowers the spreading radius and simultaneously increases the advancing contact angle. In addition, the influence of pressure on the minimum spreading ratio,  $\xi$  is investigated for four different substrates. It is also observed that the average Reynolds number of the spreading droplet decreases in a monotone fashion while the Weber number demonstrates a rapid increase.

#### **3.2 Introduction**

Refining, extraction, refrigeration, polymer foaming, oil seeds pressing, polymerization, free meniscus coating, processing of dairy products, are some examples of high-pressure engineering [1]–[3]. Especially in the context of mitigating global climate change, high pressure wetting plays

a significant role in the large scale geological sequestration of supercritical CO<sub>2</sub> in geological reservoirs [1], [4]–[6]. Regardless of the application considered, the spreading behavior of various liquid droplets on chemically treated or naturally occurring substrates, such as porous rock, is important to analyze and understand in order to generate significant gains in efficiency in these processes. To know about the wetting characteristics of a liquid droplet on a solid surface, understanding the spreading behavior is crucial. Axisymmetric drop shape analysis (ADSA) is used to record a profile of a droplet placed on a given solid or semi-solid substrate [7]. The output parameters of this technique are drop volume, contact angle, and drop radius. Contact angle defines the limit of intermolecular interactions at the three different interfaces (solid-liquid, liquid-vapor and solid-vapor) and surface tension of a fluid and surface energy of a solid represent the magnitude of interactions [8]. Motivated by the concept of Young's equation, many techniques have been developed to measure contact angle. This thermodynamically established Young's equation relates the contact angle with the three interfacial tensions [9]. The model assumes that solid surfaces are homogeneous, smooth, chemically and physically inert. A unique contact angle is expected for a given system [10]. In a non-ideal case, the contact angle depends on the liquid advancing or receding on the solid surface instead of having a unique contact angle [11]. Contact angle measured at the advancing and the receding edges of a liquid droplet is the advancing contact angle ( $\theta_a$ ) and the receding contact angle ( $\theta_r$ ), respectively [12]. By dosing or aspirating the volume of a droplet in the sessile drop method, dynamic advancing and receding contact angles are measured.

A theoretical model to predict the spreading during the advancement of the contact line by adding mass was developed through a series of iterations [7]. The relationship between dynamic contact angle and the advancing speed of the three-phase contact line (TPCL) was established [13]–[16].

A number of studies have put forward simple and semi-empirical equations. It was carried out by relating the dynamic contact angle to the capillary number with the addition of a 'shift factor' [17]. The first general theoretical model on dynamic spreading of a droplet was proposed in [18] which is called the overall energy balance model (OEB). The author suggests that the internal viscous dissipation balances the change in the total surface and kinetic energy of the drop [18]. Using this model, the spontaneous and low speed drop spreading was studied at a later date, where the initial kinetic energy of a fixed mass droplet was considered as the primary driving force [19], [20]. The spreading of a droplet with a fixed volume can be described by the power law [21]. The power law is based on the assumption that the volume and radius of the droplet is smaller than the capillary length scale. For industrial processes though, e.g., polymer processing technologies and welding, the study of a drop spreading by adding liquid to the system is essential since the primary driving force is hydrodynamic in nature. Erickson et al. [7] successfully developed a model to study drop spreading driven by hydrodynamic forces by extending the OEB model by adding mass and energy to the system. The authors developed a non-linear first-order ordinary differential equation and used previous development in this area [19], [20] to show the drop spreading phenomenon using a numerical solution to an ordinary differential equation. Spreading behavior of a spherical droplet with a constant dynamic contact angle and constant mass flow rate for a given system can be predicted by this model [7]. To model the hydrodynamic spreading phenomenon under elevated pressure, we use the extended model of OEB [7] and combine it with the Lucas empirical equation of viscosity [22].

In this investigation, we observe the phenomenon of hydrodynamic spreading of a drop at different surrounding pressure conditions, such as atmospheric and 10 megapascal (MPa), and 20 MPa above atmospheric condition. In particular, the spreading phenomenon is analyzed and discussed

during the time of advancement of a drop. Advancing contact angle, increase of volume over time, and spreading radius of water on four different substrates, i.e., Polytetrafluoroethylene (PTFE), Poly (methyl methacrylate) (PMMA), Copper (Cu) and Aluminum (Al), is investigated experimentally. Furthermore, by analyzing the variation of different non-dimensional numbers with pressure we study the influence of the drag force and cohesive force during spreading at variable pressure points. We also propose a theoretical model to predict the spreading behavior of a hydrodynamically driven droplet and compare it with the experimental values to investigate the validity of the model. We investigate theoretically and experimentally the combined effects of changes in physical properties of advancing droplets and the equally nuanced changes in wetting behavior of various solid substrates on spreading dynamics with respect to elevated surrounding pressures.

### 3.3 Theoretical model

The OEB approach describes the spreading of a droplet while considering the continual addition of mass until it reaches the equilibrium position. During the spreading process, the steady addition of liquid to the system serves as the primary driving force. The energy influx to the droplet is transformed into the internal energy of the system, surface and gravitational potential energies, boundary movement work, and work done against viscous dissipation. Thus, the OEB equation can be presented as follows [7]:

$$\frac{dE_{in}}{dt} = \frac{d}{dt} (E_{sys} + E_s + E_g) + \frac{d}{dt} (W_v + W_b) \quad (1)$$

where  $E_{in}$  is the energy entered to the system,  $E_{sys}$  is the internal energy of the system,  $E_g$  is the gravitational potential energy,  $E_s$  is the surface energy,  $W_v$  is the viscous dissipation work, and  $W_b$  is the boundary movement work. This is the generic form of the governing equation to describe

the spreading behavior of the droplet. In this case, the system is considered to be in thermal equilibrium with its surrounding and the internal heat generation is negligible. Detailed derivations of each term of equation (1) are given below.

### 3.3.1 Energy entered to the system

Due to the change of mass ( $\Delta m$ ), the energy transferred to the system can be defined by the enthalpy and kinetic energy of the liquid entering to the system [23]. Thus, the equation is as follows:

$$\frac{dE_{in}}{dt} = \frac{dm}{dt} \left[ h(T_0) + \frac{v^2}{2} \right] \quad (2)$$

where  $h(T_0)$  is the enthalpy evaluated at the system in equilibrium temperature  $T_0$ ,  $v$  is the entrance velocity. The function  $h(T_0)$  can be presented as follows:

$$h(T_0) = u(T_0) + \frac{P}{\rho} \quad (3)$$

where,  $u(T_0)$  is the internal energy of the system,  $P$  is the entrance absolute pressure of the fluid and  $\rho$  is the density of the fluid.  $u(T_0)$  is a function of pressure and temperature but the dependency is considered to be negligible as the fluid we considered in our case is incompressible (water). The droplet is assumed to be in spherical shape and the pressure difference throughout the droplet is uniform.  $P$  can be expressed as the sum of the Laplace pressure and atmospheric pressure ( $P_0$ ). This yields:  $P = \frac{2\sigma_{lv}}{R} \sin(\theta_a) + P_0$  where  $\theta_a$  is the dynamic contact angle and  $\sigma_{lv}$  is the surface tension in the liquid-vapor interface and  $R$  is the radius of the droplet. Finally, the equation can be written as:

$$\frac{dE_{in}}{dt} = \frac{dm}{dt} \left[ u(T_0) + \frac{1}{\rho} \left( \frac{2\sigma_{lv}}{R} \sin(\theta_a) + P_0 \right) + \frac{v^2}{2} \right] \quad (4)$$



### 3.3.2 Internal energy of the system

Total internal energy of the system is the sum of the internal energy and kinetic energy. By integrating the velocity profile over volume, kinetic energy can be determined. The surface area of the sessile drop is considerably larger than the area of the liquid jet and by doing order of magnitude analysis we observe that this internal kinetic energy can be ignored.

