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The undersigned certify that they have read, and recommend to the Faculty of Graduate Studies and Research, for acceptance, a thesis entitled "THE GROUND TEMPERATURE REGIME AND ITS RELATIONSHIP TO SOIL PROPERTIES, SURFACE COVER AND ENVIRONMENTAL BOUNDARY CONDITIONS" submitted by WONG, BOON-KING in partial fulfillment of the requirements for the degree of Master of Science.

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DATE: January 20th, 1975

ABSTRACT

A study is made of the problem of periodic heat flow to the ground in a freeze-thaw situation. Analytical and numerical models are developed for the purpose of studying the effects of various parameters on the ground thermal regime. In particular, the effects of surface cover, soil properties and meteorological parameters on the average ground temperature below the active layer are studied. The main objective has been toward obtaining an indication of the effects of the parameters rather than an overall predictive model.

The results obtained indicate the relative importance of the various parameters. They also indicate that under certain conditions simplifying approximations can be made to the environmental boundary conditions. Analytical methods of estimating the quantitative effects of surface disturbances are proposed.

The results are in general agreement with field observations.

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NOTATION

LETTER SYMBOLS

Symbol	Definition
c	Specific heat
d	Insulating layer thickness
D	Diffusivity
e_a	Air vapor pressure
erf	Error function
$erfc$	Complementary Error function
f	Variable associated with D
F	Constant associated with D
G	Heat flux to the ground
h	Enthalpy
h_{eff}	Effective heat transfer coefficient
k	Thermal conductivity
L	Latent heat flux
L_f	Latent heat
N	Constant associated with height of cloud cover
M_A	Molecular content
P	Total pressure
R	Radiation
r	Relative humidity
S	sensible heat flux
S_1	sensible constant

NOMENCLATURE (continued)

Symbol	Definition
t	Time
T	Temperature
U	U factor
u	Wind velocity
u _{if}	Unfrozen moisture content
w	Relative humidity
w _c	Cloud cover
x	Depth
Y	Depth of the fusion front

GREEK LETTERS

τ	Time (normalized)
α	Thermal diffusivity
ε	Thermal emissivity
η	Depth of fusion front (normalized)
θ	Temperature (normalized)
φ	Enthalpy (normalized)
ζ	Depth (normalized)
ω	Frequency
σ	Stefan-Boltzmann constant
ρ	Density

NOMENCLATURE (continued)

SUBSCRIPTS

Symbol	Definition
s_0	Soil
a	Air
n	Surface
B^{\uparrow}	Ground surface
B^{\downarrow}	Point just beneath the active layer
c	Constant
f	Freezing
f	Frozen
ul	Unfrozen

PART I

INTRODUCTION

The problem of heat flow into a thawing and freezing soil has been studied rather extensively in the past especially by those countries close to the Arctic Region namely, Russia, Canada, United States and the Scandinavian countries. Despite these efforts, an exact mathematical solution to any practical thermal soil problem is still unavailable. General solution of the temperature distribution in the soil involves analysis of the transient heat flow in the soil which is affected by the thermal properties, namely, the specific heat, latent heat and conductivity, all of which are temperature-dependent. Therefore in order to find an exact mathematical solution, the specific problem must first be idealized. Problems of this nature are boundary value problems and therefore accurate specific boundary conditions must be applied to the surface where the heat flow is known.

1.1 Review of some analytical techniques

The first mathematical solution obtained for this problem is by Neumann (1) but was later adapted to the calculation of front depth penetration in soil by Byrnes (2). The first published literature on the rate of ice formation in still water was by Stefan (3) and this became known as the Stefan Problem. Most of the classical front depth calculations used today can be shown to be a modified form or a variation of the "Byrnes Formula" (or Neumann) and the "Stefan Formula". Later Adams and Payne (4, 5) presented the "Byrnes Formula" in a modified form and that it should be adapted to transient

systems. Since then the "modified Bessel formula" has been adopted by the Corps of Engineers and the Highway Research Board in the United States with some modifications to accommodate the specific conditions.

Consider now Neumann's solution applied to a semi infinite soil initially at constant temperature, T_c . Then assume that the surface temperature drops below freezing and remains so for all $t > 0$. There exists now two phases, the frozen and the unfrozen region.

The temperature distributions for the frozen and the unfrozen regions are given by:

$$T_f = (T_f / c \lambda) \operatorname{erf}(y / 2(\alpha_f t)^{1/2}) \quad (1.1)$$

$$T_{uf} = T_c - [(T_c - T_f) / c \lambda (\alpha_f / \alpha_{uf})^{1/2}] \operatorname{erfc}(y / 2(\alpha_{uf} t)^{1/2}) \quad (1.2)$$

where T_f is the freezing temperature of the liquid and λ is the proportional constant which is determined from

$$\begin{aligned} (\alpha_f \lambda^2 / c k_f) &= \left\{ [k_{uf} (\alpha_f)^{1/2} (T_c - T_f) \alpha_f (\alpha_f / \alpha_{uf}) \lambda^2] / \right. \\ &\left. [k_f (\alpha_{uf})^{1/2} T_f \alpha_f \lambda (\alpha_f / \alpha_{uf})^{1/2}] \right\} \alpha_f T_c \quad (1.3) \end{aligned}$$

This transcendental position y is obtained from

$$y = 2\lambda (\alpha_f t)^{1/2} \quad (1.4)$$

From Neumann's (1.6) and equation (3), there only two known variables available, these are α_f and α_{uf} but they do not vary with time. Thus the only variable is λ .

used by Biot and Dughaday (18) to study the heat conduction in a melting semi infinite solid with constant properties. Goodman (19) used the Heat-Balance Integral Technique to solve cases of melting in a semi infinite solid with a fixed boundary temperature and with a given heat flux at the boundary. The Heat-Balance Integral Technique changes the energy equation from a partial differential equation into an ordinary differential equation.

$$\int_0^{\delta} \rho c \frac{\partial T}{\partial t} dx + \int_0^{\delta} \rho \cdot k \nabla T dx \quad (1.5)$$

The temperature profile is then assumed to be of a general polynomial form which is substituted into the integral equation. Integration produces an ordinary differential equation with time t as an independent variable. Peot (21) used the integral method to study a moving two dimensional solidification problem. The complexity of this heat transfer problem is increased if the latent heat effect no longer occurs at a certain single temperature, but over a temperature range. This problem was described by Rubinshtein (22) and Welner (23), both considering the solid undergoing "n" phase changes. In 1967 Tien and Geiger (27) obtained a solution for the above problem using the Neumann approach by modifying the boundary conditions.

1.2 Review of some of the numerical techniques used

As reviewed previously, there are relatively few analytical solutions available for the practical heat transfer problems involving a change in phase of the solid due to freezing and thawing. In order to obtain a more general solution, other techniques have to be adopted.

Numerical methods are the most powerful and accurate of all the techniques available.

Many of the existing numerical solutions to the practical heat transfer problem involving change of phase use a continuous description on the movement of the fusion front as well as a description of the temperature distribution. Modifying the conventional finite difference procedure, various authors developed different ways of solving the problem. Cochran (24) developed a simplified analysis by using a single lump of variable thickness to represent the frozen region. While the technique works the inaccuracy resulted from the lumping severely hindered its use. Orin (25) extended the work of Lightfoot by using the concept of a moving heat source to account for the latent heat. Forster (26) modified the conventional finite difference equation for fixed grid so that the travel of the fusion front is continuously computed, but this method is limited for well initially or finally at the fusion temperatures only. Murray and Landis (12) later developed two numerical approaches with a wider range of applicability and a greater degree of accuracy. One approach utilizes the variable space network whereas the number of grid points in each phase (frozen and unfrozen) remains constant. This numerical method was a refinement of the Forster work, it led to a more accurate resolution of the temperature distribution and it also continuously computed the fusion front movement.

More of the above methods have been developed in the literature. The methods are either a single front, constant or variable, or

or vice versa. This occurs in the case where the surface temperature is periodic and its range includes 0°C. The numerical model presented in this thesis will be free of this limitation.

1.3 Recent Interest

Building of engineering structures such as oil and gas pipelines, roads and etc. in permafrost regions such as Alaska and Northern Canada, has been and still is a big problem for any contractor especially now with the present development and discovery of large energy reserves. With the rapid growth of development, a growing concern for the arctic environment has caused both the government and industry to increase their rate of research on developing suitable methods for constructing and operating in permafrost areas. A considerable amount of work has been, and continues to be done in the area of developing comprehensive models of heat flux in freezing and thawing soils. Numerical methods have been, and continue to be developed by universities, government and industry to model ground thermal regime. The choice of numerical methods used depends on the problem under consideration. For one-dimensional analysis, finite difference methods are in common use. The finite difference methods used are ordinary forward differences, backward differences and the alternating direction. The forward differences have been used by many authors whereas the backward differences has been rarely used. The alternating-direction explicit methods were employed by Lacharriere (13) and Doherty (14) while Fleming (15) used the alternating direction implicit methods for solving the problem. The present work is towards the use of finite difference methods, an area which

by Hwang (16) and Charwood (17). These have been designed for engineering prediction of temperatures around engineering structures.

1.4 Proposed program

The purpose of this thesis will be to assess the effects of each of a number of parameters in this very complicated problem of periodic heat transfer to the ground in a freeze and thaw situation. This will be done by breaking the problem into three subproblems. They are:

- I. Ground temperature distributions with imposed surface temperatures. Here, the main objective will be to look at the effects that the various individual soil parameters have on the ground temperatures and the active layer.
- II. Ground temperature distributions with an insulating layer and an imposed insulating layer surface temperature. Here, the main objective will be to look at the effects the insulating layer have on the ground temperatures and the active layer.
- III. As in (II) but with the environmental heat fluxes at an insulating layer surface boundary. Here, the main objectives will be to look at the effects the main parameters have on the various temperatures.

PART II

IMPOSED SURFACE TEMPERATURE

2.1 Analytical model for the ground temperature distributions with imposed surface temperature

2.1.1 Formulation of the model

Assume the ground to be a semi infinite domain in which two phases exist as shown in Figure 1. Soil properties will be assumed to be uniform throughout a given phase, however conductivity, specific heat and the temperature gradient will be different for the two phases.

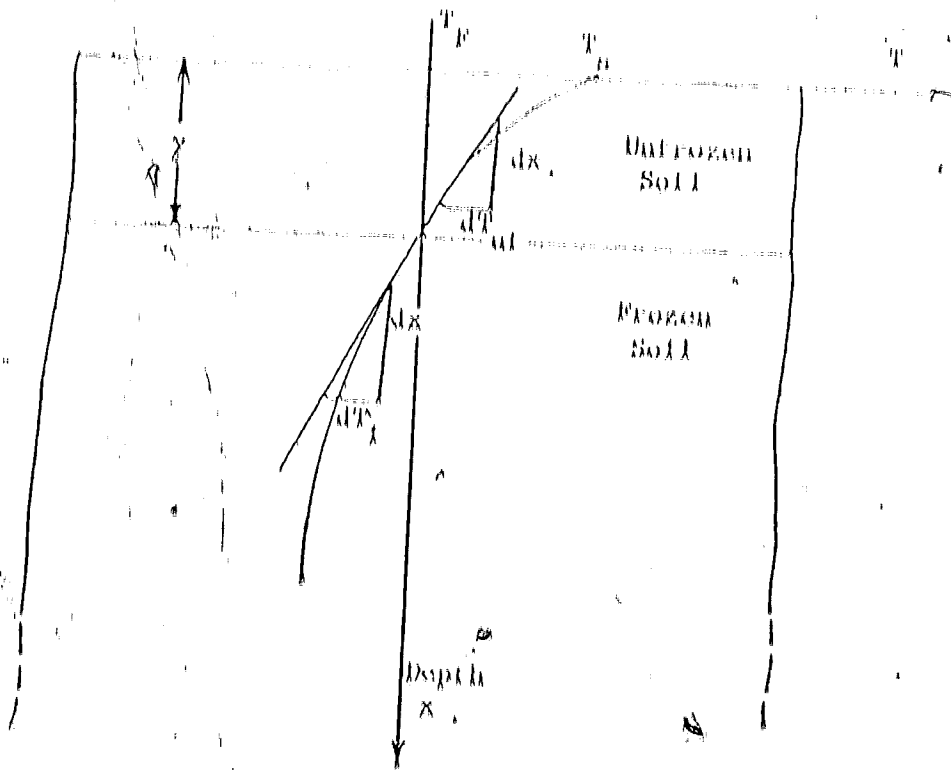


FIGURE 1

Applying the classical theory of heat transfer to the above conditions, the temperature distribution within a given phase may be calculated.

Using the Fourier approximation for one-dimensional heat conduction and the first law of thermodynamics, the equation for the temperature, T_{ul} in the unfrozen region becomes

$$\alpha_{ul} (\partial^2 T_{ul} / \partial x^2) = \partial T_{ul} / \partial t \quad (2.1)$$

where $\alpha_{ul} = k_{ul} / \rho c_{ul}$.

At the moving interface between the frozen and the unfrozen regions, the heat flow into the interface equal to the heat flow out plus the latent heat of fusion associated with the interface motion. The conditions will be satisfied by the equation

$$k_{ul} \partial T_{ul} / \partial x + \rho L (\partial y / \partial t) = k_f \partial T_f / \partial x \quad (2.2)$$

where y is the position of the interface.

In the frozen region the diffusion equation is

$$\alpha_f (\partial^2 T_f / \partial x^2) = \partial T_f / \partial t \quad (2.3)$$

The boundary condition at the ground surface will be assumed to be that of an imposed temperature varying sinusoidally during the year, that is

$$T_{fs} = \bar{T}_{fs} + \Delta T \sin 2\pi t$$

where \bar{T}_{fs} is the average temperature and

ΔT is the amplitude of variation

The boundary condition at $x = 0$ is $(T/\partial x) = 0$, the u_L zero heat flux. Non-dimensionalized equations (2.1), (2.2) and (2.3), the three equations become

$$\partial^2 \theta_{ul} / \partial \xi^2 = S_{ul} (\partial \theta_{ul} / \partial \xi) \tag{2.4}$$

$$(k_{ul}/k_l) \partial \theta_{ul} / \partial \xi = - \partial u / \partial \xi + \partial \theta_l / \partial \xi \tag{2.5}$$

and

$$\partial^2 \theta_l / \partial \xi^2 + S_l (\partial \theta_l / \partial \xi) \tag{2.6}$$

where $\theta = (T - T_c) / \Delta T$

$$\xi = x / \lambda_c$$

$$u = y / \lambda_c$$

$$S = 1 / \lambda_c^2 \left(\text{and } S_c = \lambda_c^2 \gamma \alpha \mu \right)$$

$$S_l = \rho c_l \Delta T / \rho k_l$$

$$S_{ul} = \rho c_l \lambda_c^2 / k_l \lambda_c$$

$$\lambda_c = (k_l \lambda_c \Delta T / \rho k_l)^{1/2}$$

Similarly

$$S_{ml} = \rho c_{ml} \lambda_c^2 / k_l \lambda_c$$

The boundary conditions in non-dimensional form become

$$\text{at } z = 0 \quad \theta = 0, \quad \partial\theta/\partial z = 0$$

and

$$\text{at } z = 1 \quad \partial\theta/\partial z = 0$$

For the purposes of obtaining an approximate analytical solution, the above problem can be simplified down to "Stefan Problem" which is the simplest form of approximation. It will be used to predict the front penetration depth and the average ground temperature. In this simplification it is assumed that the volumetric heat capacities of both the frozen and the unfrozen soil are negligible with respect to the latent heat of fusion of the soil water. This condition is satisfied when the Stefan Number (CAT/L) is small. When this condition is satisfied, the three governing equations become respectively

$$\partial^2 \theta_{u1} / \partial z^2 = 0 \quad (2.7)$$

$$(k_{u1}/k_f) \partial\theta_{u1} / \partial z = - \partial\theta / \partial z = \partial\theta_f / \partial z \quad (2.8)$$

$$\partial^2 \theta_f / \partial z^2 = 0 \quad (2.9)$$

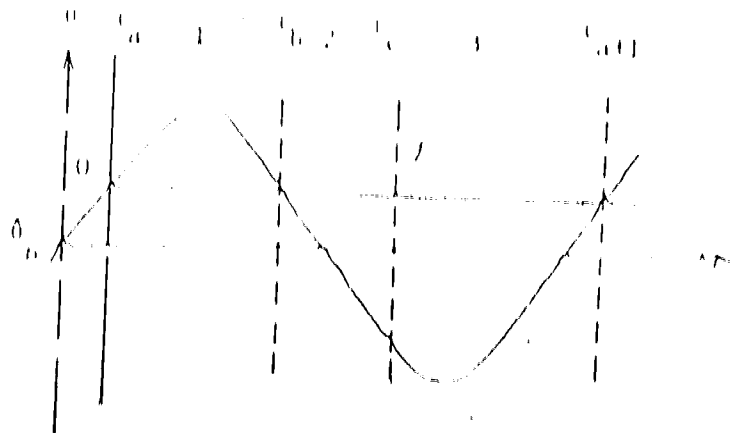
Equations (2.7) and (2.9) with the unfrozen and the frozen phases can

then be integrated to give

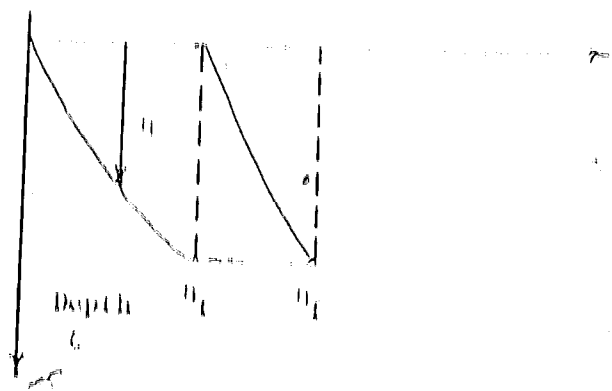
$$\theta_{u1} = C_1 + C_2 z \quad (2.10)$$

$$u_1 = C_1 + C_2 x \tag{2.1}$$

that in within a given phase the temperature profiles are linear



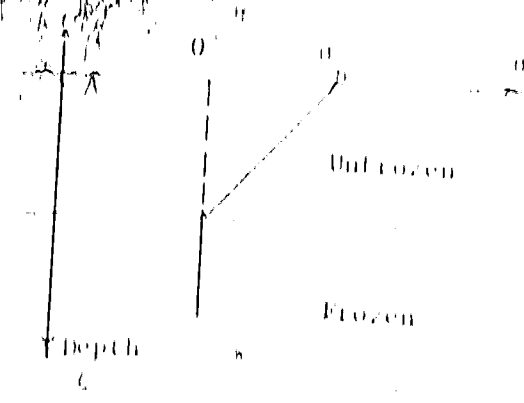
(I)



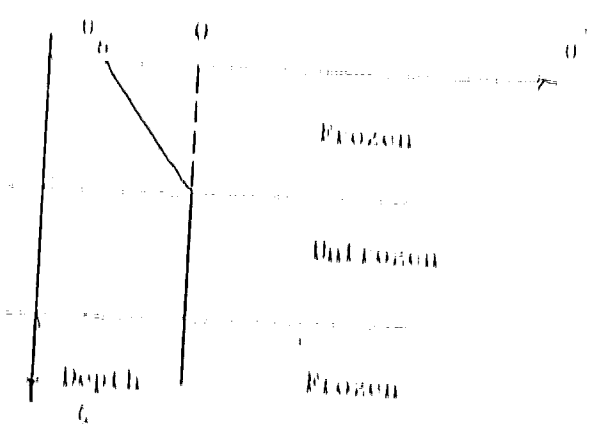
(II)

FIGURE 2

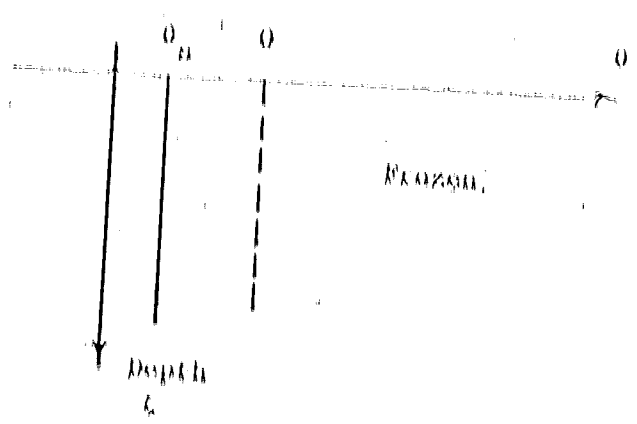
For the assumed form of the surface temperature three different time domains may exist throughout the year (Figures 2 and 3).



(I)



(II)



(III)

FIGURE 3

Applying the boundary conditions to equation (2.10) and (2.11)

$$\zeta = 0, \quad \frac{\partial \zeta}{\partial t} = 0$$

$$\zeta = \infty, \quad \partial \zeta / \partial \zeta = 0$$

$$\zeta = 0, \quad \frac{\partial \zeta}{\partial t} = 0$$

Considering first equation (2.10)

$$\frac{\partial \zeta}{\partial t} = C_1 + C_2 \zeta$$

$$\text{at } \zeta = 0, \quad \frac{\partial \zeta}{\partial t} = 0$$

$$\therefore C_1 = 0$$

$$\text{at } \zeta = \infty, \quad \frac{\partial \zeta}{\partial t} = 0$$

$$\therefore C_2 = \lambda \frac{0}{\infty} = 0$$

hence

$$\frac{\partial \zeta}{\partial t} = 0 \quad (\lambda = 0) \quad (2.12)$$

Considering now equation (2.11)

$$\zeta = C_3 + C_4 \zeta$$

$$\text{at } \zeta = 0, \quad \zeta = 0$$

$$\therefore 0 = C_3 + C_4 \cdot 0$$

$$\text{at } \zeta = \infty, \quad \partial \zeta / \partial \zeta = 0$$

implying $C_4 = 0$

$C_3 = 0$

hence $C_1 = 0$

(2.13)

Substituting equations (2.12) and (2.13) into equation (2.8), it becomes

$$(k_{ad}/k_l) \theta_{ad}/a = d\theta/dx \tag{2.14}$$

Equation (2.14) is the governing equation for this simplified problem.

2.1.2 Solution

Considering first region (1) and Figure 2(1), the boundary conditions are

at $x = 0, \quad \theta = \theta_a$

and at $x = 0_L, \quad \theta = \theta_b$

The governing equation is

$$(k_{ad}/k_l) \theta_{ad} = d\theta/dx$$

but $\theta_{ad} = \theta_{ad} + \sin 2\pi x/L$

where $\theta_{ad} = (T_{ad} - T_b)/\Delta T$

Integrating equation (2.14), it gives

$$\theta^2/2 = (k_{ad}/k_l) [\cos 2\pi x/2L + \theta_{ad}] + \text{const}$$

from the boundary conditions, when $x = 0, \theta = \theta_a$

$$\therefore \text{const.} = (k_{ul}/k_l) (\cos 2\pi t_a/2\pi + 0_B t_a)$$

hence
$$h^2/2 = (k_{ul}/k_l) \{ (\cos 2\pi t_a - \cos 2\pi t)/2\pi + 0_B (t - t_a) \}$$

when $t = t_a, \quad t = t_b$

$$\therefore h^2/2 = (k_{ul}/k_l) \{ (\cos 2\pi t_a - \cos 2\pi t_b)/2\pi + 0_B (t_b - t_a) \} \quad (2.15)$$

From Figure 2(1)

$$0_B = 0$$

when $t = t_a$ and $t = t_b$

$$\therefore \sin 2\pi t_a = -0_B$$

hence
$$t_a = \sin^{-1}(-0_B)/2\pi$$

$$t_b = 1/2 - t_a$$

$$\cos 2\pi t_a = (\lambda - 0_B^2)^{1/2}$$

$$\cos 2\pi t_b = \cos(\pi - 2\pi t_a)$$

$$= -(\lambda - 0_B^2)^{1/2}$$

Substituting the above expressions into equation (2.15), it becomes

$$h^2/2 = (2k_{ul}/k_l) \{ (\lambda - 0_B^2)^{1/2}/\pi + 0_B (\lambda/2 - \sin^{-1}(-0_B)/\pi) \} \quad (2.16)$$

Equation (2.16) gives the depth of the flow potential well. Considering

now portion (2) of Figure 2,

$$\therefore \eta_1^2/2 \sim - \int_{t_b}^{t_c} \theta_B dt$$

From region (1),

$$\eta_1^2/2 \sim k_{ul}/k_l \int_{t_a}^{t_b} \theta_B dt \quad (2.18)$$

but $\eta_1^2/2 \sim \eta_l^2/2$

$$\therefore \int_{t_b}^{t_c} \theta_B dt \sim - k_{ul}/k_l \int_{t_a}^{t_b} \theta_B dt \quad (2.19)$$

Considering all the 3 regions

$$\theta_B \sim \int_{t_c}^{t_{a+l}} \theta_B dt \quad (2.20)$$

$$\int_{t_a}^{t_{a+l}} \theta_B \sim \theta_B$$

$$\therefore \int_{t_A}^{t_B} \theta_B dt + \int_{t_b}^{t_c} \theta_B dt + \int_{t_c}^{t_{a+l}} \theta_B dt \sim \bar{\theta}_B$$

Assuming

$$\int_{t_c}^{t_{a+l}} \theta_B dt \sim - \int_{t_A}^{t_B} \theta_B dt - \int_{t_b}^{t_c} \theta_B dt + \bar{\theta}_B \quad (2.21)$$

Substituting equation (2.21) into equation (2.19), it gives

$$\bar{\theta}_B \sim - \int_{t_A}^{t_B} \theta_B dt - \int_{t_b}^{t_c} \theta_B dt + \bar{\theta}_B$$

$$\begin{aligned} \therefore \theta_B - \theta_M &= \int_{t_a}^{t_b} \theta_{tt} dt - \int_{t_b}^{t_a} \theta_{tt} dt \\ &= (1 - k_{ul}/k_l) \int_{t_a}^{t_b} \theta_{tt} dt \end{aligned}$$

but from region (1),

$$\int_{t_a}^{t_b} \theta_{tt} dt = (1 - \theta_B^2)^{1/2} / \mu - \theta_B (1/2 - \mu \lambda)^{-1} (-\theta_B) / \mu$$

$$\therefore (\theta_B - \theta_M) = (k_{ul}/k_l - 1) \left(\theta_B (1/2 - \mu \lambda)^{-1} (-\theta_B) / \mu \right)$$

$$+ (1 - \theta_B^2)^{1/2} / \mu \quad (2.22)$$

Equation (2.22) gives the average ground temperature for the case when $T_M < T_K$.

Solutions to the above problem are for the case when the average temperature T_M is less than the freezing temperature T_K . Under the same approach as above, the solutions for the case when the average temperature, T_M is greater than the freezing temperature T_K are

$$\theta_K^2 = -2 \left\{ (1 - \theta_M^2)^{1/2} / \mu + \theta_M (1/2 - \mu \lambda)^{-1} (-\theta_M) / \mu \right\} \quad (2.23)$$

and

$$(\theta_B - \theta_M) = (1 - k_l/k_{ml}) \left\{ \theta_M (1/2 - \mu \lambda)^{-1} (-\theta_M) / \mu + (1 - \theta_M^2)^{1/2} / \mu \right\} \quad (2.24)$$

Equation (2.23) and (2.24) gives the frost penetration and the average ground temperature for the case when $T_M > T_K$. These analytic results will be compared with the numerical results later in this part of the thesis.

2.1.3 Limitations of this analytic model

1. The assumption that the properties of the soil are uniform is not true in many cases. This is because of the different soil textures, moisture content, soil compositions and other soil properties at different regions in the soil. All these properties contribute to the volumetric heat capacities, latent heat of fusion and the thermal conductivity of the soil.
11. Though the air temperature is not a true periodic function of time owing to the variability of the weather, it has many features in common with a periodic function, such as the fluctuations caused by the succession of day and night or winter and summer.

2.2 Numerical Model

2.2.1 Formulation of the model

The main differences between the proposed method and most of the existing methods will be, instead of continuously computing the travel of the fusion front, this proposed method will keep track of the enthalpy h_s as well as the temperature T_s at any time t_s . Governing Equations

$$\partial h / \partial t = k (\partial^2 T / \partial x^2) \quad (2.25)$$

$$\left. \begin{aligned} h / \rho L &= (T_s - T_f) \rho c_A / \rho L & T_s < T_f \\ h / \rho L &= (T_s - T_f) \rho c_{MA} / \rho L + \rho L / \rho L & T_s > T_f \end{aligned} \right\} \quad (2.26)$$

The boundary conditions are

$$\left. \begin{aligned} \kappa &= 0, & T &= T_b \\ \kappa &= \infty, & \partial T / \partial \kappa &= 0 \end{aligned} \right\} \quad (2.27)$$

At $T = T_f$, the enthalpy is not a single valued function of temperature, thus equation (2.26) cannot be substituted directly into equation (2.25) as in the case where no phase change occurs. This difficulty can be overcome either by keeping track of the position of the fusion front or by keeping track of the enthalpy. The later method will be used.

Equation (2.26) is true when the entire phase change occurs at a single temperature. If the phase change happens to occur over a temperature range, equations (2.25) and (2.26) will be modified to take into account the unfrozen water content. The modifications will be presented at the end of this section.

2.2.2 Normalizing the equations

Writing

$$\phi = h/\rho l,$$

$$\theta = (T - T_f)/\Delta T$$

$$\xi = \kappa/\kappa_0$$

$$\tau = t/\tau_0$$

where $\kappa_0 = l/\gamma \Delta T$

Equation (2.26) becomes

$$\phi = (\rho c_{\text{I}} \Delta T / \rho L) \theta \quad \theta = 0$$

$$\phi = (\rho c_{\text{III}} \Delta T / \rho L) \theta + 1 \quad \theta = 0$$

Let

$$\phi = S_{\text{I}} \theta \quad \theta = 0$$

$$\phi = S_{\text{III}} \theta + 1 \quad \theta = 0$$

(2.28)

where

$$S_{\text{I}} = (\rho c_{\text{I}} \Delta T) / \rho L$$

and

$$S_{\text{III}} = (\rho c_{\text{III}} \Delta T) / \rho L$$

Equation (2.25) becomes

$$(\rho L / \kappa_{\text{I}}) \partial \phi / \partial t = (k \Delta T / \kappa_{\text{I}}^2) \partial^2 \theta / \partial x^2$$

$$\partial \phi / \partial t = [k_{\text{I}} \Delta T / \rho L \kappa_{\text{I}}^2] \partial^2 \theta / \partial x^2 \quad (2.29)$$

For the frozen and unfrozen regions, the respective equations are

$$\partial \phi / \partial t = [k_{\text{I}} \Delta T / \rho L \kappa_{\text{I}}^2] \partial^2 \theta / \partial x^2 \quad (2.30)$$

and

$$\partial \phi / \partial t = [k_{\text{III}} \Delta T / \rho L \kappa_{\text{III}}^2] \partial^2 \theta / \partial x^2 \quad (2.31)$$

choosing

$$[k_{\text{I}} \Delta T / \rho L \kappa_{\text{I}}^2] = 1$$

Equation (2.30) and (2.31) becomes

$$\partial \phi / \partial t = \partial^2 \theta / \partial x^2 \quad (2.32)$$

and $\partial\phi/\partial t = (k_{ul}/k_l) \partial^2\phi/\partial z^2$ (2.33)

let $\partial\phi/\partial t = k^*(\partial^2\phi/\partial z^2)$ (2.34)

where $k^* = 1$ $0 = 0$ }
 $= k_{ul}/k_l$ $0 = 0$ } (2.35)

Rewriting the governing equations with the given boundary conditions, the equations are

$\partial\phi/\partial t = k^*(\partial^2\phi/\partial z^2)$ (2.34)

$\phi = \phi_0$ $0 = 0$ }
 $\phi = \phi_1$ $0 = 0$ } (2.38)

$k = 0$, $\phi = 0$ }
 $k = m$, $(\phi_{m+1} - \phi_m)/\Delta z = 0$ } (2.36)

2.2.3 Solution
 Considering the governing equation (2.34)

$\partial\phi/\partial t = k^*(\partial^2\phi/\partial z^2)$

we can use the finite difference method to solve it

$d\phi/dt \Big|_{z_m} = (\phi^{m+1} - \phi^m)/\Delta z$

where ψ^{m+1} is only time dependent of t^m

$$k^1(x^2_0/x^2_1) = (a_{m+1} - a_m) k^1/\Delta t + (a_m - a_{m-1}) k^1/\Delta t$$

or

$$\psi^{m+1} = \psi^m + (k^1(a_{m+1} - a_m) - k^1(a_m - a_{m-1})) \Delta t / \Delta t^2 \quad (2.37)$$

but

$$\psi^{m+1} = \psi^m + \Delta \psi^m$$

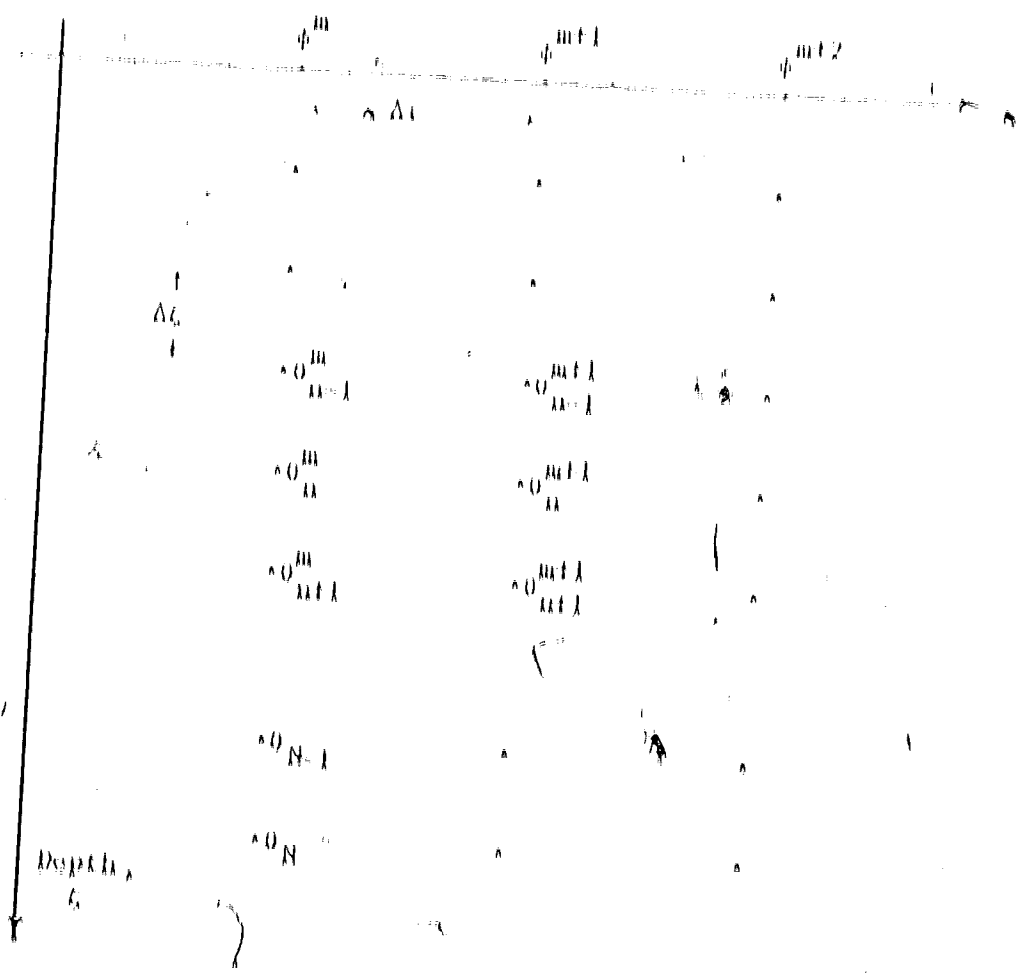
$$\Delta \psi^m = \psi^{m+1} - \psi^m = \Delta \psi^m$$

(2.38)

Hence equation (2.37) can be written as

$$\Delta \psi^m = (k^1(a_{m+1} - a_m) - k^1(a_m - a_{m-1})) \Delta t / \Delta t^2$$

(2.39)



where $\Delta t = \frac{\Delta x}{u}$ is one time step ahead of $\Delta t = \frac{\Delta x}{u}$.

Initially, all the temperatures at the nodal points were assumed to be at a constant temperature T_0 . Now enthalpies and hence the new temperatures, one time step forward, were then found by applying the boundary conditions to the governing equations. The time interval Δt was chosen to be so small that only the local temperature and the temperature of the adjacent reference points need to be considered in calculating the local temperature. It was found that if Δt was greater than $\frac{1}{2} \left(\frac{\Delta x}{u} \right)^2$, numerical instabilities resulted because of the use of an explicit computation scheme.

Briefly the steps are:

- I. Set all temperatures equal to 0 (i.e. $\theta_{in} = 0$) and then change T_0 into ψ using equation (2.28).

$$\begin{aligned} \psi^m &= S_{in} \theta + 1 & \text{if } \theta > 0 \\ &= S_1 \theta & \text{if } \theta < 0 \end{aligned}$$

- II. Apply the boundary conditions equation (2.36) and solve for the enthalpy changes at each nodal point using equation (2.39). Then using enthalpies from the last time step and these enthalpy changes, new values of the enthalpies at each nodal are calculated. Enthalpy changes only apply for nodes 2 to nodes N since this is an imposed surface temperature problem. The temperatures at the surface therefore is always equal to the imposed temperature, which in this case is the air temperature.

So for node n to $(n-1)$, using equation (2.49)

$$\Delta\phi_n^m = R11(u_{n-1}^m - u_n^m) - R12(u_n^m - u_{n+1}^m)$$

where $R11 = \Delta t / \Delta z^2$ $11 \ u_{n-1}^m + u_n^m = 0$

$$= (k_{ul} / k_l) \Delta t / \Delta z^2$$
 $11 \ u_{n-1}^m + u_n^m = 0$

and $R12 = \Delta t / \Delta z^2$ $11 \ u_n^m + u_{n+1}^m = 0$

$$= (k_{ul} / k_l) \Delta t / \Delta z^2$$
 $11 \ u_n^m + u_{n+1}^m = 0$

$$\phi_n^{m+1} = \phi_n^m + \Delta\phi_n^m$$

For node N , assuming the heat flux from deep in the ground to be negligible

$$\Delta\phi_N^m = (u_{N-1}^m - u_N^m) RT \times 2$$

where $RT = \Delta t / \Delta z^2$ $11 \ u_{N-1}^m + u_N^m = 0$

$$= (k_{ul} / k_l) \Delta t / \Delta z^2$$
 $11 \ u_{N-1}^m + u_N^m = 0$

$$\phi_N^{m+1} = \phi_N^m + \Delta\phi_N^m$$

111. Then use enthalpy ϕ_n^{m+1} to calculate u_n^{m+1} by using equation (2.28).

$$u_n^{m+1} = \phi_n^{m+1} / B_n$$
 $11 \ \phi_n^{m+1} = 0$

$$= (\phi_n^{m+1} - 1) / B_{nl}$$
 $11 \ \phi_n^{m+1} = 0$



θ_{ii}^{m+1} are then the new temperatures one time step ahead of θ_{ii}^m . Both the new temperatures θ_{ii}^{m+1} and enthalpies i_{ii}^{m+1} are stored.

iv. Proceed as before, as the time step advances, but now the initial temperatures and enthalpies are those just computed, θ_{ii}^{m+1} and i_{ii}^{m+1} respectively.