The kinetic energy entering to the system ( $E_{in,k}$ ) with the indiscernible change of time can be written as [7]:

$$\frac{dE_{in,k}}{dt} = \frac{v^2}{2} \frac{dm}{dt} \quad (5)$$

where  $v$  is has the order of  $\dot{m}/\rho A_i$ .  $A_i$  is the entrance area and considering  $\dot{m} = \frac{dm}{dt}$ , we get the following expression for the order of magnitude analysis:

$$\frac{dE_{in,k}}{dt} = O \left[ \frac{\dot{m}^3}{2A_i^2 \rho^2} \right] \quad (6)$$

By continuity, the velocity profile over a region in spherically growing droplet will remain constant since the fluid is added at a constant flowrate [7]. With this assumption we get the following internal kinetic energy equation for the drop ( $E_{sys,k}$ ) from the change in the kinetic energy equation [7]:

$$\frac{dE_{sys,k}}{dt} = \int_V^{V+\Delta V} \rho \frac{v^2}{2} dV - \int_0^V \rho \frac{v^2}{2} dV = \int_V^{V+\Delta V} \rho \frac{v^2}{2} dV \quad (7)$$

where  $V$  is the drop volume. Here,  $v$  is the region formed by the addition of mass with the order of  $\dot{m}/\rho A_{lv}$ . So, the equation (7) can be written as:

$$\frac{dE_{sys,k}}{dt} = O \left[ \frac{\dot{m}^2}{2A_{lv}^2 \rho} \Delta V \right] \quad (8)$$

Considering  $\frac{dV}{dt} = \frac{\dot{m}}{\rho}$  equation (7) becomes,

$$\frac{dE_{sys,k}}{dt} = O \left[ \frac{\dot{m}^3}{2A_{lv}^2 \rho^2} \right] \quad (9)$$

Comparing equation (6) and (9) we found that,

$$\frac{dE_{in,k}}{dt} / \frac{dE_{sys,k}}{dt} = O \left[ \frac{A_{lv}^2}{A_i} \right] \quad (10)$$

This suggest that the area through which the fluid enters is smaller than the surface area of the droplet. This means the kinetic energy of the fluid entering the drop will dominate and the internal kinetic energy can be ignored for spherically growing droplet. As a result, only the internal energy,  $u$  will be considered for hydrodynamically driven spreading. Considering the temperature to be constant, total internal energy will change with the change of mass as follows [7]:

$$\frac{dE_{sys}}{dt} = [(m + \Delta m)u(T_0) + u(T_0)m] \quad (11)$$

By considering the change of mass with respect to time is minimal, the equation can be written as:

$$\frac{dE_{sys}}{dt} = \left[ \frac{dm}{dt} u(T_0) \right] \quad (12)$$

### 3.3.3 Surface potential energy

The total surface energy of the system can be defined by considering the surface energies of the three interfacial phases, i.e., solid-liquid ( $\sigma_{sl}$ ), liquid-vapor ( $\sigma_{lv}$ ) and solid-vapor ( $\sigma_{sv}$ ) [7]. The equation can be written as:

$$E_s = \sigma_{sl}A_{sl} + \sigma_{sv}A_{sv} + \sigma_{lv}A_{lv} \quad (13)$$

where  $A_{sl}$ ,  $A_{sv}$ ,  $A_{lv}$  the area of solid-liquid, liquid-vapor and solid-vapor interface. The change in solid-vapor and solid-liquid surface areas are equal. Differentiating the surface potential energy with respect to time, we get the following equation [8]:

$$\frac{dE_s}{dt} = (\sigma_{sl} - \sigma_{sv}) \frac{dA_{sl}}{dt} + \sigma_{lv} \frac{dA_{lv}}{dt} \quad (14)$$

Young's equation for homogeneous, smooth and rigid surface can be written as follows:

$$\cos \theta_e = \frac{(\sigma_{sv} - \sigma_{sl})}{\sigma_{lv}} \quad (15)$$

The surface tensions of equation (14) can be related using Young's equation but the equilibrium contact angle ( $\theta_e$ ) mentioned here is not equivalent to  $\theta_a$ . Using the spherical shape assumption and considering the drop to be axisymmetric, change in the solid-liquid, liquid-vapor areas can be written as follows [7]:

$$\frac{dA_{sl}}{dt} = 2\pi R \frac{dR}{dt} \quad (16)$$

$$\frac{dA_{lv}}{dt} = 4\pi R h(\theta_a) \frac{dR}{dt} + 2\pi R^2 \frac{dh(\theta_a)}{dt} \quad (17)$$

where  $h(\theta_a)$  is the drop height and is expressed as:

$$h(\theta_a) = \frac{1 - \cos \theta_a}{\sin^2 \theta_a}$$

From equation (14), (15), (16) and (17) we get the following equation for the total change in the surface potential energy:

$$\frac{dE_s}{dt} = 2\pi R\sigma_{lv}[2h(\theta_a) - \cos\theta_e] \frac{dR}{dt} + 2\pi R^2\sigma_{lv} \frac{dh(\theta_a)}{dt} \quad (18)$$

### 3.3.4 Gravitational potential energy

Change in the gravitational potential energy for change of mass depends on the total mass of the system and change in the height of the center of gravity ( $\Delta z$ ). From [7] we get the gravitational potential energy equation as follows:

$$\frac{dE_g}{dt} = g \left[ m \frac{dz}{dt} + z \frac{dm}{dt} \right] \quad (19)$$

Expression for the center of mass ( $z$ ) for spherical shape droplet is:

$$z = \frac{R}{4} f(\theta_a)$$

$$\text{where } f(\theta_a) = \frac{6 \sin^2\theta_a - 3(1 - \cos\theta_a)^2 - 8(1 - \cos\theta_a) \cos\theta_a}{(2 - \cos\theta_a) \sin\theta_a}$$

The total mass of the system ( $m$ ) can be expressed by the following equation:

$$m = m_o + \int_0^t \frac{dm}{dt} dt$$

$$m_o \text{ is the initial mass of the system and } m_o = \frac{\rho\pi R_o^3(1 - \cos\theta_a)^2(2 + \cos\theta_a)}{3\sin^3\theta_a}$$

Combining all this, equation (19) becomes,

$$\frac{dE_g}{dt} = mg \left( \frac{1}{4} f(\theta_a) \frac{dr}{dt} + \frac{R}{4} \frac{df(\theta_a)}{dt} \right) + \frac{R}{4} g f(\theta_a) \frac{dm}{dt} \quad (20)$$

### 3.3.5 Viscous dissipation work

According to de Gennes [24], when hydrodynamic force is greater than the molecular force and  $Re$  is less than 1, lubrication approximation is used to model viscous dissipation work which is

true in our case. So, viscous force per unit length of the three-phase line in our case can be written as:

$$F_t = \frac{3\mu}{\theta_a} \ln(\varepsilon^{-1}) \frac{dR}{dt} \quad (21)$$

where  $\varepsilon = L_1/L$  which represents the ratio of the microscopic to macroscopic cut-off lengths and  $\mu$  is the viscosity of the liquid.  $L$  is proportional to  $R$  and  $L_1$  will vary between 1 to 5  $\mu\text{m}$  [25]. The viscous dissipation work per unit time is  $2\pi R F_t v$  and can be written as:

$$\frac{dW_v}{dt} = 6\pi \frac{R\mu}{\theta_a} \ln(\varepsilon^{-1}) \left(\frac{dR}{dt}\right)^2 \quad (22)$$

### 3.3.6 Boundary movement work

Because of the change of drop volume, the drop grows. So, the work is done due to the displacement of the atmospheric pressure. Considering the change in volume with respect to time against the atmospheric pressure, the equation for boundary movement work ( $W_b$ ) can be written as:

$$\frac{dW_b}{dt} = \frac{P_0}{\rho} \frac{dm}{dt} \quad (23)$$

By combining all this forces, the final governing equation can be obtained which relates the speed of the advancing contact radius to the material properties and the dynamic contact angle. The equation will be as follows:

$$\begin{aligned} & 6\pi \frac{R\mu}{\theta_a} \ln(\varepsilon^{-1}) \left(\frac{dR}{dt}\right)^2 + [2\pi R \sigma_{lv} [2h(\theta_a) - \cos \theta_e] + \frac{mg}{4} f(\theta_a)] \frac{dR}{dt} + \left[\frac{Rg}{4} f(\theta_a) - \right. \\ & \left. \frac{2\sigma_{lv}}{\rho R} \sin(\theta_a) - \frac{v^2}{2}\right] \frac{dm}{dt} + 2\pi R^2 \sigma_{lv} \frac{dh(\theta_a)}{dt} + \frac{R}{4} \frac{df(\theta_a)}{dt} mg = 0 \end{aligned} \quad (24)$$

Considering the  $\theta_a$  constant and  $\frac{dm}{dt} = \dot{m}$ , equation (24) can be simplified as:

$$\begin{aligned}
6\pi \frac{R\mu}{\theta_a} \ln(\varepsilon^{-1}) \left(\frac{dR}{dt}\right)^2 + \left[ 2\pi R \sigma_{lv} [2h(\theta_a) - \cos \theta_e] + \frac{(m_o + \dot{m}t)g}{4} f(\theta_a) \right] \frac{dR}{dt} \\
+ \left[ \frac{Rg}{4} f(\theta_a) - \frac{2\sigma_{lv}}{\rho R} \sin(\theta_a) - \frac{v^2}{2} \right] \dot{m} = 0
\end{aligned} \tag{25}$$

The non-dimensionalized form of equation (25) is obtained by considering the initial radius ( $R_o$ ) and initial velocity ( $v_o$ ) at the first instance as the characteristic length and velocity respectively. Considering  $\beta = R/R_o$  and  $t^* = t/(R_o/v_o)$  the resulting equation after non-dimensional analysis is then as follows:

$$\begin{aligned}
12 \frac{R^*}{\theta_a Re C(\theta_a)} \ln(\varepsilon^{-1}) \left(\frac{dR^*}{dt^*}\right)^2 \\
+ \left[ 4 \frac{R^*}{C(\theta_a) We} [2h(\theta_a) \right. \\
\left. - \cos \theta_e] + \frac{1}{6} \frac{Bo}{C(\theta_a) We} g'(\theta_a) f(\theta_a) + \frac{Bo}{2We} t^* f(\theta_a) \right] \frac{dR^*}{dt^*} \\
+ \left[ \frac{R^*}{2} f(\theta_a) \frac{Bo}{We} - \frac{4}{R^* We} \sin(\theta_a) - 1 \right] = 0
\end{aligned} \tag{26}$$

where  $g'(\theta_a) = (2 - 3\cos\theta_a + \cos 3\theta_a)/\sin\theta_a$  and  $C(\theta_a) = 2(1 - \cos\theta_a)/\sin^2(\theta_a)$ . Here,  $Re = \rho v R_o/\mu$  is the Reynolds number,  $We = \rho v^2 R_o/\sigma_{lv}$  is the Weber number and  $Bo = \rho R_o^2 g/\sigma_{lv}$  is the Bond number. To determine the change of  $\mu$  with respect to pressure we use the viscosity equation proposed by Lucas [22] which suggests that change of viscosity with increasing pressure can be estimated using:

$$\frac{\mu}{\mu_{SL}} = \frac{1 + D(\Delta P_r/2.118)^A}{1 + C\omega\Delta P_r} \tag{27}$$

where,  $\mu$  = liquid viscosity at the pressure of interest  $P$ ,  $\mu_{SL}$  = liquid viscosity of the saturated liquid at the temperature of interest  $T$ ,  $\Delta P_r = [(P - P_{VP})/P_C]$ ,  $P_{VP}$  = saturated vapor pressure at

the temperature of interest  $T$ ,  $P_C$ = critical pressure,  $\omega$ = is the acentric factor,  $A = 0.9991 - [4.674 \times 10^{-4}/(1.0523 T_R - 0.3877 - 1.0513)]$ ,  $D = [0.3257/(1.0039 - T_R^{2.573})^{0.2906}] - 0.2086$ ,  $C = -0.07921 + 2.1616T_R - 13.4040T_R^2 + 44.1706T_R^3 - 84.8291T_R^4 + 96.1209 T_R^5 - 59.8127 T_R^6 + 15.6719 T_R^7$  and  $T_r$ = reduced temperature ( $=T/T_C$ ).

The governing equation (26) is solved numerically by using Runge–Kutta (RK-4) method with appropriate initial conditions, such as the drop radius ( $R_o$ ) or the initial velocity ( $v_o$ ) at the first instance of spreading. Additionally, the values of dynamic advancing contact angles ( $\theta_a$ ) and the equilibrium contact angle ( $\theta_e$ ) are kept fixed for a specific surrounding pressure condition and are subsequently obtained from the experiments. While solving the equation (26), the  $\theta_a$ ,  $\theta_e$ , and the mass flux are known from the experiment. The dynamic contact angle remains unchanged during the entire drop deposition process for a specific surrounding pressure condition. Additionally, the mass flow rate is considered to be  $1.67 \times 10^{-10} m^3/s$  for the numerical results which are plotted and described in detail in result and discussion section.

### 3.4 Experimental setup and procedure

The schematic of the experimental setup is shown in Figure 3.1. The essential parts of the experimental arrangement include the pressurization unit and the imaging system, based on a commercial goniometer (DSA100HP, KRÜSS Scientific Instruments Inc.) with additional modifications made in-house to accommodate experimental conditions and parameters.

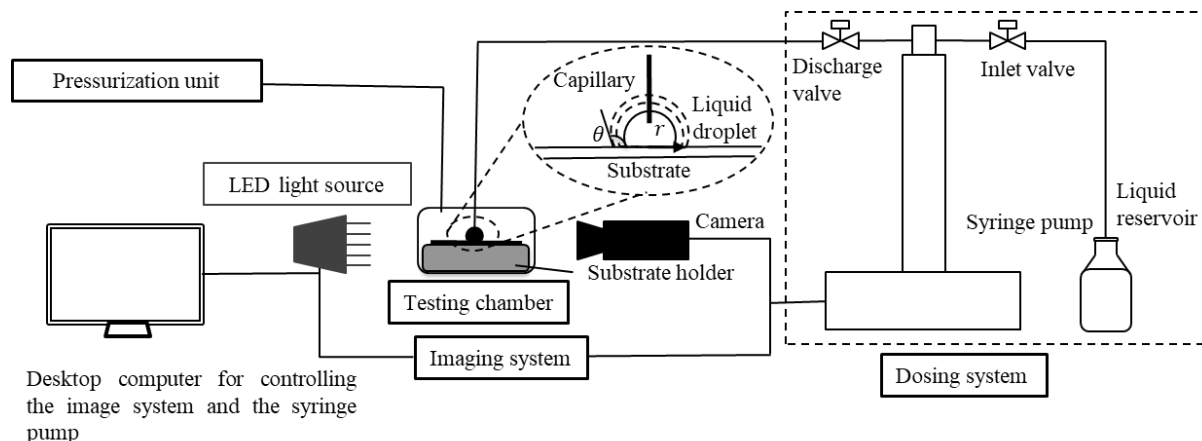


Figure 3.1 Schematic of the experimental setup to investigate the spreading of hydrodynamically driven droplets at high pressure. The figure is not drawn to scale.

The pressurization unit of a system consist of a high-pressure viewing cell (380; Eutotechnica GmbH, Bargteheide, Germany) with two sapphire windows rated to 75 MPa and 200 °C, a gas booster pump (393, Eutotechnica GmbH, Bargteheide, Germany), nitrogen gas cylinder, high pressure syringe pump (100DM, ISCO), and a control panel for flow management and distribution, built in-house. A commercial high pressure syringe pump (100DM, ISCO) controlled through in-house developed software using ANSI C programming environment (LabWindows/CVI, National Instruments) is used for continual mass addition to the sessile droplet. The software allows us to control the mass flow rate in addition to other parameters during the dosing/dispensing process at atmospheric and pressurized condition. This pump has the capacity to add volume in a drop with a constant flow rate (CFR). The drop volume is increased over time using CFR at atmospheric and pressurized conditions, such as gauge pressure of 10 and 20 MPa. In order to support substrates and keep them horizontally inside viewing cell, a customer made 3D printed substrate holder was designed and manufactured. In addition, a movable needle/ capillary of 1.1 mm outer diameter is used to control the distance between a substrate and a drop. The advantage of overhead dosage for advancing in relation to puncturing the substrate for adding mass as done by Erickson et al. [7] is that it does not adversely affect the wetting characteristics of the substrate.