As for the surface temperature, $\theta_{11}^m \approx \theta_{ii}^m$.

v. The computation is over when the average temperature distribution over a year approaches steady state. This numerical approach provides one with a continuous computation of the temperature distribution and also the travel of the fusion front with all minimum of complications.

2.2.4

For the case when the phase change occurs over a temperature range instead of at a single temperature, equation (2.34) becomes

$$\partial\phi/\partial t \approx k^t (\partial^2\theta/\partial x^2) \tag{2.40}$$

$$\text{where } \left. \begin{aligned} k^t &\approx (1 - \text{mf}/M_{ho}) + (k_{ul}/k_l) \text{mf}/M_{ho} & 0 < \theta \\ &\approx k_{ul}/k_l & 0 > \theta \end{aligned} \right\} \tag{2.41}$$

and equation (2.28) becomes

$$\left. \begin{aligned} \phi &\approx N_l \theta + \text{mf}/M_{ho} & 0 < \theta \\ &\approx N_{ul} \theta + 1 & 0 > \theta \end{aligned} \right\} \tag{2.42}$$

where mf is the unfrozen fraction and M_{ho} is the total molecular content. Proceed as before, using now equations (2.40), (2.41) and (2.42).

2.2.5 Comments

In this thesis the steady periodic solution will only be of interest, the transient behavior produced by starting from a constant ground temperature will not be considered. Rigorously speaking, the steady periodic solution is obtained only when the temperatures at all times during the yearly cycle are the same in two consecutive years. However, since this work is primarily concerned with the temperature averaged over a yearly cycle, a somewhat less rigorous test was used to determine if the steady periodic state had been reached. The criterion used was that the change of the averaged temperatures from one year to the next must be small. In practice the calculations were always carried out through five yearly cycles of temperature variation. If the change in the averaged temperatures was greater than 0.02°C between the fourth and the fifth years, the calculations were extended to the tenth or the fifteenth years as required.

Input data for this model included the ground surface temperature variation during the year and the physical properties of the soil. The ground surface temperature is assumed to be equal to the air temperature, which is approximated by the harmonic function fitted to meteorological data, Appendix I. Temperature conditions of Edmonton, Norman Wells, and Inuvik were used as the basic cases. For soil properties data compiled by Lambridon (28) was used, Appendix II. The basic conditions used for calculating effects of soil property variations were taken as a fine grained soil, dry density $1.2 \times 10^3 \text{ kg/m}^3$ with moisture contents of 15%, 30% and 45%.

2.3 Results of the numerical calculations

2.3.1 Temperature distributions in the ground

Figures (4) and (5) show the temperature distribution in the ground for two cases, Edmonton and Norman Wells, both with 30% moisture content in a fine grained soil. The temperature profiles were drawn for the various time intervals of the fourth year.

The temperature profiles for the two cases display the characteristic features of all ground temperature profiles, that is the amplitude of the temperature variation decreases with depth and the phase of the temperature variation lags behind the phase at the surface. The overall temperature profiles do not appear to resemble those for the idealized analytic model, whose Stefan number was zero. It will be noted however, for both the cases, the temperature profiles that cross the 0°C line tend to be linear from that point to the surface temperature value. This is particularly striking in the case for Edmonton where the average air temperature of 2.56°C is very close to the freezing point. Since it is the region where phase change occurs, it is thus the most important region in determining the thermal behavior of the ground. Thus, it is reasonable to expect that the analytical model will give satisfactory results for some calculations. Also, note that in the temperature profiles in Figures (4) and (5), the average temperature deep in the ground appears to be different from the average temperature at the surface. This is the result predicted by the analytical model.

Figure (6) shows the variation of the average soil temperature with depth. It will be observed that this variation occurs only in the

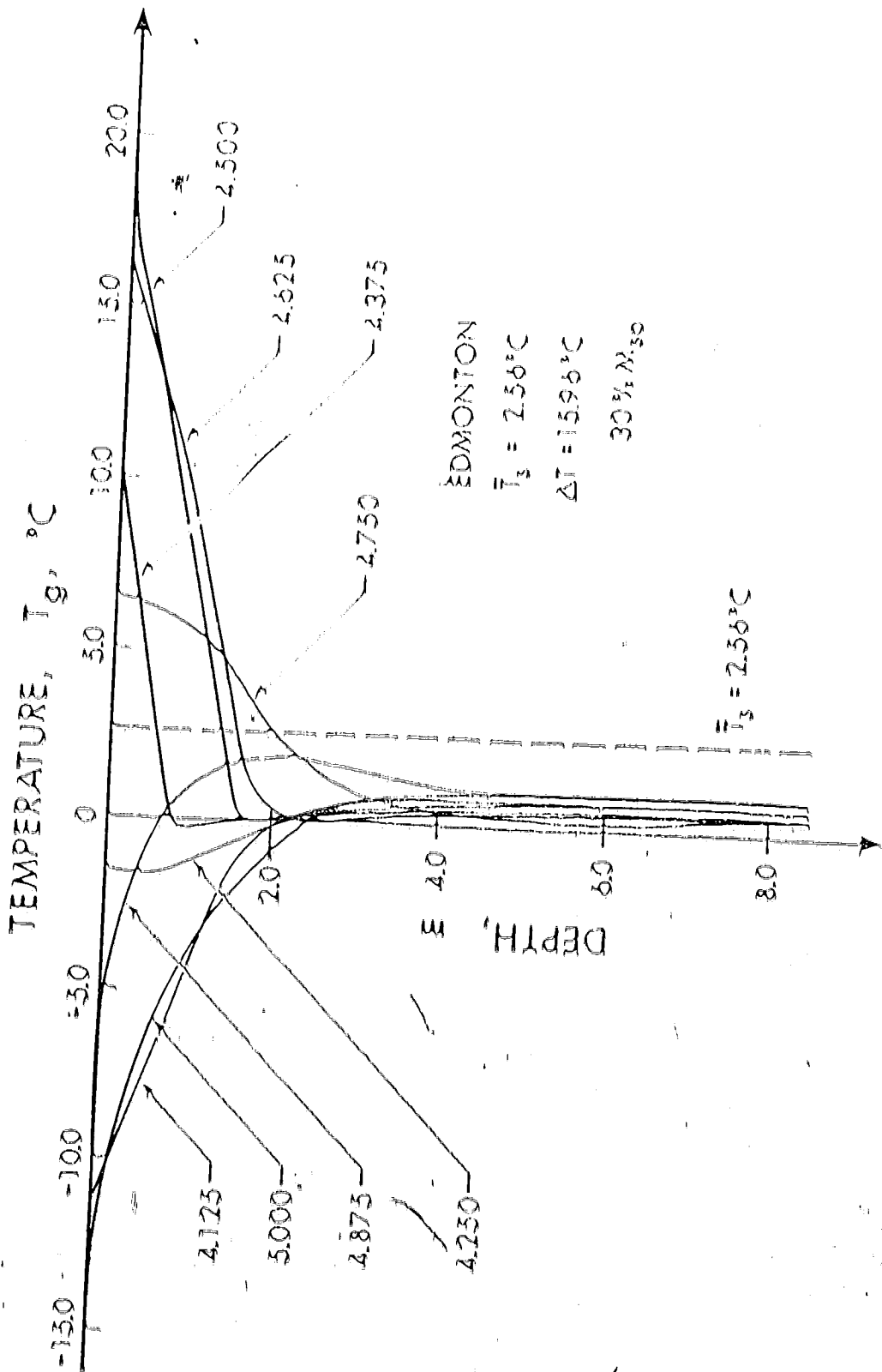


FIGURE 4 -- Temperature distributions in the ground at Edmonton with imposed surface temperature

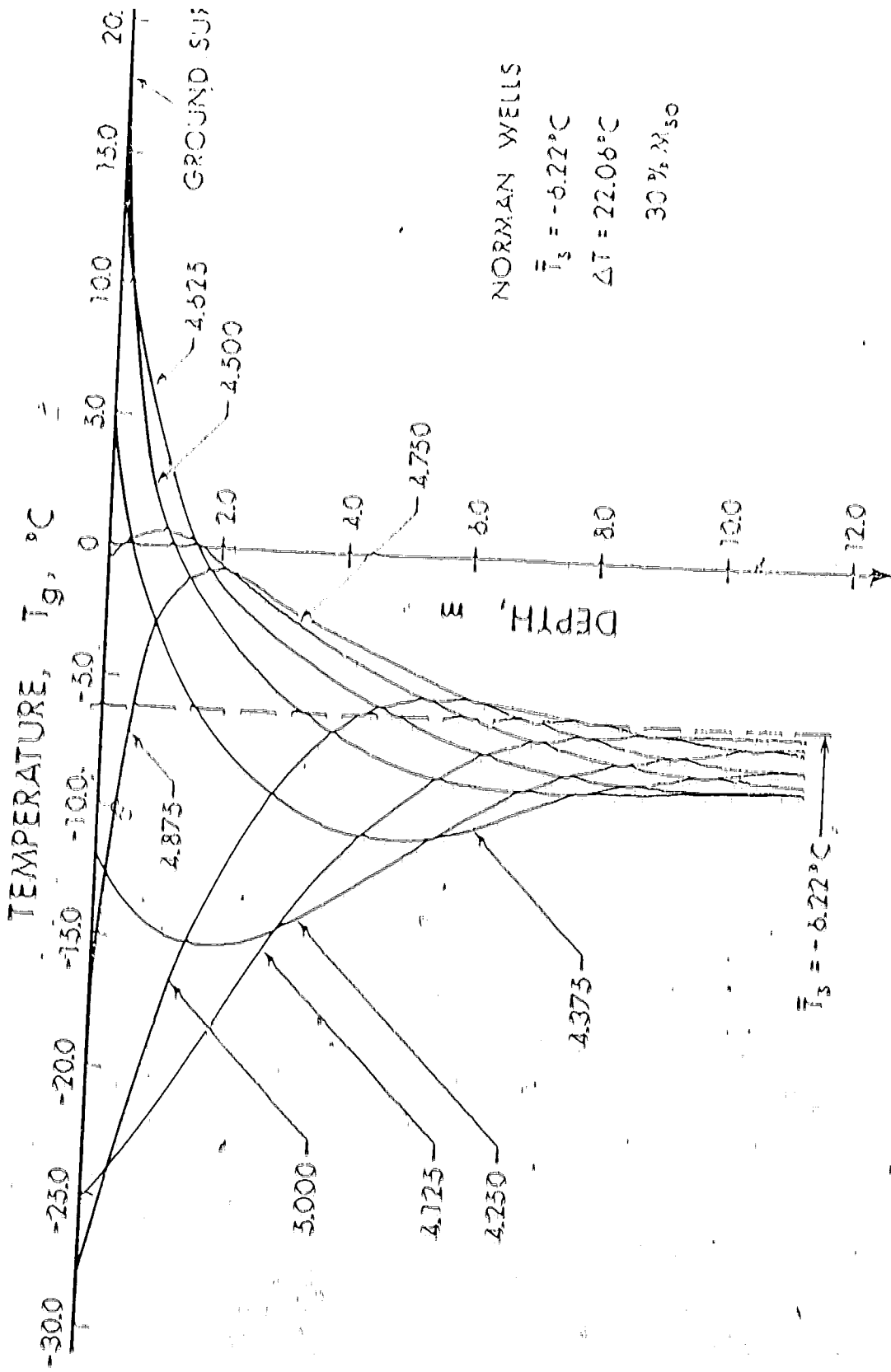


FIGURE 5 -- Temperature distributions in the ground at Norman Wells with an imposed ground surface temperature

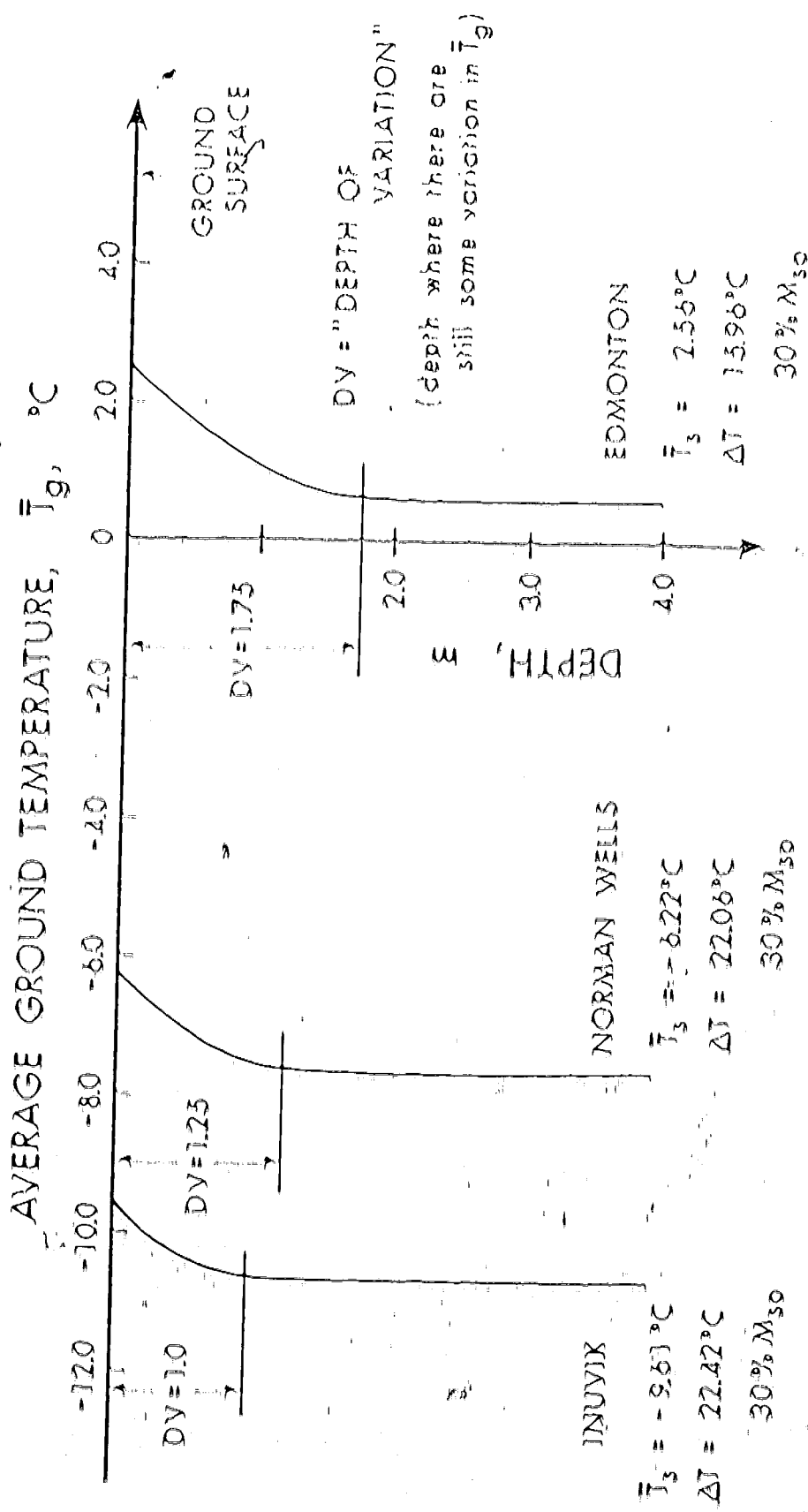


FIGURE 6 — Profiles of average temperature with imposed ground surface temperature

top few meters of the soil. This depth will be observed later to correspond to the depth of penetration of the phase change. This indicates that, the phase change or rather the conductivity change that results from phase change is responsible for the change in the average ground temperature with depth.

The absolute determination of the ground temperature in practice is almost impossible because of the large number of variables involved. However it is possible to predict the change in the ground temperature that would result from a change in one of the controlling parameters. This section will deal with the effects the main parameters namely, average surface temperature T_{M}^* , volumetric heat capacities of the soil C_{H0} , thermal heat conductivity ratio k_{MF}/k_L and the soil moisture content M_{H0} have on the difference between the average surface temperature T_{M}^* and the average ground temperature T_{G}^* .

The effects that changes in the average surface temperatures have on the temperature difference $T_{G}^* - T_{M}^*$ is shown in Figure 7. Here the temperature difference has been normalized by ΔT_s , the amplitude of the temperature oscillation at the surface. This parameter is a ratio of conductivity between the unfrozen and frozen soil, k_{MF}/k_L of 0.647 which corresponds to a fine grained soil with 30% moisture content. It will be noted that for this ratio of conductivity, the temperature below the surface layer is always colder than the average surface temperature and the maximum difference occurs when T_{M}^* is slightly greater than the freezing temperature. This theoretical analysis indicated by the solid lines in Figure 7 must be a value of $(T_{G}^* - T_{M}^*)/\Delta T_s = -0.14$.

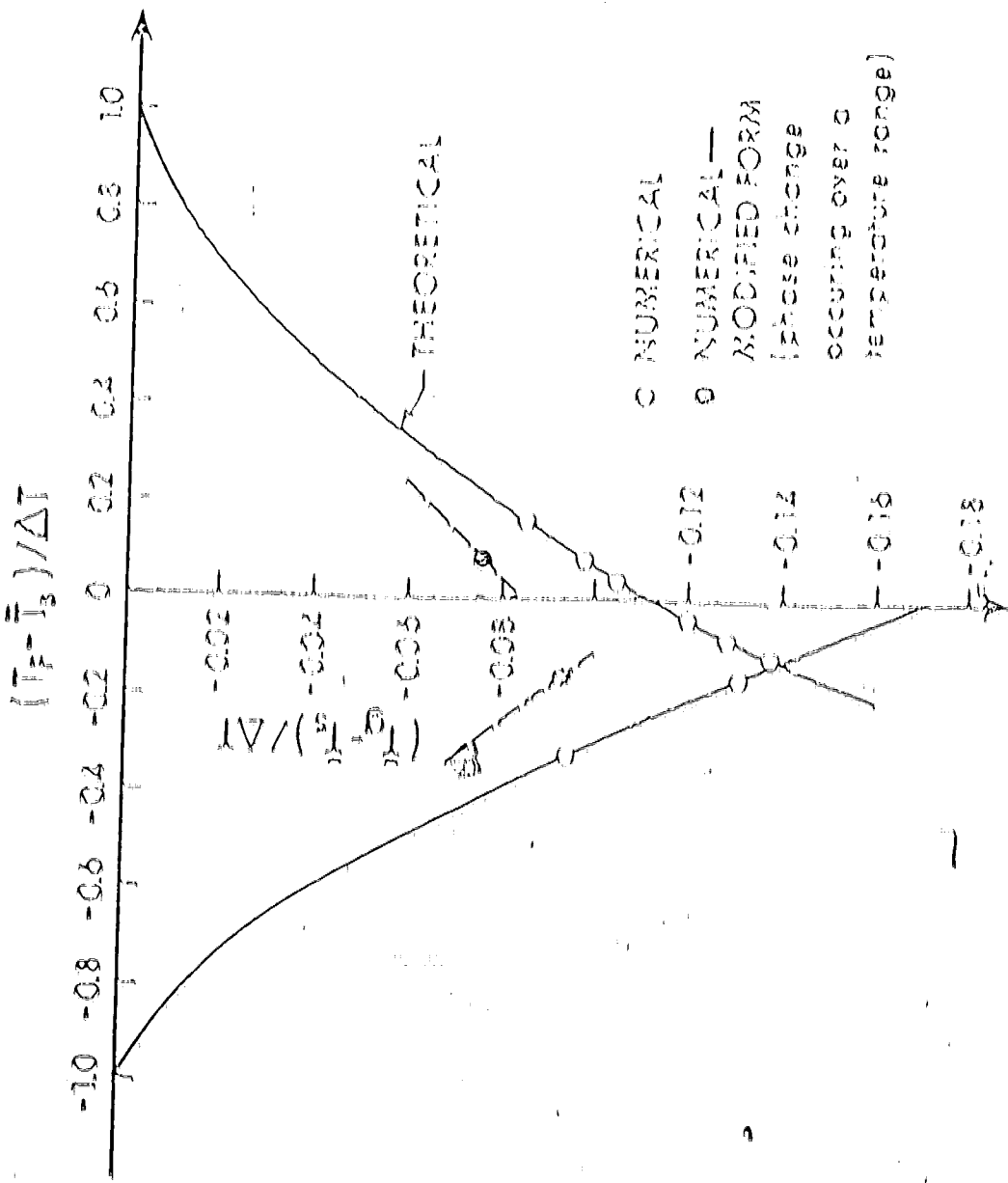


FIGURE 7 - Variation of T_2 with $(T_2 - T_3) / \Delta T$

The curve on the left corresponds to a soil temperature deep in the ground that is above 0°C and the one on the right corresponds to a soil temperature below 0°C (that is permafrost grounds). The intersection of these curves and thus the minimum value of the parameter $(T_E - T_B)/\Delta T$ corresponds to the condition under which the greatest difference exists between the average surface and ground temperatures. The minimum value of $(T_E - T_B)/\Delta T$ for this case is 0.14, which for a typical temperature range $\Delta T = 20^\circ\text{C}$ would correspond to a difference between average surface and ground temperatures of 2.8°C. The position of this minimum value and thus the minimum value of $(T_E - T_B)/\Delta T$ is a complex function of the conductivity ratio k_{H2O}/k_I . It can be calculated by equating equations (2.22) and (2.24) of the analytic model.

The results of the numerical calculations for $I_T = I_p \times 10^8$, that is $M_{tot} \sim 30\%$, agree with the analytic results except near the minimum point in Figure 7. Here calculations must be run through 15 yearly cycles before a steady periodic state is reached.

An increase in the moisture content causes a decrease in $(T_E - T_B)/\Delta T$, as indicated in Figure 7. This is due to a decrease in the range of conductivity.

A change in the moisture content of the soil affects both the latent heat of fusion I_T and the ratio of the conductivities k_{H2O}/k_I in the soil. These effects of the two parameters will have to be looked at separately.

Figure (8) indicates that if the calculations are carried out through enough yearly cycles, approximately 15, the values of $(T_E - T_B)/\Delta T$ obtained are independent of the volumetric heat capacity of the soil I_T .

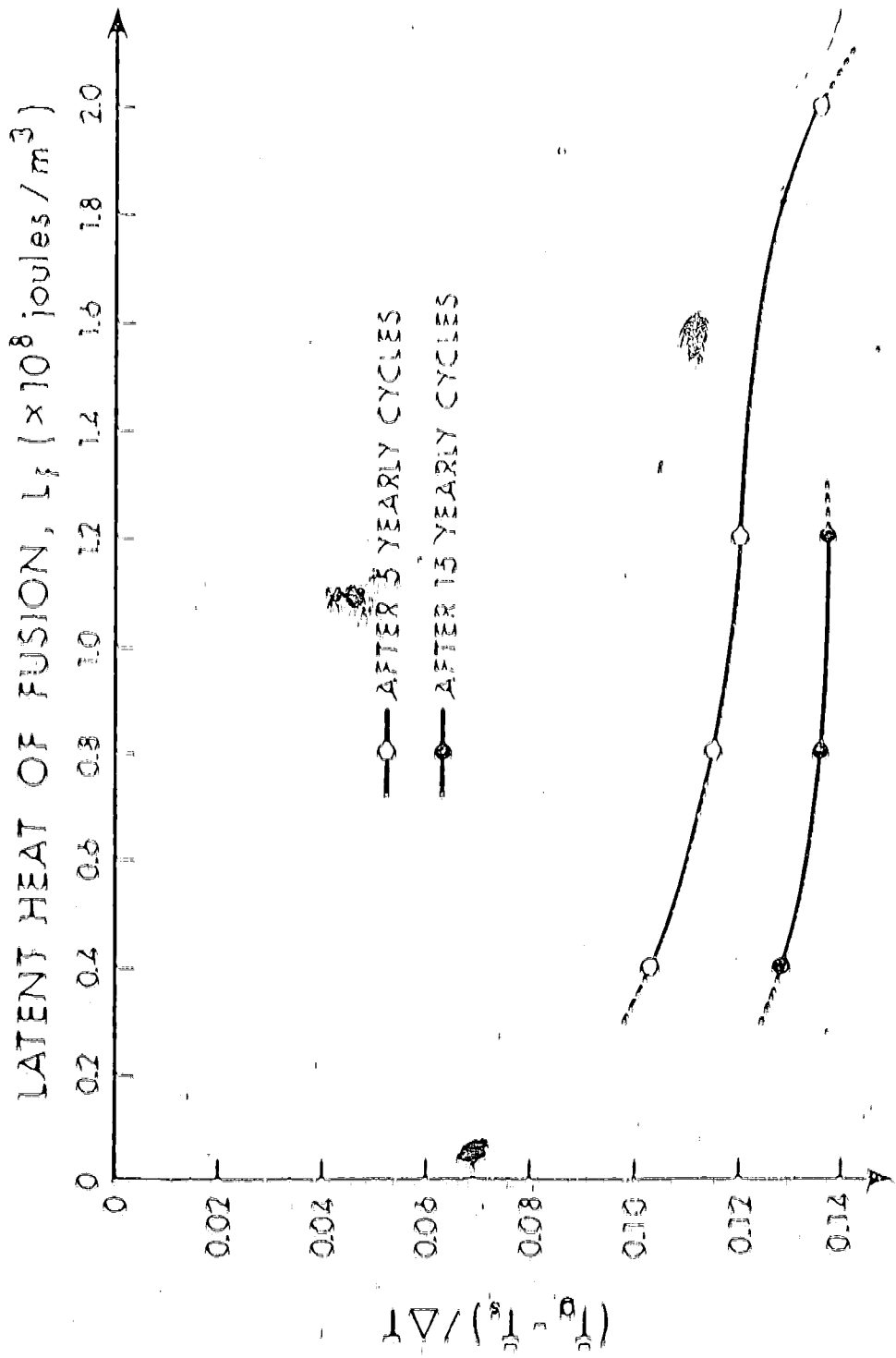


FIGURE 8 — Variation of L_f with $T_2 - T_1$ (CT)

The result is in agreement with the analytic model.

The ratio of conductivities, k_{ul}/k_f , has a much more significant effect. The relationship between k_{ul}/k_f and $(T_B - T_D)/\Delta T$ is linear, that is, as k_{ul}/k_f increases, $(T_B - T_D)/\Delta T$ increases, as indicated by Figure (9). For every unit increase in k_{ul}/k_f , there results a corresponding 0.166 units increase in $(T_B - T_D)/\Delta T$. This is at a value of $(T_B - T_D)/\Delta T = 0.341$.

The graph of $(T_B - T_D)/\Delta T$ versus the moisture content M_{50} in Figure (10) will only confirm the results of the two previous findings. The graph shows that an increase in moisture content will result in an increase in the magnitude of $(T_B - T_D)/\Delta T$. This results because of the increased difference between k_{ul} and k_f that occurs at the higher moisture content. Also changes in moisture content have less effect on the ground temperature in the colder region such as Inuvik than the comparatively warmer region such as Edmonton. This is because as shown in Figure (7), the effects of conductivity differences between frozen and unfrozen soil are the greatest when the soil is near 0°C.

2.4 Depth of the active layer

2.4.1 Depth of front of thaw penetration

Figure (11) shows the front of thaw penetration depth during the thawing year for the three localities, Edmonton, Norman Wells and Inuvik, with 30% moisture content. The method of obtaining the front

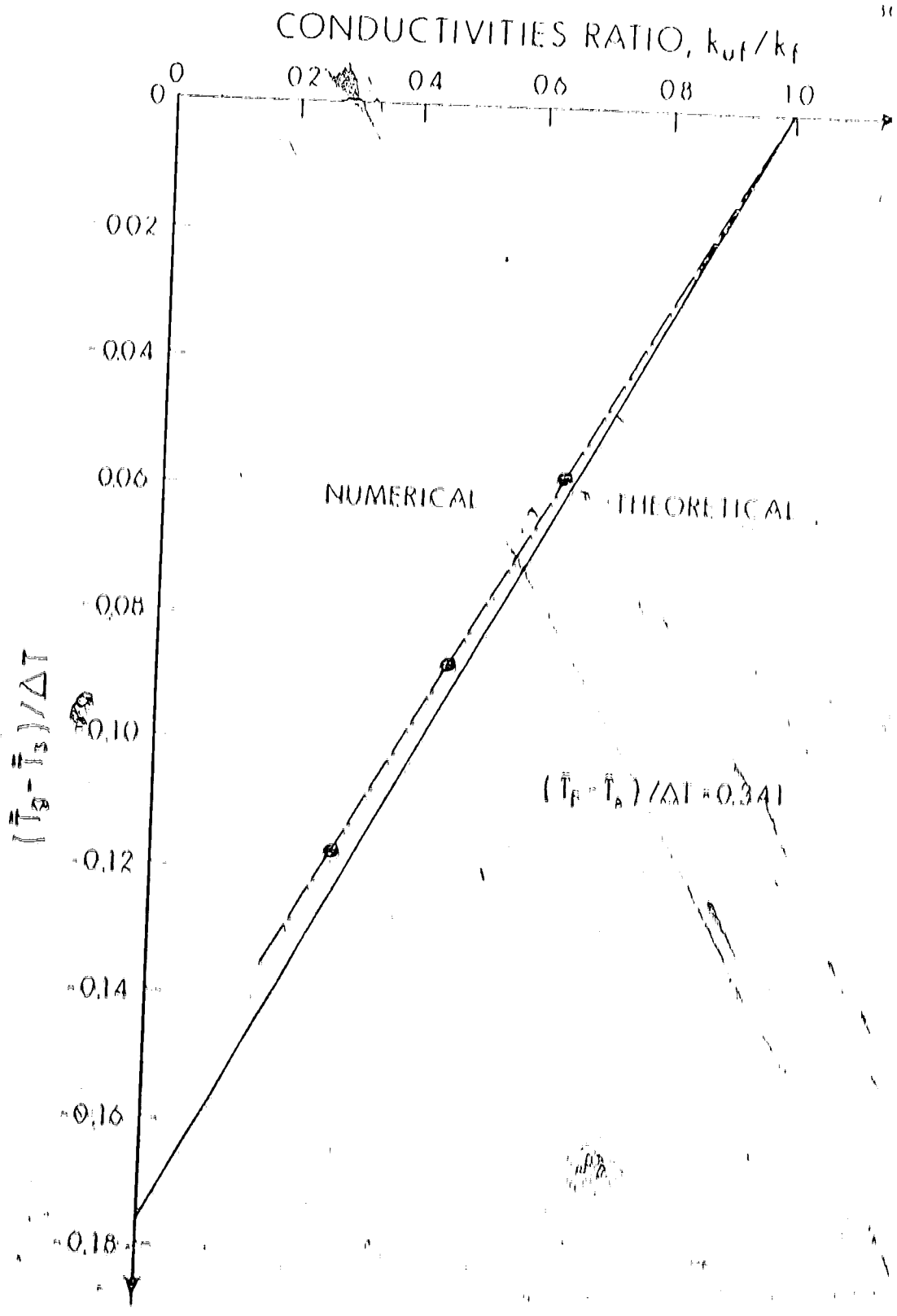


FIGURE 9 - Variation of k_{uf}/k_f with $(\bar{T}_B - \bar{T}_A)/\Delta T$

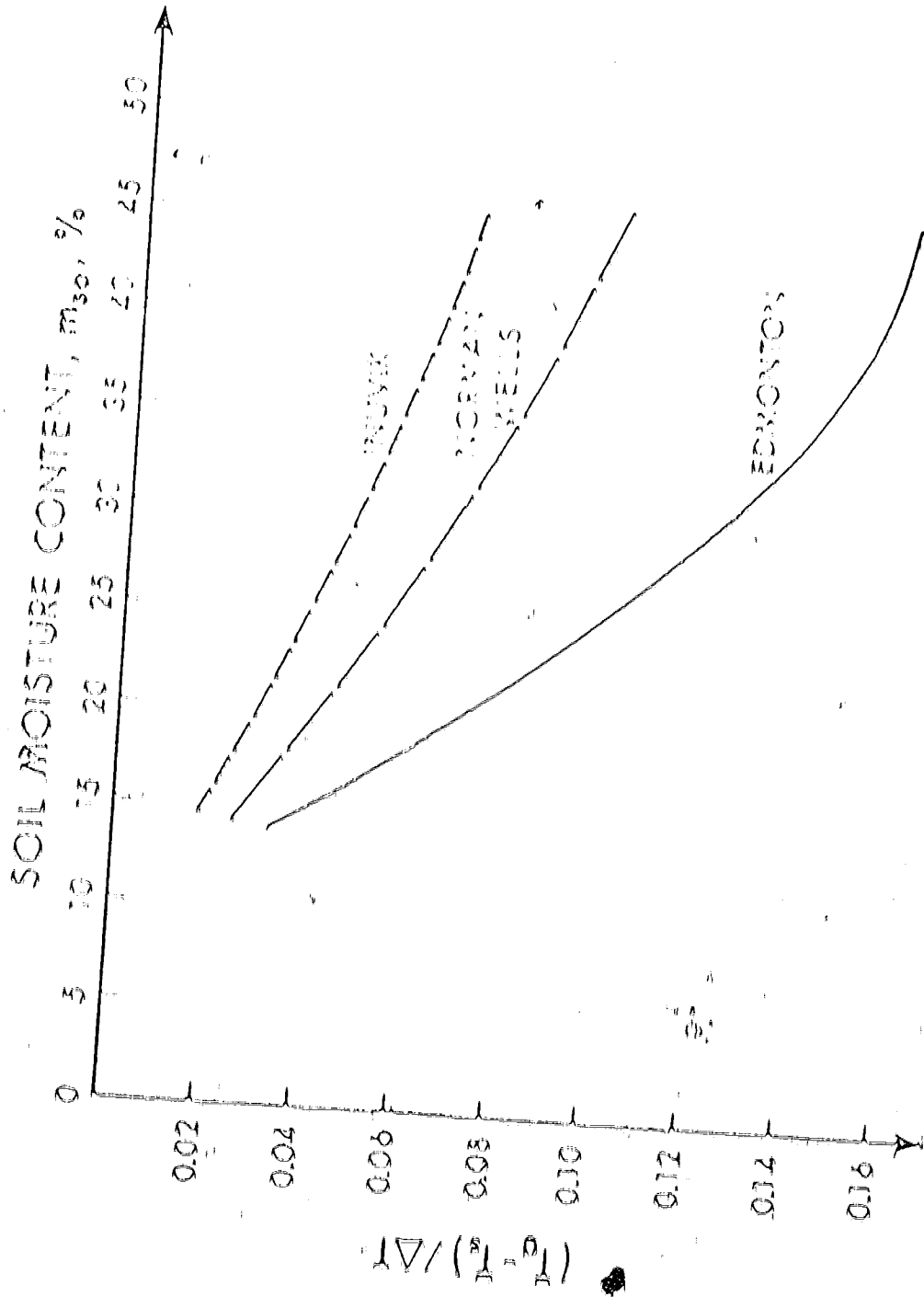


FIGURE 10 - Variation of m_{30} with $(T_1 - T_0) / \Delta T$

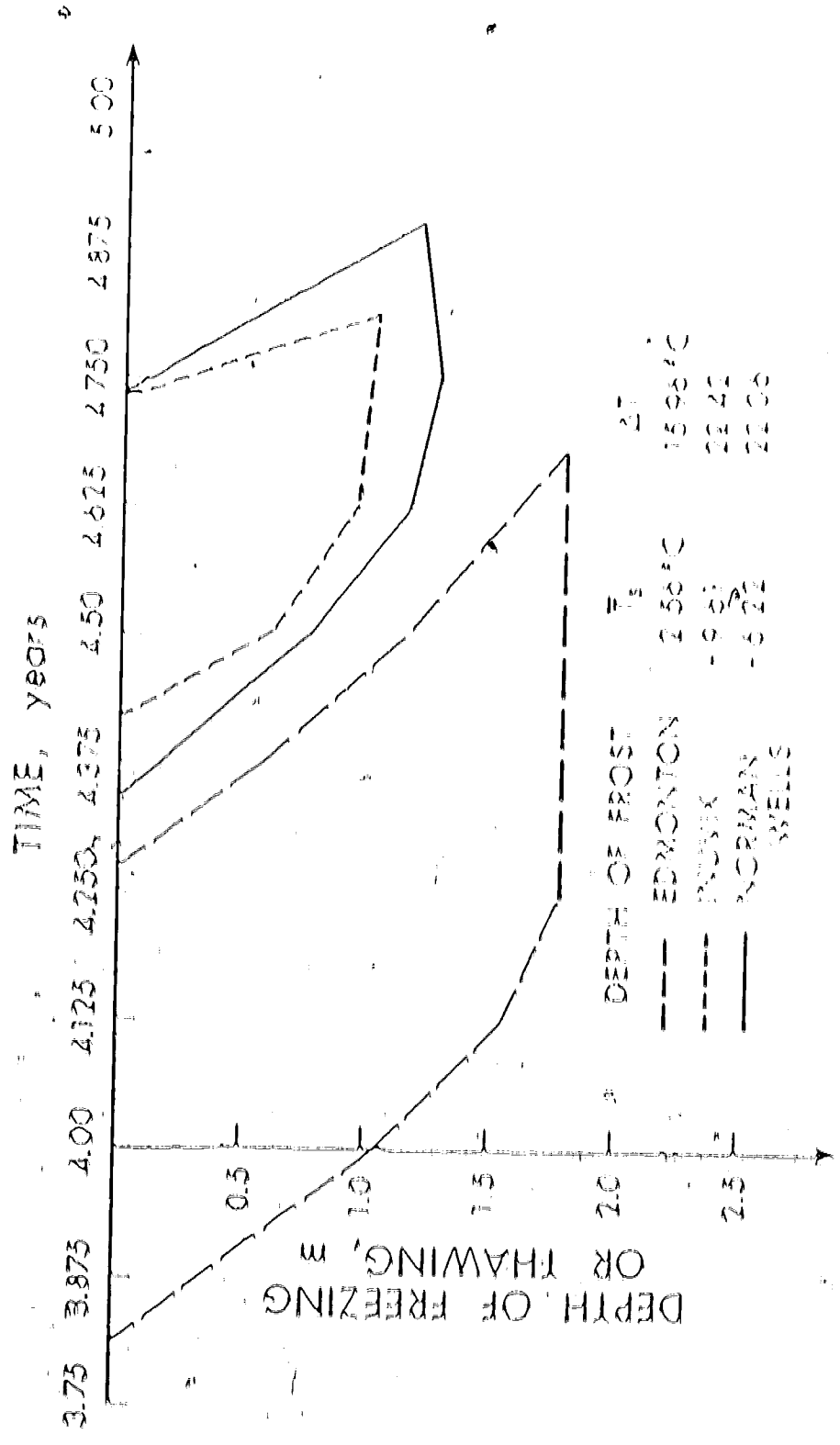


FIGURE 11 - Frost of three basin applications with imposed ground surface temperature and 30% R_s

of thaw penetration depth will be discussed briefly below.