Camera (Phantom V711, Vision Research Inc.) mounted on a tripod base (Manfrotto model 475B Pro Geared Tripod) and a tripod head (Manfrotto 229 3D Super-Pro Head) is used to record experiment at 60 fps frame rate and exposure time of 40  $\mu$ s. The LED light kit (Nila Zaila - Daylight v2) is used to provide a daylight-balanced light source and amplify the image quality. The magnification was set in a way that two third of the sensor's field of view is occupied by the generated droplet. Capillary outer diameter is used to calibrate the image magnification. A 1024 x 768-pixel field of view (FOV) is used for all the experiments. Additionally, the precision in measuring the contact angle is  $\pm 0.1^\circ$  and for the spreading radius is  $\pm 0.01$  mm in all experiments. The commercial axisymmetric drop shape analysis software (ADVANCE, KRÜSS Scientific Instruments Inc.) is used to determine the contact angle, volume of the droplet and change in a drop diameter over time.

Commercially available Polytetrafluoroethylene (PTFE) (8545K201, McMaster-Carr), 6061 grade Aluminium (Al) (89015K22, McMaster-Carr), Copper (Cu) (8963K183, McMaster-Carr), Poly (methyl methacrylate) (PMMA) (8589K11, McMaster-Carr) substrates are used as solid substrate to conduct experiments. The substrates were cut to size of 37 x 25.5 mm with 1 mm in thickness. Using commercial sonicator (model 1510, Branson Ultrasonics Corp., Danbury, CT, USA), at first the substrates are sonicated in isopropanol (>98% Food Grade, Sigma-Aldrich, Oakville, ON, Canada) for 15 minutes followed by 15 minutes in deionized distilled water sourced from water purification system (Milli-Q A10, Millipore). At the end of the cleaning procedure they were dried using the compressed air. Deionized water from the same source is used as a drop liquid to study the effect of surrounding pressure on a hydrodynamically driven droplets.

Experiments were carried out at atmospheric condition and at 10 and 20 MPa above atmospheric condition. The pump was rinsed with ethyl alcohol (99.5%, ACS reagent, ACROS Organics) and

then the whole system was flushed for several times. After that the pump was filled with water to execute the experiment. Surface tension of the water ( $72.2 \pm 0.17$  mN/m) was measured prior to each test to verify if there was any other agent present prior to each test inside the hydraulic components of experimental setup. Substrates were placed on the substrate holder inside the chamber. After that the chamber was closed using the sapphire windows on both sides. Needle was positioned approximately 1 mm away from the substrate. For the experiments at atmospheric conditions, a 20  $\mu\text{L}$  volume drop was initially generated using the remote-control software. Then, the liquid was aspirated from the pump at constant flow rate of 10  $\mu\text{L}/\text{min}$  until an additional 35  $\mu\text{L}$  were added to the system. During the addition of drop volume, a constant contact angle value was recorded, named advancing angle with the evolution of base radius of the droplet. Both static and advancing contact angles were recorded. In case of the experiments at high pressure, the testing chamber was pressurized first to the required level and after that drop deposition process was done. In order to generate accurate and repeatable set of data and minimize human errors, drop generation process was carried out through software. To ensure repeatability there were at least three trials taken at each condition of the experiment.

### **3.5 Results and discussion**

Hydrodynamic advancing contact angle is measured in this study to characterize the wettability of the substrates in the presence of the nitrogen gas. Four different substrates are used for measuring contact angle and contact radius with the increase of the volume over time. The water injection is carried out by using a syringe pump with the flow rate of 10  $\mu\text{L}/\text{min}$ . The main purpose of this study is to investigate the variation in the advancing contact angle, contact line spreading with the change of pressure on four different substrates. Contact angle and drop radius were recorded throughout the entire injection process. Drop radius starts to increase while the contact angle

named advancing angle remain constant, for the entire dispensing process, as shown in Figure 3.2. In each case study, the time was set to zero when the dispensing process was triggered. As it can be seen, the contact angle remains fairly constant through the entire process and the difference in advancing angles is recorded not only for between substrates but even at different set points for pressure within the same system, *i.e.*, PTFE, Al, Cu, and PMMA.

The small dips that are apparent in the time evolution plots of the contact angle are due to localised surface defects which exhibit a higher degree of hydrophilic behavior than the bulk of the surface. In these regions, the advancing interface is stretched momentarily as a result of high surface energy which then causes the contact angle to fall. In addition to hydrophilic defects, pinning while advancing also takes place to varying degree in certain substrates, particularly in Cu and PMMA. Here, while the triple point is pinned, the contact angle rises until the point where the newly established interface breaks down permitting further advance. As evident from tiles (a)-(d) in Figure 3.2, increasing the surrounding pressure increases the contact angle in a nonlinear fashion with the greatest change taking place for PTFE following a pressure change of 20 MPa. Stated differently, increasing the surrounding pressure facilitates rapid dewetting of the water droplet from the underlying substrate. By contrast, a pressure increase from 10 MPa to 20 MPa has a weaker influence on the wetting characteristics for PMMA and consequently produces no noticeable change in the contact angle. This suggests that the wettability remains largely unaltered, as a result of this change in surrounding conditions. The spreading radius on the other hand demonstrates a consistent monotone increase as time evolves. Hence the drop advancing largely takes place with constant contact angle growth. We observe that as the surrounding pressure increases, the contact radius falls. For Al, the spreading radius values converge near the end of the

advancing stage in the  $P_{\text{atm}}$  and 10 MPa surrounding pressure cases. In a similar fashion, the spreading radius curves coalesce at the end for Cu for pressures valued at 10 MPa and 20 MPa.