2.6.1. Method of calculating the depth of thaw or frost penetration

Consider first the case where there is frost penetration,

that is a frozen phase over an unfrozen phase.

$$H = T_{n+1} < 0, \quad \text{then} \quad T_{n+1} = 0$$

and the depth to the fusion front is

$$y^A = n + 0.5 + \phi_{11}$$

The 0.5 occurs because the first nodal point represents only $1/2(\Delta x)$.

The latent heat of the n th nodal point is ϕ_{11} and H represents the fraction of that element unfrozen. For thaw penetration, that is an unfrozen phase over a frozen phase,

$$H = T_{n+1} > 0, \quad \text{then} \quad T_{n+1} = 0$$

and the depth to the fusion front is

$$y^A = n + 1.5 + \phi_{11}$$

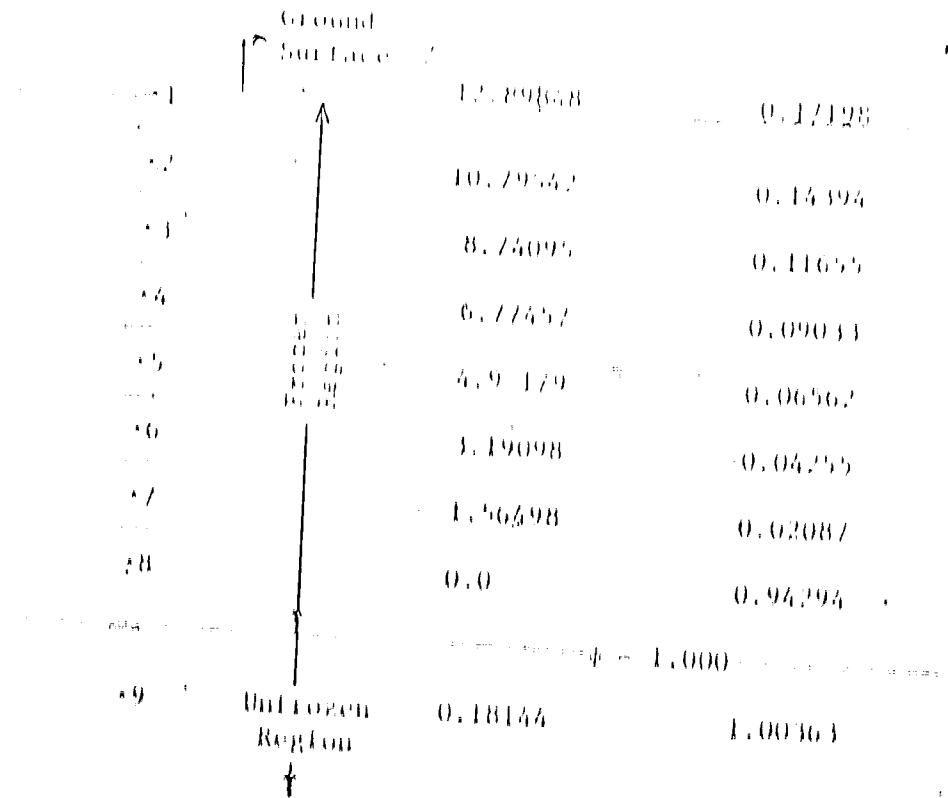
The depth of thaw or frost penetration, y , is given by

$$y = y^A \kappa_s \Delta t$$

Example:

Considering now the Edmonton case, that is, frozen/unfrozen.
 Time = 6th year

Nodal Points Temperature Enthalpy



$$y^A = 8 - 0.5 = 0.94294$$

$$= 0.56$$

$$y = y^A \cdot \kappa_v \cdot \Delta z$$

$$\Delta z = 0.06$$

$$\kappa_s = \frac{k_s \cdot c_s \cdot \Delta T}{\rho h}$$

$$= \frac{1.2 \times 3.156 \times 10^7 \times 15.96}{1.2 \times 10^8} = 7.136$$

$$\kappa_s \approx 2.6713$$

Hence the depth of Frost penetration,

$$y = (6.56 \times 0.06 \times 2.6713) \text{ meters} \\ = 1.05 \text{ meters}$$

As indicated in Figure (8), for the stated conditions, that is imposed surface temperature, average temperature $T_n = 2.56^\circ\text{C}$, amplitude of oscillation $\Delta T = 15.96^\circ\text{C}$ and with 30% moisture content, the maximum frost penetration depth is 1.8 meters, about 5.9 feet. The generally accepted and known frost penetration depth in Edmonton is about 8 feet. For the cases of Norman Wells and Inuvik, both of which are naturally permafrost locations, the maximum depths of thaw are 1.225 m (about 4 feet) and 1.00 m (about 3 feet) respectively. It will be observed that these depths of the active layer correspond to the depths over which the variation in the average ground temperature occurred in Figure (6). This is the expected result, in that it is the conductivity variations that accompany the phase changes that cause the average ground temperature to vary with depth.

2.4.2 Effects the various parameters have on the active layer

As it is possible to predict the change in the ground temperature, it is also possible to predict the change in the depth of the active layer that would result from a change in one of the controlling parameters.

Figures (12) show the effect the changes in the average surface temperature have on the depth of the active layer. A comparison with Figure (7) shows that they exhibit the same behavior with the

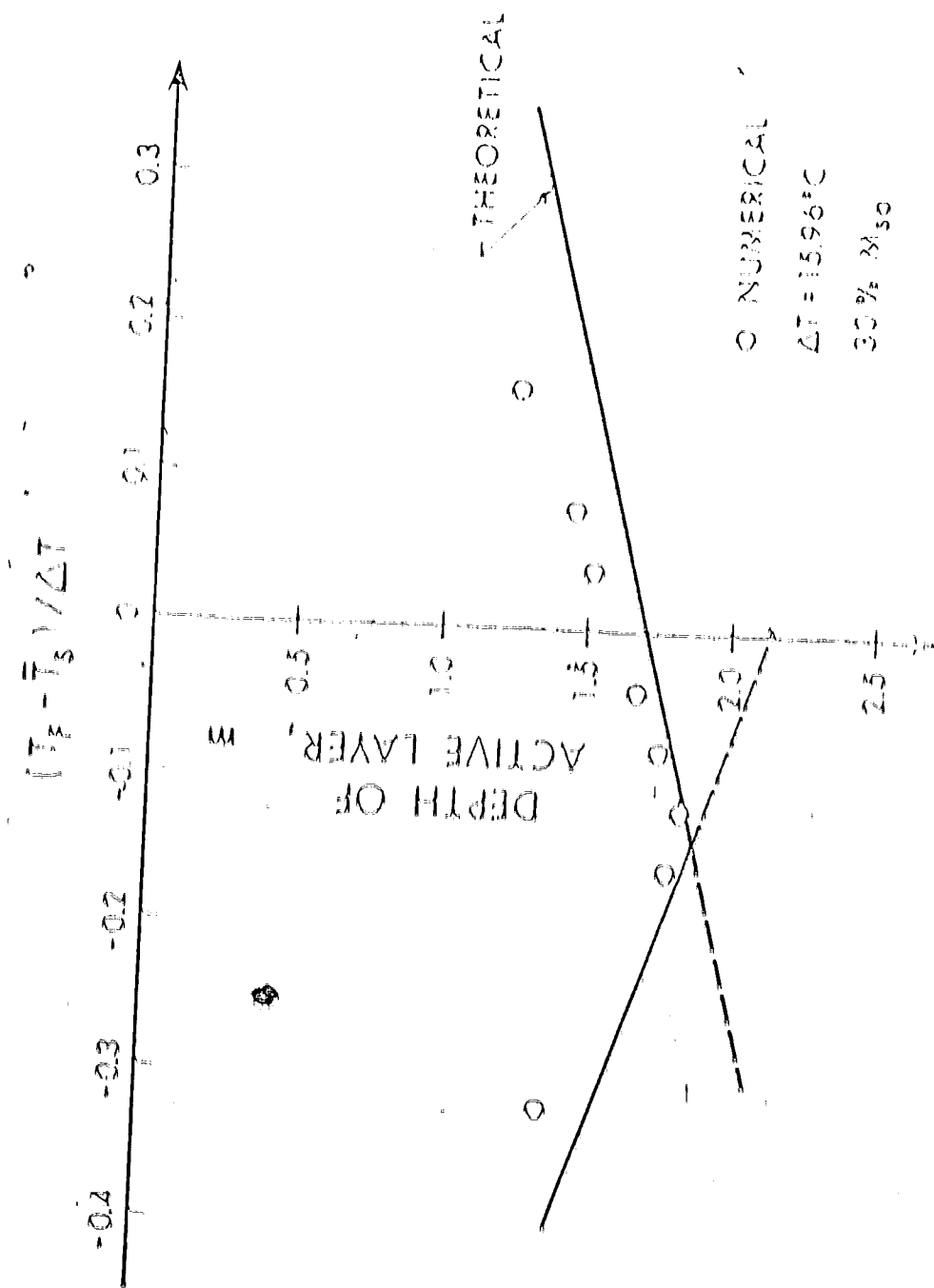


FIGURE 12 — Variation of $(T_z - T_s) / \Delta T$ with depth of active layer

maximum depth occurring when the value of $(T_F - T_B)/\Delta T = 0.14$. For a conductivity ratio k_{uf}/k_f of 0.647, which corresponds to a fine grained soil with 30% moisture content, the maximum depth of the active layer is 1.85 meters about 6.1 feet.

Again the effects of the latent heat of fusion and the conductivities of the frozen and the unfrozen soil will be examined separately and then in combination as they are related to the moisture content of the soil.

For both the naturally permafrost location and the non permafrost location, the depth of thaw and front penetration respectively decreases as the latent heat of fusion L_f increases, as indicated in Figure (13). For the Edmonton conditions, case B, the decrease in active layer depth closely follows the theoretical predictions of the analytic model. The decrease in depth is therefore proportional to $(L_f)^{-1/2}$. For the naturally permafrost conditions, case A, the numerically predicted active layer depth follows the same trend as predicted by the analytic model, however the actual values from the numerical calculations are 20 to 25% less than those predicted analytically. This accuracy of the analytical prediction is in general best, as shown in Figure (12), when the average ground temperature is near the freezing point. This is because in this condition the Biot number is near zero - the condition for which the analytical model was developed.

The analytic model would predict that the depth of the active layer for the permafrost soil is dependent only on the conductivity of the unfrozen soil and that for the non-permafrost soils the depth would depend only on the conductivity of the frozen soil. In particular the

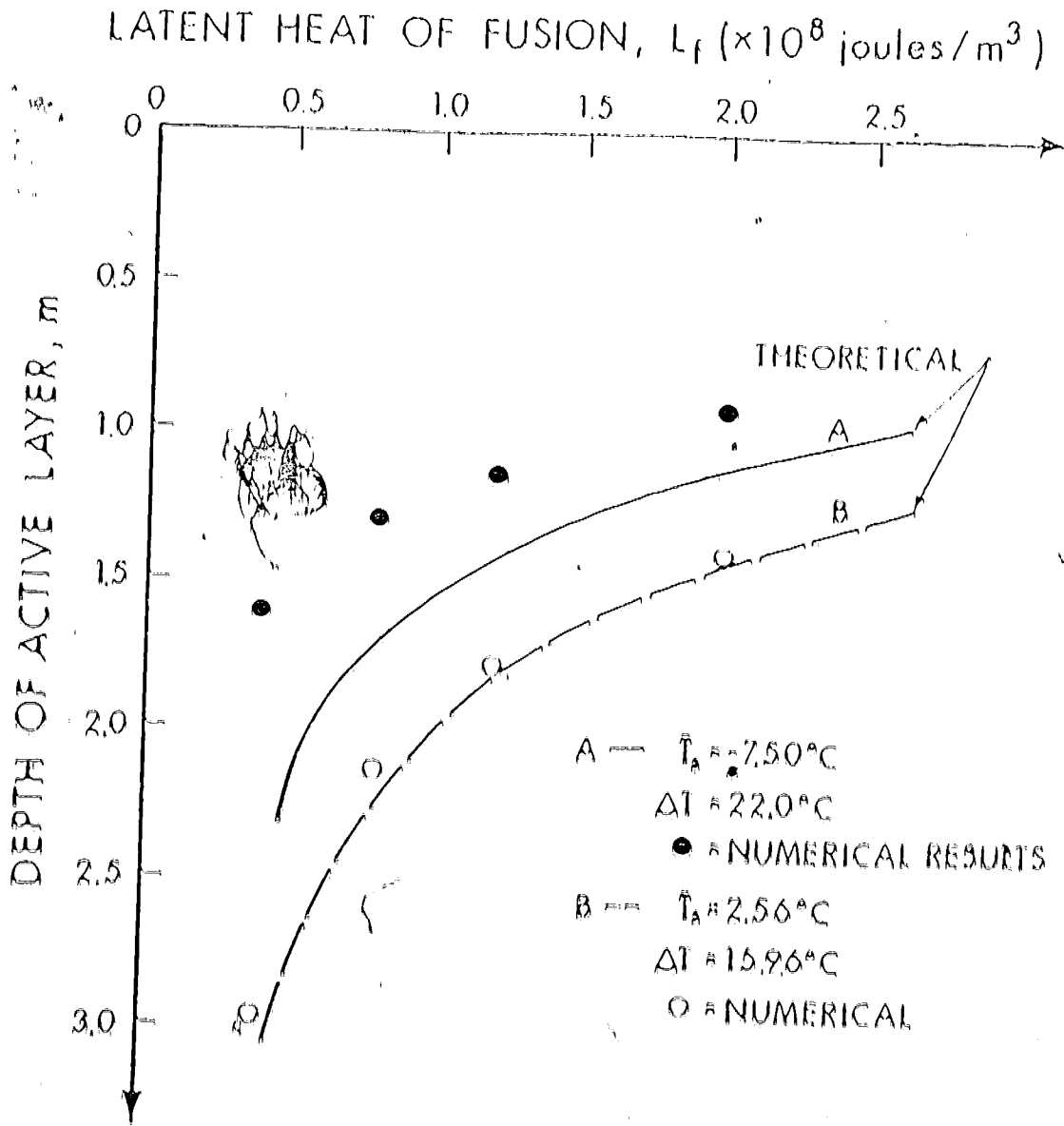


FIGURE 13 — The effects the latent heat of fusion L_f has on the depth of the active layer, for $k_{air}/k_f \approx 0.6471$

depth in each case should be proportional to the square root of the appropriate conductivity. Figure (14) shows the results of the analytic and numerical models.

Figure (15) shows the relationship between the moisture content of the soil M_{30} and the depth of the active layer for the various locations. For the naturally permafrost locations such as Inuvik and Norman Wells, the depth of the active layer decreases as the moisture content increases. As for the Edmonton case (non-naturally permafrost) it is just the opposite, that is the depth of the active layer increases as the moisture content increases.

The results for Edmonton are interesting, in that just from a consideration of the latent heat which increases with moisture content one would expect the depth of the active layer to decrease. The increase in moisture content however also increases the conductivity of the frozen ground and this acts to produce the increase in the active layer depth observed. For the two permafrost locations however, the effect of the latent heat dominates the behavior of the active layer depth. That is as the latent heat increases the active layer depth decreases.

2.5 CONCLUSIONS

This analytic model developed, based on $S_T = 0$, gives results that are in close agreement with the numerically calculated results. This result obtained demonstrates that there exists a difference between the average ground surface temperatures and the average temperatures just below the active layer and that this difference is due to conductivity changes associated with phase changes. The difference between the two

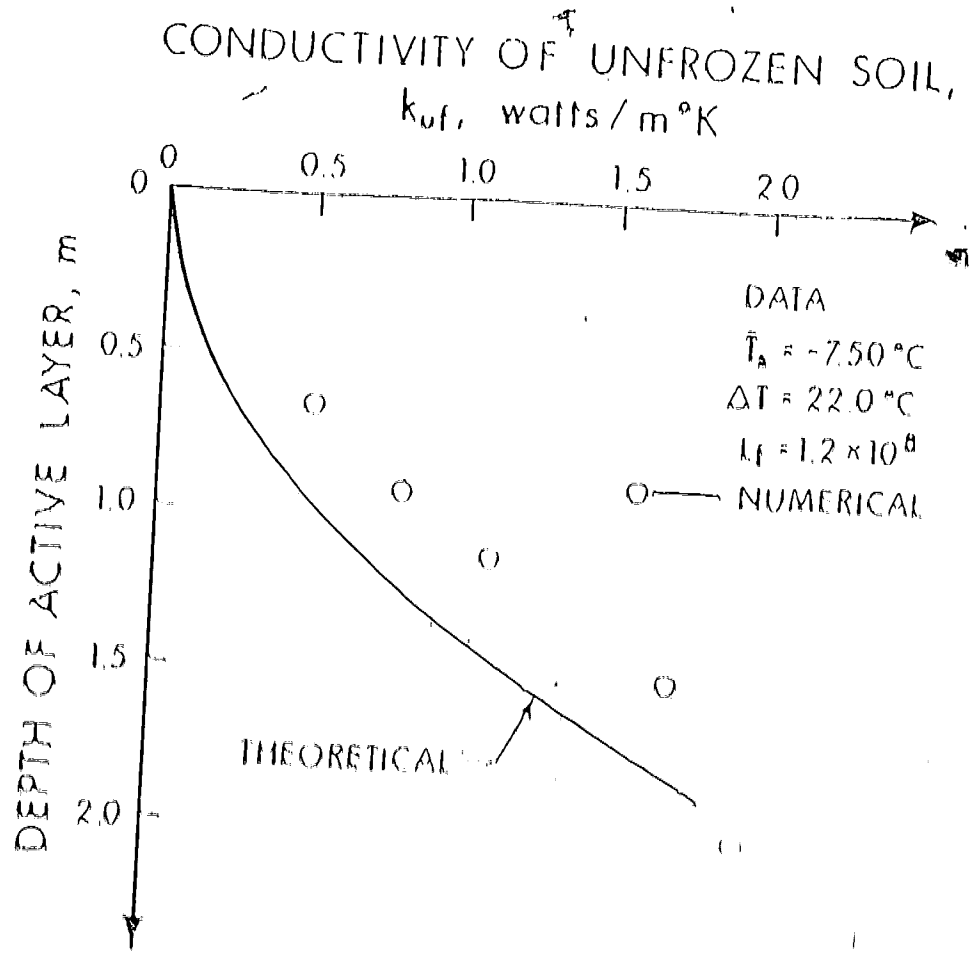


FIGURE 14 - Relationship between k_{uf} and depth of active layer in permafrost regions

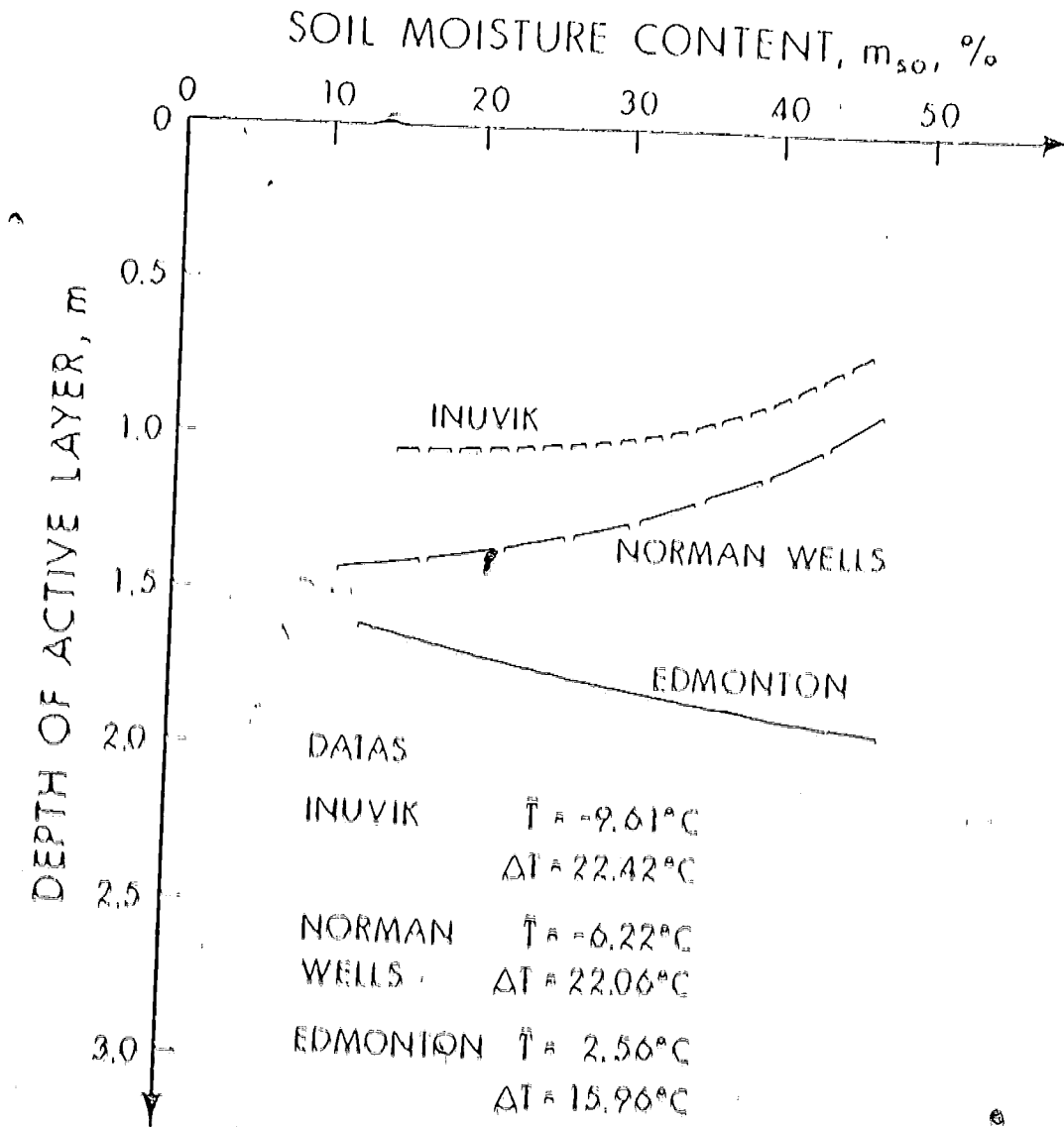


FIGURE 15 -- Variation of M_{SO} with depth of active layer

mentioned temperature is largest for soil with large moisture content and in geographical locations where the average surface temperature is near freezing.

Interestingly enough, both the analytical and the numerical models show that the average ground temperature below the active layer is independent of the latent heat of fusion. The depth of the active layer is inversely proportional to the latent heat to the one half power and would be predicted from the classical Stefan problem.

PART III

WITH AN INSULATING LAYER OF VARYING U FACTOR

3.1 Introduction

Normally, instead of the imposed surface temperature situation discussed in the last section the actual ground surface has an insulating cover (vegetation cover in the summer and snow in the winter). This insulating cover will have a major effect on the ground temperature. This section of this thesis will examine the effects of this insulating cover on the ground temperature.

The analytical model presented will neglect the effects of the latent heat of fusion on the average ground temperature and will concentrate on the effects of the insulating layer. In the numerical model all the factors will be included and then a collation will be made between the two models.

3.2 Formulation of a ground temperature model with an insulating layer of varying U-factor

3.2.1 Analytical Model

Assume that the infinite ground mass can be replaced by a plate of some effective heat capacity ρc_s . This plate is then assumed to be at the temperature of the ground surface and that no heat flows into or out of it from the bottom. The effective heat capacity of this plate would be chosen so that the plate will have the same thermal response as the ground. The effective heat capacity ρc_s will be assumed

of energy in latent heat which will affect the results as will be observed.

The model consists of a layer of insulation overlying a plate of finite heat capacity, Figure 16.

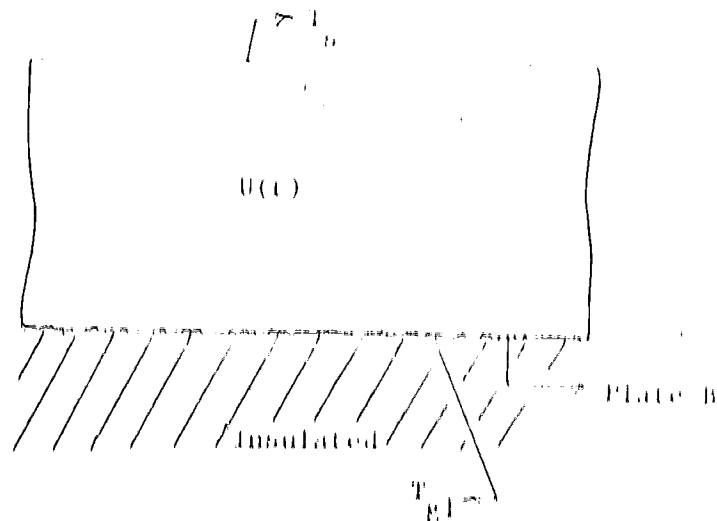


FIGURE 16

T_B ~ surface temperature

$= T_B + \Delta T_{\text{atm}}$

$U(t)$ ~ U factor of the insulating cover

$\lambda = k/d$

$U(t)$ will be assumed to be of the form $U = \lambda U_0 \sin(\omega t + \phi_M)$.

The phase angle ϕ_M is the difference in phase between the variation of the U-factor and the surface temperature. A value of $\phi_M = 0$ means that $U(t)$ is in phase with the surface temperature, that is the snow cover is winter is a better insulator than the vegetation cover in the summer. The normal case one would expect would be ϕ_M near zero. When $\phi_M = \pi$

$U(t)$ is 180° out of phase with the surface temperature, $\sin(\omega t + \pi)$ that the summer vegetation cover is a better insulator than the snow in winter. This case would not be as likely, however, it could possibly occur if the ground were covered with a thick layer of moss which became very dry during the summer months.

3.2.2. Solution.

For this model the governing equation for the ground surface temperature T_{gl} is

$$\rho c_v (dT_{gl}/dt) = U(t) [T_a(t) - T_{gl}] \quad (3.1)$$

$$\text{where } T_a(t) = T_a + \Delta T \sin(\omega t) \quad (3.2)$$

$$U(t) = U + \Delta U \sin(\omega t + \alpha) \quad (3.3)$$

and T_{gl} is a function of time, t .

Let

$$u = T_{gl}^3 - T_a^3(t)$$

$$\frac{1}{u} du/dt = dT_{gl}/dt$$

$$\text{hence } dT_{gl}^3/dt = du/dt + dT_a^3/dt$$

$$\text{but } dT_a^3/dt = \omega \Delta T^3 \cos \omega t$$

which means that

$$dT_{gl}^3/dt = du/dt + \omega \Delta T^3 \cos \omega t \quad (3.4)$$

Substituting equation (3.3) into equation (3.1), it gives

$$U \frac{dU}{dt} + U \cos(\theta) = U(t) \quad (3.4)$$

or

$$dU/dt + U(t) \cos(\theta) = U(t) \quad (3.5)$$

From equation (3.4)

$$U(t) = U(t) \sin(\theta) + \frac{dU}{dt} \cos(\theta) \quad (3.6)$$

Let

$$U/\cos(\theta) = A e$$

$$\Delta U \cos(\theta) / \cos(\theta) = B \quad (3.7)$$

$$\Delta U \sin(\theta) / \cos(\theta) = C$$

Equation (3.5) will now become

$$dU/dt + (A + B \cos(\theta) + C \sin(\theta)) U = U(t) \cos(\theta) \quad (3.8)$$

Assume

$$U = a_0 + \sum_{n=1}^m [a_n \ln(\cos(\theta)) + b_n \cos(\theta)] \quad (3.9)$$

$$\frac{dU}{dt} = \sum_{n=1}^m [a_n \sin(\theta) - b_n \sin(\theta)] \quad (3.10)$$

Substituting equations (3.9) and (3.10) into equation (3.3), the explicit form becomes

$$\begin{aligned} & \sum_{n=1}^{\infty} \left[\frac{(a_n - b_n) \cos(n\pi t)}{n} - \frac{b_n (n-1) \ln(n-1)}{n} \right] + \left[\frac{c_n}{n} + \frac{d_n}{n-1} \right] \ln(n) \\ & + \frac{1}{2} \left[\frac{c_n}{n} + \frac{d_n}{n-1} \right] \ln(n-1) + \frac{1}{2} (a_n - b_n) \cos(n\pi t) + \frac{1}{2} (a_n - b_n) \cos(n\pi t) \end{aligned} \quad (3.11)$$

Multiplying out the second term on the right hand side and then regrouping the common terms, equation (3.11) becomes

$$\begin{aligned} \Delta a_n + B a_n \ln(n) + C a_n \cos(n\pi t) + \frac{1}{2} (A a_n - m b_n) \ln(n) \\ + (A b_n - m a_n) \cos(n\pi t) + 1/2 (a_n - b_n) \ln(n-1) \\ + 1/2 (a_n - b_n) \cos(n-1) + 1/2 (b_n - B a_n) \ln(n+1) \\ + 1/2 (b_n - C a_n) \cos(n+1) + m \Delta T \cos(n\pi t) \end{aligned} \quad (3.12)$$

or simplifying further by letting

$$n = 1, 2, \dots, N, \dots$$

Constant term

$$\Delta a_0 + 1/2 (B a_1 + C b_1) \dots$$

Linear term

$$B a_1 + (A a_1 - m b_1) + 1/2 (C a_2 - B b_2) \dots$$

The matrix in equation (3.14) is of infinite dimensions and thus exact values of the coefficients cannot be obtained in this way. However, under certain conditions the matrix may be approximated by truncating it at a certain value of N . In particular it was found by expansion of the first few terms, that if ω/ω_c and ω/ω_p are small parameters, the coefficients of 'a' and 'b' become small for large values of N . The approximation obtained by setting $a_n = 0$ and $b_n = 0$ for $n \geq 2$ will be obtained here. The matrix will be reduced to

$$\begin{pmatrix} A & B/2 & C/2 \\ B & A & -\omega \\ C & \omega & A \end{pmatrix} \begin{pmatrix} a_0 \\ a_1 \\ b_1 \end{pmatrix} = \begin{pmatrix} 0 \\ 0 \\ \omega\Delta T \end{pmatrix} \quad (3.15)$$

Solving for a_0 , a_1 and b_1

$$a_0 = \begin{pmatrix} 0 & B/2 & C/2 \\ 0 & A & -\omega \\ \omega\Delta T & -\omega & A \end{pmatrix}^{-1} \begin{pmatrix} 0 \\ 0 \\ \omega\Delta T \end{pmatrix}$$

$$a_0 = \frac{\omega\Delta T(-\omega B/2 - CA/2)}{\Lambda(\Lambda^2 - \omega^2)} = B/2(\Lambda B/Ca) + C/2(Ba - CA)$$

$$\approx \Delta T B/2\Lambda \{1 + CA/Ba - 1/\omega^2(\Lambda^2 - B^2/2 - C^2/2)\}$$

hence

$$a_0 = \frac{\Delta U/U}{\omega} \approx \frac{\cos \phi_u}{\omega} \left(1 + \frac{U}{\rho C_{\infty} \omega} \cos \phi_u + \frac{U^2}{(\rho C_{\infty} \omega)^2} + \frac{2U^2}{2(\rho C_{\infty} \omega)^2} \right) \quad (3.16)$$

and finally

$$a_1 = \begin{vmatrix} \Lambda & 0 & C/2 \\ B & 0 & -\omega \\ C & -\omega \Delta T & \Lambda \\ \Lambda & B/2 & C/2 \\ B & \Lambda & -\omega \\ C & \omega & \Lambda \end{vmatrix} \quad (3.17)$$

$$a_1 = \frac{\Lambda(\omega^2 \Delta T) + 2(-B\omega \Delta T)}{\Lambda(\Lambda^2 \omega^2) - \frac{B}{2}(ABCC\omega) + \frac{C}{2}(B\omega - C\Lambda)}$$

$$a_1 \approx -\Delta T \left(1 + \frac{CB}{2\Lambda\omega} - \frac{1}{\omega^2} \left(\Lambda^2 - \frac{B^2}{2} - \frac{C^2}{2} \right) \right)$$

hence

$$a_1 \approx -\Delta T + \Delta T \left(\frac{U}{\rho C_{\infty} \omega} + \frac{U}{\rho C_{\infty} \omega} + \frac{U}{2\rho C_{\infty} \omega} \left(\frac{\Delta U}{U} \right)^2 \right) \\ = \frac{1}{4} \left(\frac{\Delta U}{U} \right)^2 \frac{1}{\omega} \ln \left(\frac{2\phi_u}{\omega} \right) \quad (3.18)$$

and finally

$$b_1 = \begin{vmatrix} \Lambda & B/2 & 0 \\ B & \Lambda & 0 \\ C & \omega & -\omega \Delta T \\ \Lambda & B/2 & C/2 \\ B & \Lambda & -\omega \\ C & \omega & \Lambda \end{vmatrix} \quad (3.19)$$

$$\begin{aligned}
 & \frac{A(\Delta\omega\Delta I_B)}{\omega^2\Lambda(1 + \frac{1}{\omega^2}(\Lambda^2 - \frac{B^2}{2} - \frac{C^2}{2}))} - \frac{B}{2}(\frac{B^2}{2} - \frac{C^2}{2}) \\
 \therefore b_1 &= \frac{\Delta T B^2}{2\omega\Lambda} \left(1 - \frac{2\Lambda^2}{B^2}\right) \times \left(1 - \frac{1}{\omega^2}(\Lambda^2 - \frac{B^2}{2} - \frac{C^2}{2})\right)
 \end{aligned}$$

Neglecting all the higher terms, i.e. terms with $(U/vc_e\omega)^2$ or higher

$$\therefore b_1 \approx \frac{\Delta T B^2}{2\omega\Lambda} \left(1 - \frac{2\Lambda^2}{B^2}\right)$$

or

$$b_1 \approx \Delta T \left(\frac{U}{vc_e\omega}\right)^2 \left(1 + \frac{1}{4}\left(\frac{\Delta U}{U}\right)^2 + \frac{1}{4}\left(\frac{\Delta U}{U}\right)^2 \cos 2\phi_u\right) \tag{3.20}$$

From equation (3.9)

$$\begin{aligned}
 0 &= a_0 + \sum_{n=1}^{\infty} (a_n \sin(n\omega t) + b_n \cos(n\omega t)) \\
 &= a_0 + a_1 \sin\omega t + b_1 \cos\omega t + \dots
 \end{aligned} \tag{3.21}$$

Substituting equations (3.16), (3.18) and (3.20) into equation (3.21), the following expression follows:

$$\begin{aligned}
 0 &= \frac{\Delta U}{U} \Delta T \frac{\cos\phi_u}{2} \left(1 + \frac{U}{vc_e\omega} \frac{\sin\phi_u}{\cos\phi_u} + \frac{U^2}{(vc_e\omega)^2} + \frac{\Delta U^2}{2(vc_e\omega)^2}\right) \\
 &+ (\Delta T) + \Delta T \frac{U}{vc_e\omega} \left(\frac{U}{vc_e\omega} + \frac{U}{2vc_e\omega} \left(\frac{\Delta U}{U}\right)^2\right) \\
 &= \frac{1}{4} \left(\frac{\Delta U}{U}\right)^2 \sin(2\phi_u) \sin\omega t
 \end{aligned}$$

$$T = (\Delta T_0) \frac{U}{\rho c_A \omega} \left(1.5 + \frac{1}{4} \left(\frac{\Delta U}{U} \right)^2 + \frac{1}{4} \left(\frac{\Delta U}{U} \right)^2 \cos 2\phi_M \right) + \text{const} \quad (3.22)$$

To obtain the difference between the average temperature $(\bar{T}_{E1} - \bar{T}_0)$, equation (3.22) may be integrated over a time cycle. The first term and the const term will drop out, leaving the following expression:

$$\int_0^{2\pi} \omega T dt = \int_0^{2\pi} \omega \frac{\Delta U}{U} \Delta T \frac{\cos \phi_M}{2} \left(1 + \frac{U^2}{\rho c_A \omega} \frac{\sin \phi_M}{\cos \phi_M} \right) dt$$

$$= \left(\frac{U^2}{(\rho c_A \omega)^2} + \frac{\Delta U^2}{2(\rho c_A \omega)^2} \right) \int_0^{2\pi} \omega dt$$

but

$$\int_0^{2\pi} \omega dt = 0(2\pi/\omega)$$

hence

$$0 = \frac{\Delta U}{U} \Delta T \frac{\cos \phi_M}{2} \left(1 + \frac{U^2}{\rho c_A \omega} \frac{\sin \phi_M}{\cos \phi_M} \right) \left(\frac{U^2}{(\rho c_A \omega)^2} + \frac{\Delta U^2}{2(\rho c_A \omega)^2} \right) \quad (3.23)$$

In the limit $U/\rho c_A \omega \rightarrow 0$, noting that ΔU must by definition be less than U , equation (3.23) gives

$$\frac{(\bar{T}_{E1} - \bar{T}_0)/\Delta T}{\cos \phi_M} \approx (\Delta U/U) (\cos \phi_M / 2) \quad (3.24)$$

Also note that from equation (3.22), that this limiting situation implies that the ground temperature variations with time are negligible. In this case the temperature difference $(\bar{T}_{E1} - \bar{T}_0)$ varies linearly with $\Delta U/U$ and in the cosine of the phase angle, ϕ_M . This implies that

For $\phi_u = 0$, that is for a snow cover of good insulating value, the average ground surface temperature T_{gl} is greater than the average surface temperature T_b . For $\phi_u = 180^\circ$, that is the summer vegetation cover is a better insulator, the average ground surface temperature T_{gl} is less than the average surface temperature T_b . There is no difference between T_{gl} and T_b if however $\phi_u = 90^\circ$. If $U/\rho c_s \omega$ is not zero, that is if the fluctuations of the ground temperature are accounted for, equation (3.23) shows that a difference between T_{gl} and T_b exists at $\phi_u = 90^\circ$. The amount of the difference is

$$(T_{gl} - T_b) = (\Delta U \kappa \Delta T) / 2 \rho c_s \omega \quad (3.25)$$

For larger values of $U/\rho c_s \omega$ and $\Delta U/U$, higher order terms become more significant and equation (3.24) is not a good approximation.