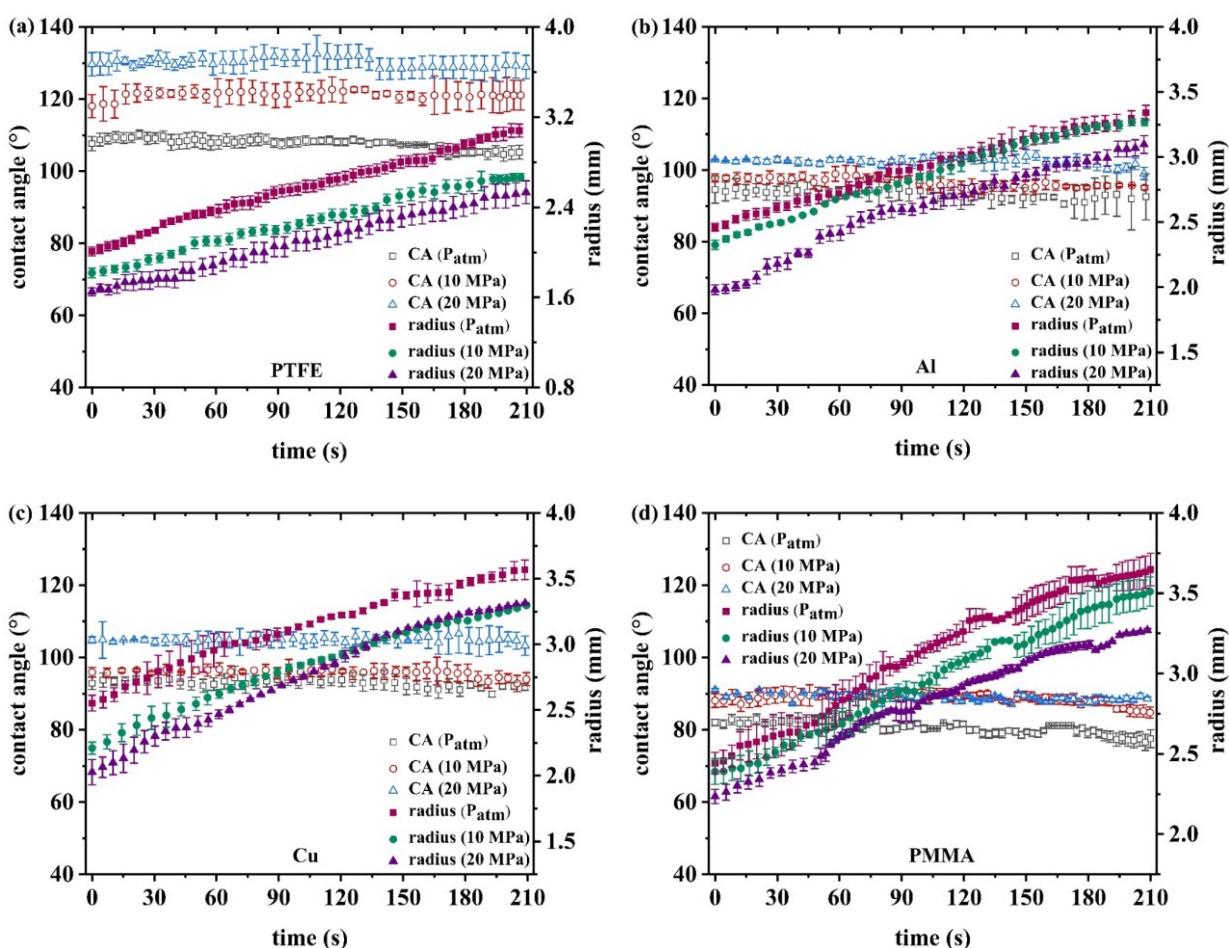


Figure 3.2 Advancing contact angle measurement with the evolution of radius of water droplet on (a) PTFE, (b) Al, (c) Cu, and (d) PMMA at  $P = P_{\text{atm}}$ , 10 MPa and 20 MPa.

The variation of the initial contact angle and minimum spreading ratio,  $\xi$  as a function of the surrounding pressure for different substrates is analyzed in Figure 3.3.  $\xi$  is defined as the ratio of the initial droplet diameter that is established upon immediate contact with the substrate to the needle diameter used for deposition. As evident from Figure 3.3 (a), the initial contact angle is an increasing function with pressure for a specific substrate. This is because elevated pressures suppress lateral spreading, leading to a marked decrease in wettability as the medium pressure is raised. Furthermore, an increase in the pressure of the surrounding fluid implies that the droplet

must overcome additional boundary movement work in order to establish a spherical cap during and after its separation from the needle. Therefore, surface free energy of the substrates falls with rising pressure causing them to experience a rise in their hydrophobicity. This proliferation in a substrate's hydrophobic behavior is most pronounced for PTFE with its initial contact angle incurring a 20.4% increase in value as pressure is raised from atmospheric conditions to 20 MPa. In contrast, Al demonstrates the least sensitivity of surface energy with respect to pressure, suffering an 8.84% increase in initial contact angle while the surrounding pressure is elevated by 20 MPa. Stated differently, low energy substrates experience the largest change in wettability in relation to medium pressure.

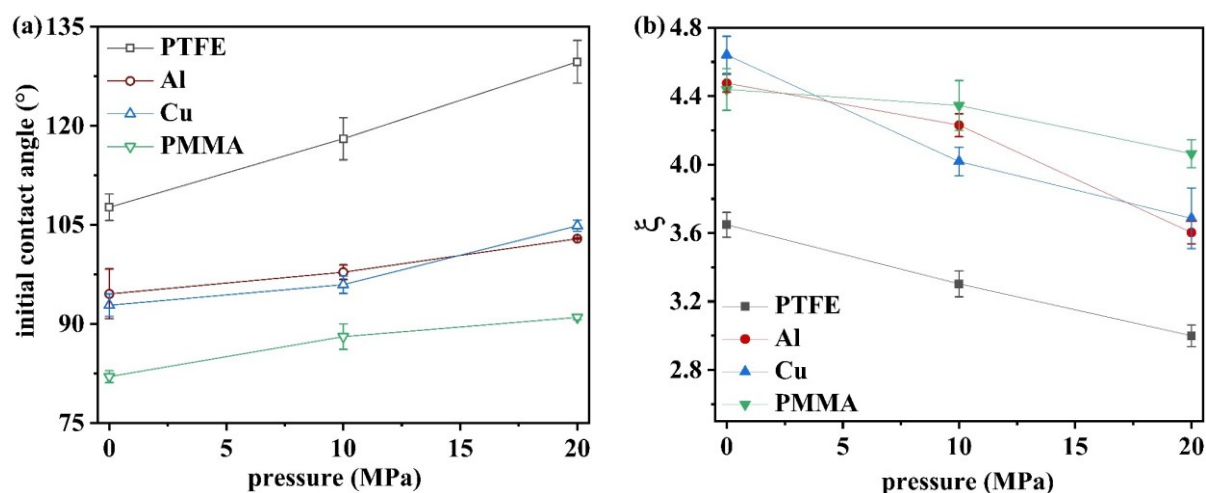


Figure 3.3 Change of initial contact angle (a) and minimum spreading ratio,  $\xi$  (b) as a function of the surrounding pressure

To further investigate the role of pressure on the dynamics immediately following drop deposition, we plot the corresponding change in  $\xi$  in Figure 3.3 (b). Firstly, we observe that  $\xi$  decreases in a monotone fashion for all four substrates, as pressure is increased. This phenomenon of falling  $\xi$  with rising pressure closely resembles the effects witnessed in Figure 3.3 (a). In other words, there is an inverse relationship between the minimum spreading ratio and the initial contact angle in that they are negatively correlated. The implication is that a fall in wettability (increasing initial contact

angle) that is accompanied with rising pressure limits the minimum spreading ratio and vice versa. Additionally, the extent to which  $\xi$  incurs a fall in with elevated pressure varies between different substrates. We observe that Cu experiences the largest change in  $\xi$  – a 20.6% decrease in value when pressure increases by 20 MPa. On the other hand, PMMA is impacted the least with an 8.5% fall in value of  $\xi$  with a similar rise in pressure. Because the initial volume of the deposited droplet is kept constant between trials, the combined effects of a growing initial contact angle and decreasing minimum spreading ratio stemming from low wettability results in a more pronounced spherical cap at the start of the spreading process.

We compare the advancing,  $\theta_a$  to equilibrium contact angle,  $\theta_e$  at different atmospheric conditions where one can find an increase in equilibrium contact angle,  $\theta_e$  for each substrate as atmospheric pressure goes up, as it is shown in Figure 3.4. In case of PTFE, there is a significant increase in angle from  $106^\circ$  to  $128^\circ$  at  $P_{atm}$  and 20 MPa, respectively. Similarly, Al, Cu and PMMA showed similar behavior with increase in  $\theta_e$  approximately with  $10^\circ$  difference between  $P_{atm}$  and 20 MPa. As expected,  $\theta_a$  were recorded at higher values with approximately  $3^\circ$  over  $\theta_e$  values in each case study. This information can be used, as a guideline for material selection within high pressure applications, where wetting or de-wetting takes place.

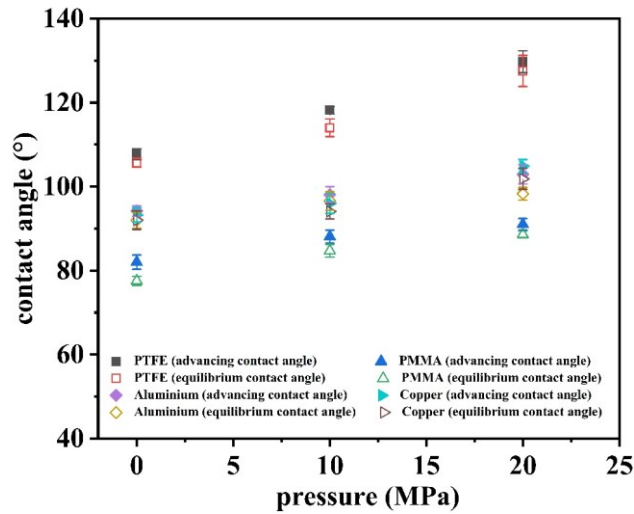


Figure 3.4 Comparison between advancing contact angle,  $\theta_a$  and equilibrium contact angle,  $\theta_e$  with different surrounding pressure for four different substrates.