3.3 Numerical Model

The method adopted here will be similar to that developed for the imposed surface temperature except for a slight modification at the soil surface level.

3.3.1 The difference between this and the previous model

(a) At the ground surface with imposed surface temperature

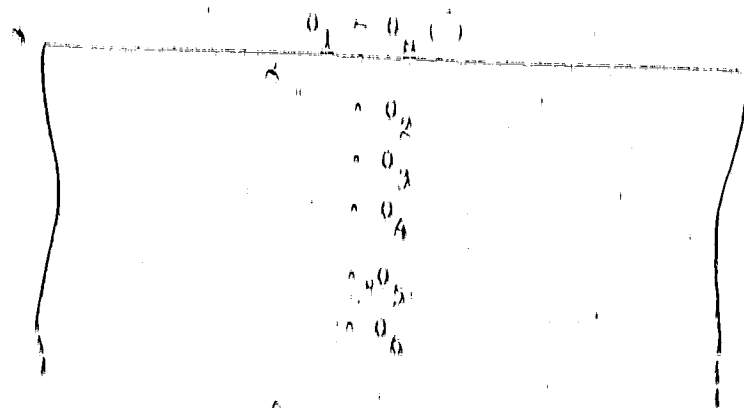


FIGURE 12 (a)

Temperature

$$0_1 = 0_2$$

Enthalpy

$$\phi_1 = 1 + 5_{01} 0 \quad \text{If } 0 = 0$$

$$= 5_1 0 \quad \text{If } 0 \neq 0$$

(b) With an Insulating Layer

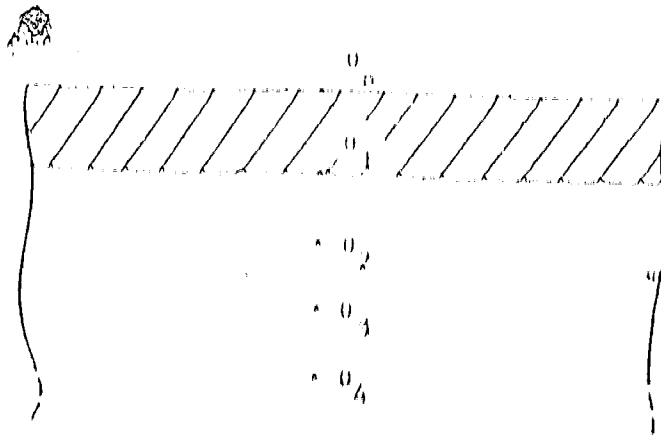


FIGURE 17(b)

At the solid surface

$$dh_1/\rho L = (U(T_B - T_1) - (T_1 - T_2)k/\Delta x) \Delta t/\Delta x = 2 \kappa \lambda/\rho L \quad (3.26)$$

or

$$\Delta \phi_1 = (U \Delta T/\rho L - (0_1 - 0_2) \Delta x) \Delta t = k \Delta T/\rho L (0_1 - 0_2) \dagger 2 \Delta t/\Delta x^2$$

or

$$= (U \Delta T \lambda/\rho L \kappa - (0_1 - 0_2) \Delta x) \Delta t = k^2 (0_1 - 0_2) \dagger 2 \Delta t/\Delta x^2$$

$$= (FUS(0_1 - 0_2) \Delta x - k^2 (0_1 - 0_2) \dagger) 2 \Delta t/\Delta x^2 \quad (3.27)$$

where

$$FUS = U \Delta T \lambda/\rho L \kappa$$

but since

$$\chi_c^2 = k_1 U_c \Delta T / L$$

$$\therefore EUB = (b_c / k_1) \quad (3.28)$$

Initially the ground temperature were all assumed to be equalled to T_0 or $\theta_1 = \theta_2 = \dots = \theta_n = 0$.

Enthalpy

$$\Delta\phi_1 = (EUB)(\theta_n - \theta_1) \frac{2\Delta c_c k RT}{L} + (\theta_1 - \theta_2) \frac{2\Delta c RT}{L} \quad (3.29)$$

where $RT = \Delta t / \Delta c_c^2$ (3.30)

and $RT = \Delta t / \Delta c_c^2$ (3.31)

$$\sim (k_{ad} / k_1) \Delta t / \Delta c_c^2$$

if $\theta_1 + \theta_2 < 0$
if $\theta_1 + \theta_2 > 0$

$$\phi_1^{m+1} = \phi_1^m + \Delta\phi_1^m$$

Temperature

*

$$\theta_1^{m+1} = \phi_1^{m+1} / \beta_1 \quad \text{if } \phi_1^{m+1} < 0$$

$$= (\phi_1^{m+1} - 1) / \beta_{ad} \quad \text{if } \phi_1^{m+1} > 0 \quad (3.32)$$

The main difference between the two problems is, instead of having the ground surface temperatures equal to the air temperatures, the ground surface temperatures will now be varying. This is achieved by the introduction of a varying amplitude layer which separates the atmosphere

from the ground surface. The temperature on the surface of the insulating layer is now assumed to be equalled to the air temperature.

So, except for these modifications at the ground surface level, all the procedures and parameters remain unchanged. The varying insulating layer is approximated by a harmonic function which may or may not be in phase with the air temperature. Again temperature conditions of Edmonton, Inuvik and Fort-in-Klein were used as the basic cases.

3.4 Results of the Numerical Calculations.

3.4.1 Effects of the insulating layer have on the various temperatures.

The temperature profile, as illustrated in Figure 13, display similar characteristics as those for the imposed surface temperature, except that the insulating layer decreases the amplitude of the temperature variation at the ground surface. The amplitude of surface and ground surface temperature variations for the three cases, together with the soil and insulating properties are listed in Table 1. The properties of the insulating layer chosen for the three cases are rather arbitrary as there is very little data available on the insulating values of the various surface covers. Some rational range of values used is given in Appendix III. In all cases the phase angle ϕ_m is 0.0, that is the situation in which the winter snow cover is better insulator than the summer vegetation cover.

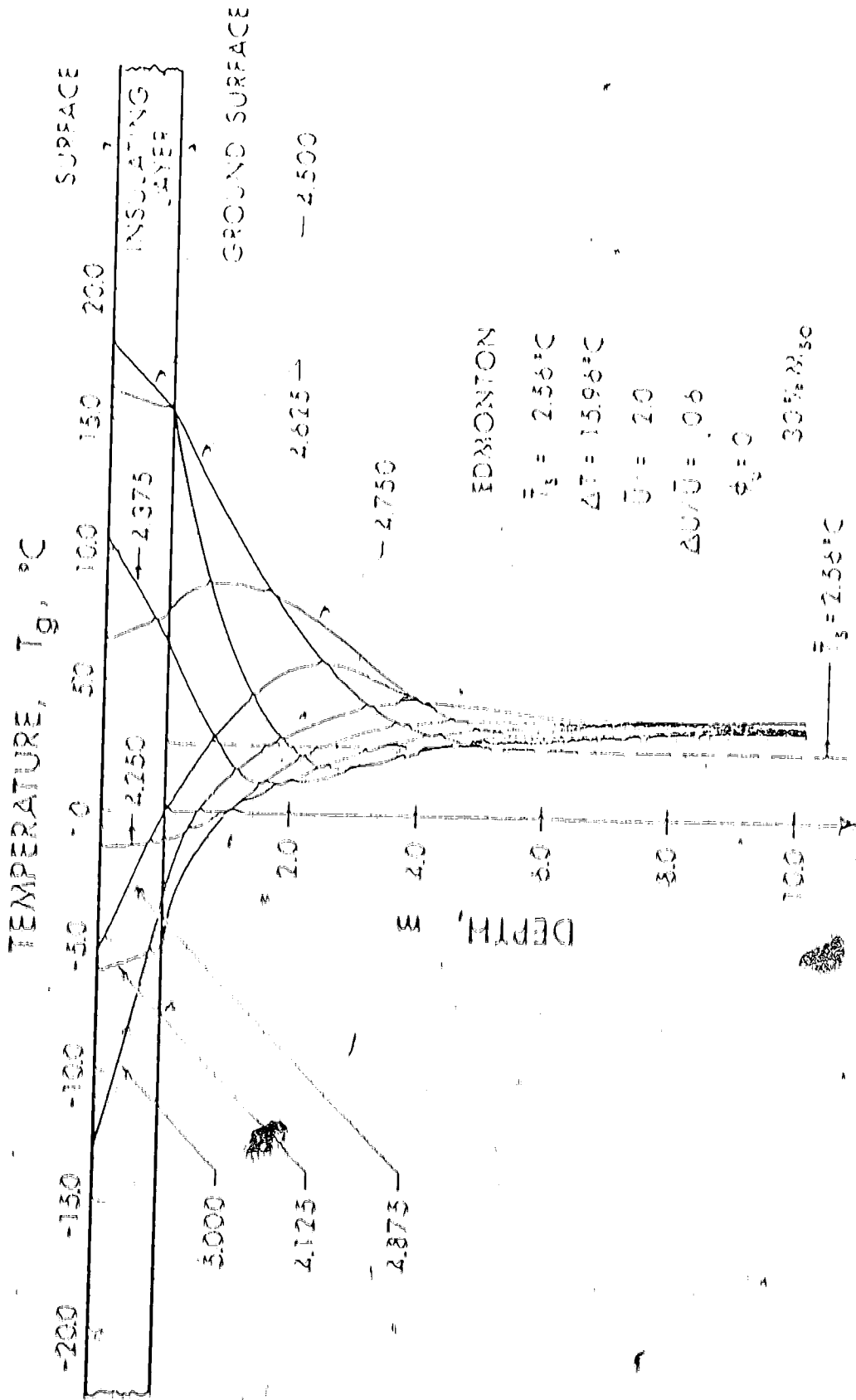


FIGURE 18 — TEMPERATURE DISTRIBUTIONS AT EDMONTON WITH IMPOSED SURFACE TEMPERATURE

TABLE 1. Amplitude of Temperature Variation

Locations, soil and insulating properties	Amplitude of surface temperature variations ($^{\circ}\text{C}$)	Amplitude of ground surface temperature variations ($^{\circ}\text{C}$)
Edmonton, $T_{\text{a}} = 2.56^{\circ}\text{C}$, $\Delta T = 15.96^{\circ}\text{C}$ $U = 2.0$, $\Delta U/U = 0.6$ $\phi_{\text{a}} = 0.0$, $H_{\text{so}} = 10$ Normal 15 E151	13.1 $^{\circ}\text{C}$ to 12.3 $^{\circ}\text{C}$	1.5 $^{\circ}\text{C}$ to 2.1 $^{\circ}\text{C}$
Normal 15 E151 $T_{\text{a}} = -0.22^{\circ}\text{C}$, $\Delta T = 22.06^{\circ}\text{C}$ $U = 0.75$, $\Delta U/U = 0.8$ $\phi_{\text{a}} = 0.0$, $H_{\text{so}} = 30$	15.1 $^{\circ}\text{C}$ to 22.6 $^{\circ}\text{C}$	6.1 $^{\circ}\text{C}$ to 2.5 $^{\circ}\text{C}$
Inuvik $T_{\text{a}} = -9.61^{\circ}\text{C}$, $\Delta T = 22.42^{\circ}\text{C}$ $U = 0.75$, $\Delta U/U = 0.8$ $\phi_{\text{a}} = 0.0$, $H_{\text{so}} = 30$	12.0 $^{\circ}\text{C}$ to 31.3 $^{\circ}\text{C}$	3.0 $^{\circ}\text{C}$ to 18.7 $^{\circ}\text{C}$

The amplitude of temperature variation at Normal 15 E151 decreases the most because the average ground temperature is closer to 0°C , the temperature where the phase change occurs. As indicated by Figure (10), the temperature deep in the ground is greater than the average surface temperature T_{a} . This is true for all the three cases. Hence with the introduction of an insulating layer in phase with the air temperature, the ground temperature becomes warmer. This is the result predicted by the analytical model.

Figure (17) shows the variation of the average ground temperature with height, with an insulating layer in phase with the air temperature. The temperature profiles show that the average temperatures close to the ground are less than the average ground surface temperature. The results are in agreement with and have been dealt with in the previous section where the surface temperatures were imposed. The temperature profiles also show that the average ground surface temperatures T_{g1} are greater than the average surface temperatures T_a . These results are predicted by the analytical model. The differences between the two average temperatures T_{g1} and T_a are very significant for both the Jordan Wells and Inuyk cases, both of which are in the naturally permafrost locations. Also indicated are that the differences between T_a and T_{g1} are very much greater than that between T_{g1} and T_g , implying that the effects of the insulating layer are much greater than the effects of the thermal properties of the soil.

3.4.2 Effects the Insulating Layer have on $(T_a - T_{g1})$

This section will deal mainly with the effects the insulating layer has on the difference between the average top surface temperature T_a and the average ground surface temperature T_{g1} . The main parameters of the insulating layer U are, the average insulating cover U , the amplitude of variation ΔU and the phase angle ϕ_u . It should be noted that the theory used in the figures would only be the "simple theory" or equation (3.24), which is the dominant term of the more complicated theory or equation (3.23).

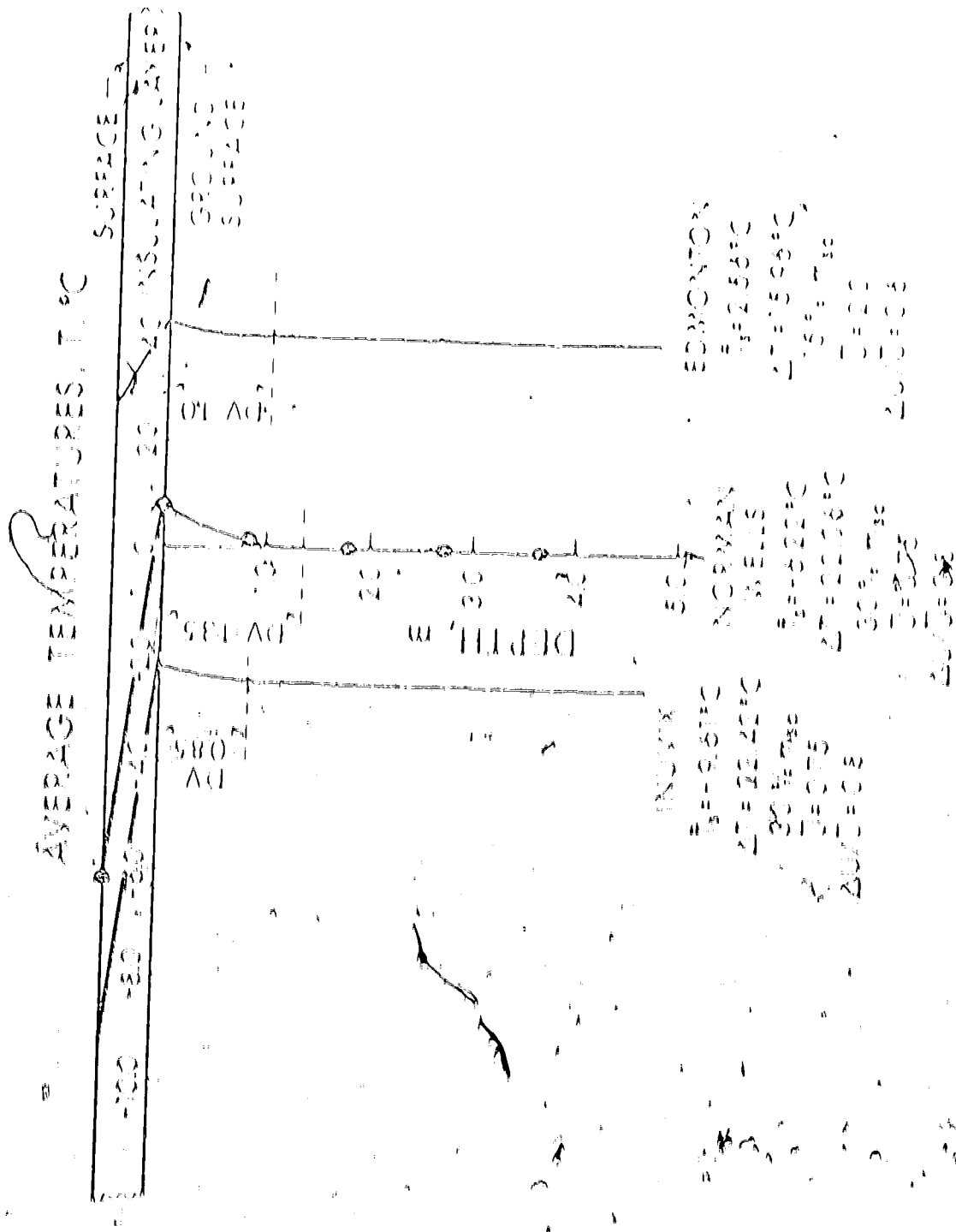


FIGURE 19 — PROFILE OF AVERAGE TEMPERATURE WITH IMPOSED SURFACE TEMPERATURE AND $t_a = 0.0$

Equation (3.23) is the "simple theory" where $T_{RL} = T_{in}$

$$T_{RL} = T_{in} - \frac{U}{h_{eff}} \Delta T$$

At $\phi_{in} = 0$ in the general case $h_{eff} = \frac{h_{in}}{1 + \frac{U}{h_{in}} \Delta T}$. In the numerical model, the more complicated theory of equation (3.23) was not used in the "simple theory" because of the difficulty in predicting the value of the effective heat capacity of the wall, C_{eff} , because the effective heat capacity, C_{eff} , varies as any one of the controlling parameters varies. Attempts were made to obtain a suitable average value of C_{eff} and substituting it into the more complicated theory to obtain a theoretical curve that would match the numerical results more closely. This was not possible in general because of the sensitivity of C_{eff} to the other parameters of the problem.

The effects the ratio AU/B has on $(T_{RL} - T_{in})/\Delta T$ are shown in Figure (10) for two cases, one when the insulating layer is in phase $\phi_{in} = 0.0$, while the other when the insulating layer is 180° out of phase ($\phi_{in} = 0.5$) with the air temperature. For the case when $\phi_{in} = 0.0$, $(T_{RL} - T_{in})/\Delta T$ increases almost linearly as AU/B increases and the numerical result is about 25% less than the results of the simple theory. In neglecting the term $\frac{U}{h_{eff}} \Delta T$ when $\phi_{in} = 0.5$, $(T_{RL} - T_{in})/\Delta T$ decreases as AU/B increases. The numerical results here are about 25% more than the results of the simple theory. The big difference between the simple theory results and the numerical results is mainly because of the assumption in the simple theory that $U/h_{eff} \Delta T = 0.0$. An inspection of the more complicated theory, equation (3.23), indicated

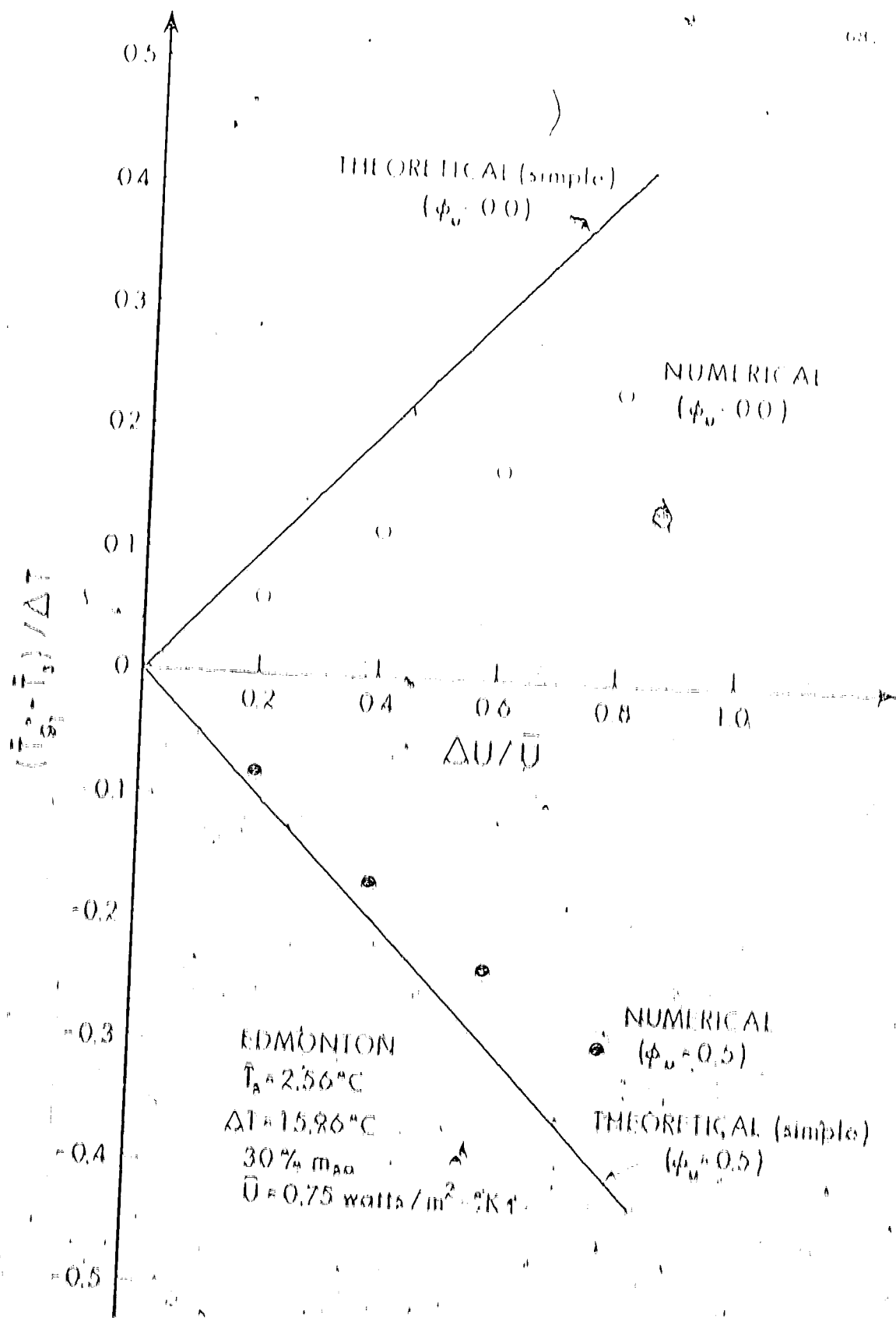


FIGURE 20 Variation of $\Delta U / \bar{U}$ with $(\bar{T}_u - \bar{T}_s) / \Delta T$.

that any finite value in $(T_{RI} - T_n)/\Delta T$ would decrease the absolute value of $(T_{RI} - T_n)/\Delta T$, hence closer to the numerical results.

Figure (21) shows the relationship between $(T_{RI} - T_n)/\Delta T$ and θ for a given value of ϕ_{II} . This insulating layer model will approach that of the imposed surface temperature model when θ approaches infinity. Analytically, the simple theory indicates that $(T_{RI} - T_n)/\Delta T$ is independent of θ but however the more complicated theory qualitatively shows a decrease in $(T_{RI} - T_n)/\Delta T$ as θ increases. The numerical results confirm this behavior. They also show that the decrease in $(T_{RI} - T_n)/\Delta T$ is greater at 15% moisture content than at 30% moisture content. This could be attributed to the fact that the effective heat capacity (c_{eff}) is greater in the 30% case.

Figure (22) shows the relationship between $(T_{RI} - T_n)/\Delta T$ and the insulating layer phase constant ϕ_{II} for the various moisture content at Inuvik and Edmonton. When phase constant $\phi_{II} = 0.0$, the value of $(T_{RI} - T_n)/\Delta T$ is positive and it increases with moisture content. For phase constant $\phi_{II} = 0.5$, the value of $(T_{RI} - T_n)/\Delta T$ is negative and it decreases with moisture content. However, when the phase constant $\phi_{II} = 0.25$, the simple theory indicated a zero value for $(T_{RI} - T_n)/\Delta T$, but the numerical results show that the value of $(T_{RI} - T_n)/\Delta T$ is positive and it decreases as the moisture content increases. On the other hand the more complicated theoretical solution predicts a positive value of $(T_{RI} - T_n)/\Delta T$ when $\phi_{II} = 0.25$. The above observations are true for both Inuvik and Edmonton cases. Generally the values of $(T_{RI} - T_n)/\Delta T$ for Inuvik cases are greater than those for Edmonton at all phase constants.

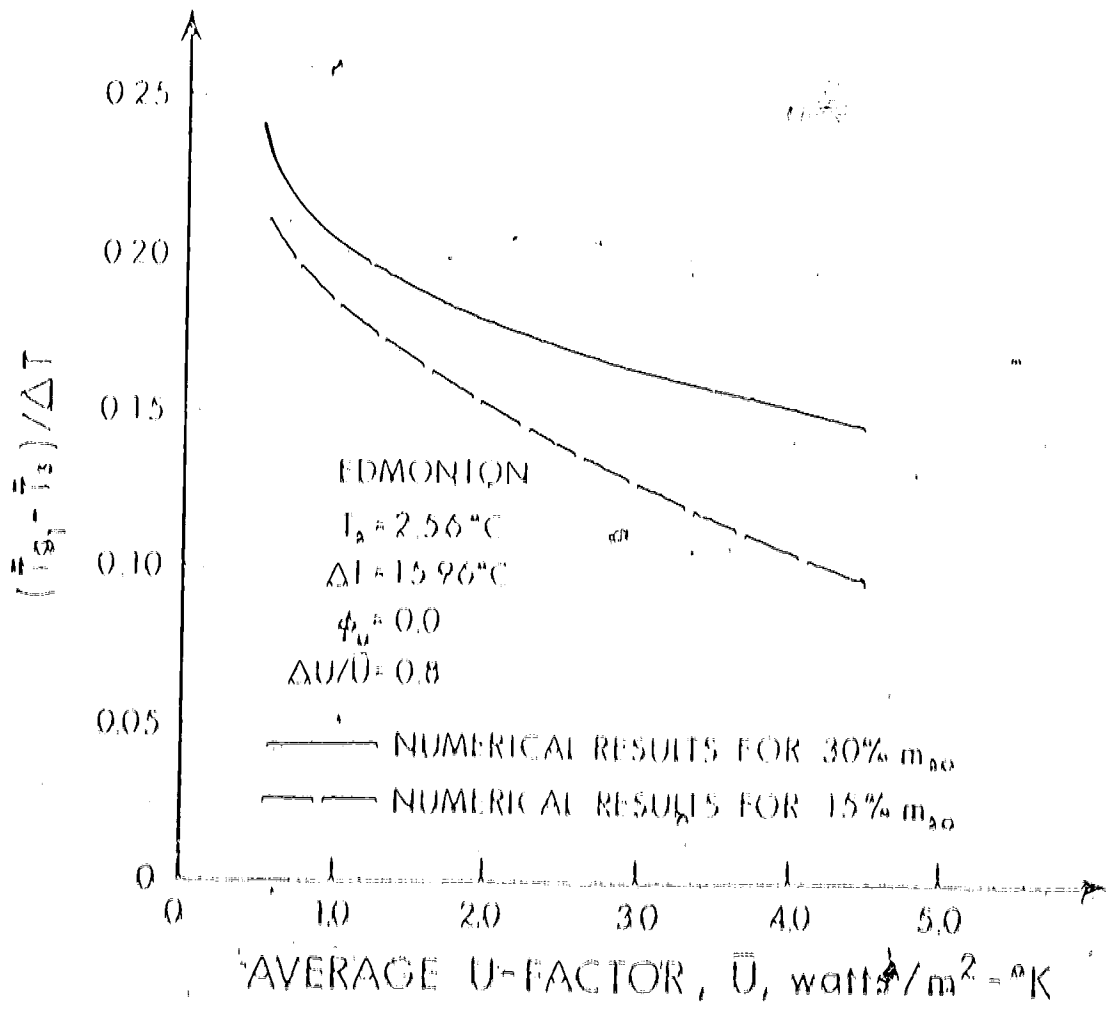
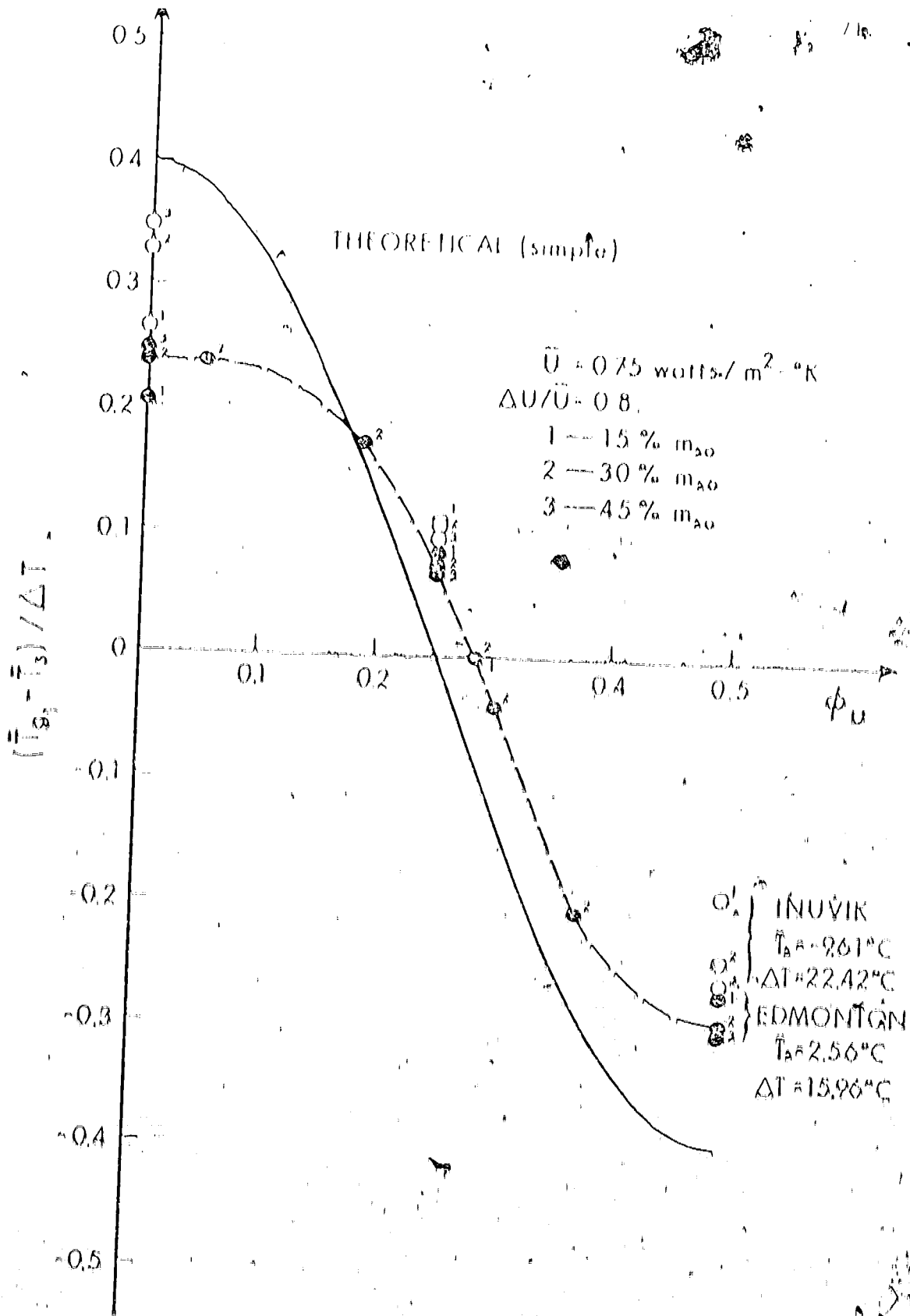


FIGURE 21 - Variation of \bar{U} with $(\bar{T}_{g1} - \bar{T}_{g2}) / \Delta T$



and that the moisture content affects the values of $(C_{eff} - 1) / C_{eff}$ at Inuvik significantly more than those at Edmonton.

3.5 Depth of the active layer

3.5.1 Depth of Frost or Thaw penetration

Shown in Figure (23) is the typical thaw penetration for Norman Wells. Depth of maximum thaw or frost penetration for the three locations under consideration are listed in Table 2. Listed also are the soil and insulating properties. The method of obtaining the depth of frost or thaw penetration is similar to the one discussed earlier in Section (2.4.1), for the imposed temperature case.

TABLE 2. Depth of frost or thaw penetration with imposed surface temperature

Location, soil and insulating properties	Depth of thaw or frost penetration (meters)
Edmonton: $T_B = 2.56^\circ\text{C}$, $\Delta T = 15.96^\circ\text{C}$ $U = 2.0$, $\Delta U/U = 0.0$ $\phi_M = 0.0$, $M_{HO} = 15\%$	0.925 m. (3.05 feet)
Norman Wells: $T_B = -6.22^\circ\text{C}$, $\Delta T = 22.06^\circ\text{C}$ $U = 0.75$, $\Delta U/U = 0.8$ $\phi_M = 0.0$, $M_{HO} = 30\%$	0.825 m. (2.71 feet)
Inuvik: $T_B = -9.61^\circ\text{C}$, $\Delta T = 22.62^\circ\text{C}$ $U = 0.75$, $\Delta U/U = 0.8$ $\phi_M = 0.0$, $M_{HO} = 30\%$	0.625 m. (2.05 feet)

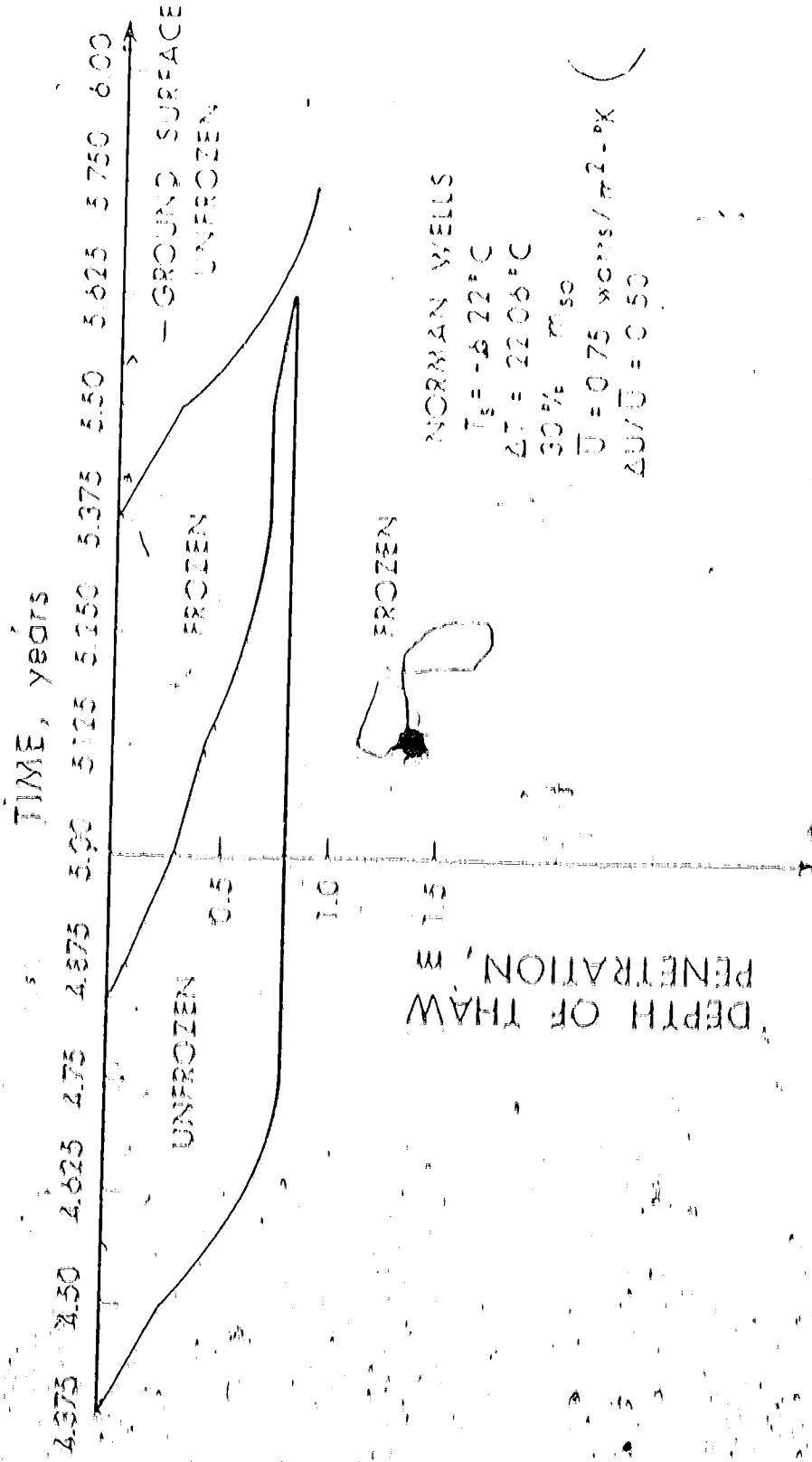


FIGURE 23 — Depth of thaw penetration in Norman Wells with the insulating layer in phase ($\rho_u = 0.0$) with the air temperature

3.5.7 Effects of the Insulating Layer on the Active Layer

Figures (24) and (25) show the relationship between the average Insulating Layer (U) and the depth of active layer for Edmonton case, with various phase constant (ϕ_u) and moisture content (M_{so}). Generally, if all the parameters remain unchanged, an increase in U will cause an increase in the depth of the active layer for the Edmonton case of the non-naturally permafrost location as indicated by both the figures. Specifically for a given U and moisture content (M_{so}), the depth of the active layer increases as the phase constant (ϕ_u) increases and for a given U and phase constant (ϕ_u), the depth increases as the moisture content decreases.

For Edmonton with 30% moisture content, $U \sim 0.75$ and phase constant $\phi_u \sim 0.0$, the depth of the active layer decreases as U/D increases as indicated by Figure (26). However, if the phase constant $\phi_u \sim 0.5$, the depth of the active layer increases at first gradually and then rapidly as U/D increases.

Figure (27) shows the relationship between the moisture content (M_{so}) and the depth of the active layer for both Inuvik and Edmonton at the various phase constant (ϕ_u). Figure (27) indicates that for the Edmonton case (non-naturally permafrost location), at a given U -factor, the depth of active layer increases as ϕ_u increases and decreases as the moisture content (M_{so}) increases, the result obtained in Figure (24) and (25). For Inuvik, a naturally permafrost location, for a given U -factor and phase constant (ϕ_u), the depth of the active layer decreases as the moisture content increases, similar to that for Edmonton but the depth decreases as the phase constant (ϕ_u) increases, opposite to

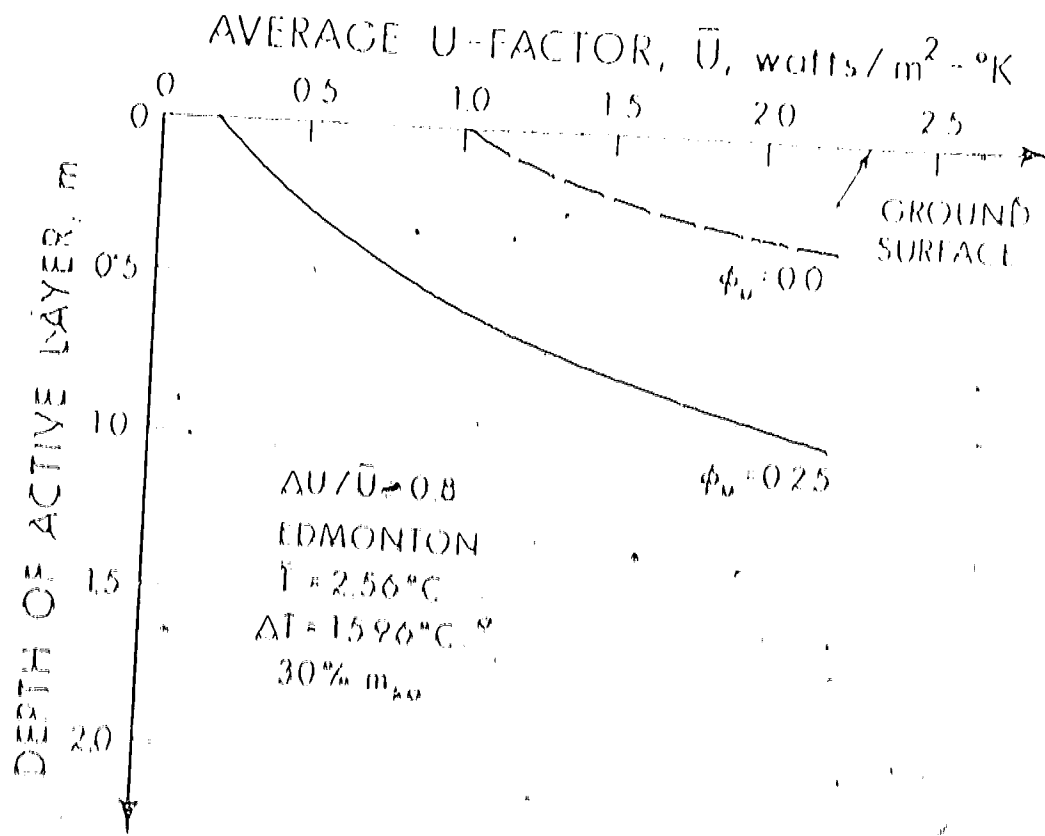


FIGURE 24 - Variation of \bar{U} and ϕ_u with depth of active layer

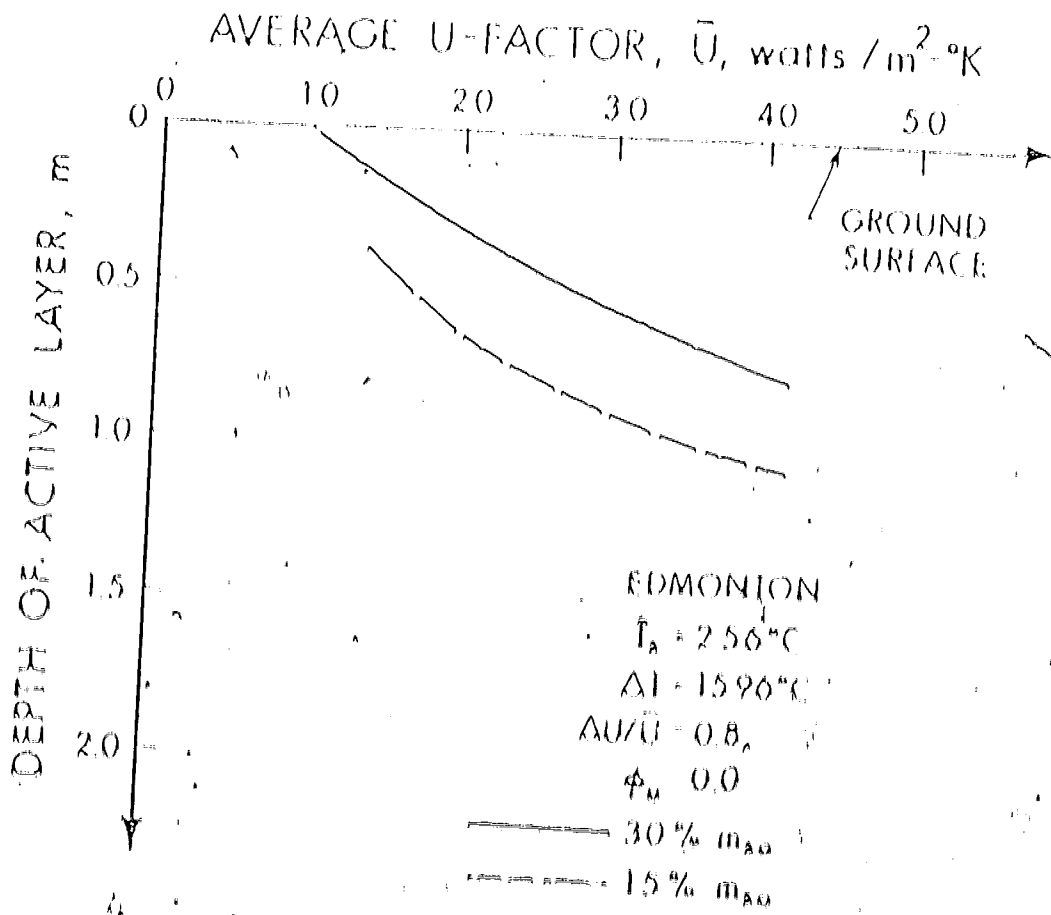


FIGURE 25 - Variation of \bar{U} and m_{AO} with depth of active layer

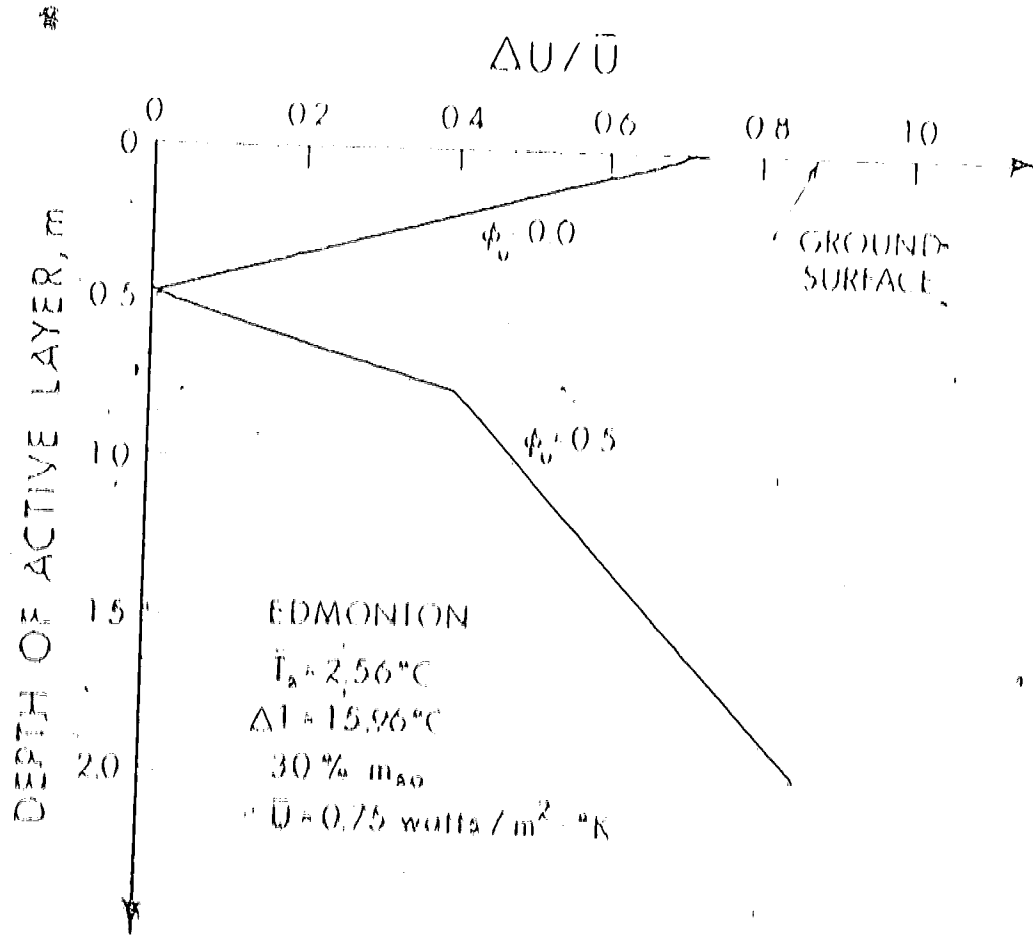


FIGURE 26 Variation of $\Delta U / \bar{U}$ with depth of active layer

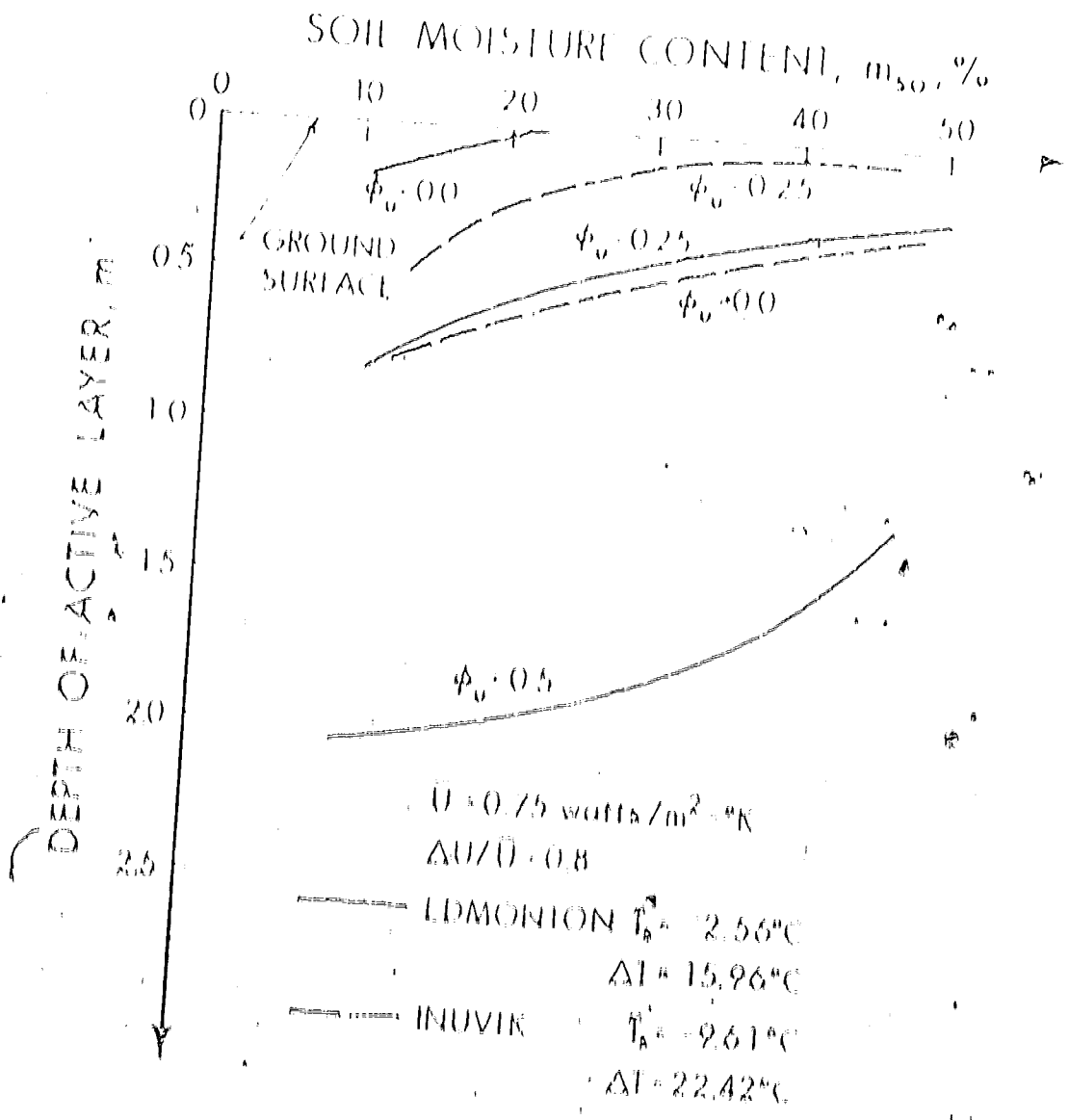


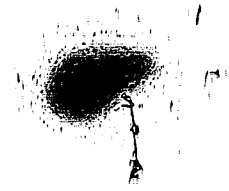
FIGURE 27 - Variation of M_{so} and ϕ_u with depth of active layer

that for Edmonton

with a mean humidity

With the addition of a ground layer, the amplitude of temperature variations at the ground surface is smaller. The greatest change occurs when the average ground temperature is close to 0°C. Except for the small amplitude of temperature variations, the profile of the ground temperature in the case of the exposed ground surface temperature curve is in phase with the air temperature. The average ground temperature is a fraction of the average air temperature. As the increase in humidity, that is, approaching this exposed ground surface temperature condition, the effect of the insulating layer on the ground temperature decreases. The effect of the insulating layer is largest for high moisture content soils.

For both Edmonton (non-permafrost location) and Inuvik (naturally permafrost location) cases, an increase in moisture content H_{so} would cause the active layer depth to decrease. However, for a given H_{so} , an increase in ϕ_u would cause the active layer depth in Edmonton to increase whereas that in Inuvik to decrease.



PART IV

DRIFTED HEAT CORRECTION WITH EFFECT
 OF FROST AND VARIOUS HEAT FLUX AT GROUND
 BOUNDARY

6.1 INTRODUCTION

In this section, instead of assuming T_a equals the air temperature T_a , T_b here will be calculated from a balance of heat fluxes at the surface. It is reasonable to assume that the calculated T_b will be different from the air temperature T_a because of the various complex heat fluxes.

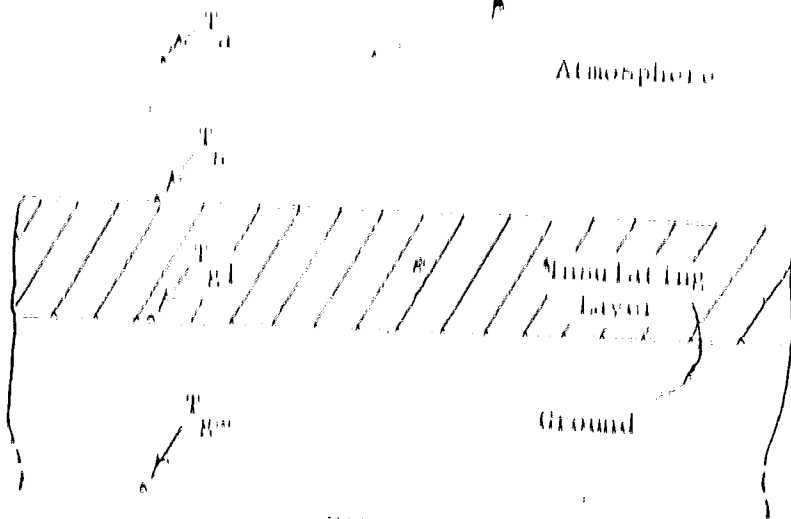


FIGURE 2B

Figure (2B) shows the various major average temperatures in the ground-atmosphere system,

- Definitely, $T_{B0} \neq T_{B1}$ because of the freezing and thawing
- $T_{B1} \neq T_B$ because of the simulating layer
- $T_B \neq T_a$ because of the heat fluxes

4.2 Disposition of Solar Radiation

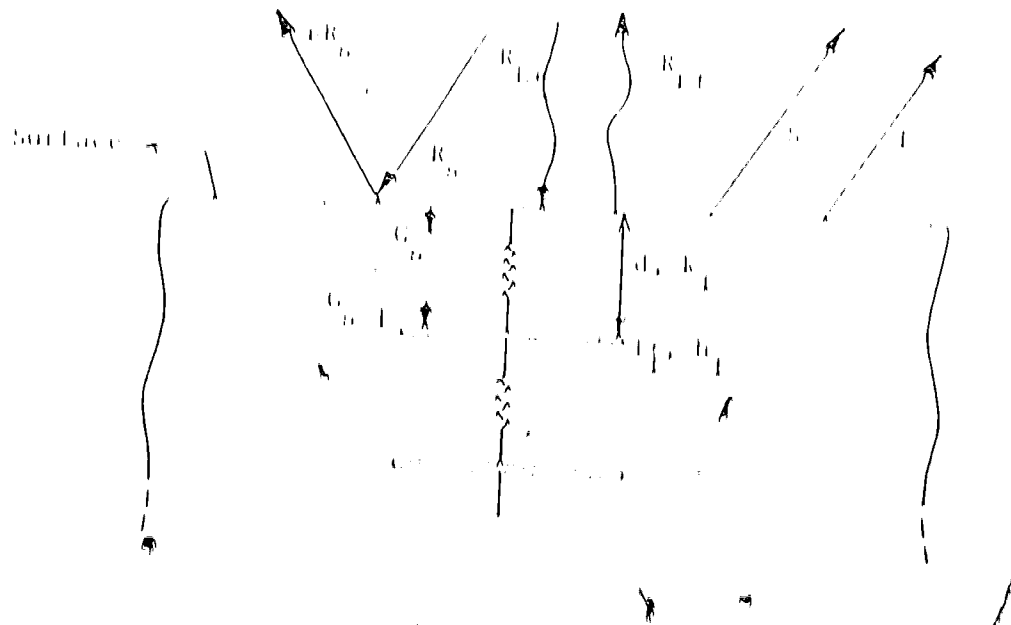


FIGURE 29

Figure (29) shows the major components of the disposition of solar radiation in the earth-atmosphere system. The components are:

- i) R_n , incoming solar radiation
- ii) rR_n , amount of R_n reflected
- iii) R_{at} , long-wave radiation down from the sky
- iv) R_{gf} , long-wave radiation (infrared radiation) from the ground
- v) S_s , sensible heat flux
- vi) L_s , latent heat flux
- vii) G_n , heat flux conducted through the tabulating surface into the ground.

All of these components will be individually discussed later in this section.

The air temperature T_a will be assumed to vary as a pure harmonic function of time around an average value. Consider only the case for Berlin W.H. The average air temperature $T_a = T_{a0} = 6.2^\circ\text{C}$ and the amplitude of oscillation $T_a = T_{a0} \pm 2.06^\circ\text{C}$.

$$T_a = T_{a0} + 2.06 \sin(\omega t + \phi)$$

The differentiation between summer and winter conditions will be determined by whether the air temperature T_a is above or below the freezing temperature 0°C . With reference to Appendix I, for Berlin W.H.

when $0.42 - \sin \alpha = 0.83$ (summer conditions)

$0.42 - \sin \alpha = 0.83$ (winter conditions)

Heat Fluxes

4.2.1 Incoming Solar Radiation (R_H)

This value of R_H is usually the measured value and is independent of the nature of the ground surface. The incoming solar radiation is made up of diffuse and direct radiation and is the total solar radiation that reaches the earth surface. On the average about half of the solar radiation intercepted by the earth eventually reaches the surface. The rest is either reflected and scattered back to space by clouds and atmospheric constituents or is absorbed by clouds and atmospheric constituents. Appendix IV shows the range of short wave radiation (R_H) for a latitude of 65°N . This data was recorded from 1943-1957.

These values are assumed to be those of Norman Wells (66° 17' N) and that the terms are assumed to vary sinusoidally about an average value.

$$R_D = R_{D0} + ER_{D0} \sin(\omega t + \phi) \quad (6.1)$$

4.2.2. Reflected Short Wave Radiation (R_D)

All solar radiation incident on the earth's surface is not absorbed there, a certain portion of that is reflected and is lost to space. The amount reflected depends primarily on the color, moisture content, texture and the composition of the surface. In general, a wet or dark surface has a lower reflectivity than the dry or light surface.

TABLE 3. Albedo of Natural Surfaces (1)

Type of Surface	Reflectivity, ρ
Fresh snow cover	0.75 - 0.95
Dense cloud cover	0.60 - 0.90
Old snow cover	0.40 - 0.70
Clean thin snow	0.50 - 0.65
High sand dunes, soil	0.30 - 0.60
Clean glacier ice	0.30 - 0.46
Dirty thin snow	0.20 - 0.50
Dirty glacier ice	0.20 - 0.30
Sandy soils	0.15 - 0.40
Meadow and fields	0.12 - 0.30
Densely built-up areas	0.15 - 0.25
Woods	0.05 - 0.20
Dark cultivated soil	0.07 - 0.10
Water surfaces, sea	0.03 - 0.10

The values given in Table 3 are obtained from Golubev (45). Observing the values given in the table, shown that the albedo of different natural surfaces vary from about 3 to 30%. This is because the albedo varies with the moisture content of the soil and also varies both with the wavelength and the angle of incidence of the solar rays. The values of the last two cases are given in Section (43). In the case of cultivated soil, albedo depends primarily on the moisture content of the soil, an increase in soil moisture decreases the albedo. For

the standard data set used in the calculation, the albedo reflectivity was assumed to have an average winter and summer values, α_w and α_s . These values were chosen to be representative of vegetation in summer and snow cover in winter, $\alpha_s = 0.16$ and $\alpha_w = 0.60$.

4.2.3 Long wave emission from the earth's surface (R_{L1})

The surface of the earth, as any other body, radiates energy. Because of the earth's temperature most of the emission lies in the invisible infrared region of the spectrum. The earth is commonly assumed to emit and absorb energy as a gray body in the infrared region. The formula for the earth's surface emission may thus be written as

$$R_{L1} = \epsilon_B \sigma (T_B + 273)^4 \quad (4.2)$$

according to Stefan-Boltzmann law

where σ is the Stefan-Boltzmann constant

$$\sigma = 5.67 \times 10^{-8} \text{ Watts/m}^2 \text{ } ^\circ\text{K}^4$$

T_B is the temperature of the surface in $^\circ\text{C}$ and ϵ_B is the infrared emissivity of the surface. The emissivity of a black body is unity. Typical infrared emissivities, expressed in percent for the various surfaces are given in Table 4. The data are from Sellers (43).

TABLE 4. Infrared Emissivities (ϵ_{IR})
(Per cent)

Surface	
Water	92 - 96
Snow, fresh	82 - 99.5
Snow, ice granules	89
Ice	96
Soil, frozen	93 - 94
Sand, dry	84 - 90
Sand, wet	95
Gravel, coarse	91 - 92
Ground, moist, bare	95 - 98
Ground, dry, plowed	90
Grass, high dry	90
Field and shrubs	90
Pine forest	90

As indicated in Table 4 the values of ϵ_{IR} for the various surfaces vary from 82 to 99.5%.

4.2.4 Long-wave counter-radiation from the atmosphere (R_{LW})

Although the atmosphere is nearly transparent to short-wave radiation, it readily absorbs the terrestrial radiation. This terrestrial absorption is due to water vapor, atmospheric gases and clouds.

As a result only about 9 per cent of the terrestrial radiation escapes directly to space. The atmosphere in turn reradiates the absorbed terrestrial radiation, partly to space and partly back to the surface (counter radiation). The counter radiation which is absorbed by the surface R_{L1} for clear sky is

$$R_{L1} = \epsilon_a \epsilon_a \sigma (T_a + 273)^4 \quad (4.3)$$

where ϵ_a is the effective emissivity of the air and has been approximated by

$$\epsilon_a = a + b e^{-c p_a}$$

a , b and c are empirical constants and p_a is the vapor pressure in millibars.

Ofier (45) uses

$$a = 0.82$$

$$b = 0.25$$

$$c = 0.094$$

The vapor pressure p_a has further been approximated by (Appendix IV)

$$p_a = 8.5 \times 10^{-3} \frac{23.4 - 5900/(T_a + 273)}{(T_a + 273)} \quad (4.4)$$

where w is the relative humidity (Appendix V).

The counter-radiation R_{L1} from a cloudy sky is expressed in terms of the value for a clear sky (45) by the equation

$$R_{L1}(\text{cloudy}) = (1 + M_w) R_{L1} \quad (4.5)$$

where H is a constant and w_c is the extent of cloud cover (Appendix VI), which for overcast skies is 1.0 and for clear skies is 0.0.

Radiation from the clouds, as indicated in equation (4.5), does not increase linearly with w_c , but almost as a quadratic function. Other than depending on the extent of the cloud cover, counter radiation also depends on the height of the cloud, as indicated by the values of M in Table 5. In general, the higher and thinner the clouds the less they will contribute to the counter radiation reaching the ground. Typical values for M are listed in Table 5(45).

TABLE 5. Values of M

Cloud Type	Height (m)	M
Cirrus	12,200	0.04
Cirrocumulus	8,390	0.08
Alto cumulus	3,660	0.17
Altostratus	2,140	0.20
Stratocumulus	1,220	0.20
Stratus	460	0.24

4.2.5 Shortwave Heat Flux (S) and Longwave Heat Flux (L)

The composition of the earth's atmosphere is usually different from the composition of the lower layer of the atmosphere. Consequently, a vertical heat flux exists between the atmosphere and the atmosphere at the surface.

by the turbulent heat exchange in the air layer near the ground. The determination of these vertical turbulent heat fluxes usually presents the greatest difficulties in any heat balance problem. Because of the complexities involved in developing suitable equations for these two fluxes, detailed descriptions will not be presented here, however they are available in Sellers (63), Budyko (64), Gelfand (65) and Peteev (72). The most common method for obtaining the formula for the vertical turbulent heat flux in the air layer near the ground is to assume that the process of turbulent diffusion is similar to that of molecular diffusion.

Hence for sensible heat flux Q_s , the equation is

$$Q_s = - \rho c_p K_H (\Delta T / \Delta z) \quad (4.6)$$

and for latent heat flux Q_L , the equation is

$$Q_L = - \rho L_e K_W (\Delta q / \Delta z) \quad (4.7)$$

K_H and K_W are the corresponding eddy diffusion and q , T and z are the specific humidity, temperature and height. Assuming Q_s and Q_L do not vary within the air layer thickness of one or two meters above the underlying surface, equations (4.6) and (4.7) can then be integrated from the surface to a height z_s , one or two meters above the ground to give

$$Q_s = \rho c_p K_H (T_s - T_A) \quad (4.8)$$

and

$$Q_L = \rho L_e K_W (q_A - q_s) \quad (4.9)$$

or

$$L = (0.622 \rho_s D_s) (e_s - e_a) / P \quad (4.10)$$

Further, if it is assumed that the diffusion of heat takes place by the same mechanism as the diffusion of water vapor, $D_H = D_w$, hence equations (4.8) and (4.10) can be written as

$$S = A(T_b - T_a) \quad (4.11)$$

and

$$L = B(e_s - e_a) \quad (4.12)$$

where

$$A = \rho_s c_p D$$

$$B = (0.622 \rho_s D_s) / P$$

$D_s = e_{s,s} - e_{s,a}$ and P are the diffusivity coefficient, surface vapor pressure, air vapor pressure and the total pressure,

From Appendix V, the saturated surface vapor pressure $e_{s,s}$ is given by

$$e_{s,s} = e_s^{*} = 23.4 \exp(-5900 / (T_s + 273)) \quad (4.13)$$

and that for the air vapor pressure in

$$e_{s,a} = w_s e_a^{*} = 23.4 \exp(-5900 / (T_a + 273)) \quad (4.14)$$

Let the top surface with a certain moisture content $M_{H,s}$ the surface vapor pressure $e_{s,s}$ is

$$c_p = M_a c_{p,a}^{3.4} \left(\frac{5900}{(1 + 0.24)} \right) \left(\frac{1}{(1 + 0.24)} \right) c_{p,a}$$

$$= M_a c_{p,a} \left(\frac{1 + 0.24}{1 + 0.24} \right) c_{p,a} \tag{6.13}$$

D , the diffusivity coefficient for heat and moisture in the air layer near the ground are presented in various forms by different authors (43, 44). The principal properties of the diffusivity coefficient are that it changes considerably with the wind velocity, increases as the surface roughness increases and that the coefficient is higher in dry regions. It also will depend on the stability of the air layer. From Sellers (43), the diffusivity coefficient is given by

$$D = 1.94 R u^2 \tag{6.14}$$

where D is m^2/sec , R is a constant with values around 0.002 , u is the wind velocity in m/sec and the coefficient 1.94 is extremely variable with values ranging from $0.0045 \text{ m}/\text{sec}$ to $0.0007 \text{ m}/\text{sec}$, depending on the type of surface. The coefficient 1.94 increases as the surface roughness increases.

Values used in the calculations are

- $1 = 0.003 \text{ m}/\text{sec}$
- $R = 0.002$
- $u = 4.5 \text{ m}/\text{sec}$

4.2.6 Heat Flux to the Ground (G_g)

The insulating layer lying over the ground was assumed to have no heat capacity, therefore, with reference to Equation (29),

$$G_{in} = G_{in,1}$$

$$G_{in} = G_{in,1} + (d/k_1) G_{in} \quad (4.15)$$

or

$$G_{in} = (k_1/d)(G_{in,1})$$

4.2.7 The Insulating Layer (U Factor)

The layer was assigned a U factor, k_1/d , having an average winter and summer values, U_w and U_s .

$$d/k_1 = d_w/k_w = 0.42 \quad (U = 0.8)$$

$$= d_s/k_s = 0.42 \quad (U = 0.8)$$

4.2.8 Energy Balance

Net heat flux to the ground surface is

$$F = (1-\epsilon) R_{in} + R_{ext} - R_{out} = \epsilon R_{in} - R_{out} \quad (4.16)$$

For energy balance at the surface, F is also equal to G_{in} , the heat flux to the ground.

4.3 The Input Variables Required for the Solution to Equation (4.2)

(a) Radiation Parameters

$$R_{in} = \Delta R_s + \phi R_s + \epsilon H_s + a_s + b_s + \dots$$

(b) Relative Humidity:

w_a

(C) Cloud Cover:

$$w_c, N$$

(D) Sensible and Latent Heat Parameters:

$$c_p, T_{c0}, P, T_s, P_s, u_s, v_p$$

(E) Air Temperature Parameters:

$$T_a, \Delta T_d, \phi_1$$

(F) Surface Conditions:

$$T_b, T_w, d_b, d_w, T_1, T_2, k_b, k_w, M_b$$

(G) Ground Parameters:

$$k_g, k_{gl}, \rho_{gl}, \rho_{gl}, c_{gl}, M_{gl}$$

4.4 Standard Input Data Set (Normal Month)

(1) Radiation Parameters:

$$R_b = 100 \text{ watt/m}^2$$

$$\Delta R_b = 100 \text{ watt/m}^2$$

$$\phi_R = 0.873$$

$$c_b = 0.90$$

$$a = 0.82$$

$$b = 0.25$$

$$c = 0.094 \text{ 1/mb}$$

$$d = 5.67 \times 10^{-8} \text{ watt/m}^2 \text{ } ^\circ\text{K}^4$$

(2) Relative Humidity:

$$w = 0.65$$

iii) Cloud Cover:

$w_c = 0.64$

$M = 0.2$

iv) Sensible and Latent Heat Parameters:

$\rho = 1.204 \text{ kg/m}^3$

$L_v = 2.31 \times 10^6 \text{ Joules/kg}$

$P_{sat} = 1018 \text{ mb}$

$U = 0.001 \text{ m/sec}$

$R = 0.002$

$u = 3.50 \text{ m/sec}$

$c_p = 1.0 \times 10^3 \text{ Joules/kg } ^\circ\text{C}$

v) ΔT Temperature Parameters:

$T_a = 0.22 \text{ } ^\circ\text{C}$

$\Delta T_a = 22.06 \text{ } ^\circ\text{C}$

$\phi_T = 0.29$

vi) Surface Conditions:

$N_{H_2O} = 0.10^3 \text{ (surface moisture content)}$

$k_H = 0.16$

$k_W = 0.60$

$d_H = 0.25 \text{ m}$

$d_W = 0.38 \text{ m}$

$k_H = 0.30 \text{ watts/m}^2\text{ } ^\circ\text{C}$

$k_W = 0.17 \text{ watts/m}^2\text{ } ^\circ\text{C}$

$k_1 = 0.42$

$k_2 = 0.83$

IV) Ground Parameters

- M_{so} 0.30 (soil moisture content)
- k_f 1.70 watts/m²°C
- k_{af} 1.10 watts/m²°C
- ρ_f 1.60×10^6 joules/m³°C
- ρ_{af} 2.50×10^6 joules/m³°C
- L_f 1.20×10^8 joules/m²°C

4.5 Introduction of the Analytical Section

For a given set of input variables, equation (4.16) along with the equations of heat transfer in the ground (Part III) can be solved for the ground temperature variation throughout the year. This will be done numerically in the following section. However, because of the large number of input variables involved and the uncertainty in some of the empirical expressions for the heat flux, the absolute determination of the ground or surface temperatures may be questionable. One can however predict the changes in the ground and the surface temperatures that would result from changes in any one of the controlling parameters. This approach which is essentially a 'sensitivity analysis' will be taken with the model presented. First this will be done analytically. The analytic calculation will require the neglect of the interaction of the ground and the surface heat fluxes.

4.5.1 Analytical Section

The main objective of this section are to obtain analytic estimation of the effects of the various heat fluxes on the

surface of the ground cover. These effects will be expressed in terms of an effective surface temperature, an effective "heat transfer coefficient", and an average influence coefficient. The values of these parameters will be compared to the values obtained numerically.

4.5.2 Effectively surface temperature and effective "heat transfer coefficient"

A linear approximation to equation (4.16) can be obtained by expanding it in a Taylor series about some temperature T_B^Δ . Thus

$$F(T_B) \approx F(T_B^\Delta) + \left. \frac{\partial F}{\partial T_B} \right|_{T_B^\Delta} (T_B - T_B^\Delta) + \dots \quad (4.17)$$

where terms in a higher order in the temperature difference $(T_B - T_B^\Delta)$ have been neglected.

If T_B^Δ is chosen such that $F(T_B^\Delta) = 0$, then equation (4.17) becomes

$$F(T_B) \approx \left. \frac{\partial F}{\partial T_B} \right|_{T_B^\Delta} (T_B - T_B^\Delta) \quad (4.18)$$

The term $\left. \frac{\partial F}{\partial T_B} \right|_{T_B^\Delta}$ in equation (4.18) is essentially an effective heat transfer coefficient which could be called h_{eff} , thus giving

$$F(T_B) \approx h_{eff} (T_B - T_B^\Delta) \quad (4.19)$$

This temperature T_B^Δ is the temperature the surface would have if there was no heat flux to the ground. It will therefore be close to the true surface temperature when there is a thick insulating layer of snow or vegetation on the ground. More specifically one would

expect equation (4.19) to be a good approximation when $h_{eff} \approx U$, where U is the U-factor or conductance of the surface insulating layer. This approximation will be checked in a latter section. A temperature similar to T_B^A when used in heat transfer literature on human comfort or other similar topics is often called an effective or radiation temperature. The former term will be used in this thesis.

To find T_B^A , equation (4.16) is set equal to zero and may be rewritten in the form

$$Y - \Lambda_0 - \Lambda_1 Y^4 - \Lambda_2 Y^{-5900/Y} \quad (4.20)$$

where $Y = T_B^A + 273$ and Λ_0 , Λ_1 and Λ_2 are known constants.

$$\Lambda_0 = \frac{1/\rho c_p D \kappa (R_B (1-\epsilon) + \epsilon h_a d_{eff})}{\epsilon h_a d_{eff}} (T_a + 273)^4$$

$$+ (0.622 \rho L_a D/P) M_B \epsilon_a + \rho c_p D (T_a + 273)$$

$$\Lambda_1 = 1/\rho c_p D \kappa (1/\epsilon)$$

$$\Lambda_2 = 1/\rho c_p D \kappa (0.622 \rho L_a D/P) M_B \epsilon_a^{23.4}$$

Defining the function

$$f(Y) = Y - \Lambda_0 - \Lambda_1 Y^4 - \Lambda_2 Y^{-5900/Y} \quad (4.21)$$

$$f'(Y) = 1.0 - 4\Lambda_1 Y^3 + (5900/\Lambda_2 Y^2) \epsilon^{-5900/Y} \quad (4.22)$$

Equation (4.20) can be solved by first assuming an initial approximation to a root, then generating successive approximations. From the iteration

$$Y_{i+1} = Y_i - \frac{f(Y_i)}{f'(Y_i)} \quad (\text{Newton's Method})$$

T_B^{\wedge} for the entire year for the standard data, given in section (4.4) was then obtained as tabulated in Table 6 along with the values of air temperature, T_a .

TABLE 6. Values of T_B^{\wedge} for the entire Year

Time, t (year)	T_B^{\wedge} ($^{\circ}\text{C}$)	T_a ($^{\circ}\text{C}$)
0.000	-31.37	-27.41
0.125	-28.22	-25.21
0.250	-13.77	-11.71
0.375	3.76	5.01
0.500	16.97	15.15
0.625	13.58	12.77
0.750	-2.03	-0.73
0.875	-21.31	-17.45
1.000	-31.37	-27.41

$$T_B^{\wedge} \approx -7.73^{\circ}\text{C}$$

$$T_B^{\wedge} \approx -0.22^{\circ}\text{C}$$

It will be noted that T_b^A is lower than T_b^S in winter and higher than T_b^S in summer. This difference may be attributed to the fact that the value of the reflectivity of summer vegetation is very much lower than the reflectivity of winter snow cover, the respective values being 0.16 and 0.6. The greatest difference between T_b^A and T_b^S occurs in the winter when the additional heat flux due to solar radiation is minimum.