The change in Reynold number ( $Re$ ) and Weber number ( $We$ ) is presented with respect to the change of the surrounding pressure, where  $Re = \rho v R_o / \mu$ ,  $We = \rho v^2 R_o / \sigma_{lv}$ , respectively. The initial radius ( $R_o$ ) and average velocity ( $v$ ) is considered as the characteristic length and velocity in this case. The average velocity ( $v$ ) is obtained from the average slope of the spreading curves in Figure 3.2 and  $\sigma_{lv}$  of water is obtained from [26] for different surrounding pressure conditions. The non-dimensional form of the pressure is used, which is referred as the reduced pressure,  $\Delta P_r = P/P_c$  ( $P_c$  is the critical pressure). The decrease in  $Re$  values is observed with the increase of surrounding pressure for PTFE, Al, Cu, and PMMA substrate, as shown in Figure 3.5. The reduction by 35% is recorded for the PTFE from  $\Delta P_r = 0$  to  $\Delta P_r = 0.5$  and only 12% reduction from  $\Delta P_r = 0.5$  to  $\Delta P_r = 1$ . For Al substrate, the value of  $Re$  reduces by 8% from atmospheric condition to  $\Delta P_r = 0.5$  and 38% from  $\Delta P_r = 0.5$  to 1. For Cu substrate and PMMA, from atmospheric to  $\Delta P_r = 1$ ,  $Re$  value reduces to an amount of 39% and 22%, respectively. On the other hand, the  $We$  number increases with the increase of the pressure, as shown in Figure 3.5 (b).

From atmospheric to  $\Delta P_r = 1$ , the exponential increase of the  $We$  is observed for all the cases. These changes are due to the changes of average velocity, initial radius, and the  $\sigma_{lv}$ , mainly due to pressure dependency. The values of  $We$  for PTFE and Al substrates are 34 times higher at  $\Delta P_r = 1$  than the value at  $\Delta P_r = 0$ . Similarly, we observe an increase in the value of  $We$  for Cu and PMMA substrates with the increase of pressure. This decrease in the  $Re$  number and increase in the  $We$  number indicate that the drag force on the droplet becomes more dominant over the cohesive force with the increase of pressure, as a result, significance of the viscous force over the inertia force increases during the spreading of the droplet. Stated differently, the decrease in the cohesive force between water molecules comes as a result of several orders of magnitude decrease in surface tension with increasing pressure. Furthermore, with increased viscous dissipation at the contact line coupled with added boundary movement work associated with rising pressure, the drag force rises significantly. These effects when combined cause a rise of the  $We$  with increasing pressure.

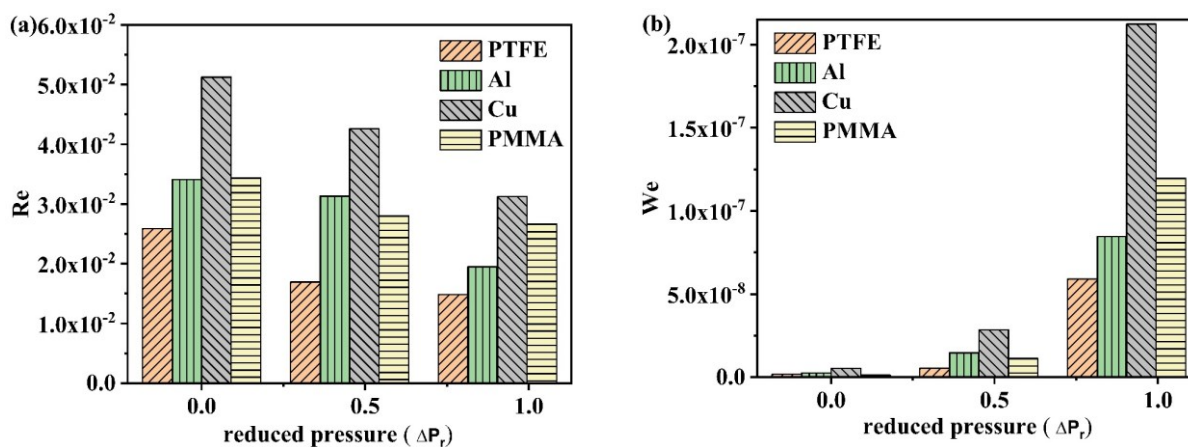


Figure 3.5 Change of the (a) Reynold number ( $Re$ ) and (b) Weber number ( $We$ ) as a function of reduced pressure ( $\Delta P_r$ ).

Results obtained from the theoretical model is compared with the experimental results performed in three different pressure conditions for water on PTFE, Al, Cu, and PMMA substrates. The



governing equation was solved using the fourth order Runge-Kutta technique [7]. The starting contact radius from Figure 3.2 are used as the initial conditions. To solve the equation, we assume that the dynamic contact angle remains constant during the entire procedure for a specific pressure condition and that the equilibrium contact angle is known. The effect of surrounding medium pressure as well as spreading behavior of a water droplet on PTFE, Al, Cu and PMMA substrate are considered in this model. A spherical cap assumption is invoked in estimating initial volume of the droplet, as well as, modelling the droplet growth over time. Additionally, the ratio of the microscopic to macroscopic cut-off length,  $\varepsilon$  is assumed to remain fixed at 0.0005.

Figure 3.6 demonstrates the comparison between the dimensionless time evolutions of the experimental versus numerical non-dimensional spreading factor,  $\beta$  of an advancing water droplet at a reduced pressure,  $\Delta P_r = 0, 0.5, \text{ and } 1$ . The model captures the complex effects of change in SFT,  $\theta_a$ , and  $\theta_e$  by considering the changes in  $E_s$  and  $W_v$ . Panels (a), (b), and (c) represent each reduced pressure setting. Each panel contains information for each substrate, namely PTFE, Al, Cu, and PMMA. To corroborate the results from the experiments with that of the theoretical ones, the analytical predictions are obtained at the reduced temperature,  $\Delta T_r = 0.5$ , which corresponds to 25 °C. Additionally, the volumetric flow rate for the experiment used is  $1.67 \times 10^{-10} \text{ m}^3/\text{s}$ . Generally, the spreading factor demonstrates a monotonic increasing trend with time as fresh injectate is continually added from the reservoir. This is observed in both the solution to the modified OEB as well the experimental data fitted over the entire range. With the exception of PMMA, other samples demonstrate close agreement between the experimental findings and the analytical predictions at atmospheric conditions, as shown in Figure 3.6 (a). Initially, the model is able to accurately predict the advancing motion of the droplet as the wetting perimeter propagates outward. This can be explained as follows, at early times, the wetting radius is modest and the

spherical cap assumption holds true. Not surprisingly, with a relatively small spreading radius, the mass and the Bond number ( $Bo$ ) of the droplet are quite small which then leads to gravitational effects being negligible. Over time, the wetting area increases, work done against surface forces becomes increasingly higher and consequently the spreading velocity of the drop falls. Simultaneously, the  $Bo$  of the droplet increasingly becomes larger as the drop expands. As the size of the drop surpasses a certain critical value, the model begins to underpredict the spreading radius and consequently the spreading velocity. The modified OEB model makes the assumption that the pressure field within the drop is uniform. This Laplace overpressure varies inversely with the contact radius whereas the hydrostatic pressure resulting from gravitational forces varies proportionately with the drop size. As the  $Bo$  increases, the Laplace pressure increasingly becomes insignificant in comparison with the gravitational pressure. Because of this, the rate of energy input on the left-hand side of the governing equation will be set too low in comparison with the actual value of  $dE_{in}/dt$ . Stated differently, the deviation from the spherical drop assumption, as the  $Bo$  crosses a critical threshold, causes the velocities to be underestimated by the current model. Furthermore, as  $Bo$  increases, the drop becomes flatter rather than being spherical which would cause the surface area growth to be underestimated. This deviation of the experimental data from the predicted findings is most prominent for PTFE in Figure 3.6 (a). Here, the fraction of time for which the model underpredicts the experimental results is approximately 0.32. Following closely with this deviation is PMMA, where the fraction of time spent undervaluing the empirical data is 0.23.