To find the effective "heat transfer coefficient", equation (4.16) must be differentiated with respect to T_b^A . This gives

$$\begin{aligned} \partial F / \partial T_b^A &= 4 \epsilon_b \sigma (T_b^A + 273)^3 + \rho c_p D \\ &= (0.622 + \rho L_v D / P) H_b \sigma^{23.4} (5900 / (T_b^A + 273))^2 + 5900 / (T_b^A + 273). \end{aligned}$$

$$h_{eff} = \left. \partial F / \partial T_b^A \right|_{T_b^A} \quad (4.23)$$

The effective "heat transfer coefficient" h_{eff} is thus the sum of the terms due to radiation, sensible heat and latent heat. These terms are labelled h_k , h_H and h_L respectively and values are given for various times during the year in Table 7. It will be noted that h_k and h_L are time dependent while h_H is not. The effective "heat transfer coefficient" h_{eff} that results ranges from a low of 18.48 $\text{watts/m}^2 \cdot ^\circ\text{K}$ in winter to a high of 30.0 $\text{watts/m}^2 \cdot ^\circ\text{K}$ in summer. The average value for the entire year is 22.51 $\text{watts/m}^2 \cdot ^\circ\text{K}$. Values of h_{eff} obtained from Williams (51) for latitude 60°N to 60°N indicate that

For average conditions of exposure, the coefficient varies between 19.39 to 29.09 $\text{watts/m}^2\text{ }^\circ\text{K}$ (16.68 to 25.02 $\text{kcal/m}^2\text{h } ^\circ\text{K}$). This analytical value of effective "heat transfer coefficient" h_{eff} , will be later compared to the value of h_{eff} obtained numerically.

TABLE 7. Values of h 's for the Entire Year

Time, t (year)	h_r (Low radiation)	h_n (Sensible)	h_L (Latent)	h_{eff} $\text{watts/m}^2\text{ } ^\circ\text{K}$
0.000	2.88	15.36	0.24	18.48
0.125	2.99	15.36	0.32	18.67
0.250	3.56	15.36	1.08	20.00
0.375	4.33	15.36	4.01	23.70
0.500	4.98	15.36	9.66	30.00
0.625	4.80	15.36	7.78	27.94
0.750	4.06	15.36	2.66	22.08
0.875	3.25	15.36	0.58	19.19
1.00	2.88	15.36	0.24	18.48

Fig. 7. Analytical average influences coefficients for the standard data set.

The main objective here is to find the changes in T_m caused by a change in x_1 where x_1 is one of the control parameters. If the heat loss to the ground is zero, then equation (4.10)

becomes

$$0 = R_B (1 - \epsilon) - \epsilon \frac{d}{dt} (T_B + T_d)^4 + \epsilon \frac{d}{dt} (T_d) (T_B + T_d)^4$$

$$\rho c_p D (T_B - T_d) = (0.622 \rho L_e D/P) (c_B - c_d)$$

which implies that

$$f(T_B, \kappa_1) = 0 \tag{4.24}$$

Taking the total derivative of equation (4.24) with respect to the independent parameter κ_1 , gives

$$df/d\kappa_1 = 0 = \partial f/\partial T_B \cdot dT_B/d\kappa_1 + \partial f/\partial \kappa_1$$

implying

$$dT_B/d\kappa_1 = - (\partial f/\partial \kappa_1 / \partial f/\partial T_B) \tag{4.25}$$

$\partial f/\partial T_B$ was obtained previously in finding the effective heat transfer coefficient. The average influence coefficient is

$$dT_B/d\kappa_1$$

which is equalled to

$$\int_0^1 dT_B/d\kappa_1 dt \tag{4.26}$$

In integrating equation (4.26), the nominal values of T_B are assumed to be T_B^0 , the values which obtain $F = 0$. Taking the partial derivatives of F with respect to the independent variables κ_1 , $\partial F/\partial \kappa_1$

$$\lambda) \quad \partial F/\partial R_B = (\lambda - F)$$

and

$$R_B = R_B (\lambda + \Delta R/R_B + \epsilon (\omega + \phi_R))$$

$$\partial R / \partial R_n = 1$$

$$\partial R / \partial (\Delta R / R_n) = R_n \ln(\omega(1+\phi_R))$$

$$\therefore \partial I / \partial R_n = (1-f)$$

and

$$\partial I / \partial (\Delta R / R_n) = (1-f) R_n \ln(\omega(1+\phi_R))$$

$$ii) \partial I / \partial I = -R_n$$

$$iii) \partial I / \partial u_n = -\alpha(T_n + 273)^4 + c_{a, \text{COFF}} \alpha(T_n + 273)^4$$

$$iv) \partial I / \partial u = \rho V_p R (T_n - T_a) = (0.622 \rho l_{v, R} / P) (\alpha_n - \alpha_a)$$

$$v) \partial I / \partial W_c = 2c_{n, D} M W_c + \alpha (T_n + 273)^4$$

$$vi) \partial I / \partial M_n = (0.622 \rho l_{v, D} / P) (\alpha_{nH} - \alpha_n)$$

$$vii) \partial I / \partial W = c_{n, D} \alpha (T_n + 273)^4 (1 + 11 W_c^2) + (\text{bc } \alpha_{nH} - \alpha_n) \alpha_{nH}$$

$$+ (0.622 \rho l_{v, D} / P) M_n \alpha_{nH}$$

$$viii) \partial I / \partial T_n = 4c_{n, D} c_{a, \text{COFF}} (T_n + 273)^3 + c_{n, D} \alpha (1 + 11 W_c^2) (T_n + 273)^3 \times$$

$$(\text{bc } \alpha_{nH} - \alpha_n) \alpha_{nH} / (T_n + 273)^2 + \rho c_p D + (0.622 \rho l_{v, D} / P) M_n \times$$

$$(\text{bc } \alpha_{nH} - \alpha_n) \alpha_{nH} / (T_n + 273)^2$$

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$$T_n = T_a (1 + \Delta T / T_a) \ln(1 + \phi_R)$$

$$\partial T_a / \partial T_a = 1$$

$$\partial T_a / \partial (\Delta T / T_a) = T_a \ln(1 + \beta T_a)$$

$$\therefore \partial T_a / \partial T_a = \partial T_a / \partial T_a \cdot \kappa \cdot \partial T_a / T_a$$

and

$$\partial T_a / \partial (\Delta T / T_a) = \partial T_a / \partial T_a \cdot \kappa \cdot \partial T_a / \partial (\Delta T / T_a)$$

$$\begin{aligned} (\kappa) \quad \partial T_a / \partial T_a &= -4 \mu_{11} \mu_{12} (T_{11} + 2/3)^3 + \mu_{12} \mu_{13} D - 0.622 \mu_{14} \mu_{15} D / P \cdot \kappa \\ &= \mu_{11} \mu_{12} (100 / (T_{11} + 2/3))^2 \end{aligned}$$

As stated earlier dT_H / dx_k can be determined by using the chain rule

$$dT_H / dx_k = \{ \partial T_H / \partial x_k / \partial T_H \}$$

The average

$$dT_H / dx_k = \int_0^1 dT_H / dx_k dt$$

where it is assumed that $T_H = T_H^a$.

The integration of dT_H / dx_k over the entire year was done by trapezoidal rule.

Define the efficiency coefficient to be the change in temperature T_H or T_H^a caused by a percentage change in the input parameters. Note, parameters $\epsilon, \mu_{11}, \mu_{12}, \mu_{13}, \mu_{14}, \mu_{15}$ and ν which represent radiation will be expressed in percentages, 1 to 100, for the purposes of uniformity in the distribution of the efficiency coefficient. That is for parameters

the standard deviation used is 0.9 which will be 90% for use in calculating the influence coefficients. The one exception is the air temperature. Its influence coefficient will be expressed in terms of a temperature change in T_a of T_b or T_g for a given temperature change in T_a .

The influence coefficients will be used later to determine which are the most important parameters in so far as their effects on the ground temperature are concerned and also they will be used to estimate changes in ground temperature that would result from given changes in the input parameters. Ideally, to be of practical use, the influence coefficient for a particular parameter should be a constant over the range of that parameter and be unaffected by changes in other parameters. That is the effects of a parameter should be linear and uncoupled. As can be seen from the equations in this section (4.5.3), for the various influence coefficients, this cannot be exactly true for any variations of the input parameter; however, it may still be a reasonable approximation for small changes in the parameters.

To get some estimate of the likely values of the influence coefficients and the ranges of these values, Tables B gives the average coefficients for the two data sets. One is the standard data set from section (4.4) and the other is a modified data set obtained by changing selected parameters within the limit of these parameters' ranges. It can be seen that most of the influence coefficients are the same for both data sets, however, some do vary. In particular the coefficient $(\Delta T_g)/\Delta T_a$ varies significantly between the two data sets. This is because it depends on the parameters $(T_M - T_m)$, which are

turn is affected by all the other parameters. The effect of the average wind velocity u is thus strongly coupled. The influence coefficient for M_H changed by about 30% between the data sets. This could be attributed primarily to strong non-linear dependence of vapor pressure on temperature.

TABLE 8. Analytical Values of the Average Influence Coefficients

Input Parameter	Average Influence Coefficients	
	Standard Data Set	Modified Standard Data Set ⁽⁴⁾
	T_b	ΔT_b
$(\Delta R_b)/R_b$	0.024	0.025
$\Delta a_R^{(1)}$	0.090	0.070
$\Delta(i)$	-0.051	-0.045
$\Delta(i_b)$	-0.047	-0.047
$(\Delta u)/u$	-0.028	-0.043
$\Delta(R_H)^{(2)}$	0.018	0.017
$\Delta(w_v)$	0.027	0.026
$\Delta(M_H)^{(3)}$	-0.017	-0.023
(ΔT_A^*)	{0.819}	{0.812}

(1) $a_R \approx \Delta R/R_b$

(2) $R_H \approx w$

(3) $M_H \approx XMS$

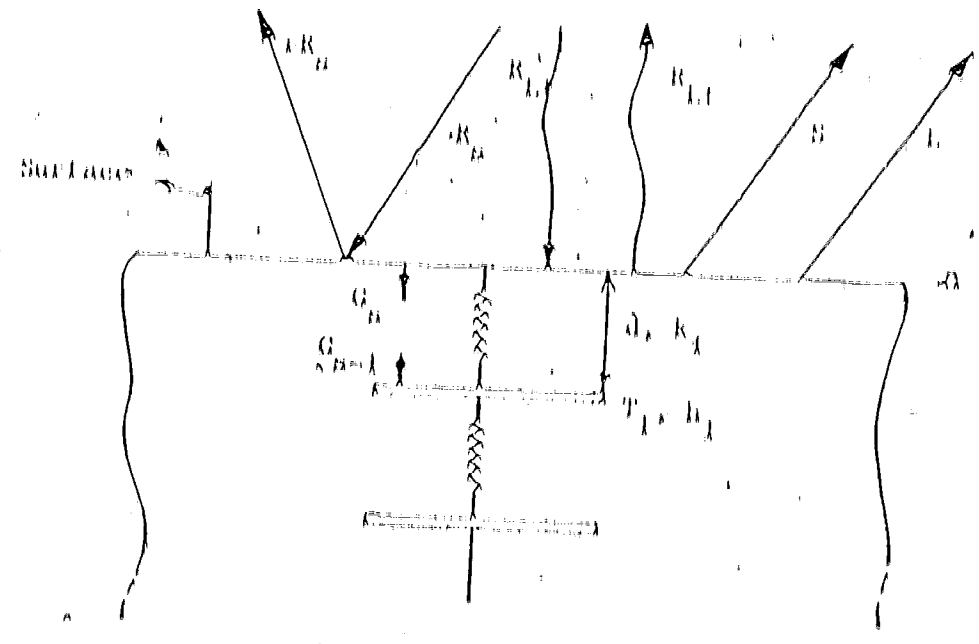
(4) Some of the parameters of the standard data set was modified with their limits from their individual nominal values.

4.6 Numerical Model

Basically the numerical model to be presented is similar to the numerical model presented in Part II of this thesis, except for the different boundary conditions and changes in the definition of the various dimensionless variables. The problem now has, instead of an imposed temperature boundary condition, a varying heat flux at one boundary. The changes in the definition of the various dimensionless variables are necessary so as to accommodate the new parameters.

The varying heat fluxes are those discussed in the early sections of this part (Part IV). The insulating layer is assumed to have negligible heat capacity and values of the insulating values are from Appendix III. The parameters in the ground remain unchanged.

4.6.1 Formulation



Governing Equations

$$G_{n+1} = (1 + \Delta t) R_{n+1} + R_{n+1} - R_{n+1} \quad (6.17)$$

$$G_{n+1} = G_n$$

$$T_{n+1} = T_n + \Delta t G_{n+1} / k_I$$

$$\Delta x = dh_I / dt = G_{n+1} - G_{n+1}$$

$$\Delta x = dh_I / dt = G_{n+1} - G_{n+1}$$

$$h = \rho c_{ul} T = \rho c_{ul} (T - T_I)$$

$$h = \rho c_{ul} T + k_I T_I$$

$$G_{n+1} = k(T_{n+1} - T_I) / \Delta x$$

where $k = k_I$ $T = T_I$
 $k = k_{ul}$ $T = T_I$

4.6.2 Normalization

Defining the various dimensionless variables,

we obtain

$$x_c = R_{n+1} c / \rho l_I$$

$$\Delta T = R_{n+1} k_c / k_I$$

$$\theta = (T - T_I) / \Delta T$$

$$\psi = h / \rho l_I$$

where $c = T_{max} - T_I$

$$\bar{v}_1 = v_1 / v_{10} = I / I_0$$

$$\bar{v}_{\text{eff}} = v_{\text{eff}} / v_{10} = I / I_0$$

$$\bar{t} = t / t_0 = \kappa / \kappa_0$$

$$\bar{c} = c / c_0$$

$$G_B^* = G_B / R_B$$

$$R_B^* = R_B / R_B$$

$$R_{L,1}^* = R_{L,1} / R_B$$

$$R_{L,1}^* = R_{L,1} / R_B$$

$$\bar{M}^* = M / R_B$$

$$L^* = L / R_B$$

$$k^* = k / k_{\text{eff}}$$

$$k_L^* = k_{\text{eff}} / k_L$$

The governing equations become

$$\int_0^1 G_B^* \bar{v}_1^2 \bar{v}_{\text{eff}}^2 (1-\bar{t}) R_B^* + R_{L,1}^* - R_{L,1}^* \bar{v}_1^2 \bar{v}_{\text{eff}}^2 \bar{M}^* \bar{L}^* \quad (4.28)$$

$$G_{B=1}^* = G_B^*$$

$$0_B = 0_L + (R_B d/k_L \Delta T) G_B^*$$

$$R_{n-1} = G_{n-1} / R_n \approx [1 + (a_{n-1} / a_n)] / R_n$$

$$\Delta \phi_{n-1} = (G_{n-1} / R_n) - (G_{n-2} / R_{n-1}) \approx \dots \quad (1-1)$$

$$\Delta \phi_{n-1} = (k^n (a_{n-1} - a_n) - k^{n-1} (a_{n-2} - a_{n-1})) / R_n \approx \dots \quad (1-1)$$

$$\phi_{n-1} = S_{n-1} = 0 \quad 0 = 0 \quad n=1, 2, 3, \dots$$

$$\phi_{n-1} = S_{n-1} = 0 + 1 \quad 0 = 0 \quad n=1, 2, 3, \dots$$

$$k^n = k_{n-1} \quad 0_{n+1} + 0_{n-1} = 0$$

$$= 1 \quad 0_{n+1} + 0_{n-1} = 0$$

$$R_{n-1} \approx 1 + (AR/R_n) + 1/n \cdot 2n(1 + \phi_R)$$

$$R_{n-1} \approx C_{n-1} / R_n \approx (T_n + 12/3)^4 / R_n$$

$$R_{n-1} \approx C_n \approx (T_n + 12/3)^4 / R_n$$

$$B^n \approx A(T_n - T_A) / R_n$$

$$L_n \approx B(C_n - C_A) / R_n$$

$$C_{n-1} \approx C_n \cdot 23.4 \cdot C_n^{-1.900} / (T_n + 12/3)$$

$$c_{ad} = c_a^{23.4} c_a^{5900/(c_{ad}^{0.73})}$$

$$c_{ad} = w c_{ad}$$

$$c_{ad} = \frac{M}{m} c_{ad} + (1 - \frac{M}{m}) c_a$$

$$c_{ad} = a + bc \cdot c_a$$

$$c_{ad} = c_a (1 + B \frac{c_a}{c_a})$$

$$A = \frac{c_a}{p} D$$

$$B = 0.622 \frac{D}{p}$$

$$D = 1 + B \frac{c_a}{c_a}$$

$$T_{ad} = T_a (1 + \frac{\Delta T}{T_a} \ln 2 \pi (1 + \phi_{ad}))$$

$$\lambda = \lambda_{ad}$$

$$\lambda = \lambda_w$$

$$d/k_a = d_{ad}/k_{ad}$$

$$d = d_w/k_w$$

$$\lambda = \lambda_{ad} (1 + \frac{\Delta T}{T_a})$$

$$\lambda = \lambda_w (1 + \frac{\Delta T}{T_a})$$

$$\lambda = \lambda_{ad} (1 + \frac{\Delta T}{T_a})$$

$$\lambda = \lambda_w (1 + \frac{\Delta T}{T_a})$$

Rewriting the governing equations in the normalized form

$$R_b^{-1} = 1 + \Delta R/R_b = \ln 2 + (1 + k_R) \quad (4.29)$$

$$R_{L+1}^{-1} = c_b c_a \cos \alpha \left(\frac{T_a + T/3}{3} \right)^4 / R_b \quad (4.30)$$

$$R_{L+1}^{-1} = c_b \left(\frac{T_a + T/3}{3} \right)^4 / R_b \quad (4.31)$$

$$S^1 = \Delta(T_b - T_a) / R_b \quad (4.32)$$

$$L^1 = B(c_b - c_a) / R_b \quad (4.33)$$

$$G_b^{-1} = (1 + L) R_b^{-1} + R_{L+1}^{-1} - R_{L+1}^{-1} - S^1 - L^1 \quad (4.34)$$

$$0_b = 0_1 + (R_b d / k_1 \Delta T) G_b^{-1} \quad (4.35)$$

$$\Delta \phi_k^m = (G_b^{-1} \Delta L + (0_1 - 0_2)) \Delta x^2 / \Delta L^2 \quad (4.36)$$

$$\Delta \phi_k^m = ((0_{k-1} - 0_k) + (0_{k+1} - 0_k)) \Delta L / \Delta L^2 \quad (4.37)$$

$$\phi_k^{m+1} = \phi_k^m + \Delta \phi_k^m \quad (4.38)$$

$$\left. \begin{aligned} 0_k^{m+1} &= \phi_k^{m+1} / \Delta L_k \\ 0_k^{m+1} &= (\phi_k^{m+1} - 1) / \Delta L_k \end{aligned} \right\} \begin{aligned} \phi_k &= 0 \\ \phi_k &= 0 \end{aligned} \quad (4.39)$$

4.6.3 Method of Solution

(a) Initially, set all the ground temperatures equal to an approximate initial temperature, θ_0 (i.e. $\theta_n = \theta_0$).

The initial enthalpies are then calculated,

$$\phi_n = S_{\text{air}} \theta_n + 1 \quad \text{if } \theta_n = 0$$

$$= S_{\text{I}} \theta_n \quad \text{if } \theta_n < 0$$

(b) Using equations (4.29) and (4.30) to compute R_n^* and $R_{L_n}^*$ respectively,

$$R_n^* = 1 + \Delta R/R_n^* \sin 2n(1+\phi_R) \quad (4.29)$$

$$R_{L_n}^* = c_{\text{air}} a_{\text{COIL}} (T_n + 273)^4 / R_n \quad (4.30)$$

(c) To solve for T_n

In equation (4.34), only R_n^* and L_n^* are functions of T_n .

On substituting these expressions into equation (4.34), the resulting expression is

$$Q_n^* = (1-\epsilon) R_n^* + R_{L_n}^* = (c_{\text{air}} a_{\text{COIL}} / R_n) (T_n + 273)^4$$

$$\left(\frac{\Delta}{R_n} \right) (T_n - T_n) = \left(\frac{B}{R_n} \right) (M_n)^{2.3 \cdot 4} (5900 / (T_n + 273))$$

$$5900 / (T_n + 273)$$

Then substitute G_b^* into equation (4.35)

$$u_b = u_1 + (R_b d/k_1 \Delta T) G_b^* \quad (4.35)$$

and writing $u_b = (T_b - T_F)/\Delta T$, equation (4.35) becomes

$$Y = A_0 + A_1 Y^4 + A_2 e^{-5900/Y} \quad (4.40)$$

where $Y = T_b + 273$ and A_0, A_1, A_2 are known constants.

$$A_0 = \frac{(1-\epsilon) R_b + \epsilon h_a \text{coeff} (T_a + 273)^4 + \text{ocp} D(T_a + 273) + (\Delta T (u_1 + 273))/d/k_1 + \text{BM}_h \epsilon_a / (\text{ocp} D(1/d/k_1))}{1}$$

$$A_1 = \epsilon_a d / (\text{ocp} D(1/d/k_1))$$

$$A_2 = \text{BM}_h \epsilon_a^{2.3,4} / (\text{ocp} D(1/d/k_1))$$

$$f(Y) = Y = A_0 + A_1 Y^4 + A_2 e^{-5900/Y} \quad (4.41)$$

$$f'(Y) = 1.0 + 4A_1 Y^3 + 5900A_2/Y^2 e^{-5900/Y} \quad (4.42)$$

Equation (4.40) is solved by first approximation at root, then by means of successive approximations from the iteration

$$X_{i+1} = X_i + f(X)/f'(X) \quad (4.43)$$

$$T_b = Y - 273$$

- (d) With the above T_b values, $R_{L, l}^*$, S^* and L^* are obtained from equations (4.31), (4.32) and (4.33) respectively.

$$R_{L, l}^* = c_{1, n} (T_b - T_a / 3)^4 / R_b \quad (4.31)$$

$$S^* = \Delta(T_b - T_a) / R_b \quad (4.32)$$

$$L^* = B(c_{2, n} - c_a) / R_b \quad (4.33)$$

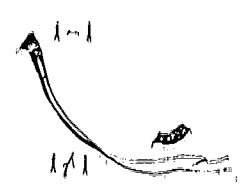
- (e) Then compute G_b^* using equation (4.34)

$$G_b^* = (1-\tau) R_b^* + R_{L, l}^* - R_{L, l-1}^* - S^* - L^* \quad (4.34)$$

- (f) Compute the enthalpy changes at each nodal point, using equation (4.36) for the top layer and equation (4.37) for all the remaining layers.

$$\Delta\phi_k^m = (G_b^* \Delta t_k - (0_{k=0, 2})) \Delta t_k^2 / \Delta t_k^2 \quad (4.36)$$

$$\Delta\phi_k^m = ((0_{k=\lambda} = 0_k) + (0_{k+1} = \phi_k)) \Delta t_k / \Delta t_k^2 \quad (4.37)$$



Then using enthalpy balance from this layer to the next and using the enthalpy changes, new values of enthalpy at each nodal point are calculated by using equation (4.38).

$$\phi_k^{m+1} = \phi_k^m + \Delta\phi_k^m \quad k=1, 2, 3, \dots \quad (4.38)$$

(g) The enthalpy ϕ_1^{m+1} is then used to calculate θ_1^{m+1} by using equation (4.39)

$$\theta_1^{m+1} = \phi_1^{m+1} / S_1 \quad \phi_1 < 0$$

$$\theta_1^{m+1} = (\phi_1^{m+1} - 1) / S_{mf} \quad \phi_1 > 0 \tag{4.39}$$

ϕ_1^{m+1} and θ_1^{m+1} are the new enthalpies and temperatures one time step ahead of ϕ_1^m and θ_1^m respectively.

(h) Proceed as before, as the time step advances, the initial enthalpies and temperatures are those just computed, ϕ_1^{m+1} and θ_1^{m+1} respectively. The computation is over when the temperature distribution for an entire year approaches a steady state.

4.7 Results of the Numerical Calculations

As stated in Part II of this thesis only the steady periodic solution will be of interest; the transient behavior produced by starting from a constant temperature will not be considered. Usually it was necessary to carry out the calculations through only two yearly cycles of temperature variation in order to obtain the steady periodic condition (refer to Part II).

4.7.1 Heat Fluxes

Figures (30) show the values of the various heat fluxes and also the net heat flux to the ground surface during the course of the year. As indicated, all the heat fluxes peak at about the same

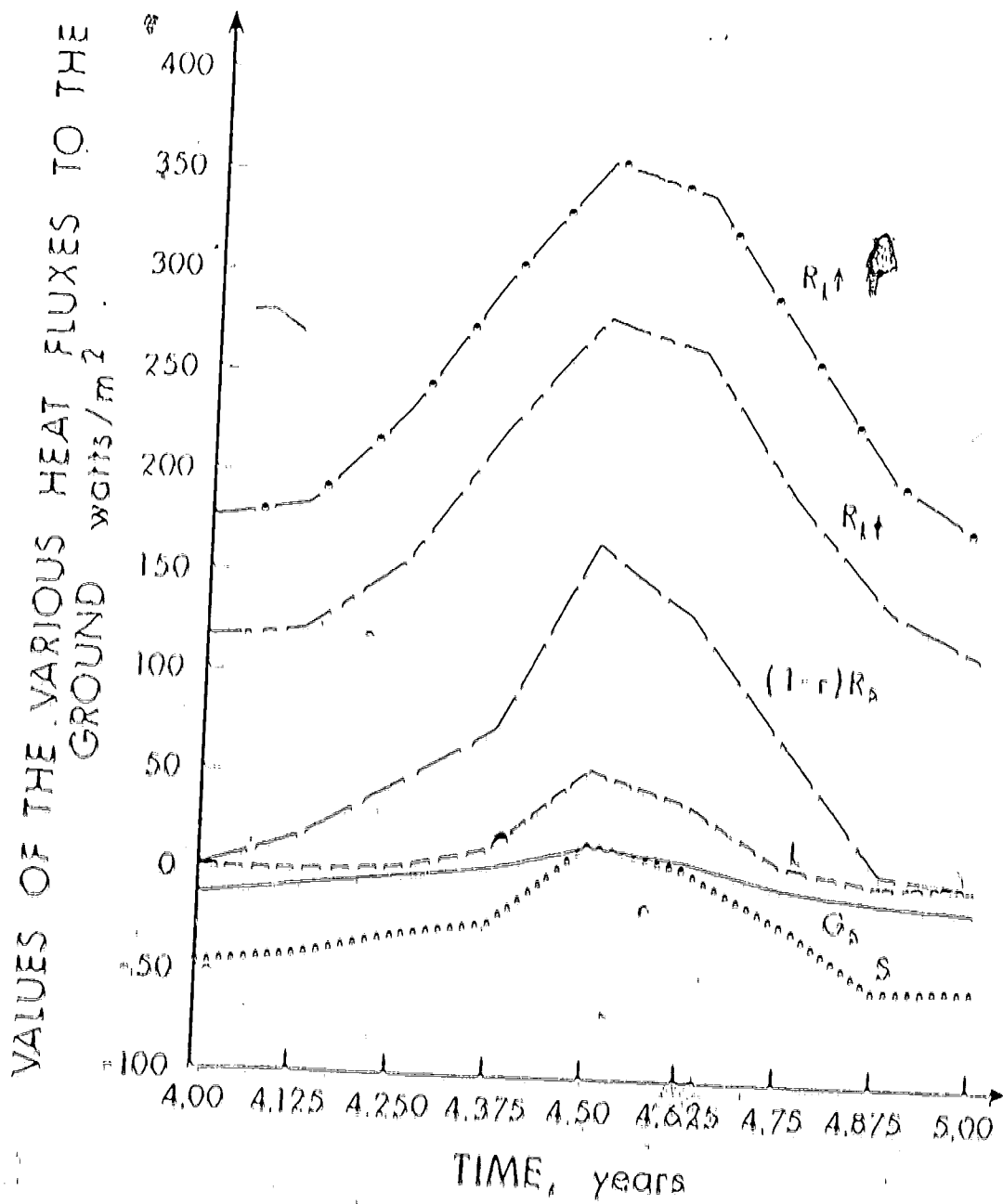


FIGURE 30 Values of the various heat fluxes to the ground

time of the year, that is, when the air and surface temperatures are maximum and correspondingly as the air and surface temperatures are minimum. The net incoming solar radiation $(1-f)R_{\text{in}}$, which is not directly temperature dependent, however, peaks during the summer months and has a minimum value during the winter. All the component heat fluxes have positive values, that is they are in the direction defined in Figure (29), throughout the entire year except for the Sensible Heat Flux, which varies from a minimum negative value to a maximum positive value. This is because during the winter, the surface temperature T_{B} is lower than the air temperature T_{A} and in summer, T_{B} is greater than T_{A} . The empirical equation for S is

$$S = \rho c_p D(T_{\text{B}} - T_{\text{A}}),$$

in which S is negative in winter and positive in summer.

The net heat flux to the ground G_{H} also varies from a minimum of -12 watt/m^2 to a maximum of 15 watt/m^2 . The empirical expression for G_{H} is

$$G_{\text{H}} = (1-f)R_{\text{in}} + R_{\text{lat}} = R_{\text{lat}} - S - L.$$

The negative value of G_{H} during the winter months indicates that heat is given off by the ground during the winter, and similarly positive value of G_{H} indicates that heat is absorbed.

TABLE 9. Comparing the Various Heat Fluxes From Various Sources For Norman Wells (65°N)

Data Source	Net Radiation R_{net} watts/m ²	Latent Heat L watts/m ²	Sensible Heat S watts/m ²
1. Sellers (44)			
Mean values for 60-70°N	26.0	18.2	7.8
2. Hare (52)	22.1		
3. Standard Data Set	6.9	15.6	-22.4
4. Modified Standard Data Set	22.7	20.1	2.6

As indicated in Table 9 the values of the various heat fluxes depend very much on the choice of the input parameters. The values from Modified Standard Data Set agree favourably to those obtained from Sellers, whereas those values from the Standard Set are significantly different. The modified net are just the values of the standard net with some variations in some of the controlling parameters such as T_a , R_H , ΔR_H , ϵ_a , S_{net} , F_{all} , d_H and d_w . The variations are however within the range of variation of each of the individual parameters.

4.7.2 Temperature Distributions

Figure (11) shows the values of the various temperatures, namely, T_a (air temperature), T_s (surface temperature), T_{BL} (boundary layer temperature) and T_{BTL} (temperature just below the melting layer) and also the surface to fluxes. This shows that in winter the

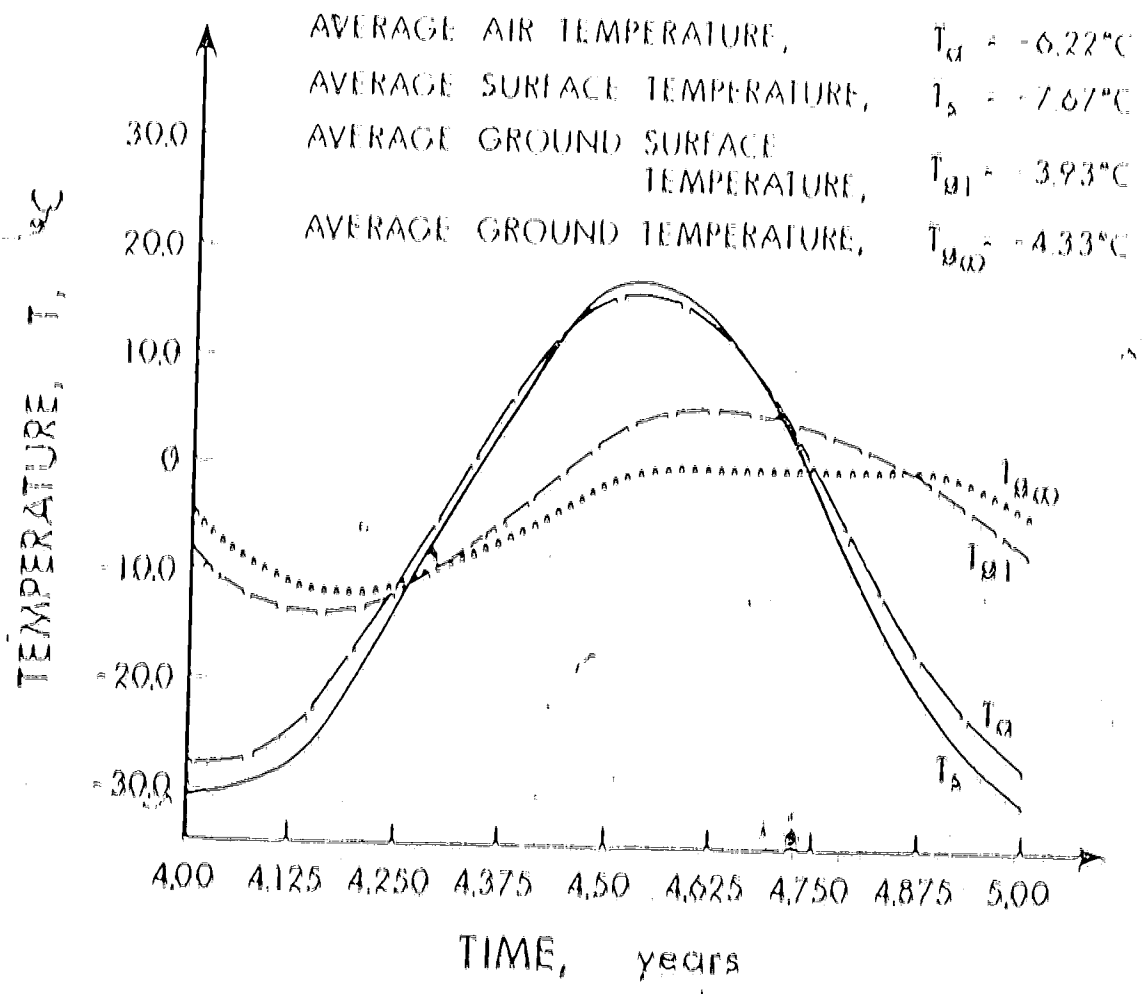


FIGURE 31 Profiles of temperature distributions with varying heat fluxes at the surface, $\bar{Q} = 0.76 \text{ watts/m}^2 \text{ } ^{\circ}\text{K}$, $(U_s - U_w)/\bar{Q} = 1.0$ and $\psi_M = 0.0$

temperature deep in the ground $T_{E\infty}$ is warmer than the ground surface temperature T_{E1} which in turn is warmer than the surface temperature T_s . The air temperature T_a is cooler than the ground surface temperature T_{E1} but is warmer than surface temperature T_s . During the summer months the reverse is true, that is $T_{E\infty}$ is cooler than T_{E1} which in turn is cooler than T_s . The air temperature T_a is warmer than T_{E1} but is cooler than T_s . It also will be noted that the averages of these temperatures over the year are not the same. The differences in the average temperatures between the various levels is due to non-linearity in the equations.

$$\Delta T_a = 6.22^\circ\text{C}$$

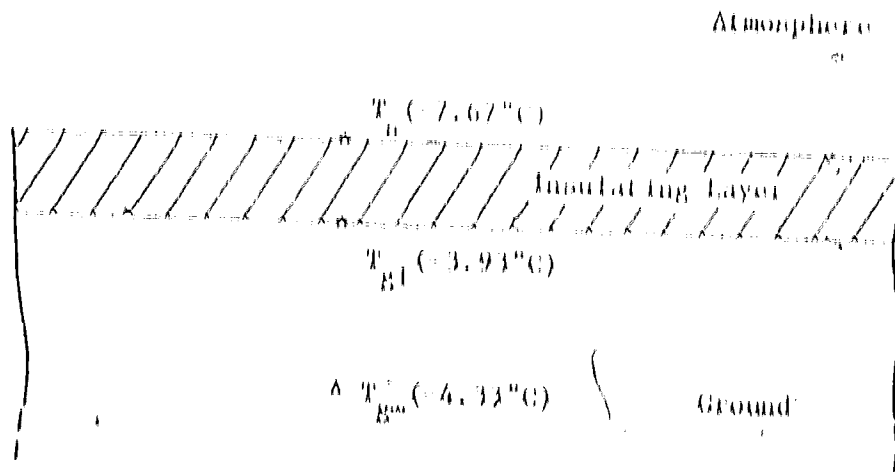


FIGURE 32

The differences in $(T_{E1} - T_{E\infty})$ was studied in Part II and the differences in $(T_s - T_{E1})$ in Part III. The differences in $T_a - T_s$ are shown in this section because of the non-linear nature of the heat flux equations. As indicated in Figure (32), the average temperature is $T_a = 6.22^\circ\text{C}$.

$T_a = 2.67^\circ\text{C}$, $T_{EI} = 3.93^\circ\text{C}$ and $T_{P^*} = 4.33^\circ\text{C}$. Note that $(T_a - T_{EI})$ is by far the largest of these differences indicating the importance of the insulating layer in the determination of the net difference between the ground temperature and the air temperature. Measured values obtained by Brown (99) for the difference between the average air temperature T_a and the average ground temperature T_{P^*} (50 to 100 feet deep) varied from 2.7 to 4.4°C (or 5 to 8°F). However in one of his other papers Brown (90) obtained another set of values of $T_a - T_{P^*}$ (12" deep) for Norman Wells which range from 0.7 to 11°C. The measurements were made under different conditions at different sites. The computed values for $(T_a - T_{P^*})$ is 1.9°C. These large variations in the value of $(T_a - T_{P^*})$ are mainly due to large number of varying influential parameters which the value of $(T_a - T_{P^*})$ are dependent on. Factors that are particularly influential are net radiation, vegetation, snow cover and ground thermal properties.

4.7.3 Depth of Thaw Penetration

The depth of thaw penetration in Norman Wells is about 0.625 m (approximately 2 feet), as indicated in Figure (33). The result compares favorably with those obtained by Gold (33).

4.7.4 Average Influence Coefficients

Earlier in section (4.5.3) the analytical model for finding the average influence coefficients was developed by assuming that there was no interaction between the ground and the surface heat flux. The values for the standard data set were presented in Table B.