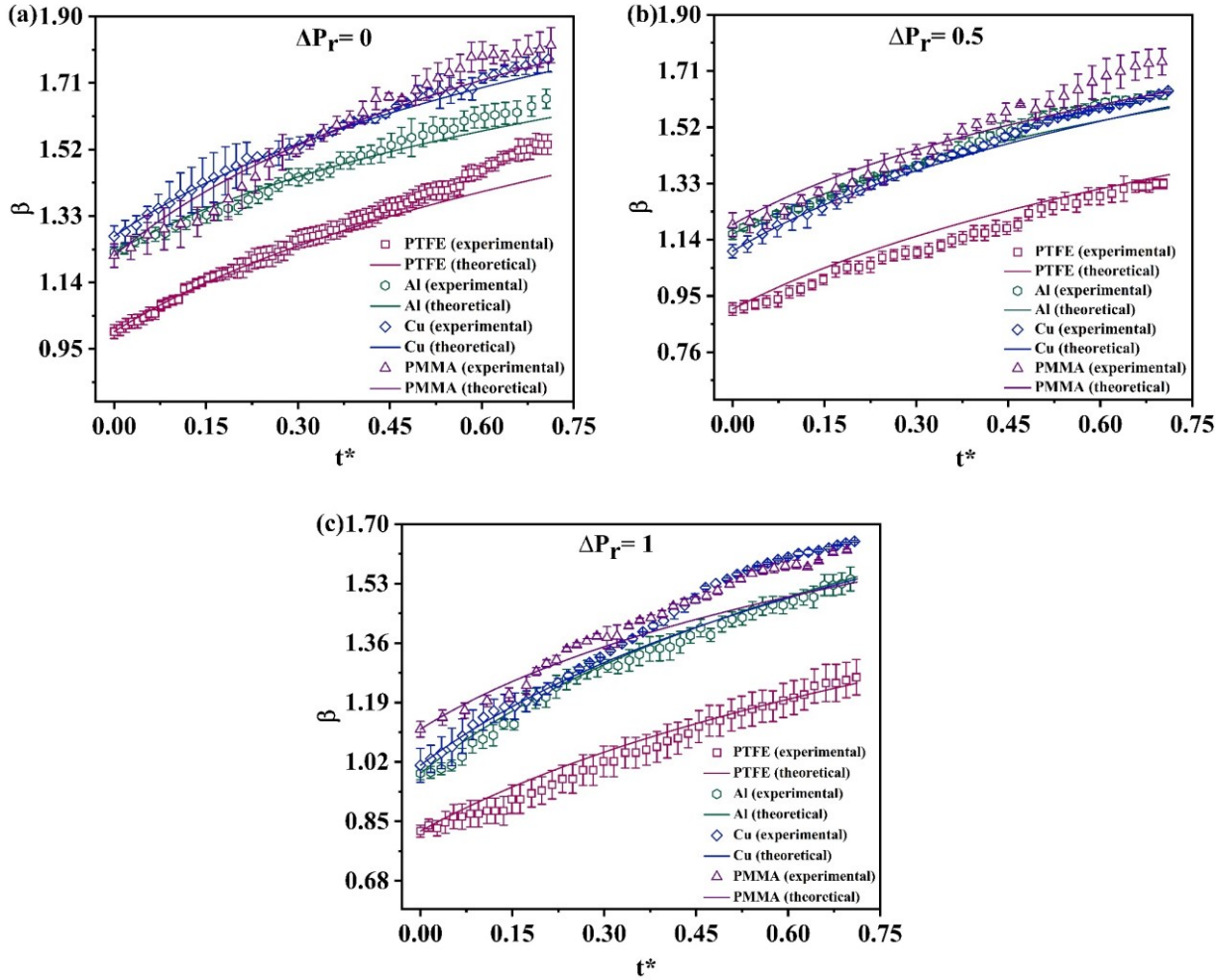


Figure 3.6 Temporal evolution of non-dimensional experimental and theoretical spreading factor ( $\beta$ ) of water on PTFE, Al, Cu, and PMMA substrate at (a)  $\Delta P_r = 0$ , (b)  $\Delta P_r = 0.5$  and (c)  $\Delta P_r = 1$ . The model prediction is performed at a reduced temperature,  $T_r = 0.5$  and ascetric factor,  $\omega = 0.344$  for water with an influx flow rate of  $1.67 \times 10^{-10} \text{ m}^3/\text{s}$ .

Similar to Figure 3.6 (a), Figure 3.6 (b) depicts the temporal evolution of  $\beta$  at a comparatively higher reduced pressure of  $\Delta P_r = 0.5$ . As the surrounding pressure increases, so does the work done in boundary movement. Additionally, with increase in pressure the fluid viscosities increase, which consequently increases the amount of energy lost from the system due to viscous dissipation. Here, the results of this effect have a direct impact of the time spent by model in overpredicting the results. This is particularly true for PTFE whereas the metals Cu and Al demonstrate little to no regions of overprediction. The additional loss in kinetic energy at elevated pressure can be

attributed to this change when compared to the results for the same materials at atmospheric conditions. On the other hand, for PMMA, the mean absolute deviation between the experimental and theoretical results falls when the pressure is comparatively larger than at atmospheric conditions whereas for PTFE the mean absolute deviation rises.

Analogous to the previous two plots, Figure 3.6 (c) illustrates the time evolution of the experimental and theoretical spreading radius at an even higher reduced pressure,  $\Delta P_r = 1$ . For PTFE, as the surrounding pressure is increased to 20 MPa, the standard deviation in the experimental results rises while the degree of overprediction in the model falls. Contrasting results for Cu at  $\Delta P_r = 0.5$  and 1, a rise in pressure results in a significant degree of underprediction by the model during the intermediate as well as the late time dynamics. This change is quite intriguing since the previous hike in pressure from  $P = P_{atm}$  to 10 MPa does not result in such a significant change. Furthermore, this can be explained by an increased degree of flattening of the drop on Cu at high pressure that results in the  $dE_{in}/dt$  value to be underestimated.

### 3.6 Conclusion

In this paper, we superpose high pressure effects on the hydrodynamically driven spreading of a water droplet that advances on a flat solid substrate. The primary contribution of this study is to incorporate into the extended Overall Energy Balance (OEB) approach the Lucas empirical model for estimating drop viscosity at elevated pressures. The model takes into account the changes in substrate wettability characterized by the advancing and equilibrium contact angle along with changes in the level of viscous dissipation work as a result of increasing pressure. The ensuing first order ordinary differential equation is solved numerically while utilizing an explicit Runge-Kutte formula. To validate the model results from theoretical study, an experimental investigation is also conducted to analyze the spreading behavior of hydrodynamically driven water droplets on

four different substrates namely: PTFE, Aluminium, Copper and PMMA. The pressure of the surrounding medium under these liquid-substrate combinations was varied stepwise in the range from atmospheric pressure to 20 MPa.

Comparing the model predictions and the experimental findings, good agreement exists in general between the two for small to intermediate drop sizes. Beyond a certain critical threshold drop radius, the spherical cap approximation does not hold true and as the drop assumes a flat pancake structure, the model underpredicts the actual drop size. The onset of this transition in drop shape is a strong function of pressure and subsequently varies with substrate combinations. With the exception of PMMA, the model effectiveness is relatively high when predicting the spreading factor,  $\beta$  at elevated pressures. For PMMA, early time stick and slip behavior which can be attributed to localized imperfections or surface heterogeneity curbs the contact line from advancing smoothly. This sticking effect is more pronounced at high pressure because increased viscous forces stemming from high viscosity leads to prolonged periods of stoppage. In other words, the model overpredicts the spreading factor at the early times for PMMA at 20 MPa. The results further demonstrate that as the pressure increases, the drag forces takes increasing precedence over the cohesive forces between neighbouring water molecules, causing a rise in the Weber number,  $We$ . Additionally, as viscous forces begin to dominate over drop inertia, this in turn leads to a corresponding fall in Reynolds number,  $Re$ . Further investigations in the future might consider the variation of surrounding medium viscosity and substrate surface temperature on advancing characteristics.