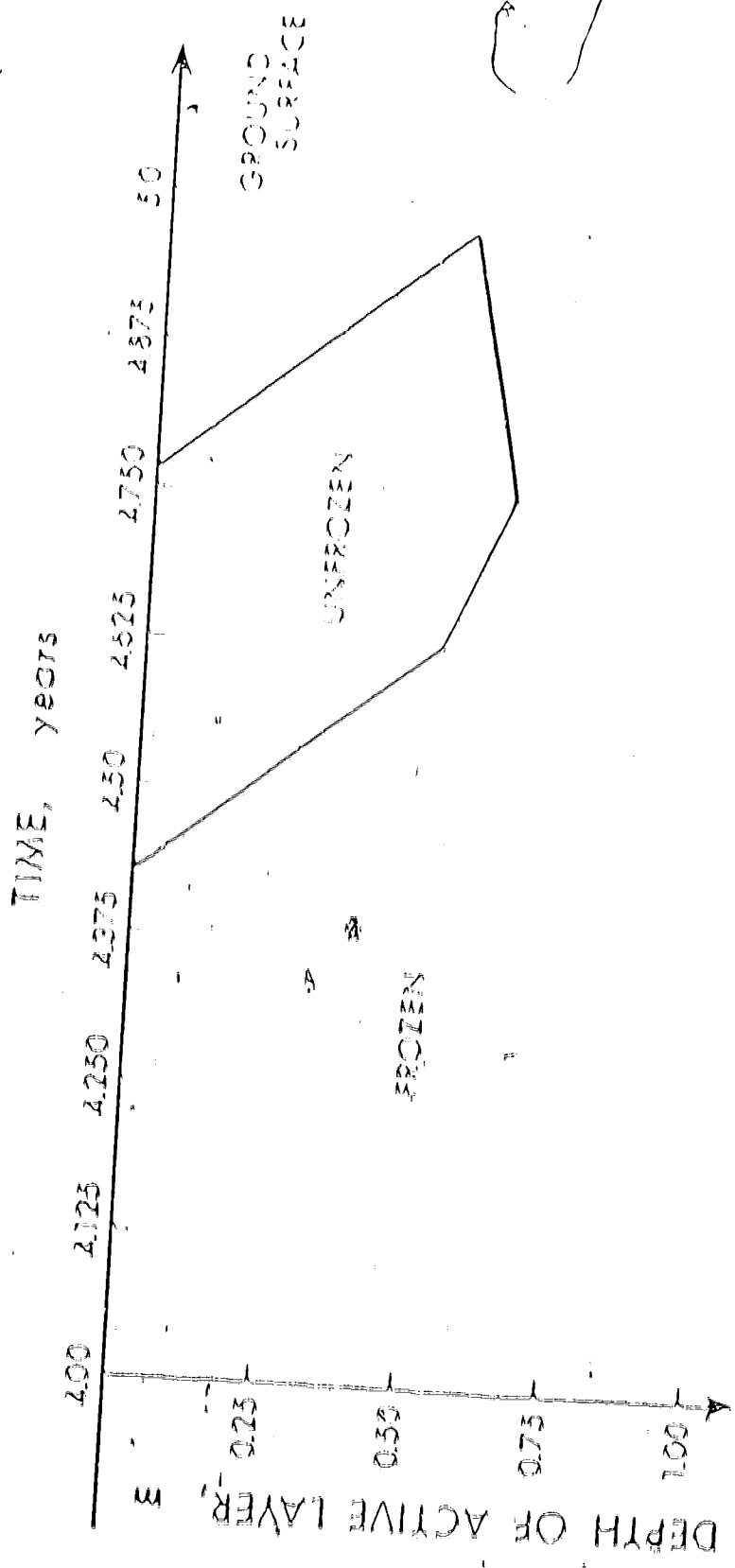


FIGURE 33 - Depth of thaw penetration at Normanville with varying heat flux at the surface

4.3

In this section the values of the average influence coefficients are obtained numerically and the interactions between the ground and the surface fluxes are taken into account. The values are obtained by varying a single parameter at a time and then divide the change in the average surface temperatures (T_B) and the change in the average ground temperatures (T_{R^m}) by the change in the varying parameter. The range of variation for each individual parameter is a carefully chosen range within the realistic climatic conditions of the location under study.

The values of the average influence coefficient for the standard data set are presented in Table 10. The table also shows that the set of values obtained analytically compares favourably with those obtained numerically. From these coefficients and an estimate of the possible range of each parameter (column 5) the magnitude of its effect on the ground temperature can be estimated (column 6). The order of importance of a parameter in determining the difference between T_{R^m} and T_B or $(T_{R^m} - T_B)$ for a given T_a will be indicated by the magnitude of ΔT_{R^m} in column 6 of Table (10). That is to say the parameter inducing the largest ΔT_{R^m} will be the most important and so on in descending order. This order of importance are $(\Delta U_w)/U_w$, $(\Delta U_B)/U_B$, M_{B0} , $(\Delta V)/V$ and $(\Delta R_B)/R_B$ and is in rough agreement with the field observations summarized by Brown (54).

THE VALUES OF T_{R^m} PRESENTED FOR THIS SET OF PARAMETERS IS 10.77°C (SEE ΔT_{R^m} IN COLUMN 6). THE VALUES OF $(T_a - T_{R^m})$ (12 DEGREES CELSIUS) FOR NORMAN WALKER OBTAINED BY BROWN (50) IS FROM 0.7 TO 11°C . THESE CAN BE COMPARED WITH THE VALUES OF $(T_a - T_{R^m})$ OBTAINED IN THIS SET OF PARAMETERS AS SHOWN IN

TABLE 10 Numerical Values of the Average Influence Coefficients

Input Parameters (1)	Average Influence Coefficients			Range of Input (5)	ΔT_{E^*} (6)
	T_b (Analytical) (2)	T_b (3)	T_{E^*} (4)		
$(\Delta R_b)/R_b$	0.024	0.025	0.030	40%	1.20
$\Delta(t_H)$	0.051	0.022	0.033	20	0.66
$\Delta(t_W)$		0.020	0.015	20	0.30
$\Delta(t_H)$	-0.047	-0.038	0.040	10	0.40
$\Delta(R_H)$	0.018	0.027	0.033	30	0.99
$\Delta(M_b)$	-0.018	0.018	-0.024	20	0.48
$\Delta(w_c)$	-0.027	0.019	0.021	20	0.42
$(\Delta V)/V$	-0.028	-0.024	-0.042	30	1.26
$\Delta(\Delta U_H)/U_H$		0.0	0.036	100	3.60
$\Delta(\Delta U_W)/U_W$		0.0	-0.056	100	5.60
(ΔT_H)	{0.819}	{0.958}	{1.089}	{3.0°C}	3.27
$\Delta(\gamma_{MOLE})$ (1)	"	0.0	-0.020	30	0.60
$\Delta(M_{MOLE})$ (2)	"	0.0	0.042	30	1.26

(1) γ_{MOLE} = density of mole

(2) M_{MOLE} = mole molecular weight

Δ MOLE = 1 MOLE

an Edmonton and isolated pockets of thawed soil as far north as Fort McPherson (67°N).

Most influence coefficients were found to be constant within 20% over the range of the input parameters. Typical results for the non-linear cases are shown in Figure (34) and Figure (35). The application of these influence coefficients will be discussed later in section (4.7.7).

4.7.5 Effective 'heat transfer coefficient'

In section 4.5.2 the analytical value for the effective 'heat transfer coefficient' was determined by differentiating equation (4.42) with respect to the T_n to give equation (4.23).

$$h_{eff} = \left. \frac{\partial F}{\partial T_n} \right|_{T_n = \Delta} \quad (4.23)$$

The value of h_{eff} was then determined by assuming the nominal value of T_n to be $T_n = \Delta$, the value which gives $F = 0$.

In the numerical model, the effective 'heat transfer coefficient' h_{eff} is determined by dividing the change in the heat flux to the ground \dot{Q}_H by the corresponding change in the average surface temperature T_n . These changes can be brought about by gradually increasing the heat flux from deep in the ground.

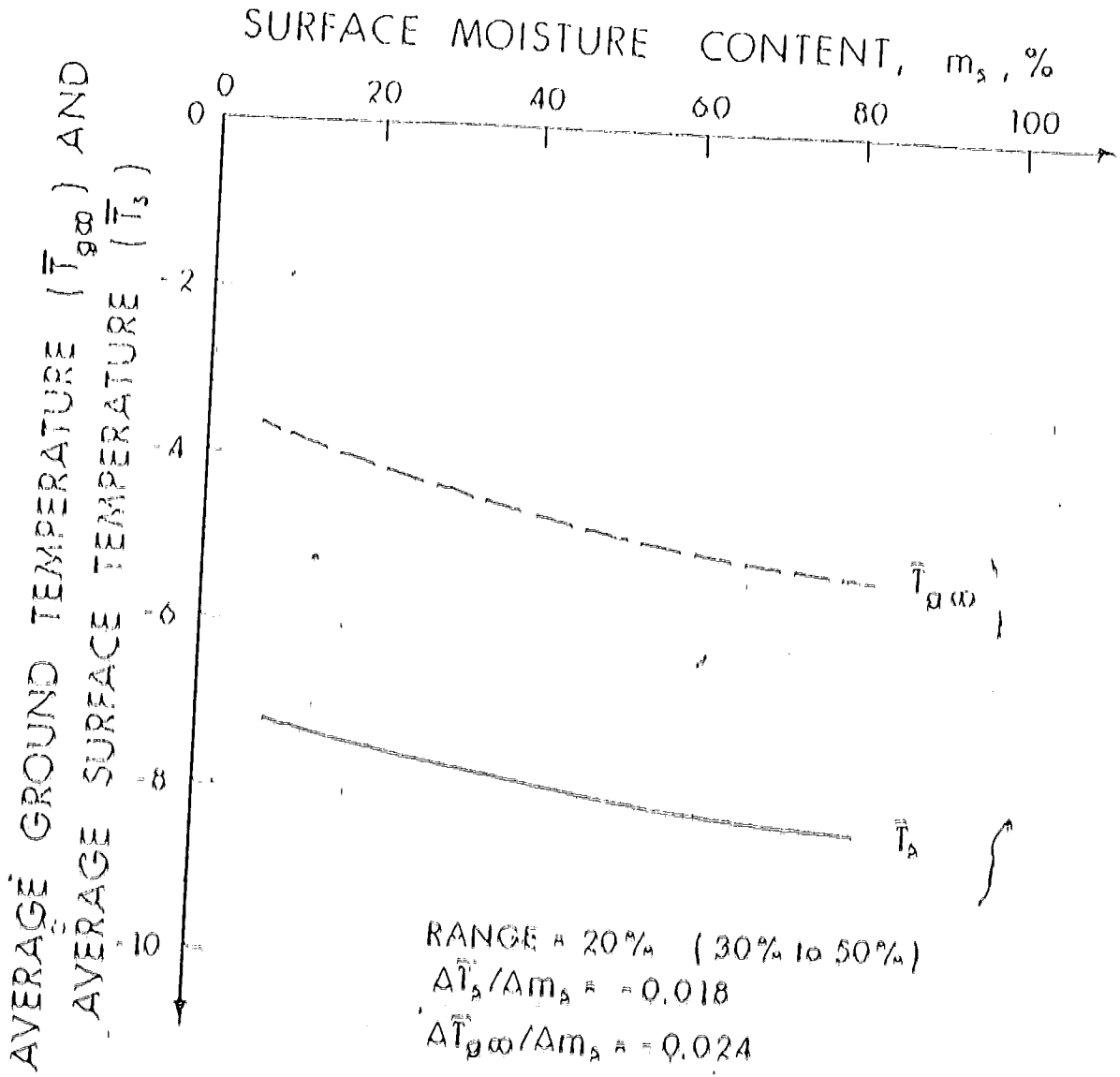


FIGURE 34 - Variation of M_s with \bar{T}_s and \bar{T}_{gw}

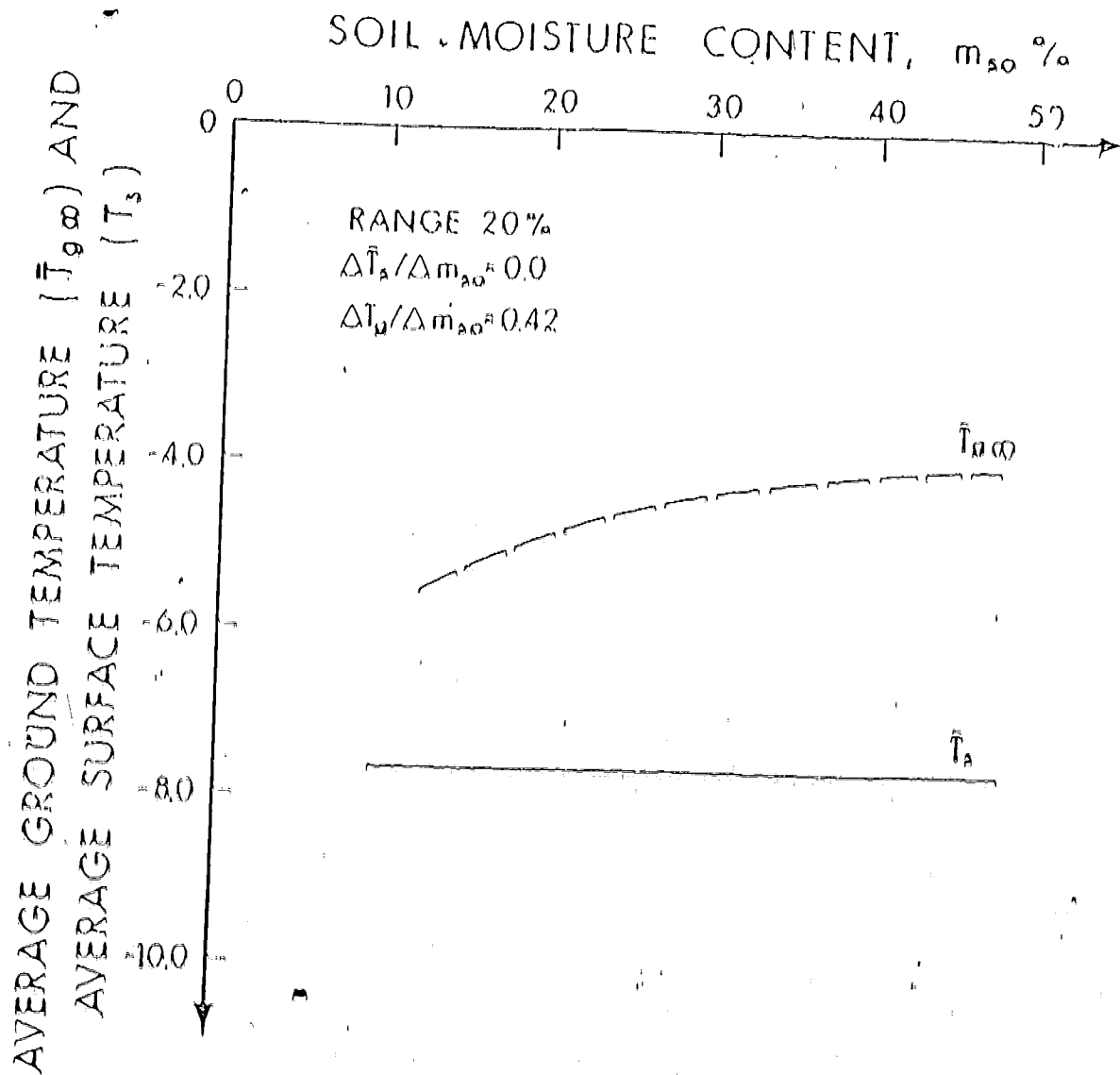


FIGURE 35 -- Variation of M_{so} with \bar{T}_s and $\bar{T}_{g\infty}$

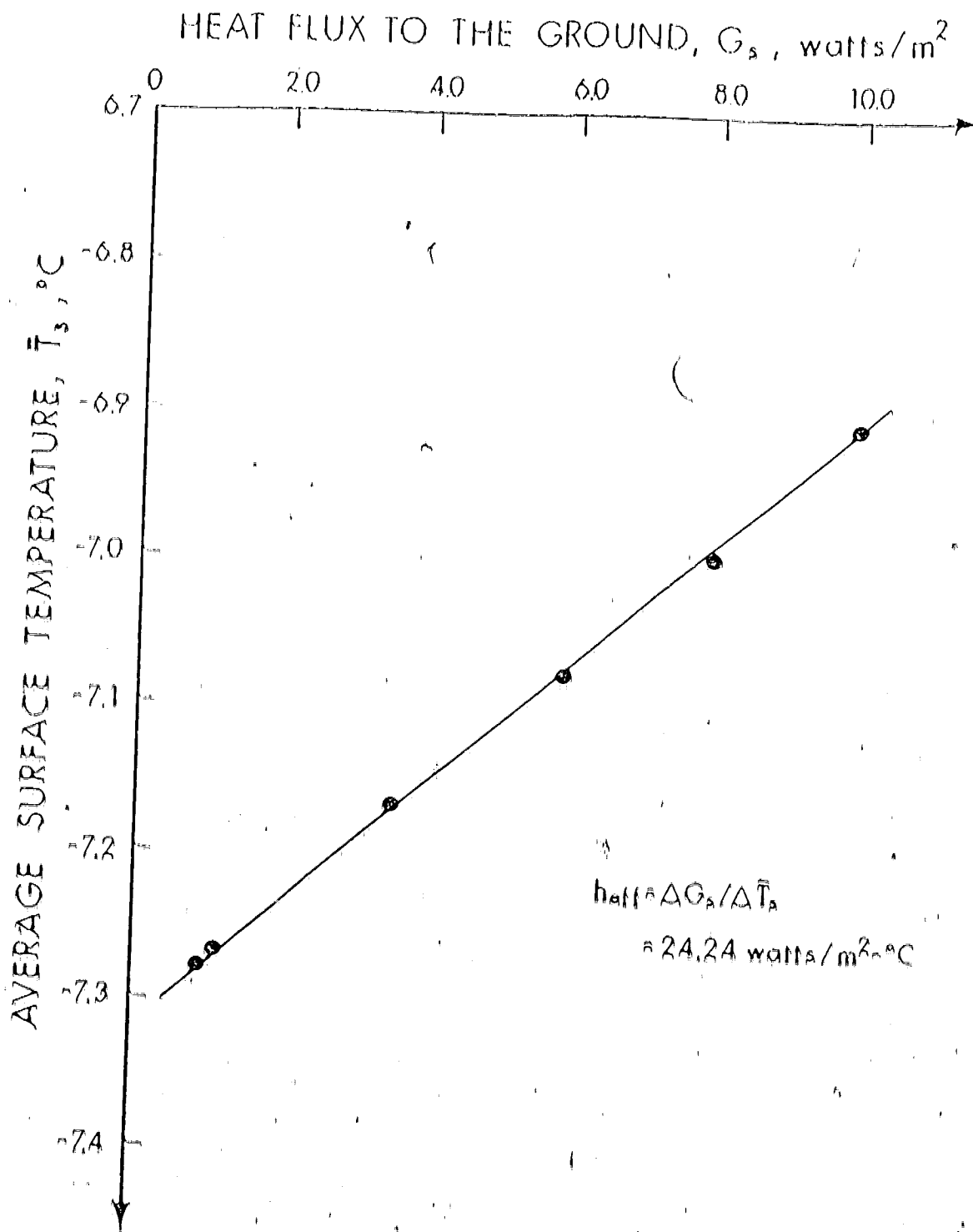


FIGURE 36 -- Relationship between the heat fluxes to the ground and the average surface temperature

Empirically the model can be written as

$$G_n = h_{c11} (T_n - T_n^A) \quad (4.44)$$

$$\frac{\partial G_n}{\partial T_n} = h_{c11}$$

$$\therefore h_{c11} = \Delta G_n / \Delta T_n$$

From Figure (36) the value of h_{c11} obtained by this numerical method is $24.24 \text{ watt/m}^2 \cdot ^\circ\text{C}$, which is slightly higher than the value of h_{c11} ($22.51 \text{ watt/m}^2 \cdot ^\circ\text{C}$) obtained analytically. However both the values are well within the range of 19.39 to $29.09 \text{ watt/m}^2 \cdot ^\circ\text{R}$ obtained by Williams (51) for latitudes 40° to 60°N . This also indicates that the analytical method used is good for a rough approximation, provides the nominal value of T_n is nearly equal to T_n^A .

4.7.6 Coupling between Air and Ground Temperature

Figure (37) shows the various temperatures and the various time intervals during the course of year. Unlike the case in Figure (31) where the average U -factor \bar{U} was 0.76 , the \bar{U} in this case is relatively large, about 6.0 . Comparing the two graphs show that except for the ground surface temperature T_{g1}^A , all the other temperatures namely, air temperature T_A^A , surface temperature T_n^A and ground temperature T_{g1}^A exhibit the same behavior. T_{g1}^A for this case closely follows T_n^A whereas in Figure (31), T_{g1}^A closely follows T_{g1}^A . This means that for $\bar{U} = 0.76$ the major thermal relationship between the air and the ground is the simultaneous cover and for $\bar{U} = 6.0$ the major relationship is between

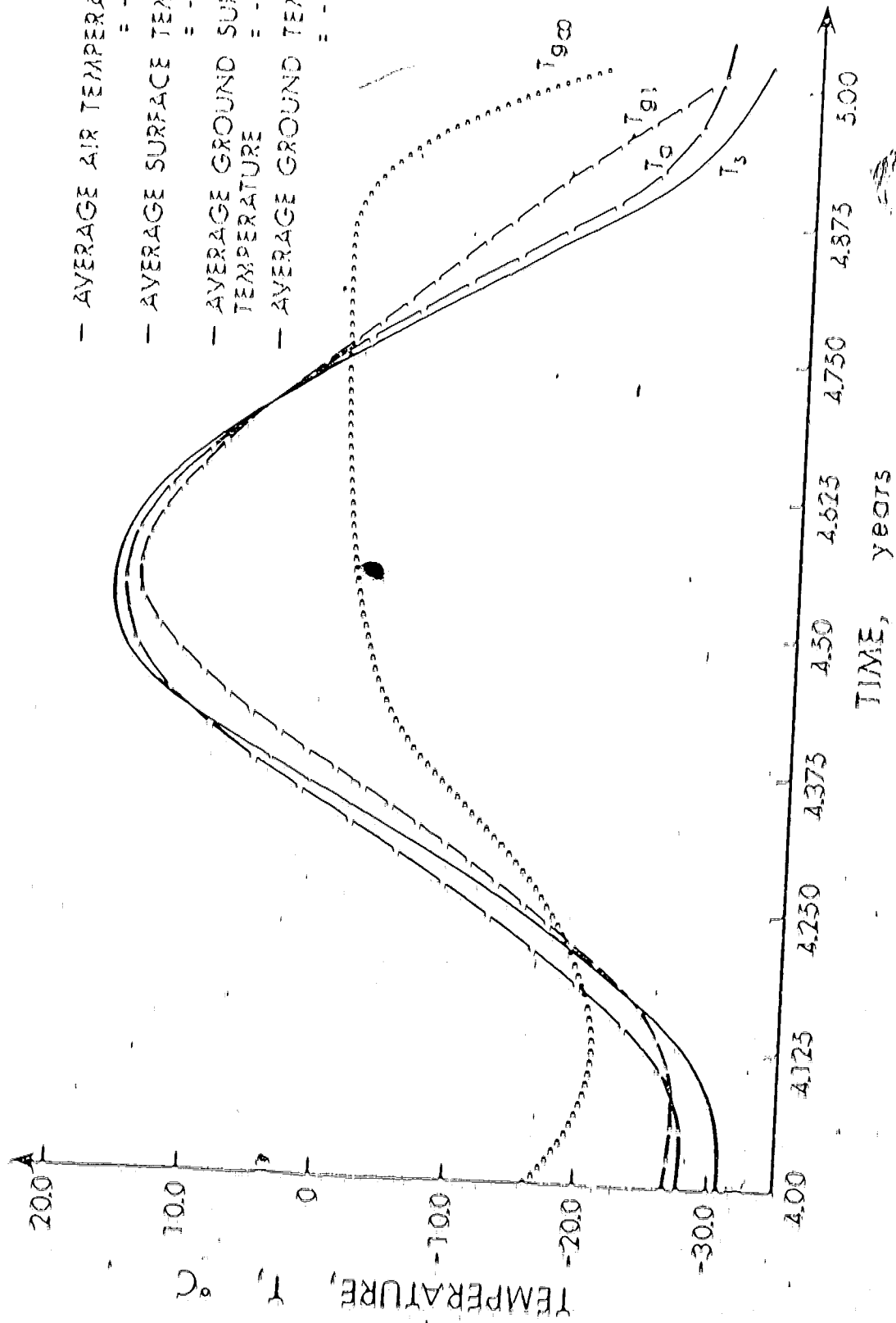


FIGURE 37 — Profiles of temperature distributions with varying heat fluxes at the surface.
 $\bar{U} = 6.0 \text{ watts/m}^2$ — X , $\sigma_0 = U_{\text{eff}} - U_{\text{eff}} = 1.0$ and $\sigma_0 = 0.0$

by the ground itself. The average surface temperature T_s remains unaffected by the change in U , however the average ground temperatures above and below the active layer, T_{E1} and T_{E2} become colder, from -3.9°C and -4.31°C to -7.16°C and -8.31°C respectively.

Figure (B) shows more clearly the effect U has on the average temperatures of the surface and the ground. For small U factor, that is high thermal resistance between the surface and the ground, the two ground temperatures T_{E1} and T_{E2} are almost the same. This occurs because the amplitude of variation in temperature T_{E1} is small and thus no difference between T_{E1} and T_{E2} can be created by the mechanism discussed in Part III. In this case the ground temperature will be determined by the characteristics of the surface cover and the climate parameters and will be unaffected by the soil type or the moisture content. Also note that the average ground temperature is higher than the average air temperature in this situation. This is because of the differential heat value effect of the insulating layer, as discussed in Part III. As U increases, that is as the insulating properties of the ground cover become poorer, the ground temperatures decrease and the difference between T_{E1} and T_{E2} increases. Two factors cause this phenomenon. First, as U increases, the effectiveness of the insulating layer as a differential heat value decreases. Thus the temperature difference $(T_{E1} - T_{E2})$ decreases resulting in the lowering of the ground temperatures. Secondly, as the U -factor increases, the amplitude of the temperature variation in T_{E1} will increase. This increases the effectiveness of the heat value effect caused by freezing and thawing of the soil and increases the difference between T_{E1} and T_{E2} .

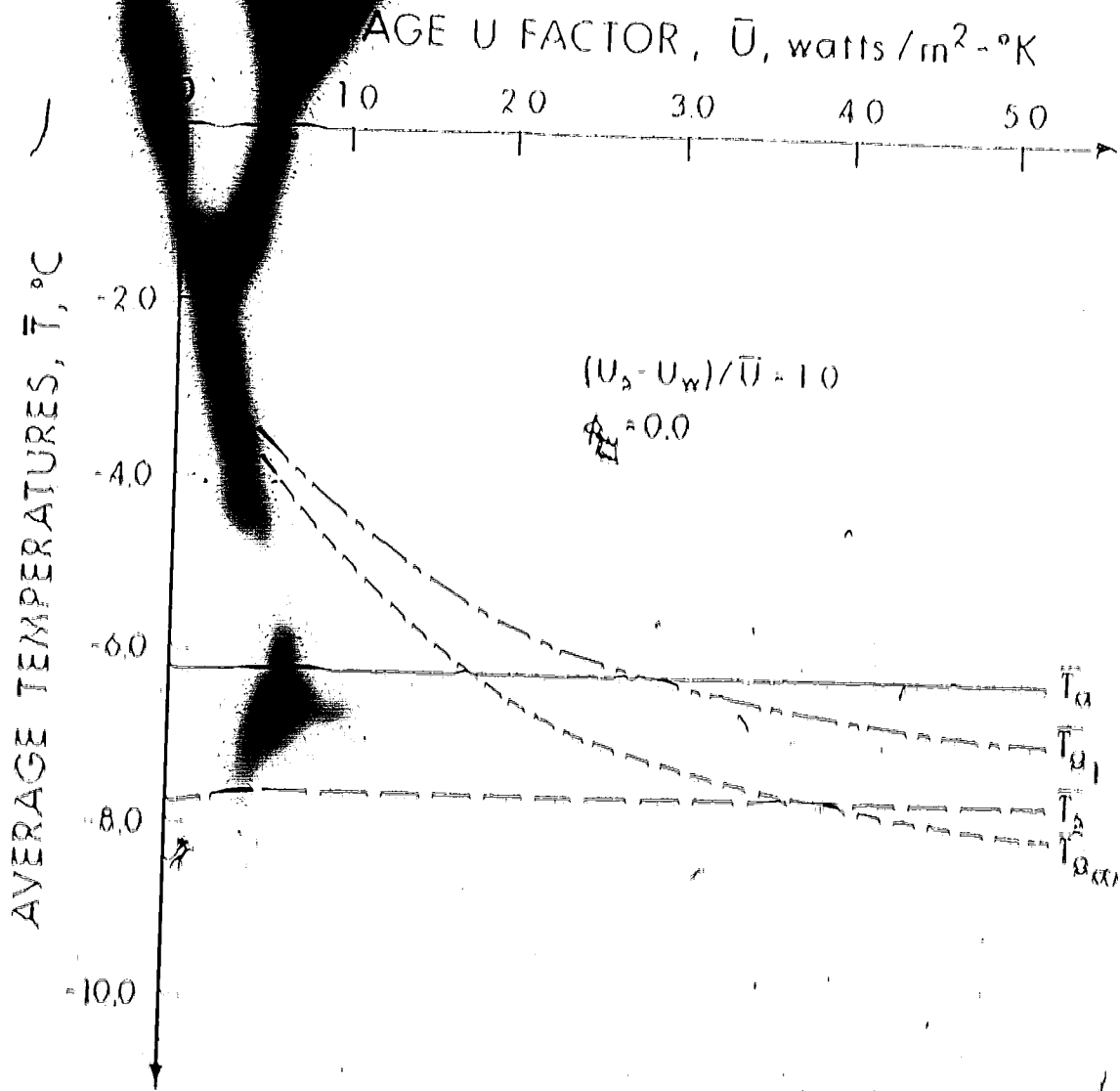


FIGURE 38 - Variation of \bar{U} with \bar{T}

An average U factor of six corresponds roughly to a surface with compacted snow, 10 cm deep in winter and a 5 cm of vegetation in the summer (Appendix III). For surfaces with less insulating value than this, the effects of the insulating cover on the average ground temperature will become negligible. In that case the ground temperature will be controlled by the soil properties and the climatic conditions.

Figure (39) shows the relationship between $(U_n - U_w)/U$ and the various average temperatures for a given U. The figure shows that the average surface temperature T_n is independent of $(U_n - U_w)/U$ but the average ground temperatures T_{E1} and T_{E2} increase very significantly as $(U_n - U_w)/U$ increases. The difference between T_n and T_{E2} is 4°C for $(U_n - U_w)/U = 1.0$. This indicates that variations in the U factor of the surface cover are the major cause of difference between the average air and ground temperatures.

It will be noted from Figure (38) that the average surface temperature is relatively insensitive to the U factor of the insulating layer. This implies that the surface temperature is controlled by the balance of the heat fluxes that occur to and from the air and that the heat flux to and from the ground is too small to have a major effect. This is basically the assumption used in deriving the analytical approximation in section (4.5.2). As stated in that section the equilibrium for application of that assumption is that $U = h_{surf}$. The value of U used is $6.0 \text{ watts/m}^2\text{-}^\circ\text{K}$ and the calculated value for h_{surf} is $24.24 \text{ watts/m}^2\text{-}^\circ\text{K}$. Listed in Table II are the values of T_n and T_{E2} for

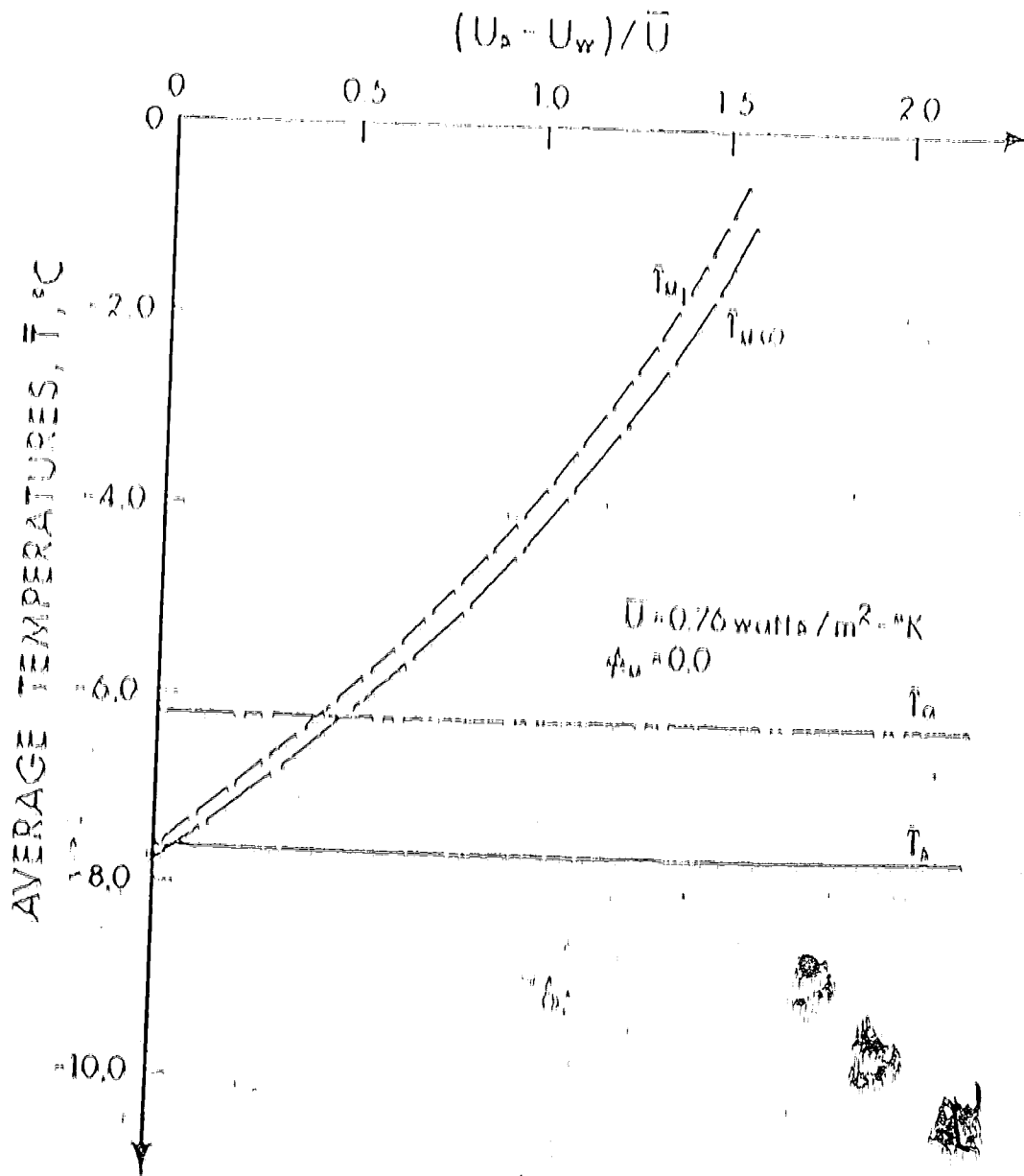


FIGURE 39 - Variation of $(U_s - U_w) / \bar{U}$ with \bar{T}

TABLE 11. Value of T_B and T_B^A for the Standard Data Set

Time (Year)	T_B^A From Section (4.5.2)	T_B $U = 0.76$ $(U_B - U_W)/U = 1.0$	T_B^A $U = 6.0$ $(U_B - U_W)/U = 1.0$
0.000	31.37	30.82	30.51
0.125	28.22	27.89	27.88
0.250	13.74	13.73	14.01
0.375	3.74	3.56	3.21
0.500	16.97	16.42	16.11
0.625	13.58	13.23	13.15
0.750	2.03	1.93	1.50
0.875	21.31	20.85	19.96
1.000	31.37	30.81	30.51

Comparing T_B to T_B^A in Table 11 shows that the difference between the two is very small, even when $U = 6.0$. It also indicates that for problems with small heat fluxes to the ground, the problem of determining the surface balance becomes uncoupled from the problem of heat flow in the ground. Under these conditions, it may also be concluded that it is adequate to assume $T_B = T_B^A$ for most cases, however a better estimate would be $T_B = T_B^A$. This is because by assuming $T_B = T_B^A$ the effects of the heat fluxes are taken into account.

Even if U for surface cover is greater than $h_{surf} T_B$ may still be close to T_B^A because of the insulating value of the ground itself. A general limit of the applicability of the approximation

$T_B - T_B^A$ is given by the equation,

$$T_B - T_B^A = q/h_{eff}$$

where q is the instantaneous value of the heat flux to the ground.

If the maximum value of the heat flux q is less than 24 watts/m^2 , the maximum difference of $(T_B - T_B^A)$ will be less than 1°C .

4.1.1 Application of the Influence Coefficient

As indicated in Table 10 the influence coefficients are relatively constant over the ranges indicated. Because they are uncoupled, the influence coefficients can be used to estimate the effects of the disturbances which simultaneously effect more than one parameter such as disturbances to the surface cover.

As an illustration of the application of the influence coefficients, the effects will be calculated for a disturbance to the surface cover which causes the following changes in the parameters:

α_B changes from 16% to 8%

ϵ changes from 90% to 97%

U_B changes from 1.2 to 2.4 ($\text{watts/m}^2 \text{ } ^\circ\text{K}$)

and M_B changes from 30% to 80%

Keeping all other parameters constant, the values predicted for the changes in T_B and T_B^A using the influence coefficients will be compared to the values obtained numerically. The changes in T_B and T_B^A caused by the change in U_B and M_B will be obtained from the graphs indicated as their relationship are nonlinear.

Changes in T_B and $T_{B^{in}}$ caused by the change in U_B .

U_B changes from 30 to 80

From Figure (34)

$$\Delta T_B = -0.7^\circ\text{C}$$

$$\Delta T_{B^{in}} = -0.9^\circ\text{C}$$

Changes in T_B and $T_{B^{in}}$ caused by the change in U_B .

U_B changed from 1.2 to 2.4

A change in U_B will cause a change in U and $(U_B - U_W)/U$. The change in U is from 0.76 to 1.25.

From Figure (38)

$$\Delta T_B = 0$$

$$\Delta T_{B^{in}} = +1.1^\circ\text{C}$$

The change in $(U_B - U_W)/U$ is from 1.0 to 1.56.