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## 4 Conclusion and future remarks

### 4.1 Summary

This thesis addresses some important aspects concerning the influence of pressure on droplet spreading dynamics and its application. We give particular attention on the initial spreading of a droplet and the hydrodynamic spreading of a droplet considering different pressure conditions. We considered the following analysis to conduct our investigation:

- The influence of pressure on the initial spreading dynamics of four different grade of silicone oil on PTFE substrate
- Determining the influence of pressure on the spreading exponent,  $\alpha$
- Analyzing the influence of pressure on the viscosity of the liquids
- Identifying the regimes and investigating the spreading velocity during initial spreading dynamics
- Experimental investigation on the effects of pressure for the hydrodynamic spreading of a water droplet on four different substrates
- Developing a generalized theoretical model for predicting hydrodynamic spreading of droplet under the effects of pressure
- Investigating the change of Reynold number, Weber number, minimum spreading ratio,  $\xi$  with pressure

In chapter 2, an experimental investigation is outlined on the study of initial spreading phenomenon of a droplet on partially wettable substrate under the influence of surrounding pressure. Results show that, pressure effects are less prominent on the advancement of the contact line in spreading for high viscosity oils. The instantaneous spreading radius and equilibrium radius

both fall on account of higher viscous dissipation with the increase of pressure. Also, for a given viscosity, the drop reaches to its equilibrium position faster in comparison to atmospheric conditions. We observe two regions in the inertia dominated early-stage spreading. In the first region, the splintering of the spreading radius curves are largely dependent on the pressure. The splintered curves collapse onto a master curve with varying levels of magnitude over time. We observe the spreading exponent,  $\alpha$  and transition time and radius to fall with the increase of pressure. Pressure effects on the time taken to reach the start of the transition regime (transition time), are negligible for highest viscous liquid (D500) whose entire spreading regime is viscosity dominated. We also observe the similar effects on the spreading exponent,  $\alpha$ . During the first stage of spreading, contact line velocity lowers significantly with the increase of pressure and in the second stage of the early inertial regime, the curves of velocities collapse upon each other. Finally, we observe that the switchover time for contact angle depends on the pressure.

In chapter 3, we conduct a theoretical and experimental investigation on the hydrodynamic spreading of droplet under the influence of pressure. We have done an experimental analysis by considering continuous addition of mass with a small flowrate to the water droplet on PTFE, PMMA, copper and aluminium substrate under the influence of pressure. We observe a change in the wettability of the substrate with the increase of pressure. The decrement in contact radius and increment in the contact angle was observed with the increase of the pressure. Both advancing and equilibrium contact angle increases with the increase of pressure. We also investigate the effects of pressure on the Reynold number and Weber number. By combining the extended Overall Energy Balance (OEB) approach and the Lucas empirical model for estimating drop viscosity at elevated pressures, we successfully predict the spreading behavior in different surrounding pressure. The

theoretical model is validated with the experimental results. The results from this model shows close agreement between the experimental findings and the analytical predictions successfully.

The findings of this study can be applied to better the understanding of the spreading mechanics of supercritical CO<sub>2</sub> in deep porous reservoirs used for curbing atmospheric emissions. Injecting the CO<sub>2</sub> at high pressure will limit the degree of lateral spreading inside the reservoir. This will subsequently lower the risk of leakage to overlying strata or shallow water formations. As a result, the trapping efficiency will likely increase, ensuring the sequestered CO<sub>2</sub> remains trapped for years on end. In a realistic scenario, the supercritical CO<sub>2</sub> will be encapsulated with surrounding brine in a porous aquifer with capillary effects. Although these complexities will result in slightly values, the underlying physics of the spreading phenomenon at high pressure should provide sufficient insights in this system to improve storage security.

## **4.2 Limitations and future recommendations**

Considering the early time dynamics, stage I has fewer data point than stage II. Hence the variation of the empirically obtained exponent with pressure is conducted using the latter regime. Due to constraints in the equipment frame rates and its relation to hardware space, a suitably high fps value of 20,000 to adequately capture both regimes was used. In the future, this can be repeated with an even higher frame rates to increase the empirical resolution. On the other hand, in terms of advancing at high pressure, molecular dynamics simulations can be conducted in the future to understand to uncover insights at the nanoscale. Additionally, the theoretical model is based on a few assumptions which in certain cases might not hold true when compared to real spreading. The model does not account for pinning which might take place in local hydrophilic zones on a smooth surface. Further, the mass addition rate must be done in an adiabatic fashion to keep the temperature constant. Otherwise, the assumption of thermal equilibrium will not be applicable. In

the future, the impact of temperature changes can be considered and as well as the effect of outer medium viscosity on advancing behavior.

### **4.3 Challenges and learning opportunities**

Repeatability and reproducibility are crucial to ensure precision while performing experiments with the high-pressure unit and the current study presents several learning opportunities to sharpen skills related to these criteria and avoid common pitfalls. In particular, the several preparation steps involved in cleaning the substrates and pressurizing the unit before executing additional trials are quite rigorous and delicate. After completing all the trials for one test liquid, the liquid lines and reservoir must be cleaned thoroughly with isopropanol (IPA) and ethyl alcohol before forcing out any remnant droplets with pressurized air. Next, the temperature inside the system is increased to 40°C with all the valves and chamber kept open for at least two days so that the remaining liquid evaporates. Then after, the unit is allowed to dry for two or three times with the pressurized air before filling the reservoir with water to verify if there are any impurities left inside by observing the SFT of water. After ensuring the cleanliness inside, water is flushed out and the drying procedure is repeated before filling the system with the next test liquid. Failure to adhere to these procedures results in test liquid contamination and a subsequent alteration in its surface tension. The prime reason for contamination comes as a result of the test liquid mixing with the cleaning liquids and strenuous efforts must be invested in the cleaning procedure to prevent this. Additionally, special attention must be given to the cleaning routines of the substrates before proceeding with a trial. This is because trace amounts of dirt or grease adversely affects the spreading dynamics, resulting in asymmetric advance which the OEB model cannot take into account. Furthermore, these surface impurities obscure the baseline of the droplet, thus making it harder to detect. In fact, a slight misinterpretation of the actual baseline in the KRÜSS Advance

software will largely alter the contact angle made by the droplet on the substrates. Drop deposition problems can arise as well if the needle isn't thoroughly cleaned with IPA and ethyl alcohol. Because of this, the drop tends to adhere to the needle which makes it difficult to ensure that the initial volume is kept the same between trails for a specific substrate and pressure combination. This tendency to stick with the needle during the advancing stage is more pronounced at high pressure when the drop viscosities are large.

The narrow channels in the unit in which the liquid passes need to be cleaned properly and air bubbles should be removed before conducting the experiments. Otherwise, the air bubbles can be observed inside the droplets which will give erroneous results. To reduce drop impact velocity, the distance between the substrate and the needle needs to be small and fixed in every trial of the experiment. Special attention was given on the mass flow rate while investigating the influence of pressure on the hydrodynamically driven spreading to ensure no internal heat generation and so that the system remains in isothermal conditions. Otherwise, the thermophysical properties will be affected by a change in temperature. In addition, the work bench needs to be in perfect horizontal position to reduce vibration and to prevent lopsided spreading due to gravity. Lights need to be adjusted accordingly because excess brightness or darkness hinders the detection of the sharp interfaces of the droplet. In the end, different kind of hazard can occur while working with the high-pressure unit such as a gas leak in the unit itself or from the booster pump. Pieces could come out with high velocity from high pressure unit due to explosion, chemical spill, etc. To reduce the associated risks, protective goggles need to be used and standard operating procedures needs to be followed while doing the experiments.

#### 4.4 Scope of the future work

The findings of this thesis bring out new opportunities which can be investigated further. Size of the droplet, speed, properties of the droplet and the interaction between the droplets to the substrate defines the effectiveness of the impact of the droplet on the surface. Lots of aspects are yet to analyze in the early time spreading and hydrodynamic spreading of a droplet considering the effect of pressure. Investigation of the initial and hydrodynamic spreading dynamics under the influence of pressure brings interest to investigate the initial spreading and the hydrodynamic spreading of a droplet in high viscosity liquid surrounding medium. We observe a change in the spreading exponent,  $\alpha$  value with the increase of pressure in air medium. Effect of surrounding liquid viscosity with the rise in pressure on the spreading exponent would be interesting to investigate. This investigation can be conducted further in different room temperature as well since temperature will change the viscosity of the liquid significantly. Furthermore, molecular level interactions with molecular dynamics simulations can be investigated to understand the early time spreading in molecular level considering the effects of pressure. Also, in case of hydrodynamic spreading of a droplet, we use water on four different substrates and validated the theoretical model results with water. This study can be further investigated with different liquids and substrate combination as well.

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