From Figure (39)

$$\Delta T_B = 0$$

$$\Delta T_{B^{in}} = +3.1^\circ\text{C}$$

TABLE 12. Changes in T_B and $T_{B^{in}}$

Parameter	Original Value	New Value	Δ	ΔT_B	$\Delta T_{B^{in}}$
U	90	97	+7	-0.2	-0.2
U_B	16	18	+2	+0.2	+0.3
U_B	1.2	2.4	ΔU	0.0	+1.1
			$\Delta(U_B - U_W)/U$	0.0	+3.1
M_B	30	80	50	-0.7	-0.9
TOTAL				-0.7	+1.2

TABLE 13. Values of Computed T_b and $T_{B''}$

	Original Values (°C)	New Values (°C)	
		Numerically Computed	Computed by Using Influence Coeff.
T_b	1.1	8.4	-8.4
$T_{B''}$	4.4	3.5	-3.2

As indicated the values obtained for T_b and $T_{B''}$ by using the influence coefficients and graphs compare favourably with those actually computed.

4.8 Conclusions

The problem of periodic heat flow to the ground in a freeze-thaw is very complicated because it involves about 30 to 40 parameters. The values of the heat fluxes to and from the ground are very sensitive as demonstrated by their significant difference between the standard data set and the modified data set in Table 9. Thus the absolute determination of the ground temperature is open to doubt, however it is possible to predict the change in the ground temperatures that would result from a change in any one of the controlling parameters. It is however impractical to solve the whole problem every time there is a change in one of the parameters. A much more practical method is to make use of the influence coefficients presented in Table 10. From these coefficients and an estimate of the possible change of each parameter, the magnitude of its effect on the ground temperatures can

be estimated. The most important parameter is the parameter with the biggest effect. The order of importance of the parameters agree with those obtained by Brown (53) from field observations. The most important parameter being the winter insulating layer, then follow the summer insulating layer, soil moisture content, wind velocity and so on. For instance the predicted quantitative effect of winter insulating layer on the ground temperature is 5.6°C whereas the effects caused by the winter reflectivity, r_w is 10.3°C .

Two sets of values of the influence coefficients were obtained, one was obtained by the analytical method where the heat flux to the ground was neglected while the other was obtained numerically. The values obtained analytically for most cases agree with those obtained numerically. These values of influence coefficients, as demonstrated in section (4.7.7), could be used to predict the changes in T_B and T_{Bm} caused by simultaneous variations in some parameters.

The results suggested that various approximations to the surface boundary condition could be applied. First and the simplest would be to assume $T_B = T_A$. This is a common assumption in some calculations of thermal regimes in the ground. For the standard data set used, it was found that the maximum values of $|T_B - T_A|$ was 3.4°C . The second approximation which would include the effects of radiation, sensible and latent heat fluxes would be to assume $T_B = T_A^*$. T_A^* is the value that T_B would be if heat flow in the ground is neglected. This approximation involves the problem of determining the heat fluxes at the surface from that of heat flow in the ground. This surface

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criteria for using this approximation is that U -factor $\approx h_{eff}$. The maximum value of $|T_B - T_B^A|$ was less than 1°C for $U = 6.0 \text{ watt/m}^2\text{-}^\circ\text{K}$. The third approximation would be the assumption that instead of an imposed surface temperature, a Neumann boundary condition exists at the surface. That is that the surface temperature is related to the heat flow to the ground by the equation

$$q = h_{eff} (T_B - T_B^A)$$

where both h_{eff} and T_B^A are time dependent. Both the analytical and the numerical value of the effective 'heat transfer coefficient', h_{eff} obtained agree favourably with the value obtained from field observations by Williams (51). In this approximation, the very complicated heat flux relationships at the ground surface have been simplified by a linear dependence. This last approximation would appear to have very general applicability.

9

BIBLIOGRAPHY

- (1) Carslaw, H.S. and Jaeger, J.C., "Conduction of Heat in Solids",
Clarendon Press, Oxford, London, 1959.
- (2) Berggren, W.P., "Prediction of Temperature Distribution in Frozen
Soil", Transactions, American Geophysical Union, Volume 24,
Part 3, 1943.
- (3) Stefan, J., "Über die Theorie der Eisbildung insbesondere über die
Eisbildung in Polarmaree", Annalen der Physik und Chemie,
Volume 42, 1891, pp. 269.
- (4) Aldrich, H.P. and Paynter, H.M., "Analytical Studies of Freezing
and Thawing of Soils", First Interim Report, Arctic Construction
and Front Effects Laboratory, Corps of Engineers, 1953.
- (5) Moulton, L.R., "Prediction of the Depth of Front Penetration",
Report 5, Engineering Experiment Station, West Virginia
University, 1969.
- (6) Muchlbauer, J.C. and Sunderland, J.E., "Heat Conduction with
Freezing and Thawing", Applied Mechanics Review, Volume 18,
No. 12, 1965.
- (7) Scott, R.F., "Heat Transfer in Soil Involving the Change in State",
Geotechnique, Volume 11, 1961.
- (8) Kesteven, M.B., "Front Penetration: Relationship to Air Temperature
and Other Factors", Bulletin 225, Highway Research Board, 1959.
- (9) Carlson, H., "Calculation of Depth of Thaw in Frozen Ground", Special
Report No. 2, Highway Research Board, 1952.

- (10) Gundersen, J.R., "A Study of Heat Conduction with Phase Change",
M.Sc. Thesis, University of Alberta, 1966.
- (11) Lock, G.S.H., Gundersen, J.R., Quon, D. and etc., "A Study of
one-dimensional Ice Formation with particular reference to
Periodic Growth and Decay", *Int. J. Heat Mass Transfer*,
Volume 12, 1969, pp. 1343-1352.
- (12) Murray, W.D. and Landin, F., "Numerical and Machine Solution of
Transient Heat Conduction Problems Involving Melting and
Freezing", *Trans. ASME*, Volume 81, 1959, pp. 106.
- (13) Lachenbruch, A.H., "Some estimates of the Thermal Effects of a
Heated Pipeline in Permafrost", *U.S. Geol. Surv. Circ. 632*,
1970.
- (14) Roberts, P.C., Geological Survey Computer Contribution No. 4,
"Hot Pipes", *U.S. Geol. Surv., Computer Center Div., Washington,
D.C.*, 1970.
- (15) Fleming, A.K., "The Numerical Calculation of Freezing Processes",
Presented at International Institute of Refrigeration Meeting,
Washington, D.C., 1971.
- (16) Howard, C.T., Murray, D.W. and Brooks, E.W., "A Thermal Analysis
for Melting in Permafrost", *Can. Geotech. J.* 9: 33-40, 1972.
- (17) Chaitwood, R.G. and Suss, O., "Nonlinear Pipelines: An Application
for Numerical Analysis, Part II", Symposium on applications
of solid mechanics, University of Waterloo, Waterloo, Ontario
1972.

- (18) Blot, M.A. and Daughaday, H.S. "Variational Analysis of Ablation",
J. of Aerospace Sciences, Volume 29, 1962, pp. 227.
- (19) Goodman, T.R., "The Heat-Balance Integral and Its Application to
 Problems Involving a Change of Phase", *Trans. ASME*, Volume 80,
 1958, pp. 335.
- (20) Goodman, T.R., "Application of Integral Methods to Transient Non-
 Linear Heat Transfer", *Advances in Heat Advances*, Volume 1.
- (21) Pootn, G., "An Approximate Treatment of a Heat Conduction Problem
 Involving a Two-Dimensional Solidification Front", *Int. J.*
Heat and Mass Transfer, Volume 5, 1965, pp. 339.
- (22) Rubinshtein, L.I., "On Heat Conduction in a Multilayer Medium with
 Phase Transitions", *Doklady Akademii Nauk Rossijskoi Serii*,
 Volume 79, 1951, pp. 221.
- (23) Welner, J.H., "Transient Heat Conduction in Multiphase Media",
Bull. J. of Applied Physics, Volume 6, 1955, pp. 361.
- (24) Cochran, D.L., "Rate of Solidification, Application and Extension
 of Theory", Technical Report No. 24, Stanford University,
 Stanford, California, 1955.
- (25) Olin, D.R., "Solving the Melting Problem using the Electric Analogy
 to Heat Conduction", *Heat Transfer and Fluid Mechanics Institute*,
 Stanford University, Stanford, California, 1956.
- (26) PEARSON, G.A., "Finite Differences Approach to some Heat Conduction
 Problems Involving Changes of State", Report of English Electric
 Co., Luton, England.

- (27) Tien, R.H. and Geiger, G.E., "A Heat Transfer Analysis of the Solidification of a Binary Eutectic System", *J. Heat Transfer*, 89, pp. 230-235, 1967.
- (28) Lamardini, V.J., "The Thermal Properties of Permafrost", 23rd Canadian Geotechnical Conference, Nov. 1970, Banff, Alberta.
- (29) Brown, R.J.E., "Relation between Mean Annual Air and Ground Temperatures in the Permafrost Region of Canada", *Nat. Res. Comm., Canada, Div. Bldg. Res., Res. Paper No. 296*, 1966.
- (30) Burns, B.M., "The Climate of the Mackenzie Valley - Beaufort Sea, Volume I", *Climatological Studies No. 24*, 1973, Atmospheric Environmental Service - Canada.
- (31) Annual Meteorological Summary for Edmonton, Alberta, 1973, Atmospheric Environmental Service - Canada.
- (32) Jansson, L.E., "Ground Frost Penetration in Sweden", *Nat. Res. Comm., Canada, Tech. Trans. 1542*, 1972.
- (33) Gold, L.W., Johnston, G.H. and Goodrich, L.E., "Thermal Effects in Permafrost", *Nat. Res. Comm., Canada, Div. Bldg. Res., Tech. Paper No. 376*, 1972.
- (34) Wijk, W.R. Van, "Physics of Plant Environment", North Holland Publishing Company - Amsterdam, 1966.
- (35) Carnahan, B., Luther, H.A. and Wilkes, J.O., "Applied Numerical Methods", John Wiley & Sons Inc., Toronto, 1969.
- (36) Gold, L.W., "Influence of the Snow Cover on the Average Annual Ground Temperatures at Ottawa, Canada", *Nat. Res. Comm., Canada, Div. Bldg. Res., Paper No. 203*.

- (37) Potter, J.C., "Snow Cover", Climatological Studies, Number 3, Department of Transport, Meteorological Branch, Canada, 1965.
- (38) Brown, R.J.E., "Occurrence of Permafrost in Canadian Peatlands", Nat. Res. Coun., Canada, Div. Bldg. Res., Res. Paper No. 432.
- (39) MacFarlane, I.C., "Munksg Engineering Handbook", University of Toronto Press, Toronto, 1969.
- (40) Bader, H., "Review of the Properties of Snow and Ice", Corps of Engineers, U.S. Army, SMPRE Report No. 4, 1951.
- (41) Mellor, M., "Properties of Snow", U.S. Army Material Command, CRREL, Hanover, New Hampshire.
- (42) Gavrilova, M.K., "Radiatsionnyy klimat Arktiki (The Radiation Climate of the Arctic)", edited by M.I. Budyko, Gidrometeoizdat, 1967.
- (43) Sellers, W.D., "Physical Climatology", The University of Chicago Press, Chicago, 1965.
- (44) Budyko, M.I., "Teplotnyy balans zemnoi poverkhnosti (Heat Balances of the Terrestrial Surface)", Gidrometeoizdat, Leningrad, 1956. Translated by Stepanova, N.A., Office of Climatology, U.S., 1958.
- (45) Gillett, R., "The Climate over the Ground", Harvard University Press, Massachusetts, 1966.
- (46) May, J.E., "Modeling the Radiation Climate of Canada", Canadian Association of Geographers Annual Conference, 1970.
- (47) Matveyev, L.T., "Fundamentals of General Meteorology, Physics of the Atmosphere", Leningrad Program for Secondary Technical Schools, Leningrad, 1967.

- (48) "Climate Normals", Volume 4, Humidity, Canada Department of Transport,
Meteorological Branch, Toronto, 1968.
- (49) "Climate Normals", Volume 3, Canada Department of Transport,
Meteorological Branch, Toronto, 1968.
- (50) Brown, R.J.E., "Some observations of the Influence of Climate and
Terrain Features on the Permafrost at Norman Wells, N.W.T.,
Canada", Nat. Res. Coun., Canada, Div. Bldg. Res.
- (51) Williams, G.P., "Heat Transfer Coefficients for Natural Water
Surfaces", Nat. Res. Coun., Canada, Div. Bldg. Res., Res.
Paper 202, 1964.
- (52) Hare, F.R. and Hay, J.E., "Aspects of the Large Scale Annual Water
Balance over Northern North America", Canadian Geographer,
July, 1971.
- (53) Brown, R.J.E., "Permafrost in Canada", University of Toronto Press,
Toronto.

APPENDIX 1

Data used for the AT_a temperatures.

Edmonton:

From Figure (30), the average air temperature and the amplitude of variation for Edmonton are:

$$T_a = 2.56^{\circ}\text{C}$$

$$\Delta T = 15.96^{\circ}\text{C}$$

The values for Edmonton are obtained from the Annual Meteorological Summary for Edmonton (31). Adopting the same procedure, similar values were also obtained for Norman Wells and Inuvik and are those tabulated in Table 14.

TABLE 14. T_a and ΔT for Norman Wells and Inuvik

Location	Average Temp. T _a (°C)	Amplitude of Variation ΔT (°C)
Norman Wells	6.22	22.06
Inuvik	9.61	22.42

The values for Norman Wells and Inuvik are from Burns (30).

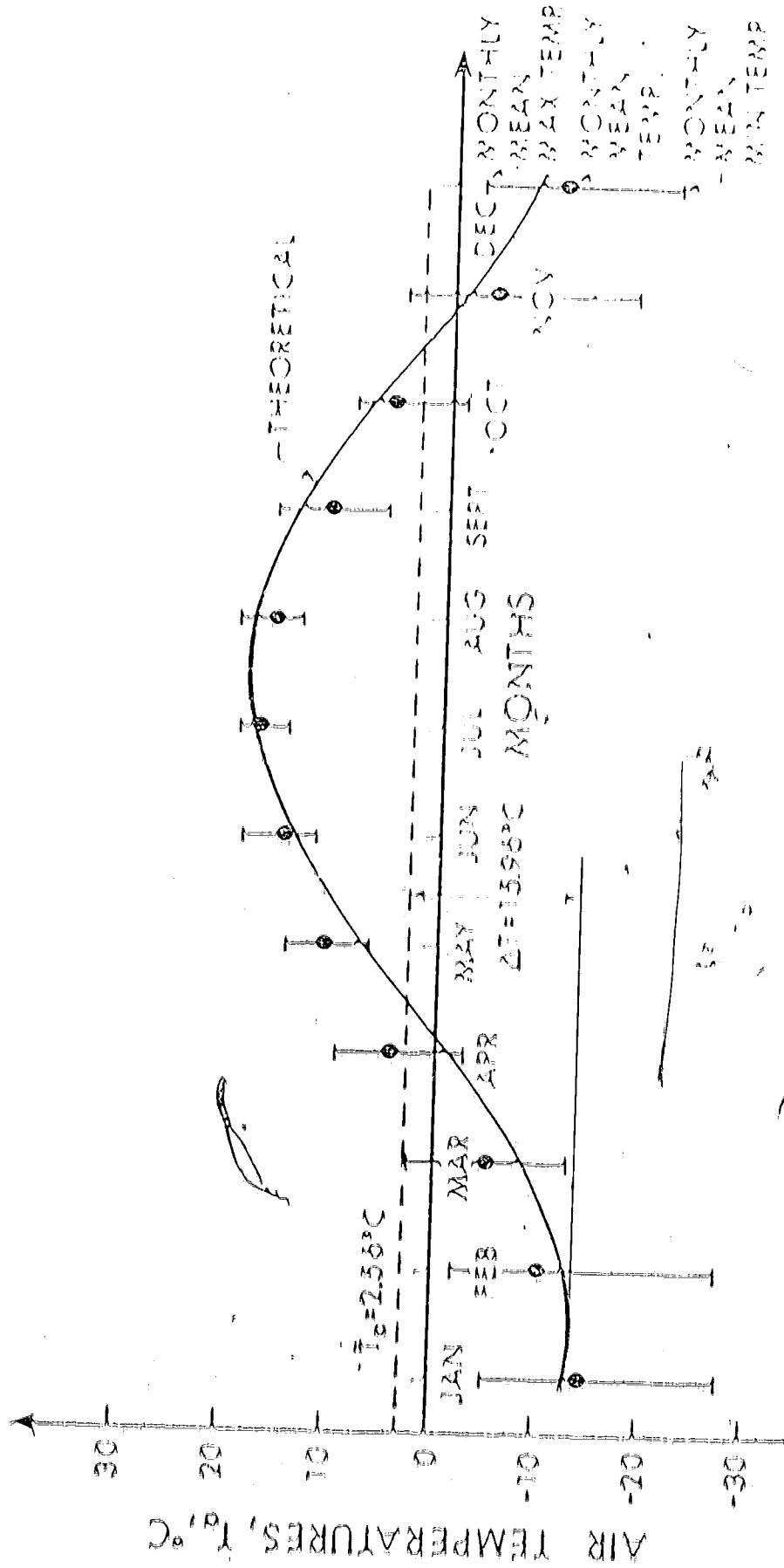


FIGURE 40 - Monthly mean temperature - Edmonton 1880 - 1972
 Source: Annual Meteorological Summary for Edmonton

APPENDIX II

Soil Properties from 'The Thermal Properties of Permafrost' by VITTORE L. LOMBARDO (1961).

Fine-grained soils.

TABLE 15. Values of k_f and c_f for Frozen Soil, $T = -1.39^\circ\text{C}$

Moisture Content, M_{30} (% of dry wt.)	Thermal Conductivity, k_f (watt/m ² /m ² °K)	Heat Capacity, c_f (Joules/m ³ °K)
15	0.82	1.2×10^6
30	1.70	1.6×10^6
45	2.70	1.9×10^6

TABLE 16. Values of k_{uf} and c_{uf} for Unfrozen Soil, $T = 4.44^\circ\text{C}$

Moisture Content, M_{30} (% of dry wt.)	Thermal Conductivity, k_{uf} (watt/m ² /m ² °K)	Heat Capacity, c_{uf} (Joules/m ³ °K)
15	0.70	1.6×10^6
30	1.10	2.4×10^6
45	1.05	3.0×10^6

TABLE 17. Values of F_1

Moisture Content, H_{100} (% of dry wt.)	Heat of Fusion, F_1 (Cal/cm ³)
15	60×10^6
30	120×10^6
45	200×10^6

APPENDIX III

U Factor for snow cover

The average density of the snow cover is 250 kg/m^3 and its average thermal conductivity is $0.13 \text{ watt/m}^\circ\text{C}$ (Gold, 1964).

TABLE 18. Snow Cover Depth for Norman Wells From Potter (17)

Year	Snow Depth (m.)		U Factor of Snow Cover	
	Minimum	Maximum	U_{SN} ($\text{watt/m}^2\text{ }^\circ\text{C}$)	
	Depth	Depth	U_{SN} (Min)	U_{SN} (Max)
0.0	0.28	0.71	0.46	0.18
0.083	0.28	0.76	0.46	0.17
0.167	0.46	1.02	0.28	0.14
0.250	0.48	0.97	0.27	0.14
0.333	0.0	0.61	"	0.21
0.417	0.0	0.0	"	"
0.500	0.0	0.0	"	"
0.583	0.0	0.0	"	"
0.667	0.0	0.0	"	"
0.750	0.0	0.15	"	0.87
0.833	0.0	0.31	"	0.49
0.917	0.15	0.46	0.87	0.28
1.00	0.28	0.71	0.46	0.18

D Factor for Vegetation (Peat) Layer

TABLE 19. Values of k_{ul} and k_l for Peat

Dry Weight kg/m^3	Moisture Content % of dry wt.	Thermal Conductivity $watts/m^{\circ}C$	
		k_{ul}	k_l
250	50	0.067	0.075
	150	0.20	0.33
	250	0.32	0.75
100	50	0.037	0.045
	150	0.058	0.077
	250	0.08	0.15

Values of the thermal conductivity were obtained from Lamardini (28).

Consider only the conditions that give the largest and the smallest values of k_{ul} and k_l for peat.

TABLE 20. Values of U_{af} and U_f

Dry Weight kg/m^3	Moisture Content % of dry wt.	Depth of Peat (m)	U Factor: $watt/m^2 \cdot ^\circ C$	
			U_{af}	U_f
250	250	0.1	3.20	7.50
		0.2	1.60	3.75
		0.3	1.07	2.50
100	50	0.1	0.37	0.45
		0.2	0.18	0.22
		0.3	0.12	0.15

From the table, the conditions that best fit the values used in the numerical calculations are

Dry density = 250 kg/m^3

Moisture content = 250%

Depth of peat = 0.25 m

The corresponding U factor values are

$$U_{af} = 1.28 \text{ watt/m}^2 \cdot ^\circ C$$

$$U_f = 3.0 \text{ watt/m}^2 \cdot ^\circ C$$

U-factor in Summer and Winter

$$U\text{-factor in summer, } U_B = \frac{U_{af}}{1 + \frac{U_{af}}{U_f}}$$

$$U\text{-factor in winter, } U_W = \frac{U_{af}}{1 + \frac{U_{af}}{U_f}}$$

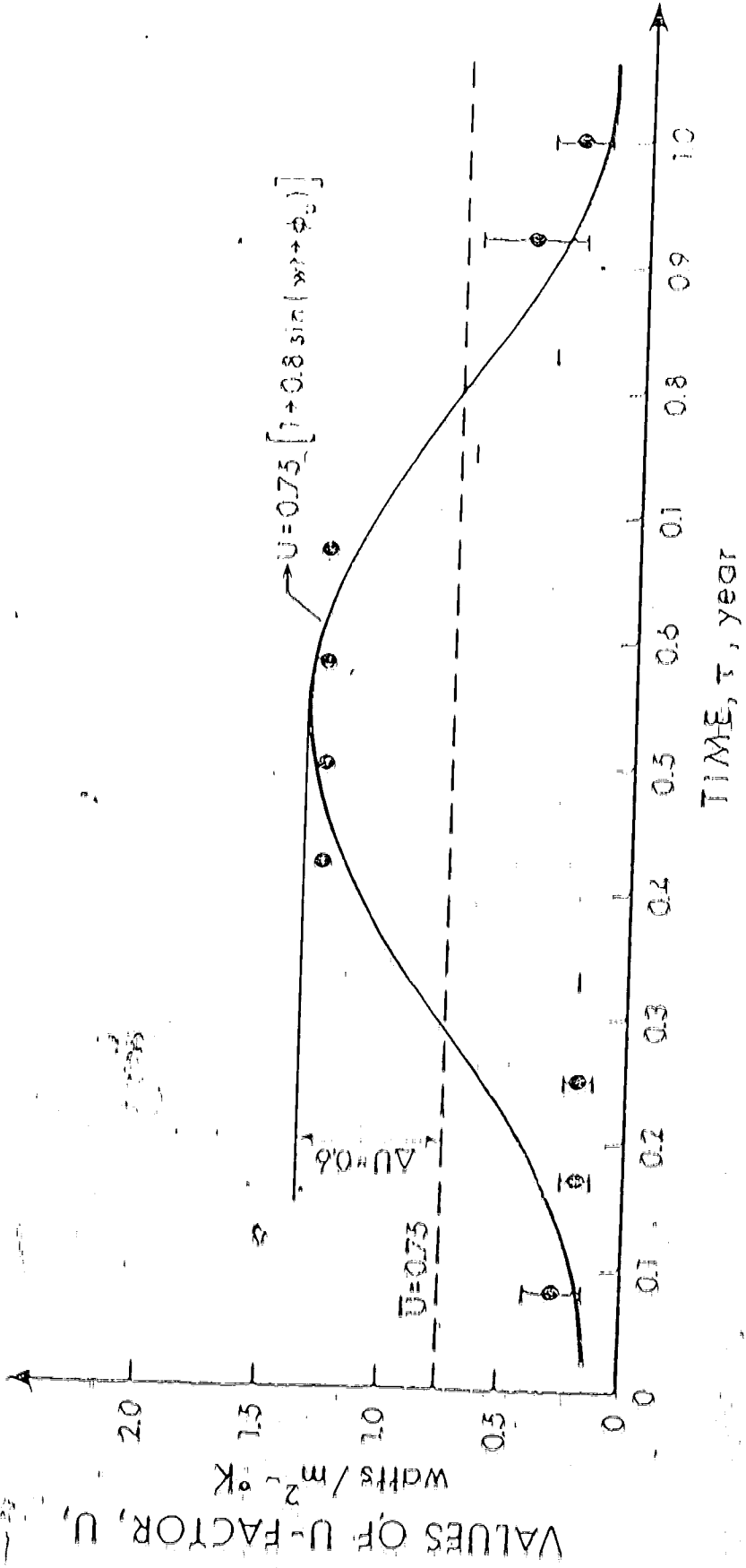


FIGURE 41 - Approximate values of the U-Factor for Norman Yields

TABLE 21. Values of U_b and U_w for the year

Year	U_b watt/m ² °C	U_w watt/m ² °C	
		U_w (max)	U_w (min)
0.08		0.40	0.16
0.17		0.26	0.13
0.25		0.25	0.13
0.33		"	0.20
0.42	1.28		
0.50	1.28		
0.58	1.28		
0.67	1.28		
0.75	"		0.68
0.83	"		0.37
0.92	"	0.68	0.26
1.00	"	0.40	0.17

The thermal properties of the snow cover used are 250 kg/m^3 for the density and $0.13 \text{ watt/m} \cdot ^\circ\text{C}$ for the thermal conductivity. These properties are dependent on many factors, such as, temperature, moisture content, fresh or melted snow and the method used in determining them, as such their values varied over a wide range. For example, its density ranges from 72 to 400 kg/m^3 (40, 41) and its thermal conductivity ranges from 0.018 to $0.5 \text{ watt/m} \cdot ^\circ\text{C}$. These wide ranges in values indicate

variability in the thermal properties of snow. The thermal properties of peat as those for the snow cover are also dependent on many factors, such as, moisture content, porosity, plant structure, temperature and drainage. Peat, because of its nature, has a great ability to soak up water many times its own weight. The moisture content as such determines the thermal properties of peat to a great extent. The moisture content of the peat varies considerably, being less in the more decomposed types than with more fibrous types, and it also varies with season and the corresponding position of the ground water table. For pure peats, when saturated, the water content is more than 600 per cent of dry weight, generally varying between 750 and 1500 per cent but extremes of 50% and 3000% have been noted. However a small percentage of inorganic material in the peat will markedly affect the water content value, which will drop sharply as the mineral contamination increases. The dry unit weight of the peat also varies considerably ranging from 40 to 400 kg/m^3 depending on the type of peat (39). The observed thickness of the peat ranges from 30 cm to more than 6 m in the alluvial alluvium permafrost zone. It usually does not exceed 1 m in the continuous permafrost zone. (4)

APPENDIX IV

Climatic Conditions

Solar Short Wave Radiation

The range of values of short wave radiation (R_u) for a latitude of 65°N from 1943-1957 (62).

TABLE 22. Values of R_u for Norman Wells ($65^\circ 17' \text{N}$)

Month	Year	Mean Radiation (watts/m ²)	Mean Absolute Deviation (watts/m ²)
January	0.08	6.94	1.39
February	0.17	27.78	4.17
March	0.25	90.28	6.94
April	0.33	158.33	8.33
May	0.42	201.39	15.28
June	0.50	208.33	16.67
July	0.58	193.06	11.11
August	0.67	138.89	13.89
September	0.75	75.00	8.33
October	0.83	33.33	5.56
November	0.92	11.11	2.78
December	1.00	2.78	

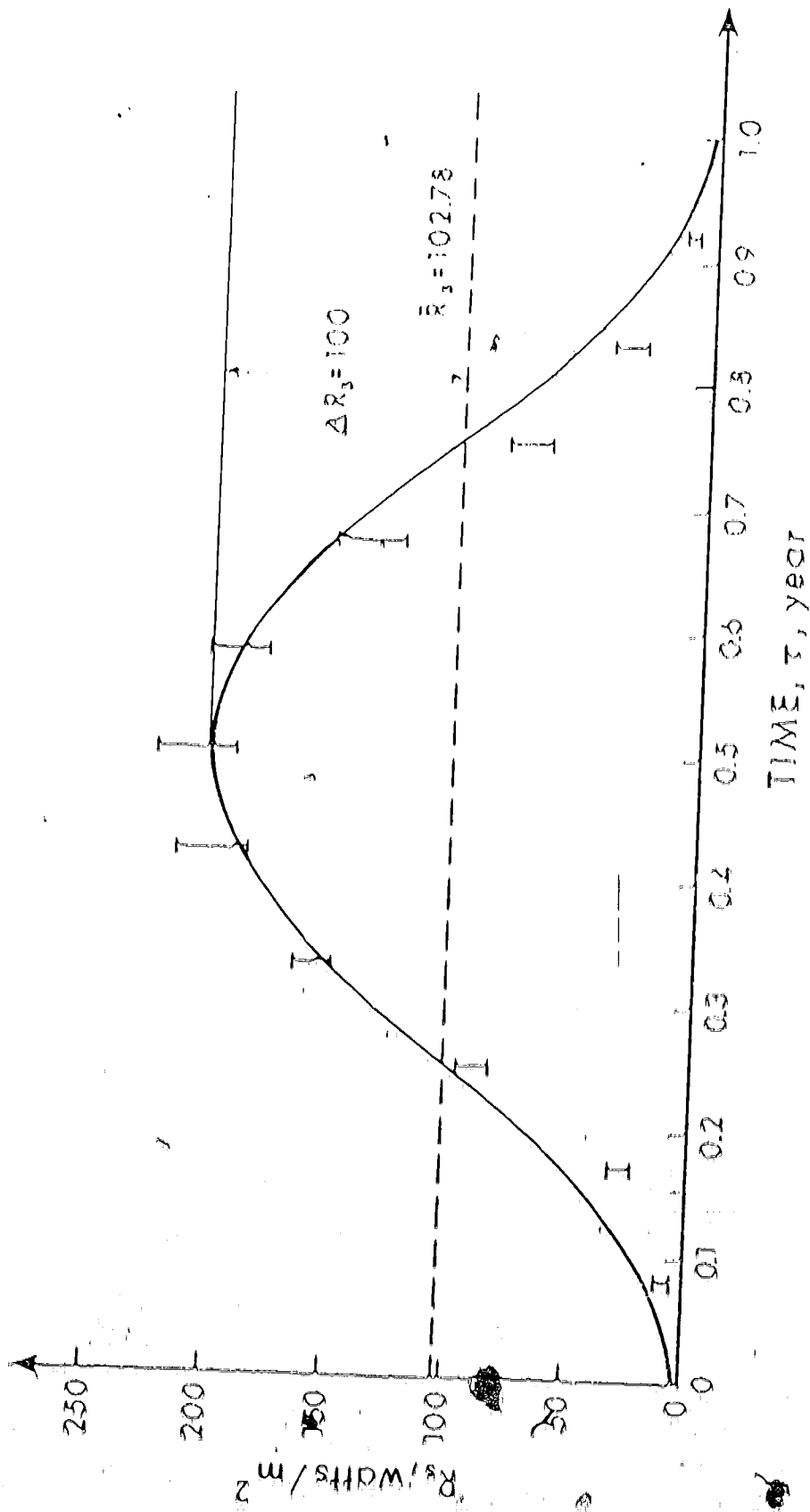


FIGURE 42 — Total radiation (Lat 65° N)
 Source: Radiation climate of the Arctic

APPENDIX V

Clausius-Clapeyron equation expresses the relationship between the temperature T and the saturated vapor pressure e_{sb} .

The equation is

$$\frac{de_{sb}}{e_{sb}} = X \frac{dT}{(T/273)^2} \quad (1)$$

Integrating equation (1) gives

$$\ln(e_{sb}) = -X/(T/273) + Y \quad (11)$$

where X and Y are constants.

TABLE 24. Values of T and e_{sb}

$T(^{\circ}\text{C})$	$1/(T/273)$ ($^{\circ}\text{K}^{-1}$)	Sat. Vapor Press. (M_b)
20	0.00341	23.37
10	0.00353	12.27
0	0.00366	6.11
-10	0.00380	2.86
-15	0.00388	1.91
-20	0.00395	1.25
-25	0.00403	0.81
-30	0.00412	0.51
-40	0.00429	0.19

X and Y are obtained by obtaining values of e_{hb} and $1/(1.273)$ from Figure (33) and then substituting them into equation (11). The calculated values for X and Y are

$$X = 5900 \quad \text{and} \quad Y = 23.4$$

hence

$$e_{hb} = e^{23.4} e^{-5900/(T_b(1.273))}$$

The air vapor pressure is given by

$$e_a = w e_{hb}$$

or

$$e_a = w e^{23.4} e^{-5900/(T_b(1.273))}$$

where w is the relative humidity.

The surface vapor pressure is given by

$$e_b = M_b e_{hb} + (1 - M_b) e_a$$

where M_b is the molal ratio content,

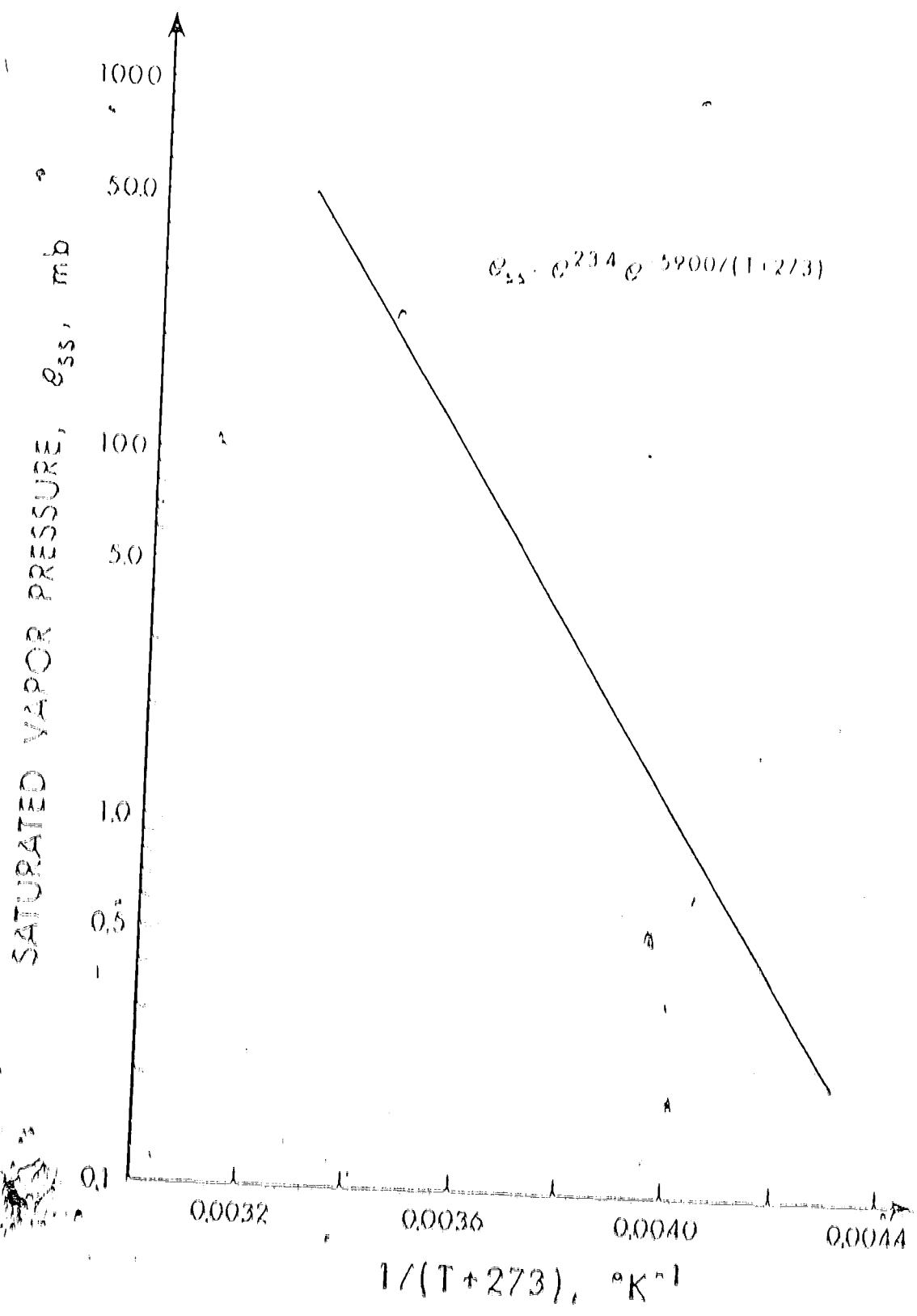


FIGURE 43 - Saturated vapor pressure (mb) vs $1/T (^{\circ}K^{-1})$

TABLE 24. Mean Relative Humidity for Norman Wells from Climate Journal, 1923

Month	Mean Relative Humidity
January	0.73
February	0.73
March	0.65
April	0.59
May	0.66
June	0.72
July	0.80
August	0.84
September	0.79
October	0.79
November	0.79
December	0.79

Considering only the relative humidity in the summer months, the average relative humidity is 0.65.

TABLE VI

Mean Monthly Cloudiness
 Between 1911-1920
 of each 10 Normal Years

Month	Mean Monthly Cloudiness
January	0.53
February	0.56
March	0.57
April	0.55
May	0.63
June	0.64
July	0.67
August	0.69
September	0.72
October	0.74
November	0.67
December	0.60

The mean annual cloudiness was 0.63.

TABLE 26. Mean Monthly Wind Velocity (m/sec.)
at Kororan Velli, Eror, Bahr (10)

Month	Mean Monthly Wind Velocity (m/sec.)	Standard Deviation (m/sec.)
January	3.00	3.26
February	2.77	2.91
March	3.31	2.82
April	3.74	3.13
May	3.80	2.82
June	4.07	2.61
July	3.71	2.59
August	3.62	2.56
September	3.71	2.77
October	3.67	2.86
November	3.08	2.82
December	2.73	3.04

Mean annual wind velocity is 3.43 m/sec.
 Mean annual standard deviation is 2.82 m/sec.
 This value used in the calculation is 4.8 m/sec.

TABLE 27. Mean Sea Level Pressure, P^s (mb) for
 Normal Years From 01 Jan 1950 to Normal 1959

Month	P^s (mb)
January	1022
February	1024
March	1022
April	1029
May	1018
June	1014
July	1013
August	1013
September	1014
October	1012
November	1018
December	1018

Annual mean sea level pressure P^s is 1018 mb. This corresponds to a true air pressure at Normal MeLH of 1000 mb.